

Model-Based Optimization for a Campus District Cooling System

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Overview

- Background and motivation
- Models for district cooling systems
- Case study
- Conclusion

Background and Motivation

Why District Cooling?

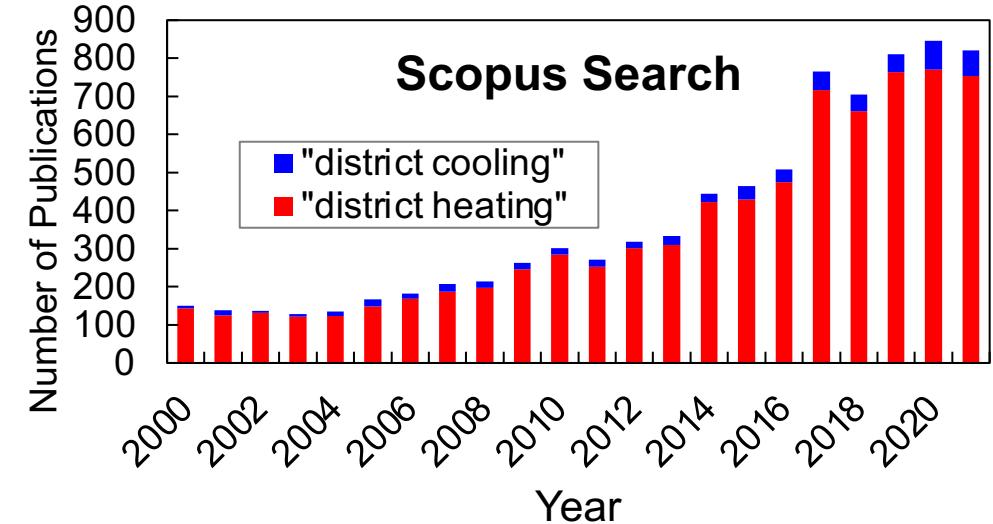
- Space cooling is growing faster than any other building end use¹
- None had modeled complete district cooling systems (plant + distribution) featuring hydraulics nor waterside economizers

Objectives

- Demonstrate how Modelica can enable complete district cooling energy analysis
- Identify investment-free energy efficiency strategies for a real-world case study
- Evaluate carbon and operational cost savings due to energy retrofits

Gaps in Scientific Literature

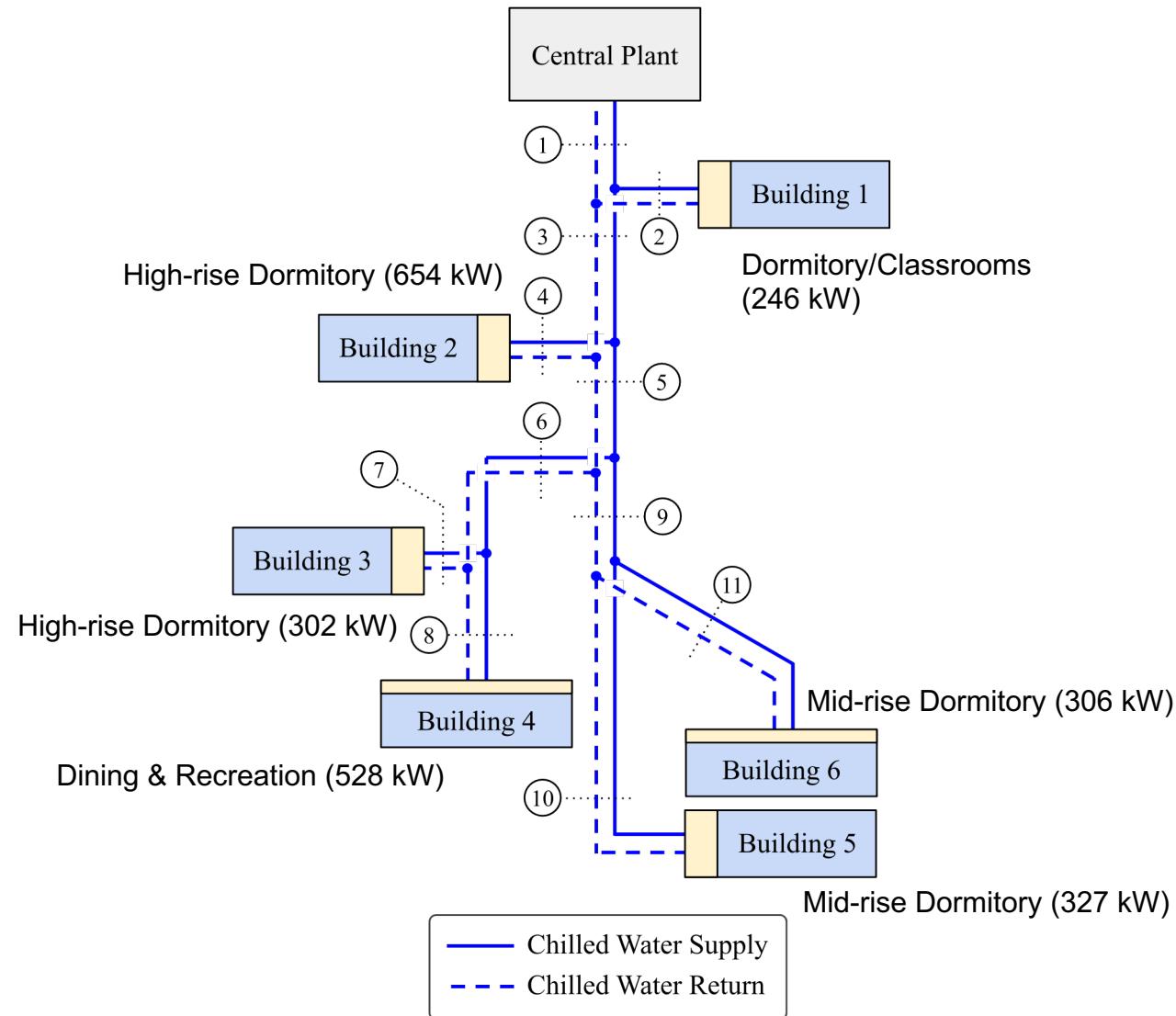
- District cooling studies are generally limited



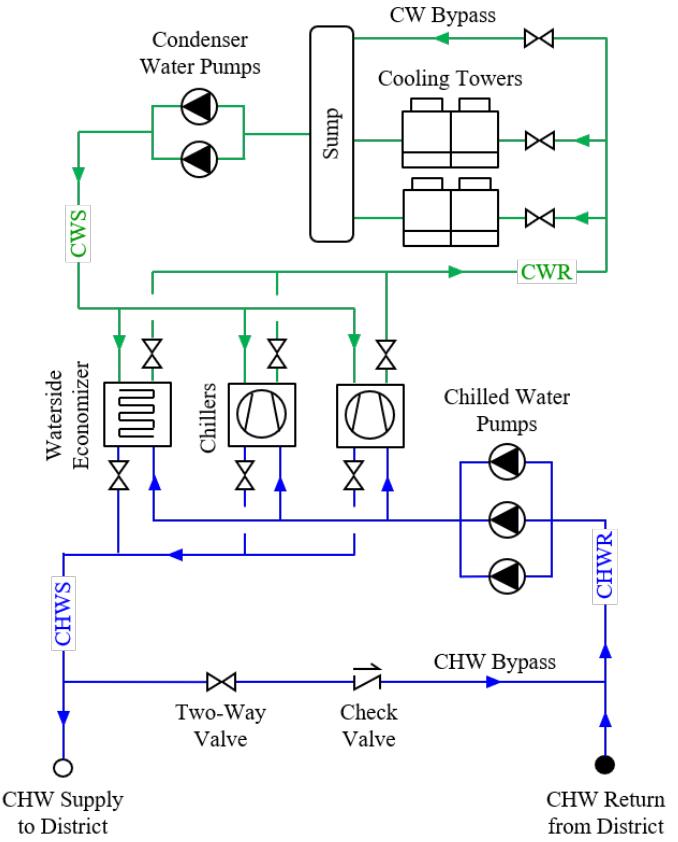
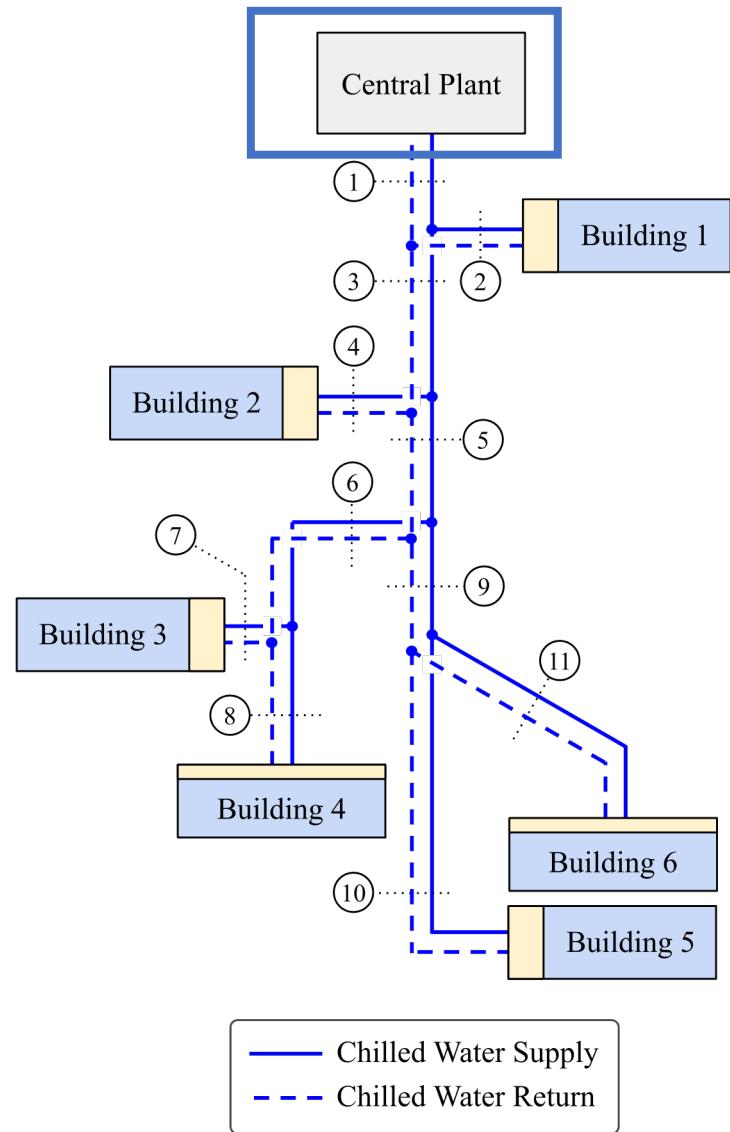
Case Study



