

Several examples of heat engines in KK, but going with a different one.

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Lipscombe and Mungan, Phys. Teach. 58, 150 (2020)

Breathtaking Physics: Human respiration as a heat engine.

Air is a diatomic gas: N_2, O_2 so it has 5 degrees of freedom (mostly)

(you derived the rotational in the exam)

3 translational, 2 rotational

$$U = \frac{5}{2} N \tau$$

using a body plethysmograph

Assume ideal gas, $U = \frac{5}{2} PV$, let V be volume of lungs

and P the gauge pressure. esophageal catheter

Empirical model of the volume of the lungs: (measured, fit

$$V = A(1 - e^{-KP})$$

fitting parameters to the data)

A_I, K_I during inhalation; A_E, K_E during exhalation

$$-KP = \ln\left(\frac{A-V}{A}\right) \Rightarrow \ln\left(\frac{A}{A-V}\right) = KP$$

$$P = \frac{1}{K} \ln\left(\frac{A}{A-V}\right)$$

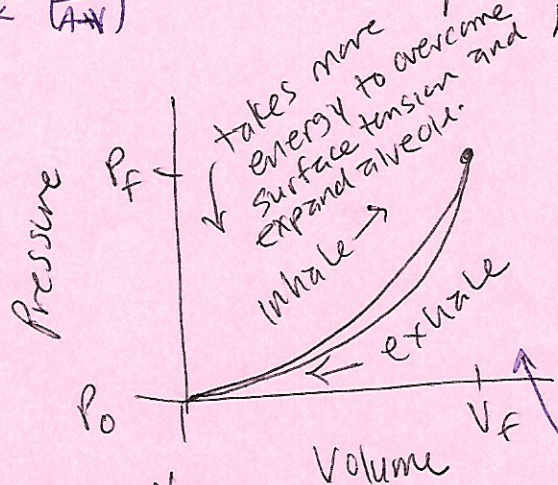
inverting

$$P = \frac{1}{K} \ln\left(\frac{A}{A-V}\right)$$

K is coefficient of compressibility

$$V_f = A_I(1 - e^{-K_I P_f}) = A_E(1 - e^{-K_E P_f})$$

$K_E > K_I$ since the alveoli are easier to compress when they are "inflated"



Refrigerators are heat engines in reverse:

they consume work to move heat from low T_c to high T_h

Body is higher T_h than outside

In case you are interested

$$A_I = 3.57 \times 10^{-3} \text{ m}^3$$

$$K_I = 0.964 \times 10^{-3} \text{ l/Pa}$$

$$A_E = 3.35 \times 10^{-3} \text{ m}^3$$

$$K_E = 1.644 \times 10^{-3} \text{ l/Pa}$$

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so $V_f = 3.31 \times 10^{-3} \text{ m}^3$

$P_f = 2730 \text{ Pa}$ ← gauge pressure, so relative to atmospheric pressure

$dW = PdV$; $V = A(1 - e^{-KP}) = A - Ae^{-KP}$

$$\Rightarrow dV = -Ae^{-KP}(-KdP) = AK e^{-KP} dP$$

$$W = \int P \cdot AK e^{-KP} dP = AK \int_0^{P_f} P e^{-KP} dP = AK e^{-KP} \left[\frac{P}{-K} - \frac{1}{K^2} \right]_0^{P_f}$$

~~Ans~~ ~~W =~~ $W = -\frac{AK e^{-KP}}{K} P - \frac{AK e^{-KP}}{K^2} = -\frac{A}{K} \left[K P e^{-KP} + e^{-KP} \right]$

$W = \frac{A}{K} \left[-(1 + KP) e^{-KP} \right]_0^{P_f} = -\frac{A}{K} \left[(1 + KP) \left(1 - \frac{V}{A} \right) \right]$

~~Ans~~ $W = -\frac{A}{K} \left[(1 + KP_f) e^{-KP_f} \right] + \frac{A}{K} \left[(1)(1) \right]$

