

An Introductory Course in Duct Acoustics in the context of Aeroacoustics

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- 1 General Introduction
- 2 Equations and Boundary Conditions
- 3 Duct Modes, Cut-off, Flow Effects
- 4 Circular Ducts: Specific Details
- 5 Surface Waves and Other Behaviour
- 6 Some Applications
- 7 Slowly Varying Modes
- 8 Summary

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Modes in General

MODE =

Modes: LEGO bricks to build general solutions.

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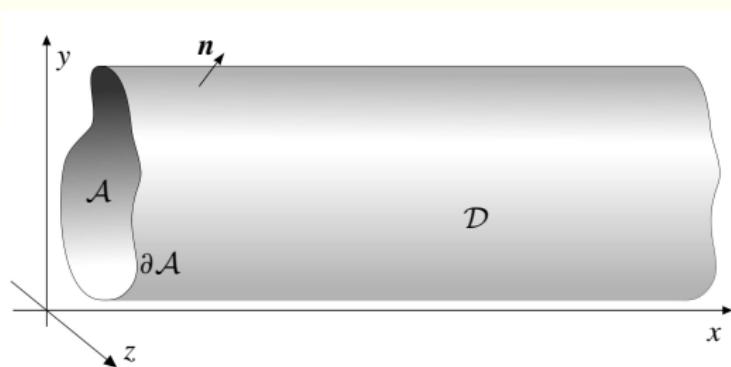
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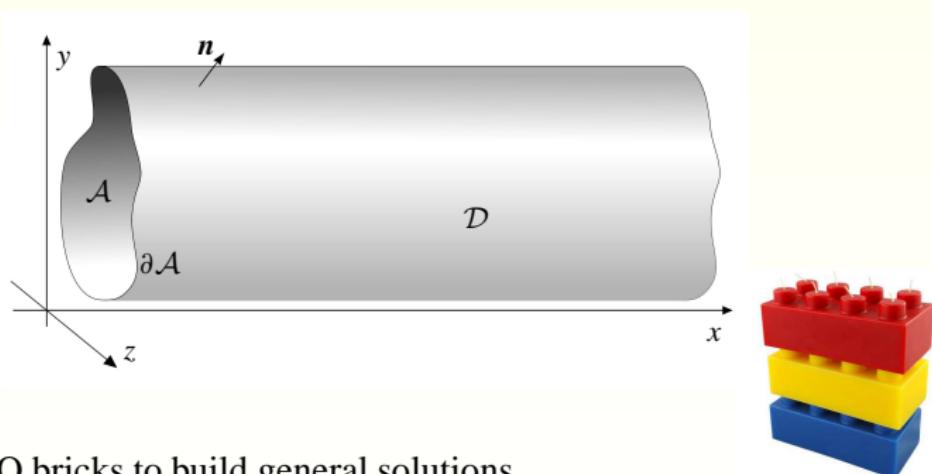
Typical environment: a duct with constant conditions in axial direction:



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Basic example

$$\nabla^2 p - \frac{1}{c_0^2} \frac{\partial^2 p}{\partial t^2} = 0 \quad \& \quad \text{BC}$$

$$p(x, t) = f(x - vt, y, z) = e^{i\omega(t-x/v)} P(y, z) = e^{i\omega t - i\kappa x} P(y, z)$$

$\nabla^2 P + \alpha^2 P = 0$ is analytically solvable for symmetric geometries.

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2D eigenvalue problem: $\nabla^2 P = -\left(\frac{\omega^2}{c_0^2} - \kappa^2\right)P = -\alpha^2 P \quad \& \quad \text{BC}$

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$\nabla^2 P + \alpha^2 P = 0$ is analytically solvable for symmetric geometries.

- For given ω , all possible $\alpha = \alpha_n$ form a countable discrete set.
- By Fourier synthesis in ω we can construct general time dependent “modal” solutions.

$$\psi(x, y, z, t) =$$

With ω -independent BCs and P independent of ω and x

$$p(x, y, z, t) = \sum_{n=0}^{\infty} A(\omega_n) e^{j\omega_n t - k_n(x)} p(y, z)$$

For example: hard walls.

With ω -dependent BCs and P depend on ω and x

$$p(x, y, z, t) = \int_{-\infty}^{\infty} A(\omega) e^{j\omega t - k(\omega)x} P(y, z, \omega) d\omega$$

For example: impedance walls.

- For given ω , all possible $\alpha = \alpha_n$ form a countable discrete set. Corresponding modes P_n, κ_n form a complete basis*, and the general harmonic solution can be constructed as

$$p(x, t) = e^{i\omega t} \sum_n A_n e^{-i\kappa_n x} P_n(y, z)$$

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$\omega = \omega_0$

For example, boundary condition at $x=0$

$$p(0, y, z, t) = \sum_n A_n e^{-i\kappa_n t} P_n(y, z)$$

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For example, impermeable walls:

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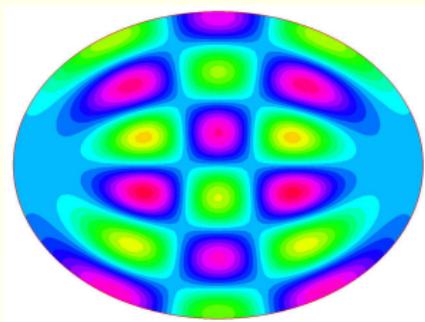
$$p(x, y, z, t) = \int_{-\infty}^{\infty} A(\omega) e^{i\omega t - i\kappa(\omega)x} P(y, z, \textcolor{red}{\omega}) d\omega$$

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Modes: examples

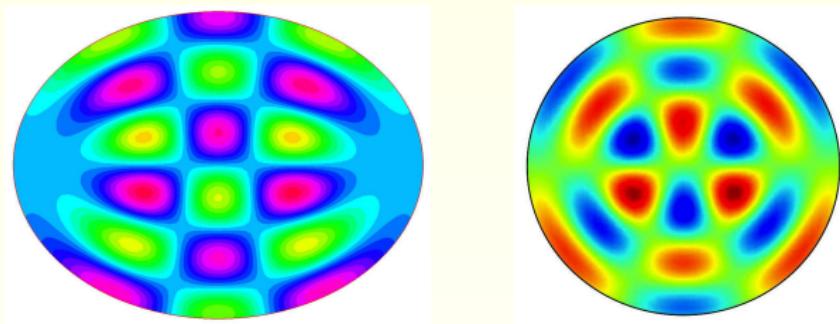
Snapshot of cross section, modal shape function $P(y, z)$



Snapshot of side view, modal pressure distribution $\text{Re}\left(e^{i\omega t - i\kappa x} P(y, z)\right)$

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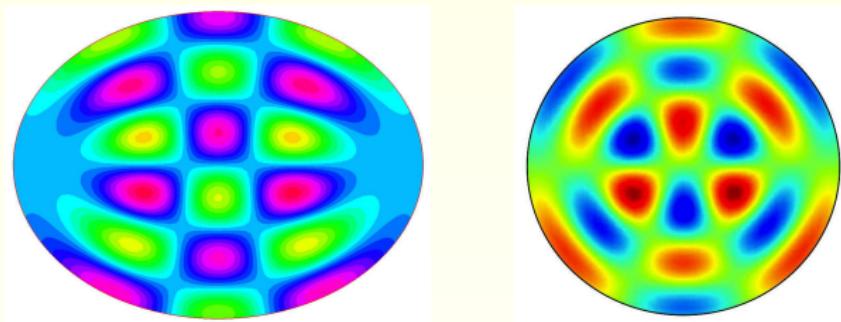
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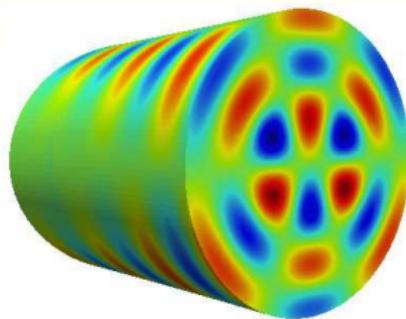
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Mode patterns for decreasing frequency