- A satellite campus of University of Colorado in Boulder, CO
- Six Buildings:
 - Floor area: 93,990 m² (1,011,699 ft²)
 - Peak load: 2.4 MW
- Radial network with 1.5 km pipes

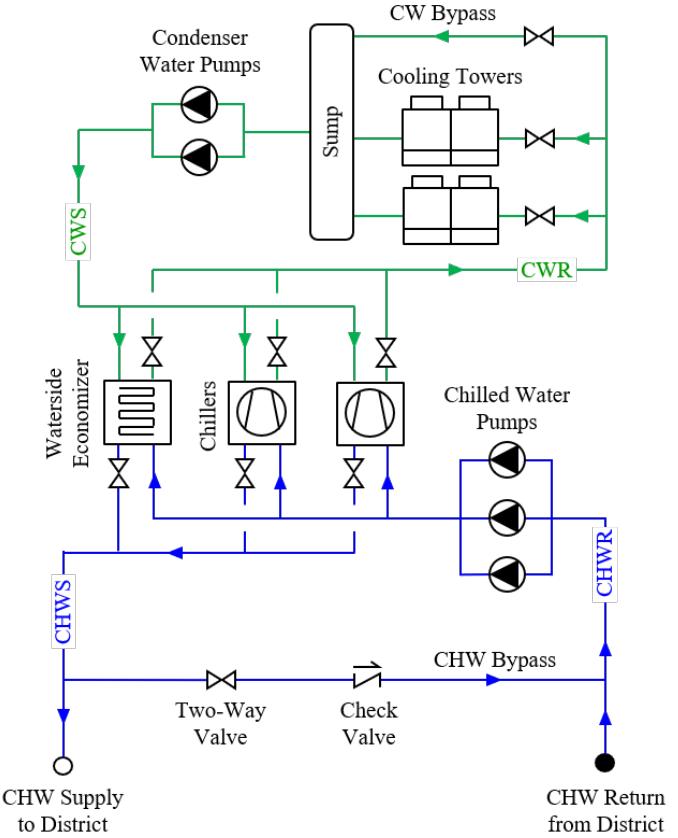
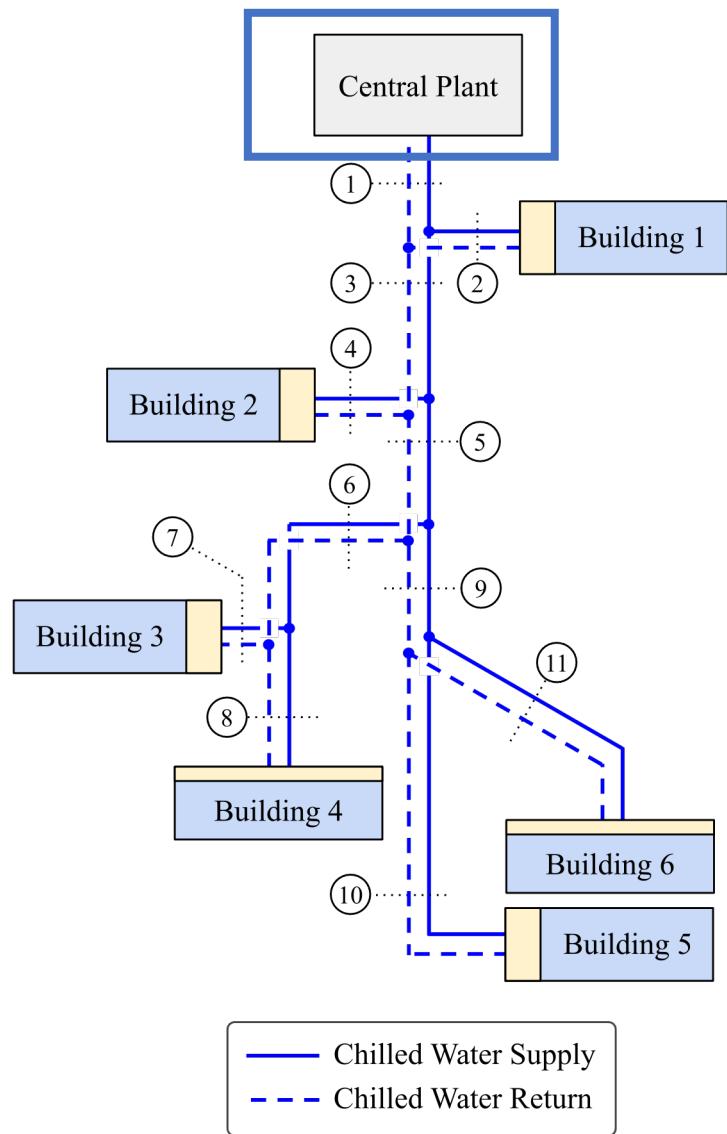


Central Plant



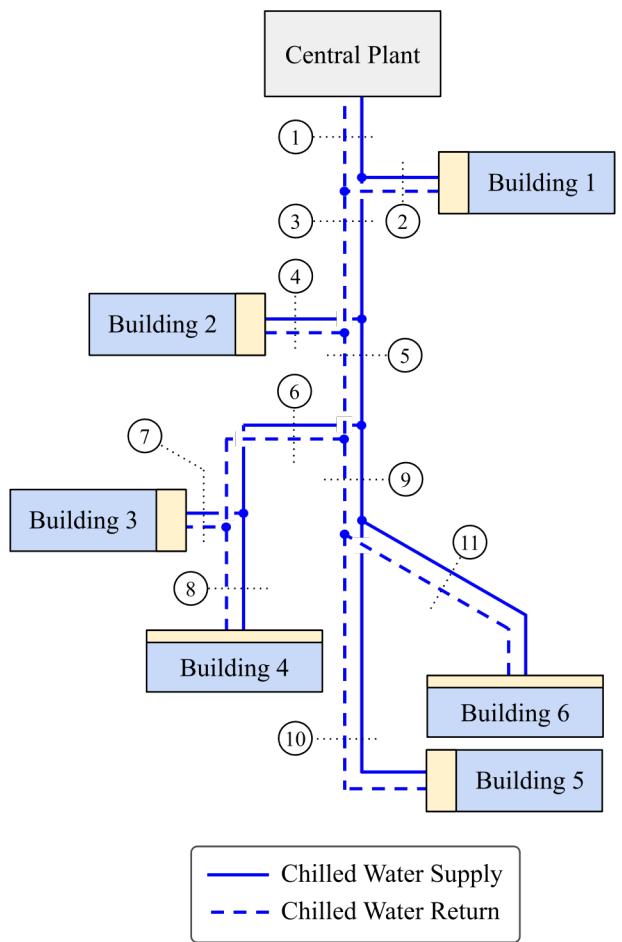
- Two single compressor chillers (2455 kW each)
- Three chilled water pump (30 kW each)
- Two condenser water pump (56 kW each)
- Non-integrated water side economizer
- Four cooling tower units (22 kW each)

