Control Computarizado final exam (29%)

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November 15, 2017

Time 2017-11-13 07:00 - 2017-11-14 17:30

Permitted aids Whatever books, notes, web pages, videos you can find. Except help from another living person. For questions about the exam I can be contacted at kjartan@itesm.mx or if acute by WhatsApp (556219 4048).

All answers should be readable and well motivated (if nothing else is written). Hand in (on paper) at the time of the class Tuesday November 14 at 17:30.

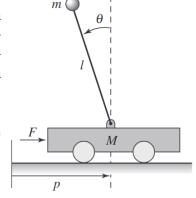
Good luck!

An inverted pendulum

The cart-pendulum system is a common example of an unstable system. If we are not concerned with the position of the cart, but only with the angle of the pendulum, then the system can be simplified and written as

$$\frac{d}{dt} \begin{bmatrix} \theta \\ \dot{\theta} \end{bmatrix} = \begin{bmatrix} \dot{\theta} \\ \frac{mgl}{J} \sin \theta + \frac{1}{J} \cos \theta u \end{bmatrix}, \qquad y = \theta, \quad (1)$$

where J is the moment of inertia of the pendulum.



From the textbook

Problem 1

(a)

Show that linearizing the model about the vertical equilibrium state ($\sin \theta \approx \theta$, $\cos \theta \approx 1$) gives the linear state-space system

$$\frac{d}{dt} \begin{bmatrix} \theta \\ \dot{\theta} \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ \frac{mgl}{J} & 0 \end{bmatrix} \begin{bmatrix} \theta \\ \dot{\theta} \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{1}{J} \end{bmatrix} u$$

$$y = \begin{bmatrix} 1 & 0 \end{bmatrix} \begin{bmatrix} \theta \\ \dot{\theta} \end{bmatrix}.$$
(2)

(b)

Show that the linear state space model corresponds to the transfer function

$$G(s) = C(sI - A)^{-1}B = \frac{1/J}{s^2 - \frac{mgl}{J}},$$
 (3)

with poles in $\pm \omega_0$ where $\omega_0 = \sqrt{\frac{mgl}{J}}$.

(c)

Here, and in the rest of the exam let J = 1. Show that the zero-order-hold discretization of the state space model (2) gives the discrete-time state space model

$$x(kh+h) = \underbrace{\begin{bmatrix} \cosh(\omega_0 h) & \frac{1}{\omega_0} \sinh(\omega_0 h) \\ \omega_0 \sinh(\omega_0 h) & \cosh(\omega_0 h) \end{bmatrix}}_{\Phi} x(kh) + \underbrace{\begin{bmatrix} \frac{1}{\omega_0^2} (\cosh(\omega_0 h) - 1) \\ \frac{1}{\omega_0} \sinh(\omega_0 h) \end{bmatrix}}_{\Gamma} u(kh)$$

$$y(kh) = \underbrace{\begin{bmatrix} 1 & 0 \end{bmatrix}}_{C} x(kh).$$

$$(4)$$

Problem 2

The open-loop discrete-time system has poles in

$$z = e^{\omega_0 h}$$
 and $z = e^{-\omega_0 h}$.

We want to design a controller that gives a closed-loop system with both poles in $z = e^{-\omega_0 h}$.

- 1. Let $\omega_0 = 1$, choose a reasonable sampling period h (motivate!) and write your state space system with numerical values.
- 2. Calculate the state feedback gain L in

$$u(kh) = -l_1x(kh) - l_2x(kh) = -Lx(kh)$$

so that the closed loop systems has the desired poles.

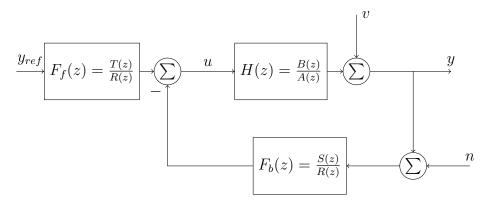
- 3. Verify your calculations using matlab. If you don't have matlab on your computer you can use the online-service https://octave-online.net/ which includes a control toolbox. At least, you should calculate the eigenvalued of the closed loop system by eig(Phi-Gamma*L). You can also try to use the functions acker or place.
- 4. If the state is not available, we must use an observer to estimate the state. Suggest a reasonable choice of observer poles (numerical values) in the z-plane.

Problem 3

For certain values of ω_0 and h we obtain the following discrete-time model of the inverted pendulum

$$H(z) = \frac{B(z)}{A(z)} = \frac{0.02(z+1)}{z^2 - 2.04z + 1}.$$

Instead of using state feedback with observer, consider using RST control as in the figure below.