Propagating modes: real κ large, real κ small

Resonance: $\kappa = 0$. Decaying mode: imaginary κ

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Summary of equations for fluid motion

Acoustics is a branch of fluid mechanics!

$$\text{mass: } \left(\frac{\partial}{\partial t} + \mathbf{v} \cdot \nabla \right) \rho = -\rho \nabla \cdot \mathbf{v}$$

$$\text{momentum: } \rho \left(\frac{\partial}{\partial t} + \mathbf{v} \cdot \nabla \right) \mathbf{v} = -\nabla p + \nabla \cdot \boldsymbol{\tau}$$

$$\text{energy: } \rho T \left(\frac{\partial}{\partial t} + \mathbf{v} \cdot \nabla \right) s = -\nabla \cdot \mathbf{q} + \boldsymbol{\tau} : \nabla \mathbf{v}.$$

where

$$T ds = de + p d\rho^{-1} = dh - \rho^{-1} dp, \quad h = e + \frac{p}{\rho}, \quad p = \rho RT,$$

$$de = C_V dT, \quad dh = C_P dT, \quad ds = C_V \frac{dp}{p} - C_P \frac{d\rho}{\rho}, \quad c^2 = \gamma RT, \quad \gamma = \frac{C_P}{C_V}$$

for a perfect gas (i.e. ideal + C_V, C_P constant).

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for a perfect gas (i.e. ideal + C_V, C_P constant). No heat conduction, no viscosity:

$$\boxed{\frac{d\rho}{dt} = -\rho \nabla \cdot \mathbf{v}, \quad \rho \frac{d\mathbf{v}}{dt} = -\nabla p, \quad \frac{ds}{dt} = 0}$$

Mean flow and small perturbations:

$$\mathbf{v} = \mathbf{V}_0 + \mathbf{v}', \quad p = p_0 + p', \quad \rho = \rho_0 + \rho', \quad s = s_0 + s'$$

linearise,

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$$\nabla \cdot (\rho_0 \mathbf{V}_0) = 0, \quad \rho_0 (\mathbf{V}_0 \cdot \nabla) \mathbf{V}_0 = -\nabla p_0, \quad (\mathbf{V}_0 \cdot \nabla) s_0 = 0,$$

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and

$$\left(\frac{\partial}{\partial t} + \mathbf{V}_0 \cdot \nabla \right) \rho' + \rho' \nabla \cdot \mathbf{V}_0 + \nabla \cdot (\rho_0 \mathbf{v}') = 0$$

$$\rho_0 \left(\frac{\partial}{\partial t} + \mathbf{V}_0 \cdot \nabla \right) \mathbf{v}' + \rho_0 (\mathbf{v}' \cdot \nabla) \mathbf{V}_0 + \rho' (\mathbf{V}_0 \cdot \nabla) \mathbf{V}_0 = -\nabla p'$$

$$\frac{\partial p'}{\partial t} + \mathbf{V}_0 \cdot \nabla p' - c_0^2 \left(\frac{\partial \rho'}{\partial t} + \mathbf{V}_0 \cdot \nabla \rho' + \mathbf{v}' \cdot \nabla \rho_0 \right) = 0$$

With source and force:

Conservation laws

mass: $\frac{\partial}{\partial t} \rho + \nabla \cdot (\rho \mathbf{v}) = q$

momentum: $\frac{\partial}{\partial t} (\rho \mathbf{v}) + \nabla \cdot (\rho \mathbf{v} \mathbf{v}) + \nabla p = \mathbf{f}$

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After linearisation

$$\left(\frac{\partial}{\partial t} + \mathbf{V}_0 \cdot \nabla \right) \rho' + \rho' \nabla \cdot \mathbf{V}_0 + \nabla \cdot (\rho_0 \mathbf{v}') = q'$$

$$\rho_0 \left(\frac{\partial}{\partial t} + \mathbf{V}_0 \cdot \nabla \right) \mathbf{v}' + \rho_0 (\mathbf{v}' \cdot \nabla) \mathbf{V}_0 + \rho' (\mathbf{V}_0 \cdot \nabla) \mathbf{V}_0 + \nabla p' = \mathbf{f}' - \mathbf{V}_0 q'$$

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with $\mathbf{V}_0 = 0$

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$$\rho_0 \frac{\partial}{\partial t} \mathbf{v}' + \nabla p' = \mathbf{f}'$$

Linearise around parallel shear flow:

PARALLEL SHEAR FLOW SATISFIES MEAN FLOW EQUATIONS

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$$\begin{aligned}\mathbf{v} &= U_0(y, z)\mathbf{e}_x + \mathbf{v}', & p &= p_0 + p', \\ \rho &= \rho_0(y, z) + \rho', & c_0 &= c_0(y, z).\end{aligned}$$

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ACOUSTIC FIELD REDUCE TO:

$$D_0^3 p' + 2c_0^2 \frac{\partial}{\partial x} (\nabla U_0 \cdot \nabla p') - D_0 \nabla \cdot (c_0^2 \nabla p') = 0,$$

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*Most general form of acoustic equation in parallel flow.
Includes various special cases.*

Modes: special cases with flow

If we assume MODES $p'(x, y, z, t) = e^{i\omega t - i\kappa x} P(y, z)$ and $\Omega = \frac{\omega - \kappa U_0}{c_0}$, then

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In ducts, κ is determined by the boundary condition (eigenvalue problem).

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Variable sound speed c_0 , no flow ($U_0 = 0$):

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A Lorentz transformation:

Note that the convected wave equation (uniform mean flow)

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$$p'(x, y, z, t) = \tilde{p}(X, y, z, T), \quad x = \beta X, \quad T = \beta t + \frac{M}{c_0 \beta} x, \quad \beta = \sqrt{1 - M^2}$$

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Similar in frequency domain.

Impedance boundary conditions (no mean flow)

Frequency domain:

$$\hat{p} = Z(\omega)(\hat{\mathbf{v}} \cdot \mathbf{n})$$

$$\downarrow \mathbf{n}$$

- Hard wall : $Z = \infty$ and $\hat{\mathbf{v}} \cdot \mathbf{n} = 0$.
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so $Z(\omega) = Z^*(-\omega)$.

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Passive walls : $\operatorname{Re}(Z) \geq 0$ for real ω .