Central Plant

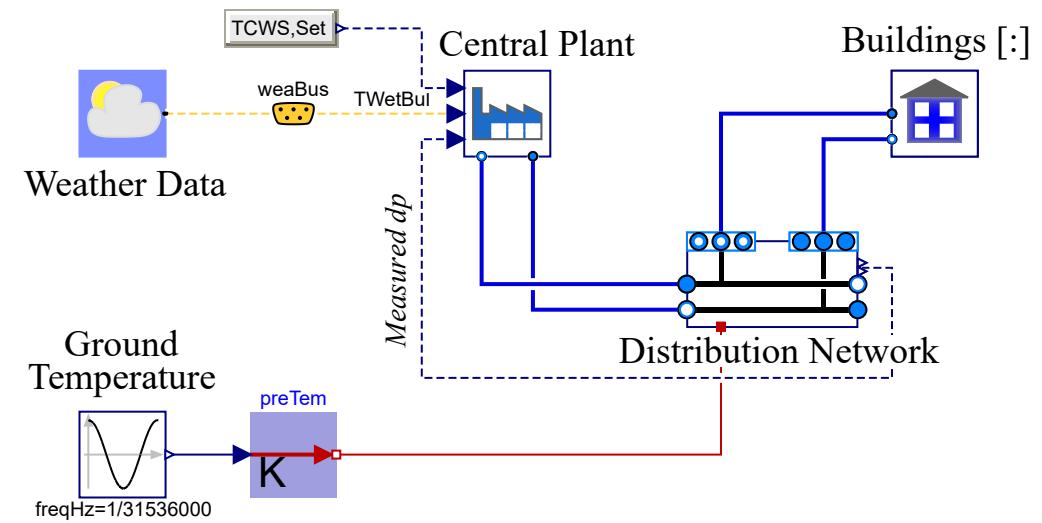


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Modeling: District Cooling Systems

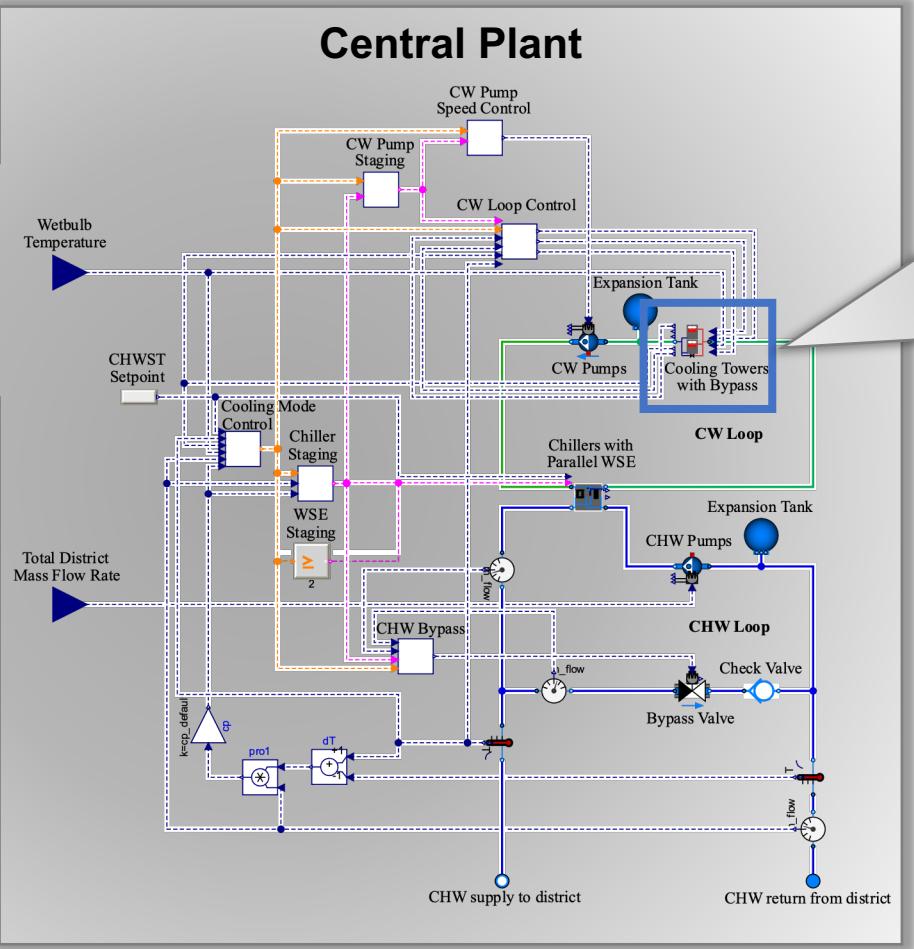
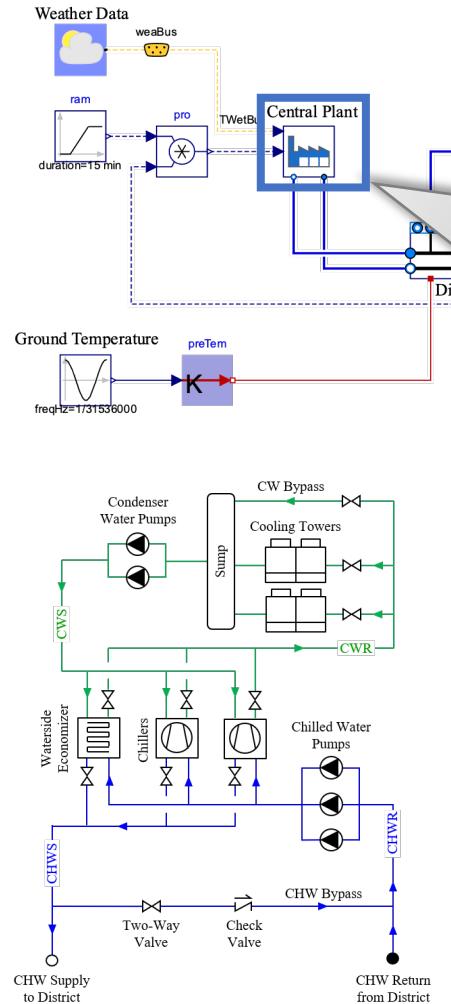