We desire a closed-loop system from the reference signal y_{ref} to the output y with pulse-transfer function

$$H_c(z) = \frac{0.1643(z+1)}{z^2 - 1.637z + 0.6703}$$

The S- and R- polynomials have been found from the Diophantine equation

$$A(z)R(z) + B(z)S(z) = (z^2 - 1.637z + 0.6703)(z - 0.67)$$

to be

$$R(z) = z - 0.368$$

 $S(z) = 4.98z - 4.076$

Determine the feedforward (prefilter) part of the controller $F_f(z) = \frac{T(z)}{R(z)}$ so that the desired closed-loop pulse-transfer function is obtained.

Solutions

Problem 1

(a)

With the small-angle approximations we get the system of first-order differential equations

$$\frac{d}{dt} \begin{bmatrix} \theta \\ \dot{\theta} \end{bmatrix} = \begin{bmatrix} \dot{\theta} \\ \frac{mgl}{I}\theta + \frac{1}{I}u \end{bmatrix}, \qquad y = \theta$$

which written on matrix-vector form is

$$\frac{d}{dt} \begin{bmatrix} \theta \\ \dot{\theta} \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ \frac{mgl}{J} & 0 \end{bmatrix} \begin{bmatrix} \theta \\ \dot{\theta} \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{1}{J} \end{bmatrix} u$$
$$y = \begin{bmatrix} 1 & 0 \end{bmatrix} \begin{bmatrix} \theta \\ \dot{\theta} \end{bmatrix}.$$

(b)

The expression

$$G(s) = C(sI - A)^{-1}B$$

for how to go from a state-space model to a transfer function is obtained by taking the Laplace-transform of the state-space model (this part is not needed to show). Note that

$$(sI - A)^{-1} = \begin{bmatrix} s & -1 \\ -\frac{mgl}{J} & s \end{bmatrix}^{-1} = \frac{1}{s^2 - \frac{mgl}{J}} \begin{bmatrix} s & 1 \\ \frac{mgl}{J} & s \end{bmatrix}.$$

Inserting into the expression we get

$$G(s) = C(sI - A)^{-1}B = \frac{1}{s^2 - \frac{mgl}{J}} \begin{bmatrix} 1 & 0 \end{bmatrix} \begin{bmatrix} s & 1 \\ \frac{mgl}{J} & s \end{bmatrix} \begin{bmatrix} 0 \\ \frac{1}{J} \end{bmatrix} = \frac{1/J}{s^2 - \frac{mgl}{J}}.$$

The poles are the roots of the characteristic polynomial $s^2 - \frac{mgl}{J}$, which clearly are $\pm \omega_0$ where $\omega_0 = \sqrt{\frac{mgl}{J}}$.

(c)

The discrete-time state-space model is given by

$$x(kh + h) = \Phi(h)x(kh + h) + \Gamma(h)u(kh)$$
$$y(kh) = Cx(kh),$$

where

$$\Phi(h) = e^{Ah}, \qquad \Gamma(h) = \int_0^h e^{A\tau} d\tau.$$

We get

$$\Phi(h) = e^{Ah} = \mathcal{L}^{-1}\{(sI - A)^{-1}\} = \mathcal{L}^{-1}\{\begin{bmatrix} s & -1 \\ -\omega_0^2 & s \end{bmatrix}^{-1}\}$$

$$= \mathcal{L}^{-1}\{\frac{1}{s^2 - \omega_0^2} \begin{bmatrix} s & 1 \\ \omega_0^2 & s \end{bmatrix}\} = \begin{bmatrix} \mathcal{L}^{-1} \frac{s}{s^2 - \omega_0^2} & \frac{1}{s^2 - \omega_0^2} \\ \frac{\omega_0^2}{s^2 - \omega_0^2} & \frac{s}{s^2 - \omega_0^2} \end{bmatrix}$$

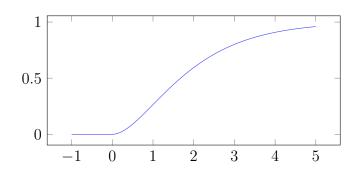
$$= \begin{bmatrix} \cosh(\omega_0 h) & \frac{1}{\omega_0} \sinh(\omega_0 h) \\ \omega_0 \sinh(\omega_0 h) & \cosh(\omega_0 h) \end{bmatrix}.$$

$$\Gamma(h) = \int_0^h e^{A\tau} d\tau = \int_0^h \left[\frac{1}{\omega_0} \sinh(\omega\tau) \right] d\tau = \begin{bmatrix} \frac{1}{\omega_0^2} \cosh(\omega_0\tau) \Big|_0^h \\ \frac{1}{\omega_0} \sinh(\omega_0\tau) \Big|_0^h \end{bmatrix}$$
$$= \begin{bmatrix} \frac{1}{\omega_0^2} \left(\cosh(\omega_0 h) - 1 \right) \\ \frac{1}{\omega_0} \sinh(\omega_0 h) \end{bmatrix}.$$