Physical conditions on $Z(\omega)$:

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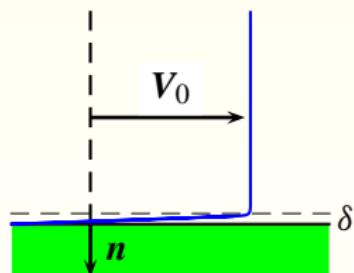
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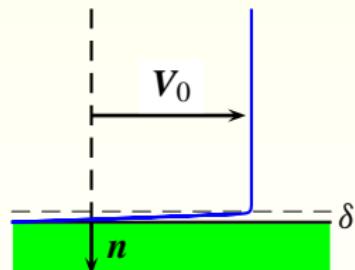
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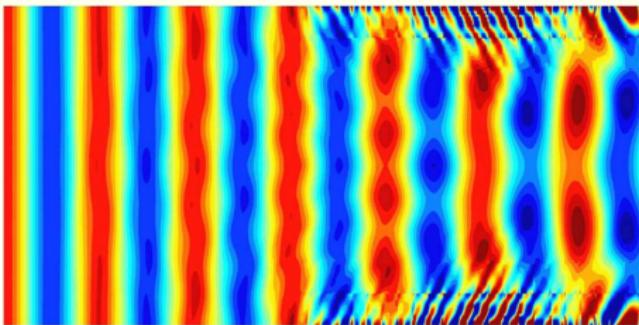
- For a point near the wall but still (just) inside the mean flow (Ingard 1959)



$$i\omega Z(\hat{\mathbf{v}} \cdot \mathbf{n}) = \left(i\omega + U_\infty \frac{\partial}{\partial x} \right) \hat{p}$$

Instability problems

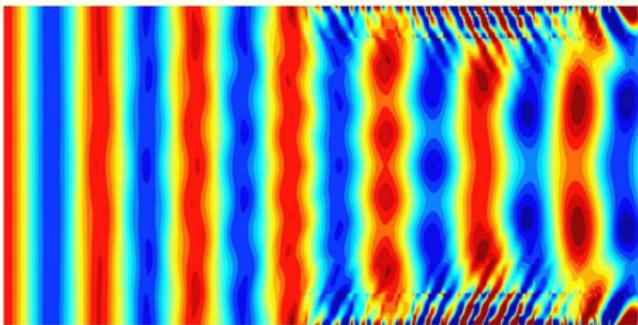
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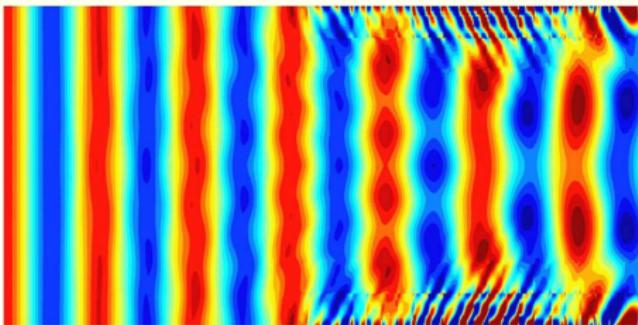
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Outline

- 1 General Introduction
- 2 Equations and Boundary Conditions
- 3 Duct Modes, Cut-off, Flow Effects
- 4 Circular Ducts: Specific Details
- 5 Surface Waves and Other Behaviour
- 6 Some Applications
- 7 Slowly Varying Modes
- 8 Summary

Duct modes 2D, no flow

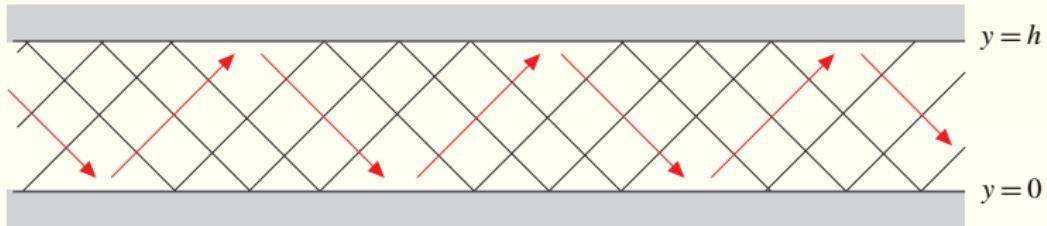
$$p(x, y, z, t) = e^{i\omega t - i\kappa x} P(y), \rightarrow P_{yy} + (k^2 - \kappa^2)P = 0, k = \frac{\omega}{c_0}$$

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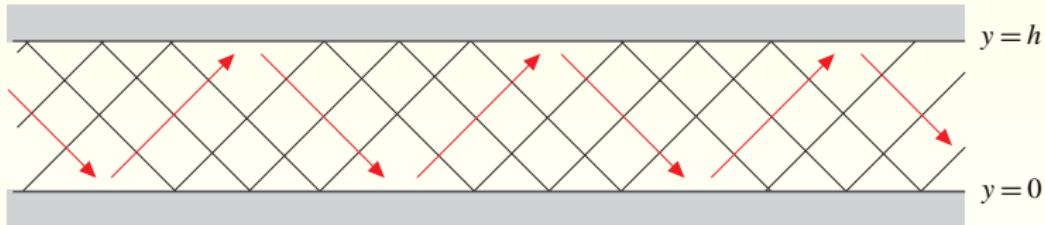


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Hard wall boundary conditions:

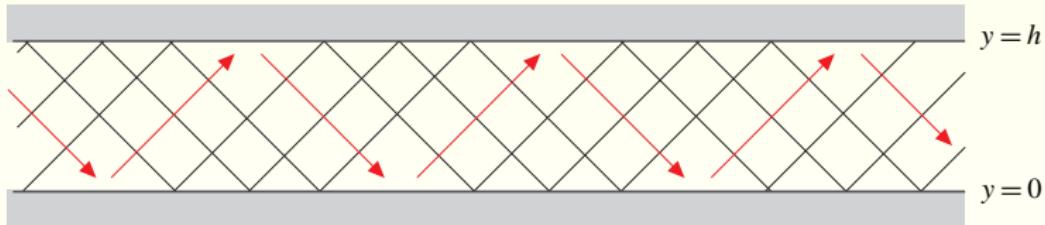
$$\left. \begin{aligned} p(x, y, t) &= e^{i\omega t - i\kappa x} (A e^{i\gamma y} + B e^{-i\gamma y}) \\ p_y(x, 0, t) &\sim i\gamma(A - B) = 0 \\ p_y(x, h, t) &\sim i\gamma(A e^{i\gamma h} - B e^{-i\gamma h}) = 0 \end{aligned} \right\} \rightarrow \begin{cases} \sin(\gamma h) = 0 \rightarrow \gamma h = n\pi \\ \rightarrow \kappa_n^\pm = \pm \frac{1}{h} \sqrt{\left(\frac{\omega h}{c_0}\right)^2 - n^2 \pi^2} \end{cases}$$

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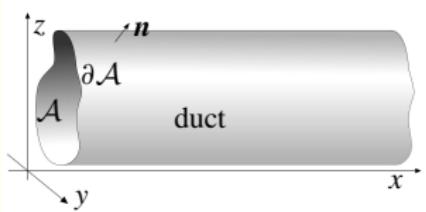


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Helmholtz Number

... generalised to 3D (no flow)

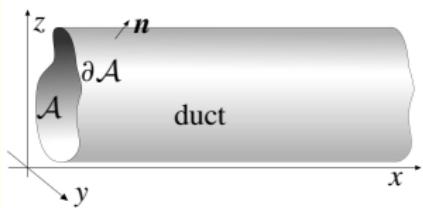


$$\nabla^2 p + k^2 p = 0, \text{ with } Z(\nabla p \cdot \mathbf{n}) = -i\omega\rho_0 p \text{ at wall}$$

with $k = \frac{\omega}{c_0}$ and solution

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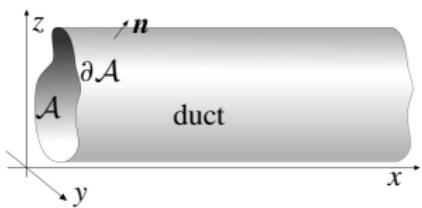
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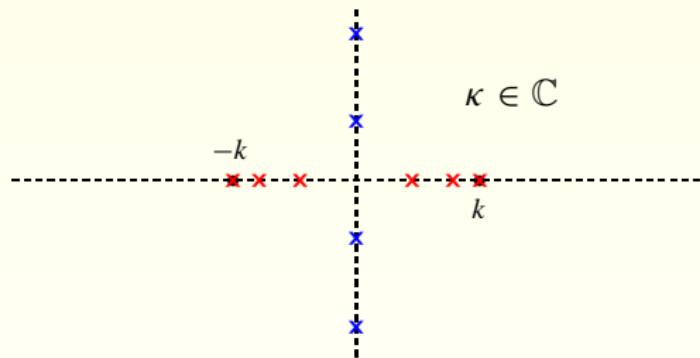
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Note that $Z = Z(\omega)$ and therefore $\alpha = \alpha(\omega)$, $\psi = \psi(y, z; \omega)$.