System Schematics

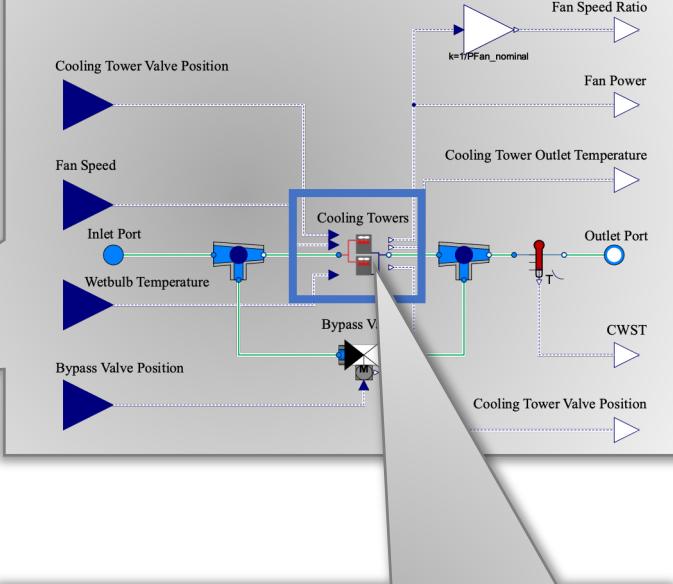


Top-Level Model

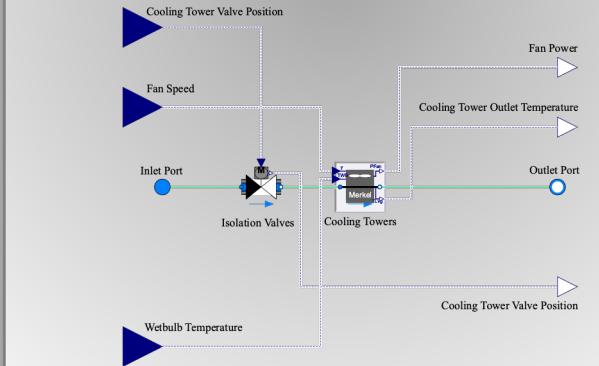
Central Plant



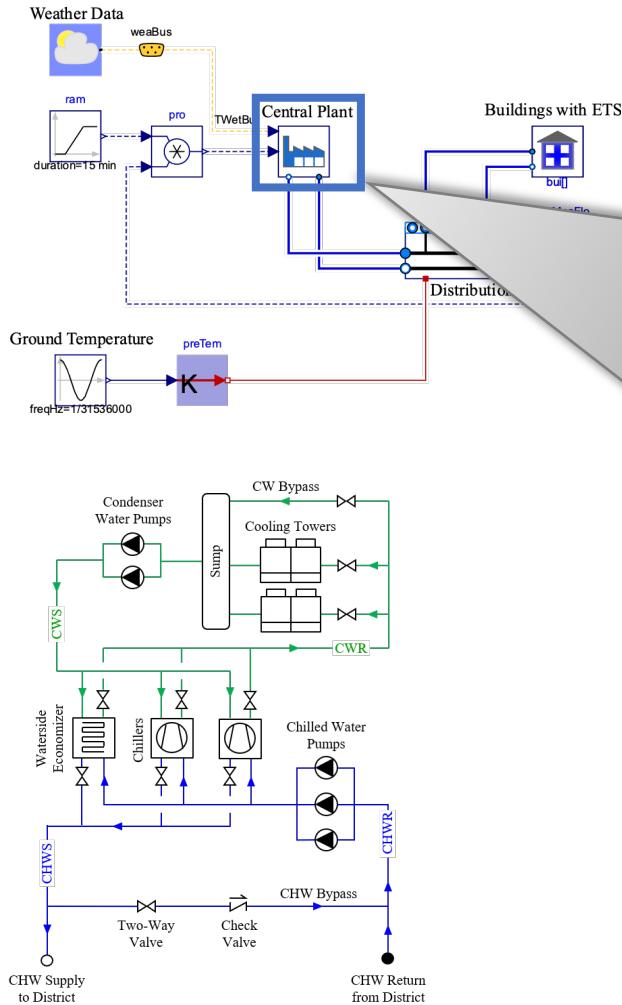
Cooling Tower with Bypass



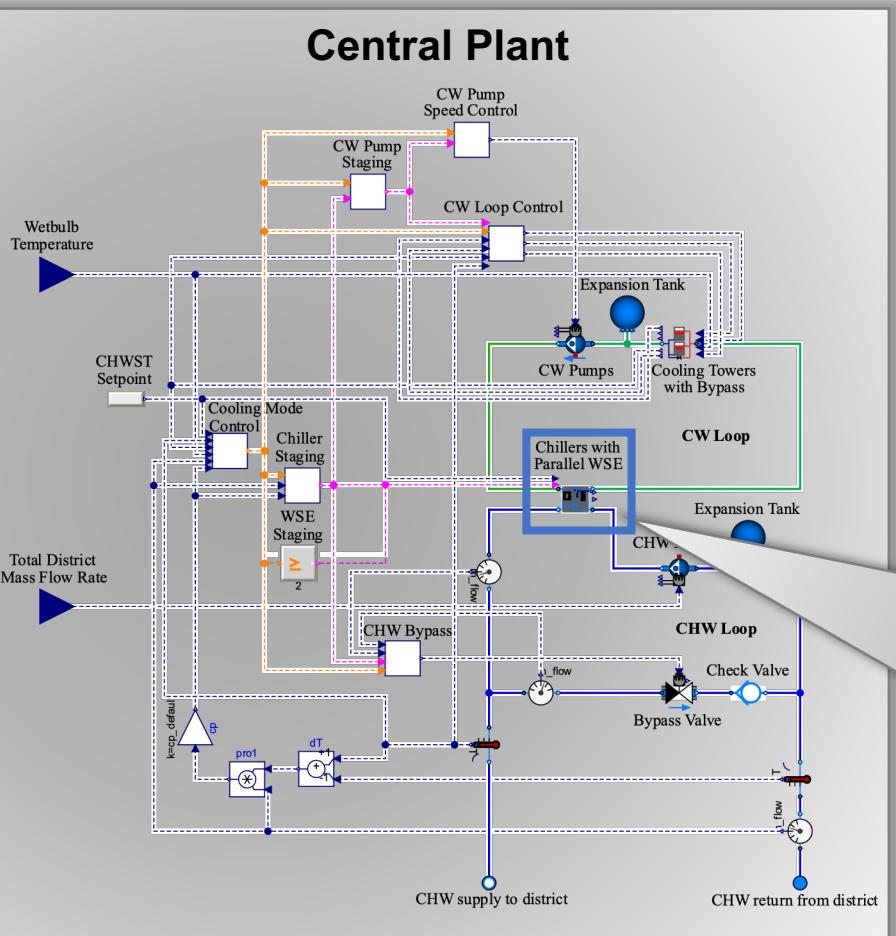
Cooling Tower with Isolation Valve



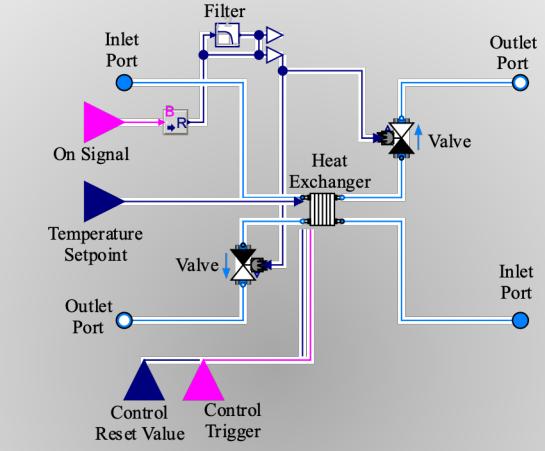
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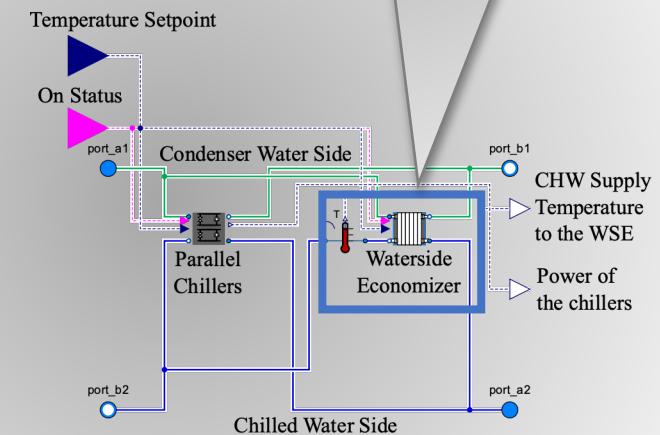
Central Plant



Water Side Economizer

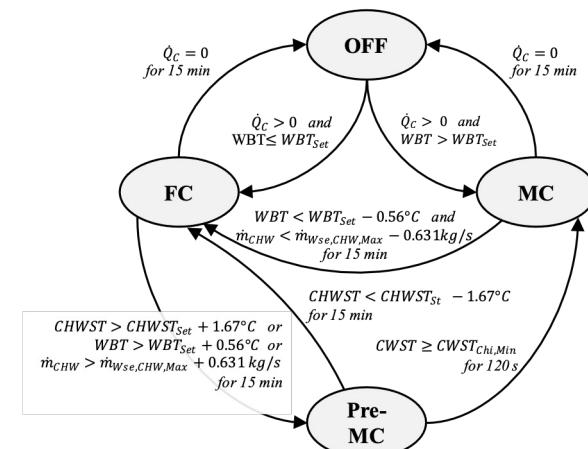
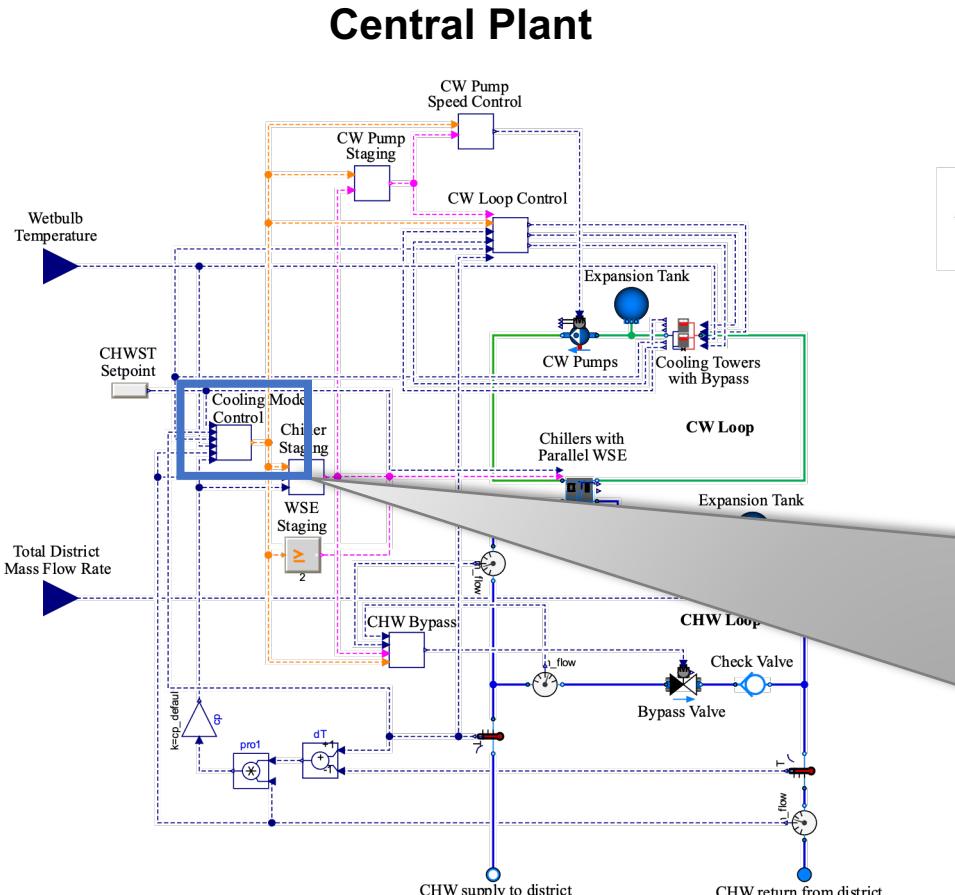
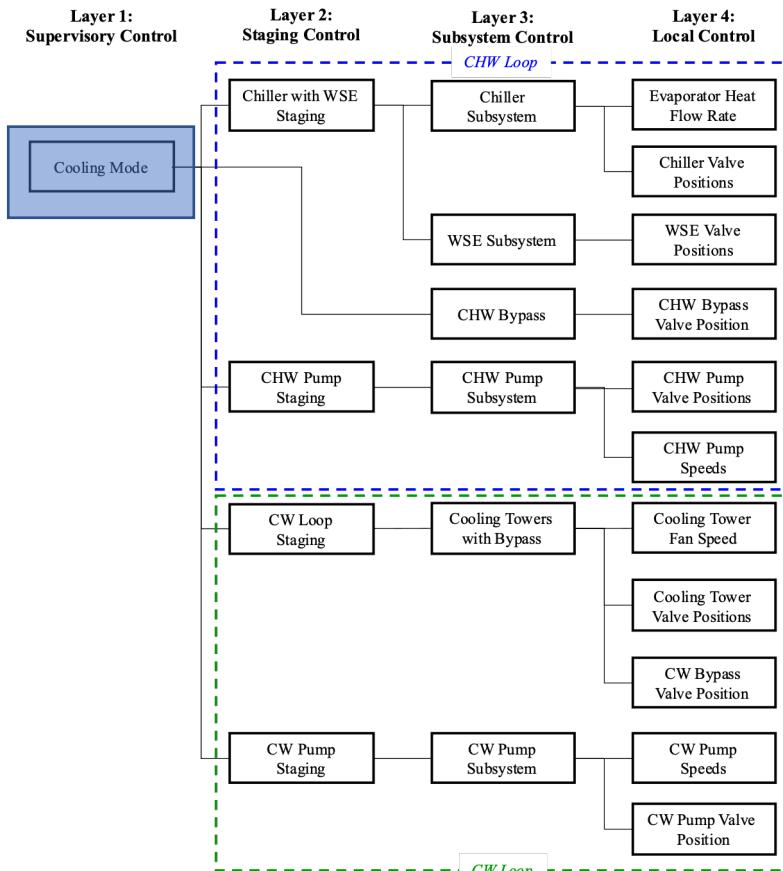


