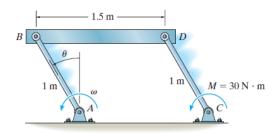
(a) (6) t=35 t=0.55 x = 0.04 sin 11t = 0.04 X = 0.0477 cos nt = -0.0477 X = -0.047 3in nt = 0 -0.04 172 w = 2 sin # t = -2 is = 17 cos # t = ac = ag + wxr + wx (wxr) + 2wx Urel + arel (a) Q= 0.2(-2)(-j) +0i = -0.8j m/52 wxr = 0, ωx(ωxr) = -2kx(-2kx0) = 0 2wx Vrel = 2(-2k)x(-0.04mi) = 0.503j m/s2, a rel =0 Substitute & get Q = - 0.297 j m/s2 (b) a= -0.2 \(\frac{7}{2} \) = -0.444i -0.4; m/s2
\(\omega \times r = \pi/\frac{7}{2} \) \(\times \times 0.04i = 0.0889; \(\omega \times \frac{7}{2} \) wx(wxr)=12kx(12kx0.04i)=-0.08i m/s2 2wx Vrol = 2V2kx0=0, arel = -0.0472 =-0.395 = m Substitute & set Qc =- 0.919 i - 0.3114 m/s2

$$k = 0.175 \text{ m}$$
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18-31.

The linkage consists of two 6-kg rods AB and CD and a 20-kg bar BD. When $\theta=0^\circ$, rod AB is rotating with an angular velocity $\omega=2$ rad/s. If rod CD is subjected to a couple moment of M=30 N·m, determine ω_{AB} at the instant $\theta=90^\circ$.



SOLUTION

Kinetic Energy. The mass moment of inertia of each link about the axis of rotation is $I_A = \frac{1}{12}(6)(1^2) + 6(0.5^2) = 2.00 \, \text{kg} \cdot \text{m}$. The velocity of the center of mass of the bar is $v_G = \omega r = \omega(1)$. Thus,

$$T = 2\left(\frac{1}{2}I_A\omega^2\right) + \frac{1}{2}M_bv_G^2$$
$$= 2\left[\frac{1}{2}(2.00)\omega^2\right] + \frac{1}{2}(20)[\omega(1)]^2$$
$$= 12.0\,\omega^2$$

Initially, $\omega = 2 \text{ rad/s}$. Then

$$T_1 = 12.0(2^2) = 48.0 \text{ J}$$

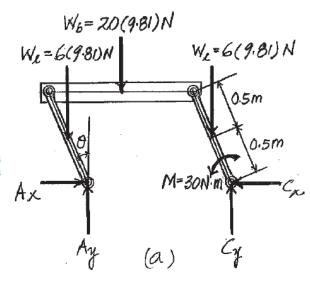
Work. Referring to the FBD of the assembly, Fig. a, the weights W_b , W_c and couple moment M do positive work when the links undergo an angular displacement θ . When $\theta = 90^{\circ} = \frac{\pi}{2}$ rad,

$$\begin{split} &U_{W_b} = W_b s_b = 20(9.81)(1) = 196.2 \text{ J} \\ &U_{W_c} = W_c s_c = 6(9.81)(0.5) = 29.43 \text{ J} \\ &U_M = M\theta = 30 \bigg(\frac{\pi}{2}\bigg) = 15\pi \text{ J} \end{split}$$



$$T_1 + \Sigma U_{1-2} = T_2$$

 $48.0 + [196.2 + 2(29.43) + 15\pi] = 12.0 \omega^2$
 $\omega = 5.4020 \,\text{rad/s} = 5.40 \,\text{rad/s}$



Ans.