Physical relevance of modal wave numbers

Hard walls: all α_n real $\rightarrow \kappa_n = \pm\sqrt{k^2 - \alpha_n^2}$: real (**cut-on**) or imaginary (**cut-off**)

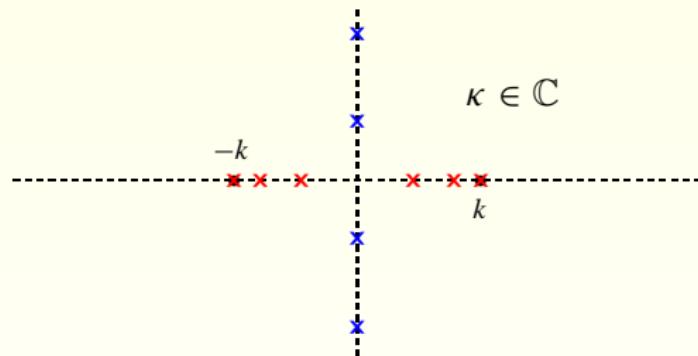
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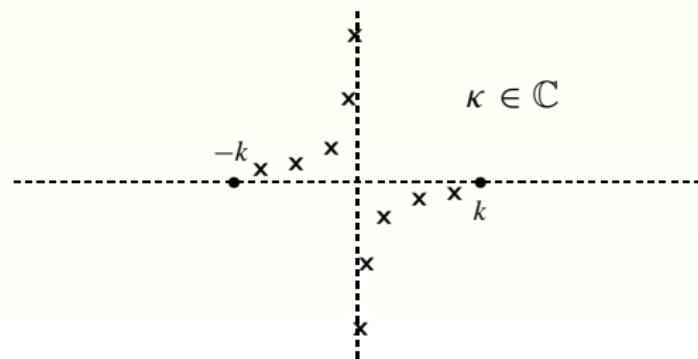
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Soft walls: α_n and κ_n complex



Low frequency waves in ducts

Hard-walled ducts allow **plane wave** solutions (right-, left-running):

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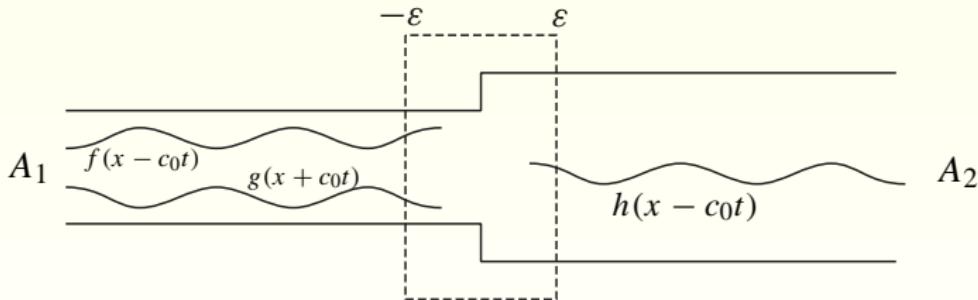
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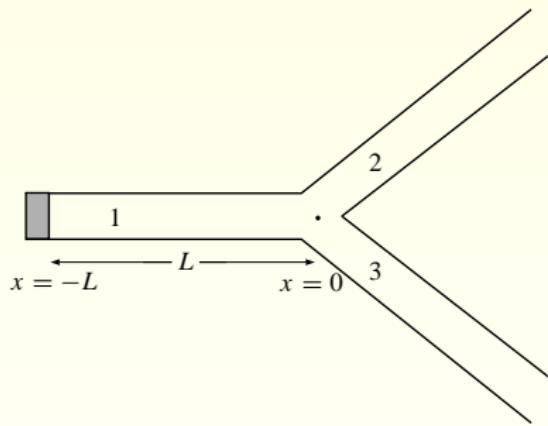


Coupling is approximately possible:

$$\int_{-\varepsilon}^{\varepsilon} [i\omega\rho_0 \mathbf{v} + \nabla p] dx = 0 \quad \Rightarrow \quad p(\varepsilon) - p(-\varepsilon) \rightarrow 0$$

$$\iiint_{\Omega} [i\omega\rho + \rho_0 \nabla \cdot \mathbf{v}] dx = 0 \quad \Rightarrow \quad A_2 \rho_0 u(\varepsilon) - A_1 \rho_0 u(-\varepsilon) \rightarrow 0$$

Example: piston at $x = -L$ and branching at $x = 0$



$$1 : \quad p_1 = f_1(x - c_0 t) + g_1(x + c_0 t)$$

$$2 : \quad p_2 = f_2(x - c_0 t)$$

$$3 : \quad p_3 = f_3(x - c_0 t)$$

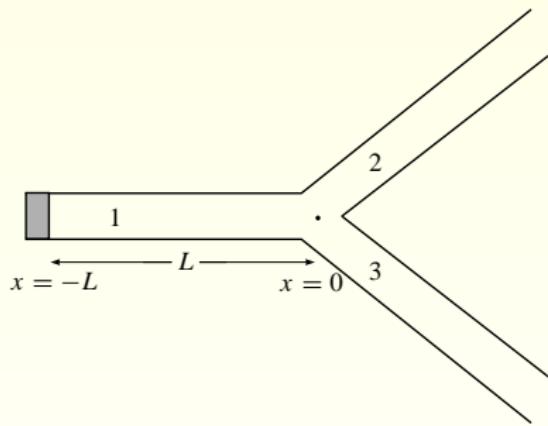
$$\text{at } x = -L : \quad \frac{1}{\rho_0 c_0} (f_1(-L - c_0 t) - g_1(-L + c_0 t)) = U_0(t)$$

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results into

$$f_1(-c_0 t) = -3g_1(c_0 t) = \frac{3}{2}f_2(-c_0 t) = \rho_0 c_0 \sum_{n=0}^{\infty} \left(-\frac{1}{3}\right)^n U_0\left(t - (2n+1)\frac{L}{c_0}\right)$$