Chillers with Parallel WSE

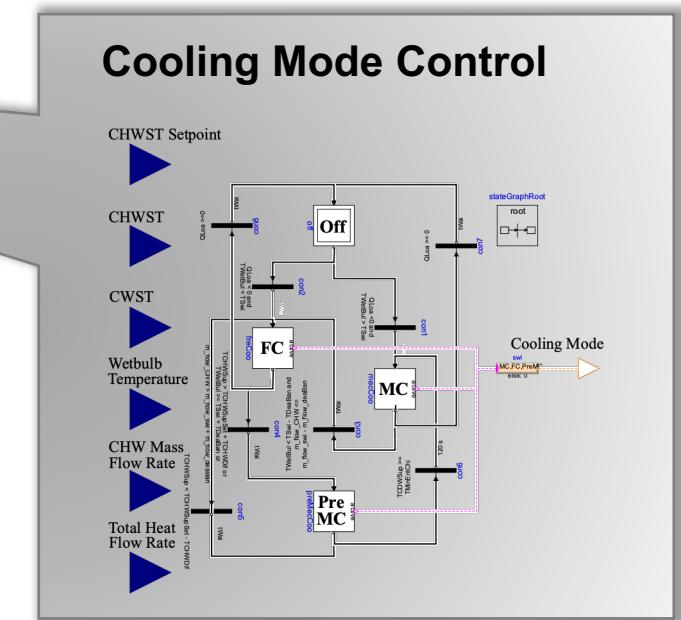


System Schematic

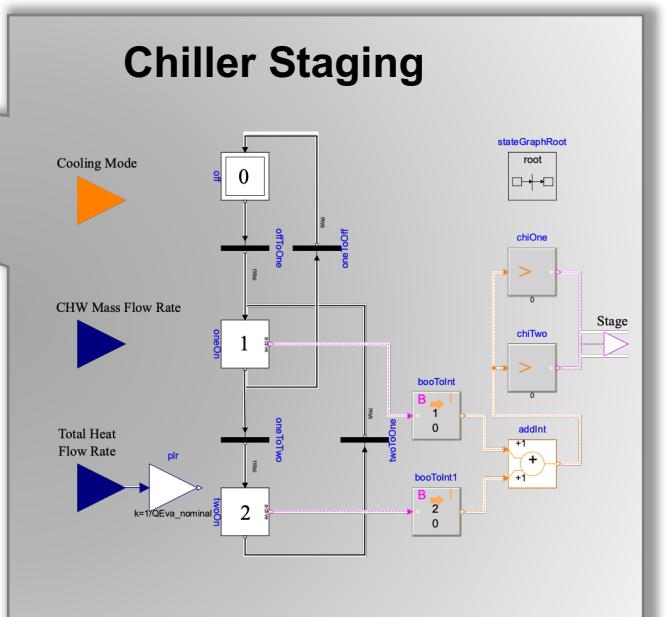
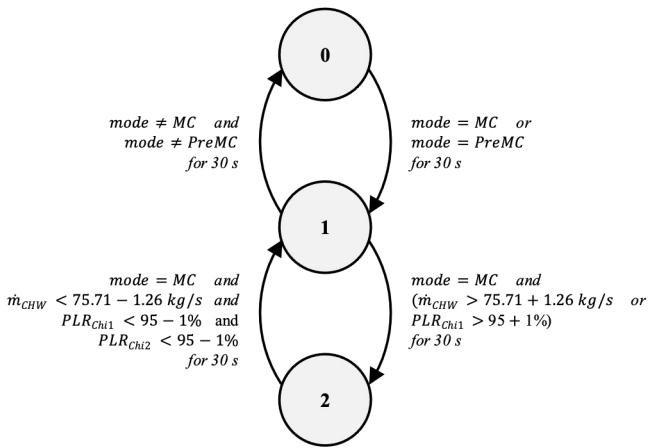
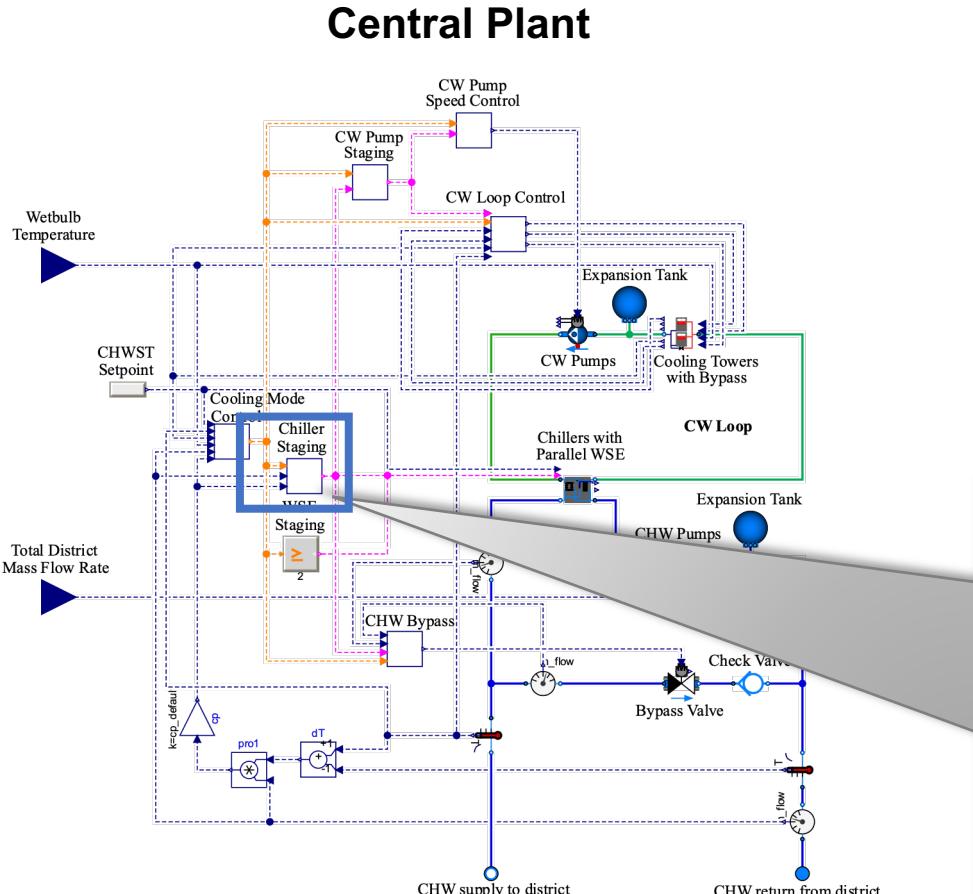
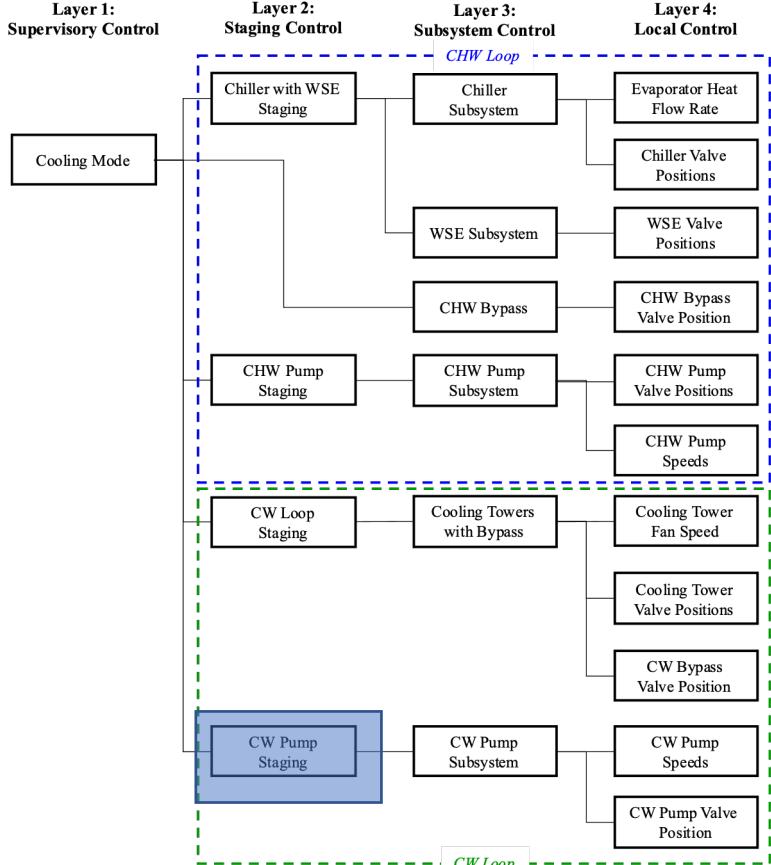
Control Layer 1: Cooling Mode Control