Problem 2

1. The sampling period should be related to the speed of the closed-loop system, since we have given the desired closed-loop poles. In continuous-time this is a second order critically-damped system with double-pole in $s = -\omega_0 = -1$. The step-response of such a system is given by

$$y(t) = 1 - (1+t)e^{-t}$$



The rise-time is about 4 seconds, and using the rule-of-thumb that the number of sampling perdiods should be 4-10, we may for instance choose h = 0.5. This gives the state-space model

$$x(k+1) = \begin{bmatrix} 1.13 & 0.52 \\ 0.52 & 1.13 \end{bmatrix} x(k) + \begin{bmatrix} 0.13 \\ 0.52 \end{bmatrix} u(k)$$
$$y(k) = \begin{bmatrix} 1 & 0 \end{bmatrix} x(k).$$

2. Calculate the state feedback gain L in

$$u(kh) = -l_1x(kh) - l_2x(kh) = -Lx(kh)$$

so that the closed loop systems has the desired poles. With the negative state feedback we get a closed-loop system with characteristic polynomial given by

$$\det (zI - (\Phi - \Gamma L)) = \det \begin{pmatrix} \begin{bmatrix} z & 0 \\ 0 & z \end{bmatrix} - \begin{pmatrix} \begin{bmatrix} 1.13 & 0.52 \\ 0.52 & 1.13 \end{bmatrix} - \begin{bmatrix} 0.13l_1 & 0.13l_2 \\ 0.52l_1 & 0.52l_2 \end{bmatrix} \end{pmatrix}$$

$$= \det \begin{bmatrix} z - 1.13 + 0.13l_1 & -0.52 + 0.13l_2 \\ -0.52 + 0.52l_1 & z - 1.13 + 0.52l_2 \end{bmatrix}$$

$$= (z - 1.13 + 0.13l_1)(z - 1.13 + 0.52l_2) - (-0.52 + 0.52l_1)(-0.52 + 0.13l_2)$$

$$= z^2 + (0.13l_1 + 0.52l_2 - 2.26)z + 0.1235l_1 - 0.52l_2 + 1.0065.$$

Setting each coefficient equal to those of the desired characteristic polynomial $(z - e^{-0.5})^2 = (z - 0.607)^2 = z^2 - 1.21z + 0.368$ gives the system of equations

$$\begin{bmatrix} 0.13 & 0.52 \\ 0.1235 & -0.52 \end{bmatrix} \begin{bmatrix} l_1 \\ l_2 \end{bmatrix} = \begin{bmatrix} -1.21 + 2.26 \\ 0.368 - 1.0065 \end{bmatrix}$$

with solution

$$l1 = 1.61$$
 $l2 = 1.61$

3. Verify your calculations using matlab.

```
omega0 = 1;
h = 0.5;
Phi = [cosh(omega0*h), 1/omega0*sinh(omega0*h)
    omega0*sinh(omega0*h), cosh(omega0*h)]
Gamma = [1/omega0^2*(cosh(omega0*h) - 1); 1/omega0*sinh(omega0*h)]
L = [1.61, 1.61];
eig(Phi-Gamma*L)
ans =
    0.606530659712637
    0.604280024872918
```

4. The observer poles should be at least twice as fast as the closed-loop poles. Twice as fast means two poles in $z = e^{-2\omega_0 h} = e^{-1} \approx 0.368$.

Problem 3

We only need to determine the polynomial T(z). The closed-loop transfer function from the reference signal to the output is, according to the block-diagram and using Mason's rule

$$H_c(z) = \frac{\frac{T}{R} \frac{B}{A}}{1 + \frac{S}{R} \frac{B}{A}} = \frac{TB}{AR + BS} = \frac{T(z)B(z)}{A_{cl}(z)}$$
$$= \frac{T(z)0.02(z+1)}{(z^2 - 1.637z + 0.6703)(z - 0.67)},$$

which should be equal to the desired closed-loop transfer function

$$H_c(z) = \frac{0.1643(z+1)}{z^2 - 1.637z + 0.6703}.$$

This gives

$$\frac{T(z)0.02(z+1)}{(z^2 - 1.637z + 0.6703)(z - 0.67)} = \frac{0.1643(z+1)}{z^2 - 1.637z + 0.6703}.$$

Solving for T(z) gives

$$T(z) = \frac{0.1643}{0.02}(z - 0.67).$$

You would arrive at the same result if you identified (z-0.67) as the observer polynomial, and followed the usual route of setting $T(z) = t_0 A_o(z)$ and solving for t_0 so that the closed-loop system have unit static gain.