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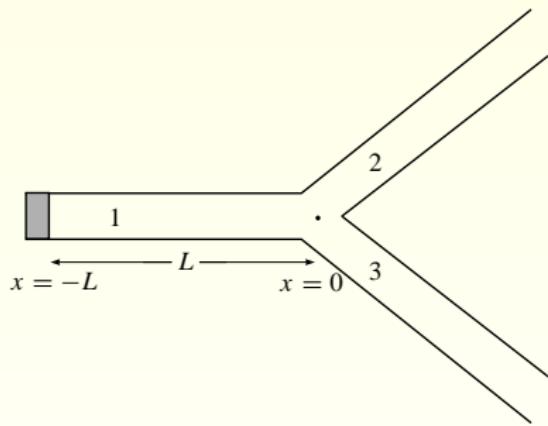
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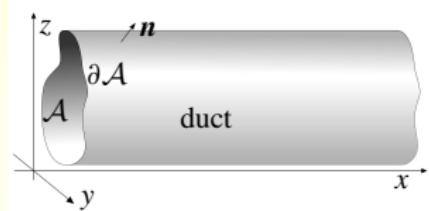
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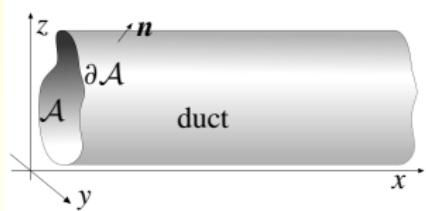
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Straight duct with uniform mean flow



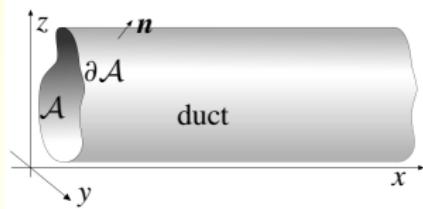
$$\nabla^2 p - \frac{1}{c_0^2} \left(i\omega + U_0 \frac{\partial}{\partial x} \right)^2 p = 0, \text{ with } i\omega Z(\nabla p \cdot \mathbf{n}) = -\rho_0 \left(i\omega + U_0 \frac{\partial}{\partial x} \right)^2 p \text{ at wall.}$$

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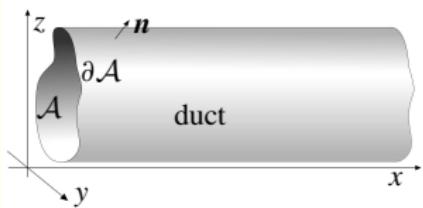


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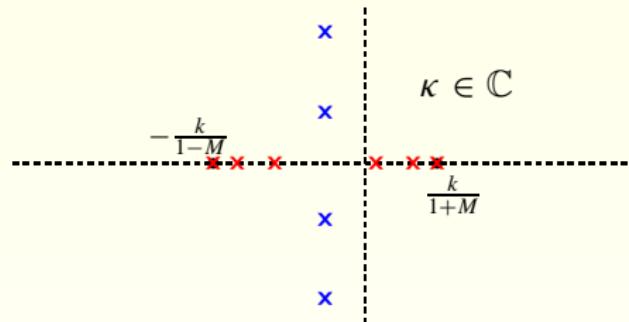
$$\nabla^2 \psi_n = -\alpha_n^2 \psi_n, \quad (+ \text{B.C. containing } \alpha_n)$$

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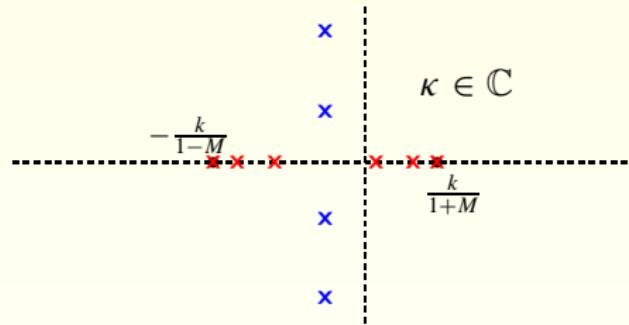
Modal wave numbers

Hard walls: all α_n real $\rightarrow \kappa_n$ real (cut-on) or (Doppler-shifted) complex (cut-off)

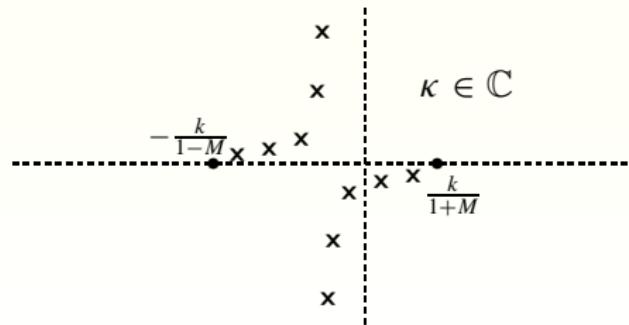


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Soft walls: α_n and κ_n complex (but there is more ...)



Modal Phase and Group Velocity (no flow)

(Wave crest and signal velocity.)

- Hard wall, cut-on.
- $v_f v_g = c_0^2$.
- Dispersion (a wave packet breaks up into distinct modes) $\longleftrightarrow v_f \neq v_g$.
 - $v_f^- < -c_0 < v_g^- < 0$ on $\alpha_n c_0 < \omega$
 $0 < v_g^+ < c_0 < v_f^+$ on $\alpha_n c_0 < \omega$
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Modal Phase and Group Velocity (with flow)

(Wave crest and signal velocity.)

- Hard wall, cut-on.
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$$\nabla^2 \psi = \frac{\partial^2}{\partial r^2} \psi + \frac{1}{r} \frac{\partial}{\partial r} \psi + \frac{1}{r^2} \frac{\partial^2}{\partial \theta^2} \psi = -\alpha^2 \psi$$

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$$F'' + \frac{1}{r} F' + \left(\alpha^2 - \frac{m^2}{r^2} \right) F = 0 \quad \rightarrow \quad F(r) = J_m(\alpha r)$$

Circular symmetry $\psi = F(r)G(\theta)$: modes explicitly given by:

$$\psi_n(y, z) = J_m(\alpha_{m\mu} r) e^{-im\theta}, \quad m \in \mathbb{Z}.$$

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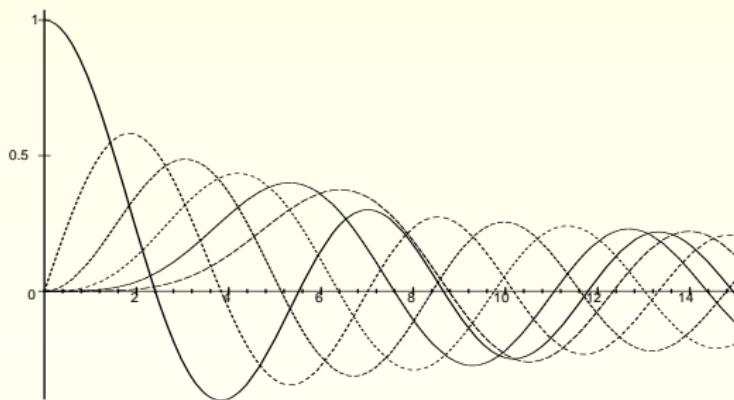
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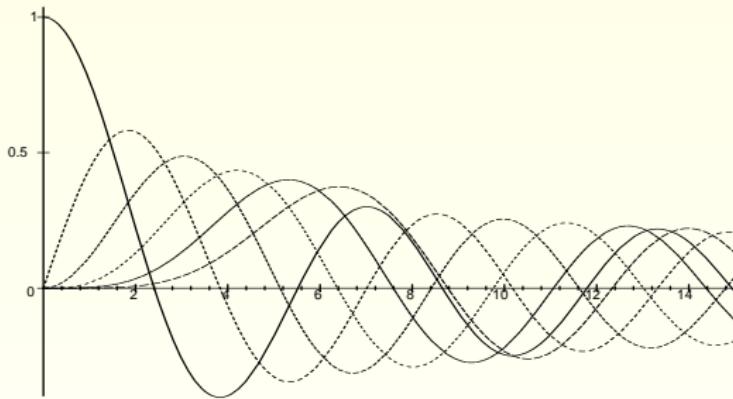
$$i\omega Z\alpha_{m\mu}J'_m(\alpha_{m\mu}R) = (\omega - U_0\kappa_{m\mu})^2 J_m(\alpha_{m\mu}R) \rightarrow \alpha_{m\mu}(Z)$$

Bessel functions



Bessel function $J_m(x)$ as function of order and real argument.