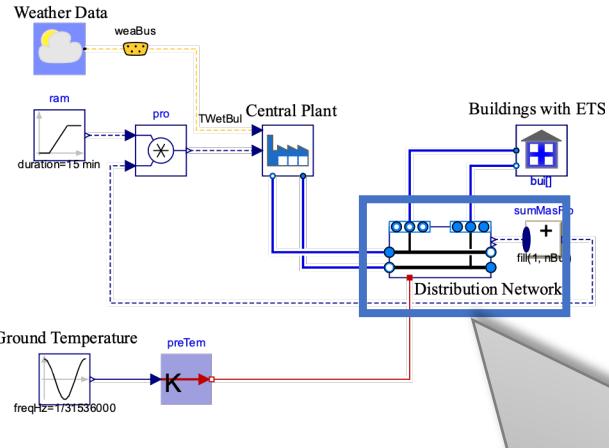
State Graph



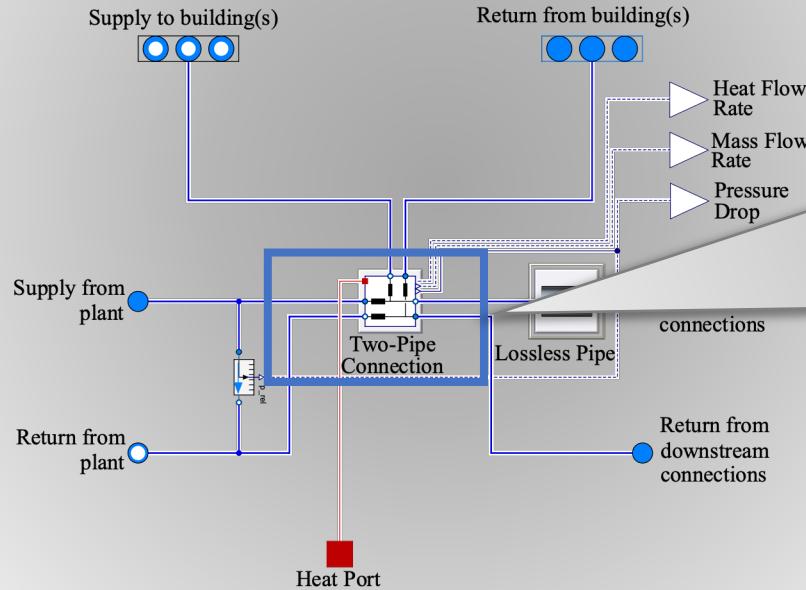
Control Layer 2: Chiller Staging Control



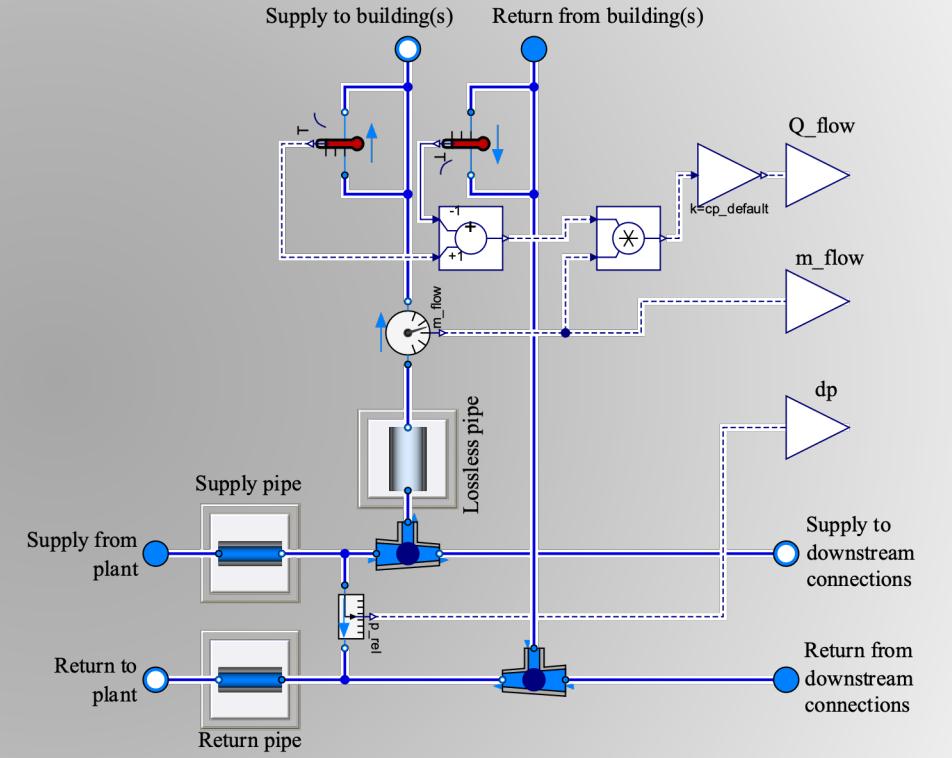
Distribution Network



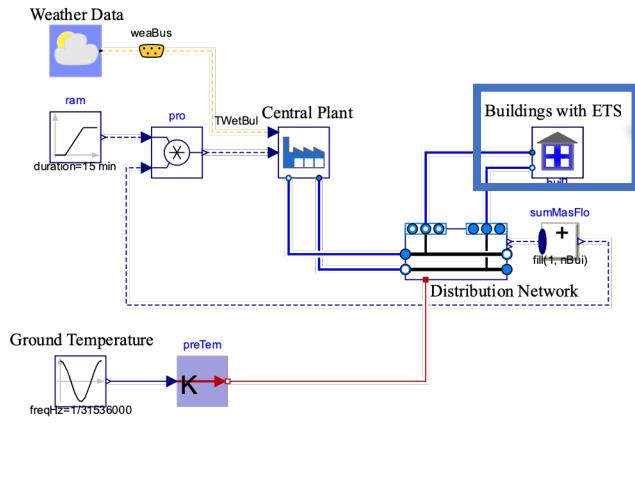
Distribution with Two Pipes



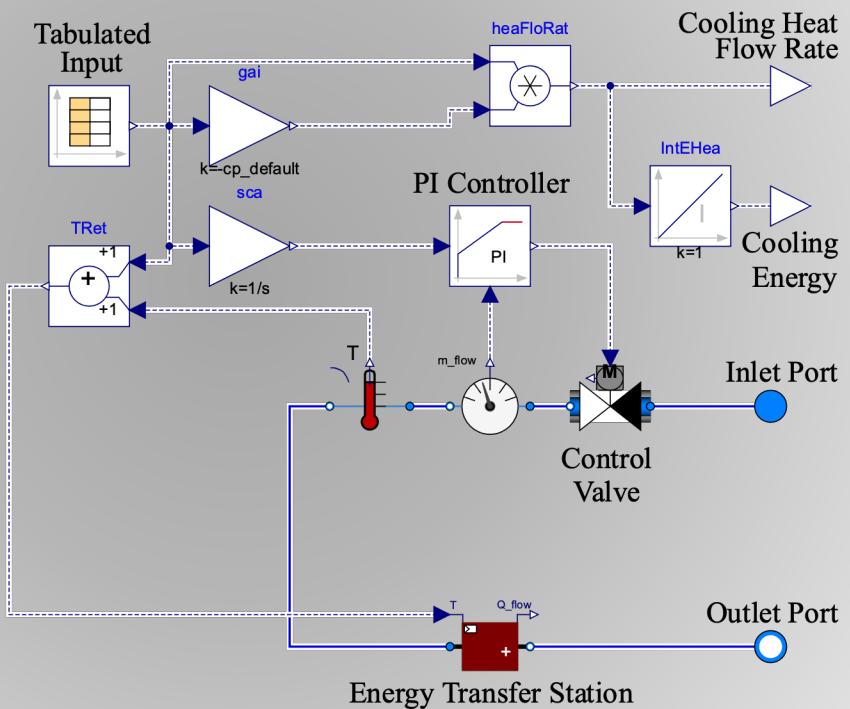
Connection with Two Pipes



Buildings with Energy Transfer Station



Buildings with Energy Transfer Station



Validation of Models

Location	CVRMSE (%)				NMBE (%)			
	Acceptable range: [0,30%]				Acceptable range: [-10,10%]			
	\dot{Q}_{CHW}	\dot{m}_{CHW}	T_{CHWS}	T_{CHWR}	\dot{Q}_{CHW}	\dot{m}_{CHW}	T_{CHWS}	T_{CHWR}
Plant	18.8	12.9	0.3	0.2	9.7	7.4	-0.1	-0.1
Chiller	22.2	15.5	0.2	0.3	8.7	7.4	-0.1	-0.1
Building 1	2.2	0.7	0.2	0.2	0.04	1.1	0.1	0.2
Building 2	2.4	0.1	0.2	0.2	0.02	0.6	-0.01	-0.02
Building 3	3.6	0.4	0.3	0.3	0.02	0.8	0.2	0.2
Building 4	1.3	0.7	0.2	0.2	-0.02	-0.1	0.04	0.04
Building 5	1.6	0.4	0.2	0.2	0.04	0.4	0.08	0.07
Building 6	2.2	0.5	0.2	0.2	-0.05	0.5	0.01	0.01