$W = \frac{A}{K} \left[1 - (1 + KP_f) e^{-KP_f} \right]$ functional form

with $e^{-KP_f} = 1 - \frac{V_f}{A}$

$$W = \frac{A}{K} \left[1 - (1 + KP_f) \left(1 - \frac{V_f}{A} \right) \right] = \frac{A}{K} \left[1 - \left(1 + KP_f + \frac{V_f}{A} + \frac{KP_f V_f}{A} \right) \right]$$

$$W = \int p A k e^{-kp} dp = AK \int_0^{p_F} p e^{-kp} dp$$

Let $u = p$ $dv = e^{-kp} dp \Rightarrow v = \int e^{-kp} dp$
 $du = dp$

$$a = -kp$$

$$da = -k dp \Rightarrow dp = -\frac{da}{k}$$
~~$$v = \frac{1}{-k} \int e^a da = -\frac{1}{k} e^a = -\frac{1}{k} e^{-kp}$$~~

$$\int u dv = uv - \int v du$$

$$= \frac{p}{k} e^{-kp} + \int \frac{1}{k} e^{-kp} dp = -\frac{p}{k} e^{-kp} - \frac{1}{k^2} e^{-kp}$$

~~W = AK~~

$$W = AK e^{-kp} \left[-\frac{p}{k} - \frac{1}{k^2} \right]_0^{p_F}$$

$$W = - \frac{AKP_f}{K} + \frac{AV_f}{AK} + \frac{AKP_f V_f}{AK} = \frac{V_f}{K} - AP_f + P_f V_f \quad (150)$$

SO $W_{\text{net}} = W_I - W_E = \frac{V_f}{K_I} - AP_f + P_f V_f - \frac{V_f}{K_E} + A_E P_f - P_f V_f$

Area under the curve Inhale curve exhale curve

$$W_{\text{net}} = V_f \left[\frac{1}{K_I} - \frac{1}{K_E} \right] + P_f [A_E - A_I]$$

$$W_{\text{net}} = V_f \left[\frac{1}{K_I} - \frac{1}{K_E} \right] - P_f [A_I - A_E] \quad \text{Eq. 10}$$

using fitted values, $W_{\text{net}} = 0.820 \text{ J}$.

$$\cancel{dU} \neq dU + PdV = \frac{5}{2} d(PV) + PdV = dQ$$

$$Q_I = \Delta U_I + W_I = \frac{5}{2} P_f V_f + \frac{V_f}{K_I} - A_I P_f + P_f V_f$$

$$Q_I = \frac{V_f}{K_I} - A_I P_f + \frac{7}{2} P_f V_f = 25.3 \text{ J}$$

$$\text{Coefficient of performance} = \frac{Q_I}{W_{\text{net}}} = \frac{25.3 \text{ J}}{0.820 \text{ J}} = 31$$

Each individual breath requires little energy, but we do breathe 20,000/day, so breathing is about 6% of the $9 \times 10^6 \text{ J}$ we spend every day

Laws of thermodynamics - revisited

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Zeroth law. If two thermodynamic systems are in contact with a third one, they are in thermal equilibrium with each other.

You must play the game

$$\left(\frac{\partial \ln g_1}{\partial U_1}\right)_{N_1} = \left(\frac{\partial \ln g_3}{\partial U}\right)_{N_3}; \left(\frac{\partial \ln g_2}{\partial U_2}\right)_{N_2} = \left(\frac{\partial \ln g_3}{\partial U_3}\right)_{N_3}$$

~~First law~~

$$T_2 = T_3 \text{ \& } T_1 = T_3 \Rightarrow T_1 = T_2$$

First law.

$$dW = dU - dQ \quad \text{Conservation of energy}$$

You can't do more work than the amount of heat you take in

You can't win

Second law.

$$\eta = \frac{T_h - T_c}{T_h} = 1 - \frac{T_c}{T_h}$$

You can't break even unless you quit

Your efficiency is always less than 100%, unless T_c is absolute zero

Third law.

$$C_v = T \frac{\partial \sigma}{\partial T}; \text{ For } C_{v,r} = \delta T$$

You can't quit

$$\Rightarrow d\sigma = \frac{\delta T}{T} dT$$

$$\sigma = \frac{\sigma}{T} + C \quad \sigma = \int d\sigma = \int \frac{\delta T}{T} dT = \delta \ln T + C$$

As $T \rightarrow 0$, you still have C (can $C = 0$?)

Just like PHYS 3331

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You can't reach absolute zero. Also from QM