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Note: $j'_{01} = 0$, $j'_{m1} \simeq m + 0.8m^{\frac{1}{3}}$ \rightarrow $\kappa_{m1} = \sqrt{k^2 - \alpha_{m1}^2} = \frac{1}{R} \sqrt{\left(\frac{\omega R}{c_0}\right)^2 - j'_{m1}^2}$

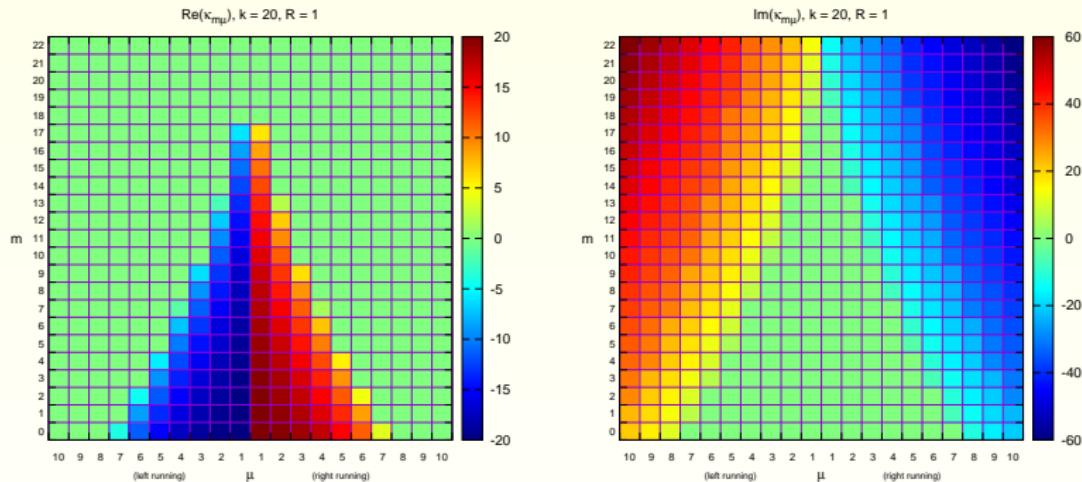
$m\text{-modes cut-on if } \frac{\omega R}{c_0} \gtrsim m$

Surface of constant modal phase

Surface of constant phase $\omega t - m\vartheta - \operatorname{Re}(\kappa_{m\mu})x = \text{constant}$

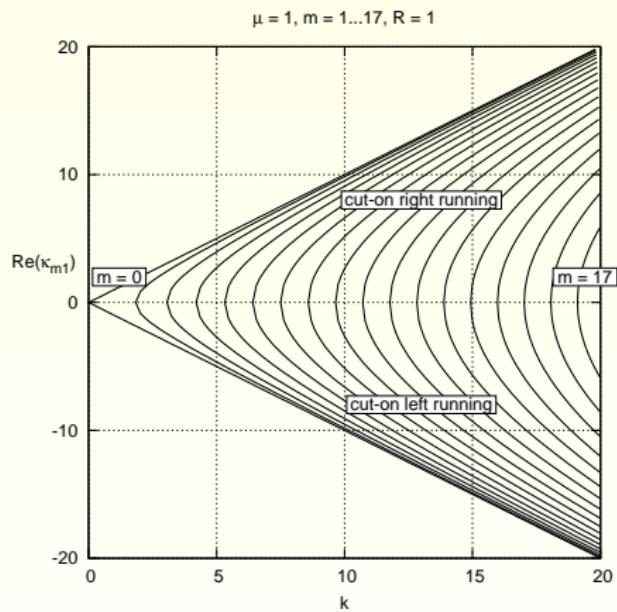
is a spiralling helicoid of pitch $2\pi m / \operatorname{Re}(\kappa_{m\mu})$

Modal wave numbers $\kappa_{m\mu}$, no flow



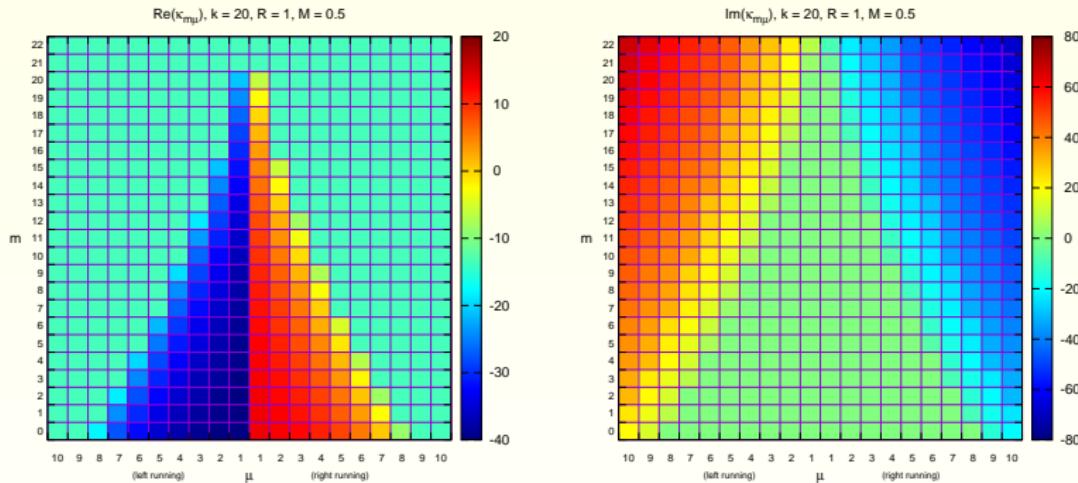
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Modal wave numbers $\kappa_{m1}(k)$, no flow



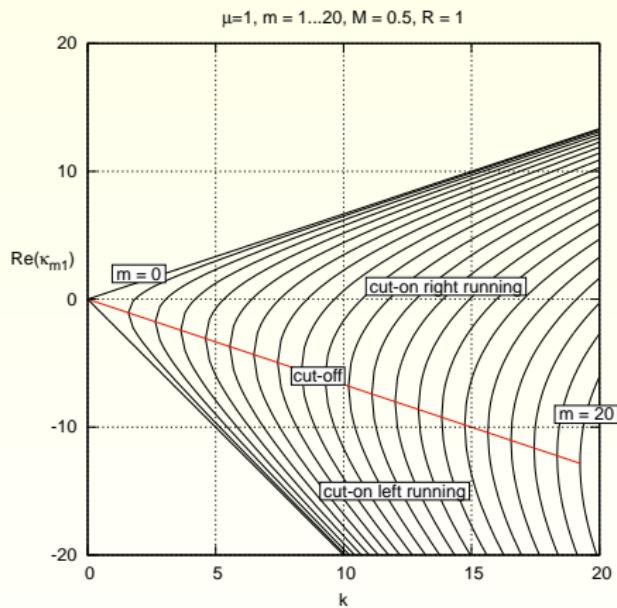
$$\kappa_{m1} = \pm \sqrt{k^2 - \alpha_{m1}^2}$$