CVRMSE (Coefficient of Variation of the Root Mean Square Error)

$$\text{CVRMSE} = \frac{\sqrt{\frac{\sum (y_i - \hat{y}_i)^2}{N-1}}}{\bar{y}}$$

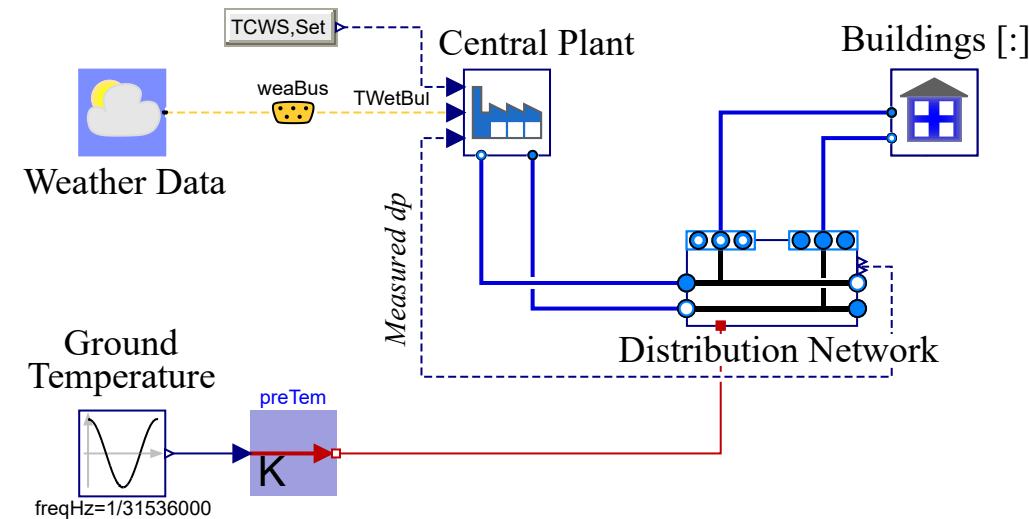
Normalized Mean Bias Error

$$\text{NMBE} = \frac{\sum (y_i - \hat{y}_i)}{(N - 1)\bar{y}}$$

Acceptable range is based on ASHRAE Guideline 14

Model-Based System Optimization

- Condenser Water Supply Temperature
- Condenser Water Flow Rate
- Waterside Economizer



CW Supply Temperature Setpoint Optimization

Optimization Problem

$$\min_{x \in [\underline{x}, \bar{x}]} E_{Pla,i}(T_{CW, set}(x))$$

Total plant energy
Condenser water supply temperature setpoint

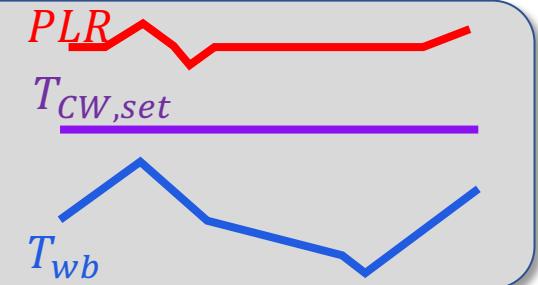
$$E_{Pla,i} = \int (P_{CH}(T_{CW, set}(x), s) + P_{CWP}(T_{CW, set}(x), s) + P_{CHWP}(T_{CW, set}(x), s) + P_{CT}(T_{CW, set}(x), s)) ds$$

Power of the:
(chillers)
(CW pumps)
(CHW pumps)
(cooling towers)

Setpoint Methods

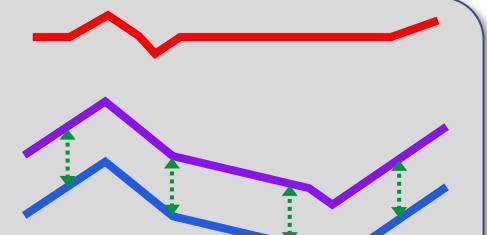
Constant Setpoint

$$T_{CW, set} = \text{constant}$$



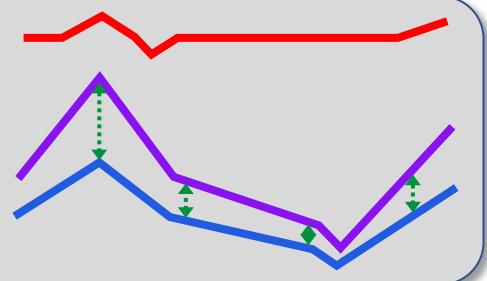
Fixed Approach

$$T_{CW, set} = T_{wb} + T_{app}$$



Adjusted Approach

$$T_{CW, set} = T_{wb} + (T_{app} + r_{PLR} PLR)$$



Results of Optimizing Condenser Water Supply Temperature

$$T_{CW, \text{set}}(x_1) = x_1,$$

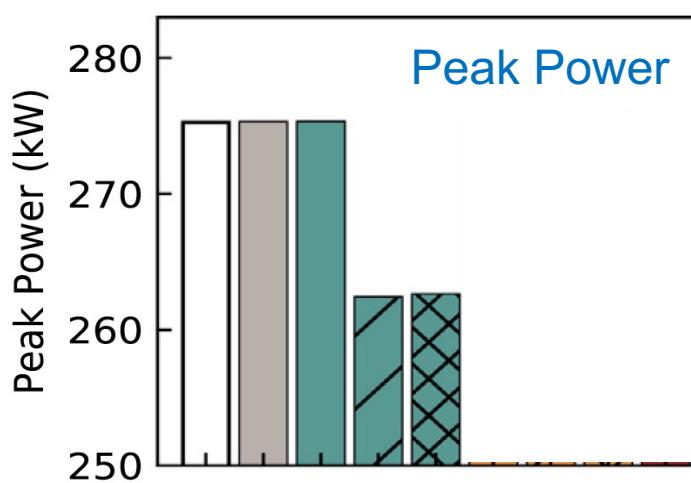
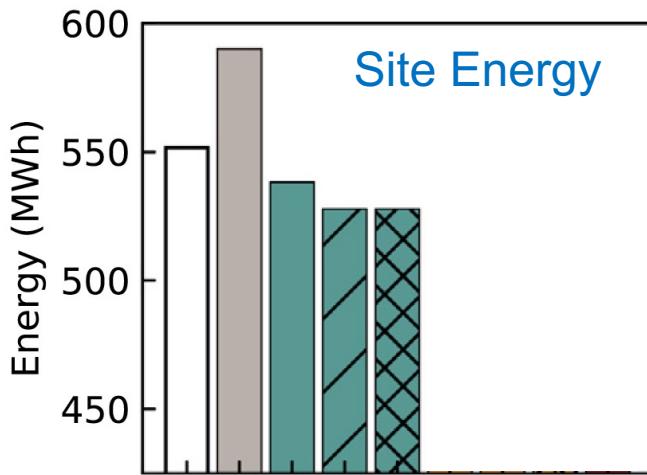
$$T_{CW, \text{set}}(x_1) = T_{wb} + x_1,$$

$$T_{CW, \text{set}}(x_1, x_2) = T_{wb} + x_1 + x_2 \text{ PLR}$$

Table 4
Condenser water supply temperature optimization results.

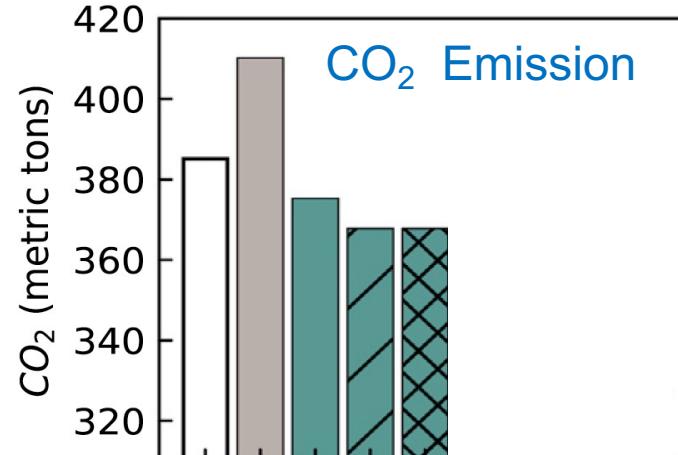
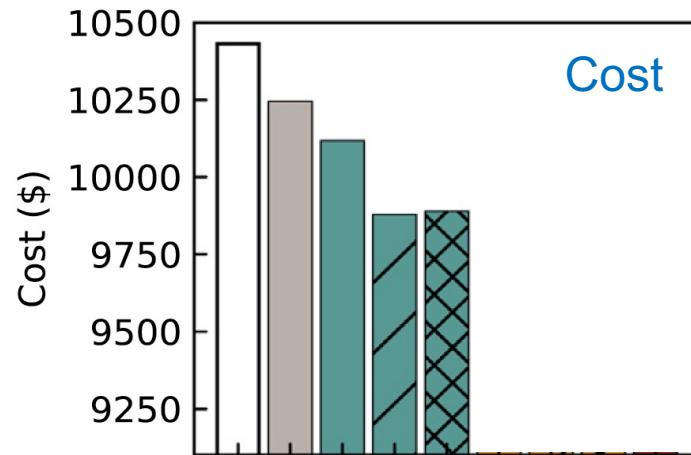
Case	Optimized x		Energy (MWh)	Savings (%)
	Variable	Value		
Baseline (no optimization)	x_1	15.6 °C	551.8	–
Fixed $T_{CW, \text{set}}$	x_1	18.7 °C	537.9	2.5
Fixed T_{app}	x_1	1.9 °C	527.5	4.4
Adjusted T_{app}	x_1	2.1 °C	527.5	4.4
	x_2	-0.44		

Summary of Results

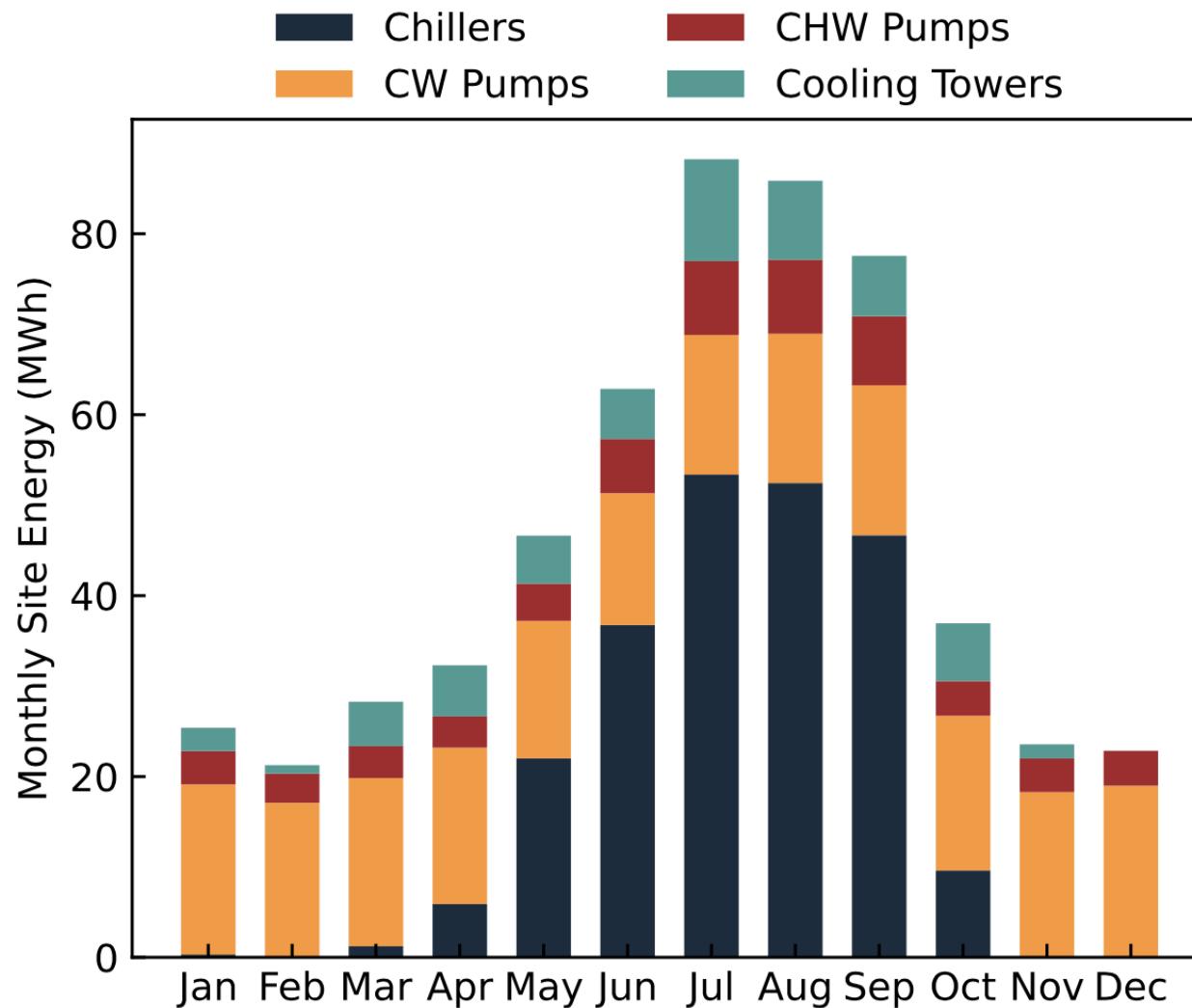


Legend:

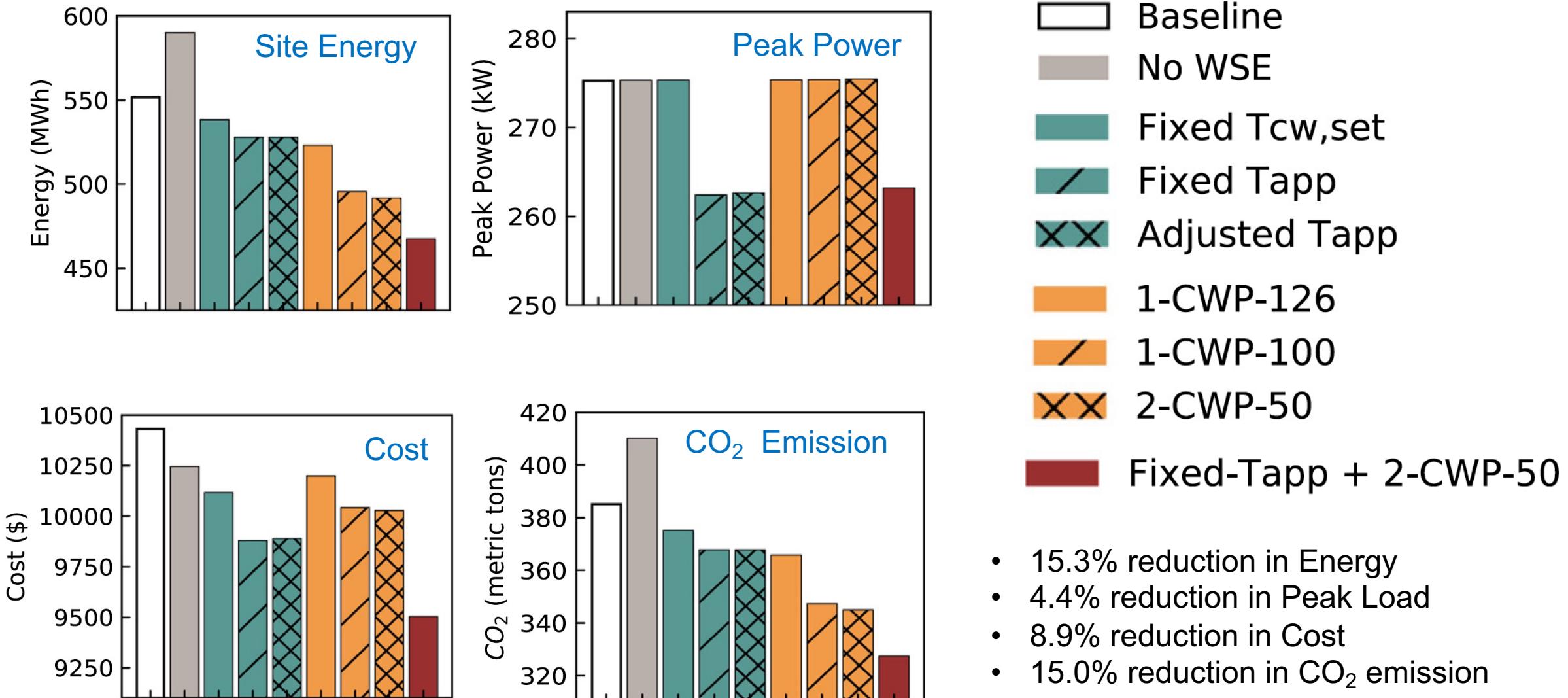
- Baseline
- No WSE
- Fixed Tcw,set
- Fixed Tapp
- Adjusted Tapp



In baseline, pumps contribute significantly to site energy use



Summary of Results: Condenser water pump flow reduction



- 15.3% reduction in Energy
- 4.4% reduction in Peak Load
- 8.9% reduction in Cost
- 15.0% reduction in CO₂ emission

Conclusion

- Developed open source models for the Modelica Buildings for the design and operation of district cooling systems
- Case study shows significant reductions in terms of energy (15.3%), cost (8.9%) and CO₂ emission (15%).

Reference

K. Hinkelmann, J. Wang, W. Zuo, A. Gautier, M. Wetter, C. Fan, N. Long. 2022. "Modelica-Based Modeling and Simulation of District Cooling Systems: A Case Study." Applied Energy, 311, pp.118654.

Questions?

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