Modal wave numbers $\kappa_{m\mu}$, with flow



$$\kappa_{m\mu} = -\frac{kM}{\beta^2} \pm \frac{1}{\beta^2} \sqrt{k^2 - \beta^2 \alpha_{m\mu}^2}$$

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Normalised modes

A complete solution may be written like

$$p(x, r, \vartheta) = \sum_{m=-\infty}^{\infty} \sum_{\mu=1}^{\infty} (A_{m\mu} e^{-i\kappa_{m\mu}x} + B_{m\mu} e^{i\kappa_{m\mu}x}) U_{m\mu}(r) e^{-im\vartheta}.$$

$$U_{m\mu}(r) = N_{m\mu} J_m(\alpha_{m\mu} r), \quad N_{m\mu} = \left\{ \frac{1}{2}(R^2 - m^2/\alpha_{m\mu}^2) J_m(\alpha_{m\mu} R)^2 \right\}^{-\frac{1}{2}}$$

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In a hard-walled duct $U_{m\mu} e^{-im\theta}$ are **orthogonal**. Normalise such that:

$$\int_0^{2\pi} \int_0^R U_{m\mu}(r) e^{-im\theta} U_{n\nu}(r) e^{in\theta} r dr = 2\pi \delta_{\mu\nu} \delta_{mn}$$

Modal power in hard-walled duct

Time-averaged axial intensity

$$\langle \mathbf{I} \cdot \mathbf{e}_x \rangle = \frac{1}{2} \operatorname{Re}(pu^*)$$

Transmitted acoustic power

$$\mathcal{P} = \frac{\pi}{\rho_0 \omega} \sum_{m=-\infty}^{\infty} \sum_{\mu=1}^{\infty} \left(\operatorname{Re}(\kappa_{m\mu}) (|A_{m\mu}|^2 - |B_{m\mu}|^2) + 2 \operatorname{Im}(\kappa_{m\mu}) \operatorname{Im}(A_{m\mu}^* B_{m\mu}) \right).$$

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Tunneling: all modes cut-off & $\mathcal{P} \neq 0$.

Source expansion

If $p(0, t, \theta) = p_0(r, \theta)$ is source in hard-walled duct, then for $x > 0$

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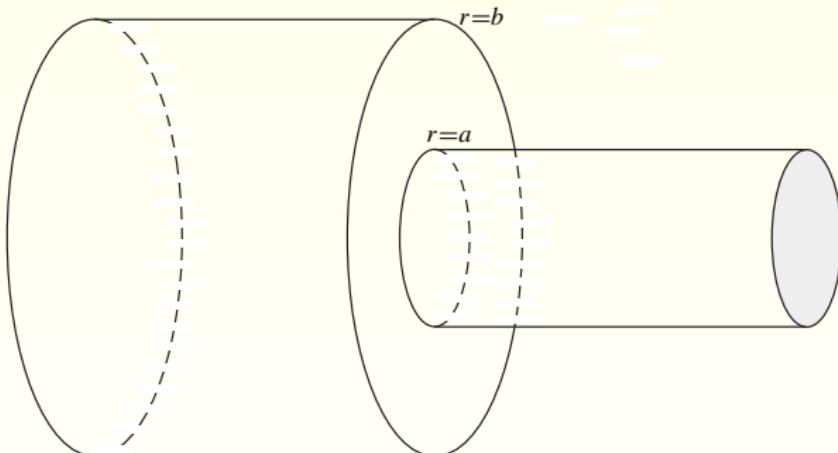
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A finite number of modes (cut-on modes) survive at large distances.
Just 1 mode if $kR \ll 1$: only A_{01} important.

Mode matching

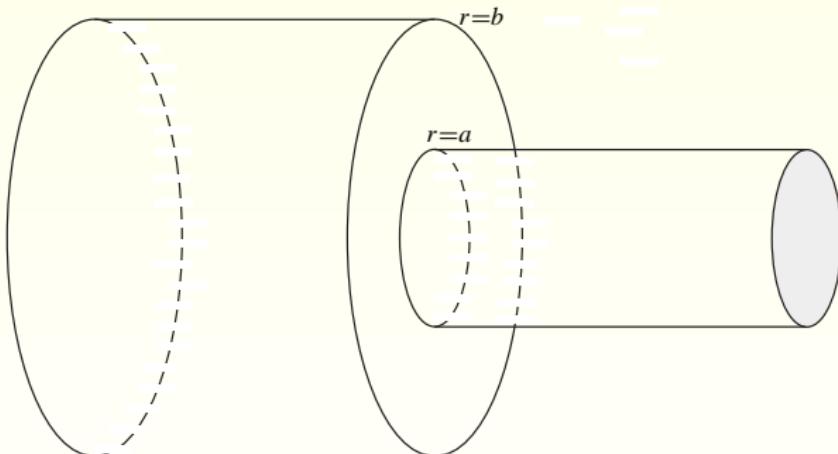
Using similar ideas we can connect duct sections by mode matching



Duct with discontinuous diameter

Mode matching

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Duct with discontinuous diameter

*Exercise in linear algebra. Know how to **chop off** the infinite series*

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Surface wave behaviour (“strange modes”)

From the asymptotic behaviour

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$\operatorname{Im}(\alpha_{m\mu} R) \sim 1$: $J_m(\alpha_{m\mu} r)$ oscillatory, $O(1)$: **acoustic wave.**

Surface wave behaviour (“strange modes”)

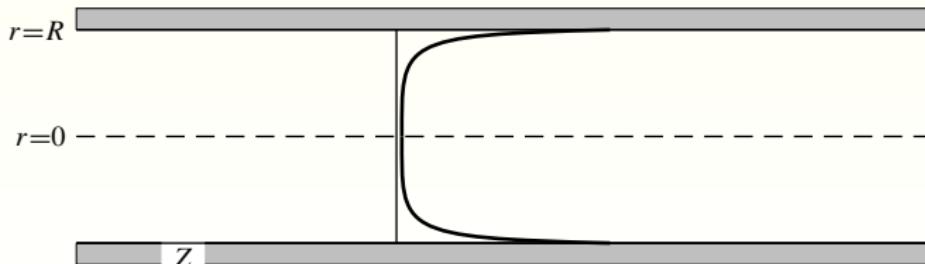
From the asymptotic behaviour

$$J_m(\alpha_{m\mu} r) \simeq \frac{e^{i\alpha_{m\mu} r - \frac{1}{2}m\pi i - \frac{1}{4}\pi i}}{\sqrt{2\pi\alpha_{m\mu} r}}, \quad \alpha_{m\mu} r \rightarrow \infty, \operatorname{Im} \alpha_{m\mu} < 0$$

we have typically two classes of behaviour:

$\operatorname{Im}(\alpha_{m\mu} R) \sim 1$: $J_m(\alpha_{m\mu} r)$ oscillatory, $O(1)$: **acoustic wave.**

$\operatorname{Im}(\alpha_{m\mu} R) \gg 1$: $J_m(\alpha_{m\mu} r)$ exponentially close to $r = R$: **surface wave.**



Mode patterns for impedance walls

Acoustic modes: $\text{Im } \alpha$ not large ($Z = 1.2 - 0.5i$, $M = 0$ and $Z = 0.4 - 0.4i$, $M = 0.5$)

Surface waves: $\text{Im } \alpha$ large ($Z = 0.001 - i$, $M = 0$ and $Z = 0.4 - 6i$, $M = -0.5$)

In more detail: we recall the theory for lined wall modes

equations

$$\begin{cases} \left(ik + M \frac{\partial}{\partial x}\right)^2 p(x, r, \theta) - \nabla^2 p(x, r, \theta) = 0 \\ r = 1 : \quad \rho_0 c_0 \left(ik + M \frac{\partial}{\partial x}\right)^2 p = -ikZ \frac{\partial}{\partial r} p \end{cases}$$

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eigenvalues

$$\begin{cases} \alpha = \sqrt{(k - M\kappa)^2 - \kappa^2}, \quad \text{Im } \alpha \leq 0 \\ \rho_0 c_0 (k - M\kappa)^2 J_m(\alpha) - ikZ \gamma J'_m(\alpha) = 0 \quad \rightarrow \quad \kappa_{m\mu} \in \mathbb{C} \end{cases}$$

Lorentz Transformation

LORENTZ TRANSFORMATION CLEANS UP THE FORMULAS:

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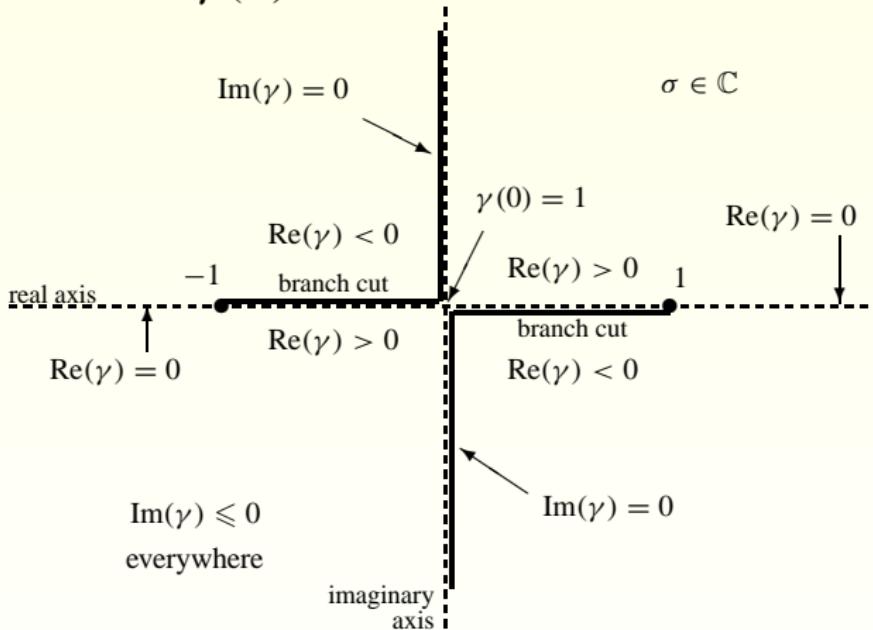
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Important square root γ

SQUARE ROOT $\gamma(\sigma)$ WITH BRANCH CUTS AND BRANCH:



Branch cuts and signs of $\gamma = \sqrt{1 - \sigma^2}$ in complex σ -plane.

The **definition** adopted is the branch with $\text{Im}(\gamma) \leq 0$ for all σ .

$\text{Im}(\gamma) = 0$ along the branch cuts. $\gamma(\sigma) \simeq -i\sigma \text{ sign}(\text{Re } \sigma)$ if $|\sigma| \gg 1$.

Analysis of eigenvalue equation: with/without flow

$$(1 - M\sigma)^2 - i\beta^3 \tilde{Z}\gamma \frac{J'_m(\Omega\gamma)}{J_m(\Omega\gamma)} = 0$$

$\text{Im}(\Omega\gamma) = O(1)$: $J_m(\Omega\gamma), J_m(\Omega\gamma r)$ oscillatory, $O(1)$: **acoustic wave**.

$\text{Im}(\Omega\gamma) \gg O(1)$: $J_m(\Omega\gamma) \sim$ exponentially increasing: $\frac{J'_m(\Omega\gamma)}{J_m(\Omega\gamma)} \rightarrow i$
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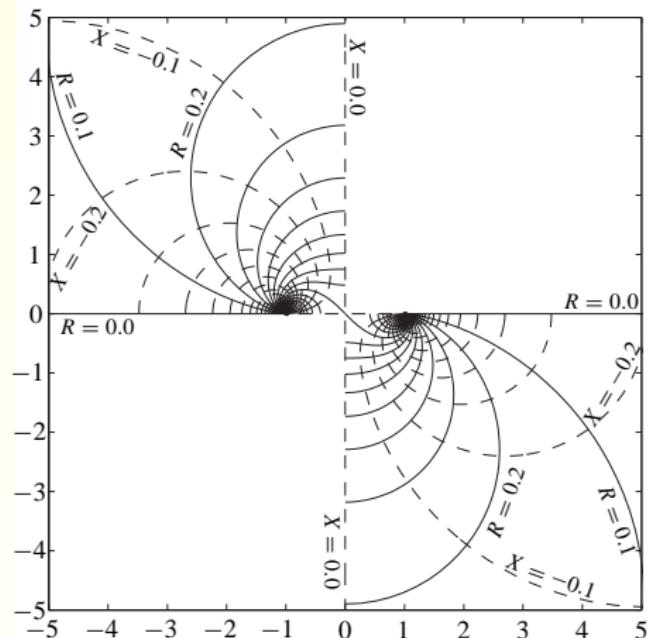
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Occurrence only for $\text{Im } \tilde{Z} \leq 0$, existence in 2nd and 4th quadrant of σ -plane.

Trajectories of 2D surface waves (no flow):



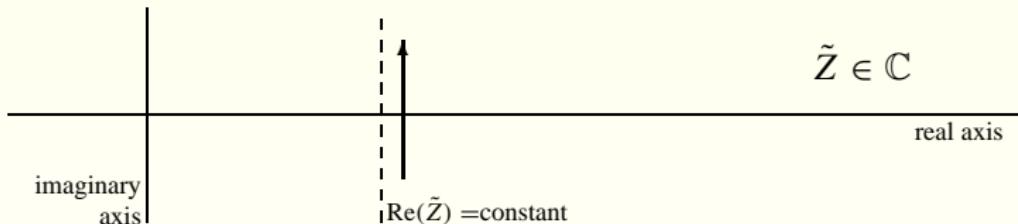
Trajectories of σ for varying $\tilde{Z} = R + iX; M = 0$.

Fixed R & $X = 0 : -0.1 : -\infty$ —————

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Behaviour of duct modes $\kappa_{m\mu}$ as a function of \tilde{Z} .

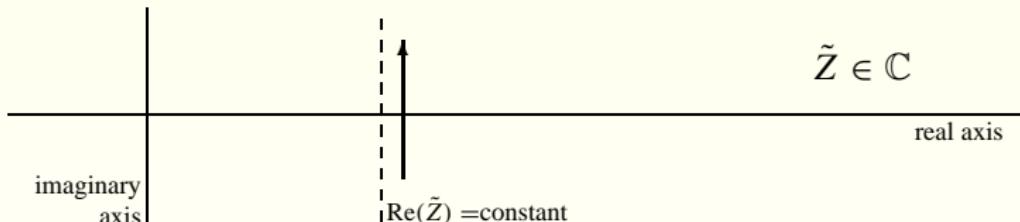
Consider \tilde{Z} along paths parallel to imaginary axis, i.e., $\text{Re}(\tilde{Z}) = \text{constant}$:



Complex impedance plane.

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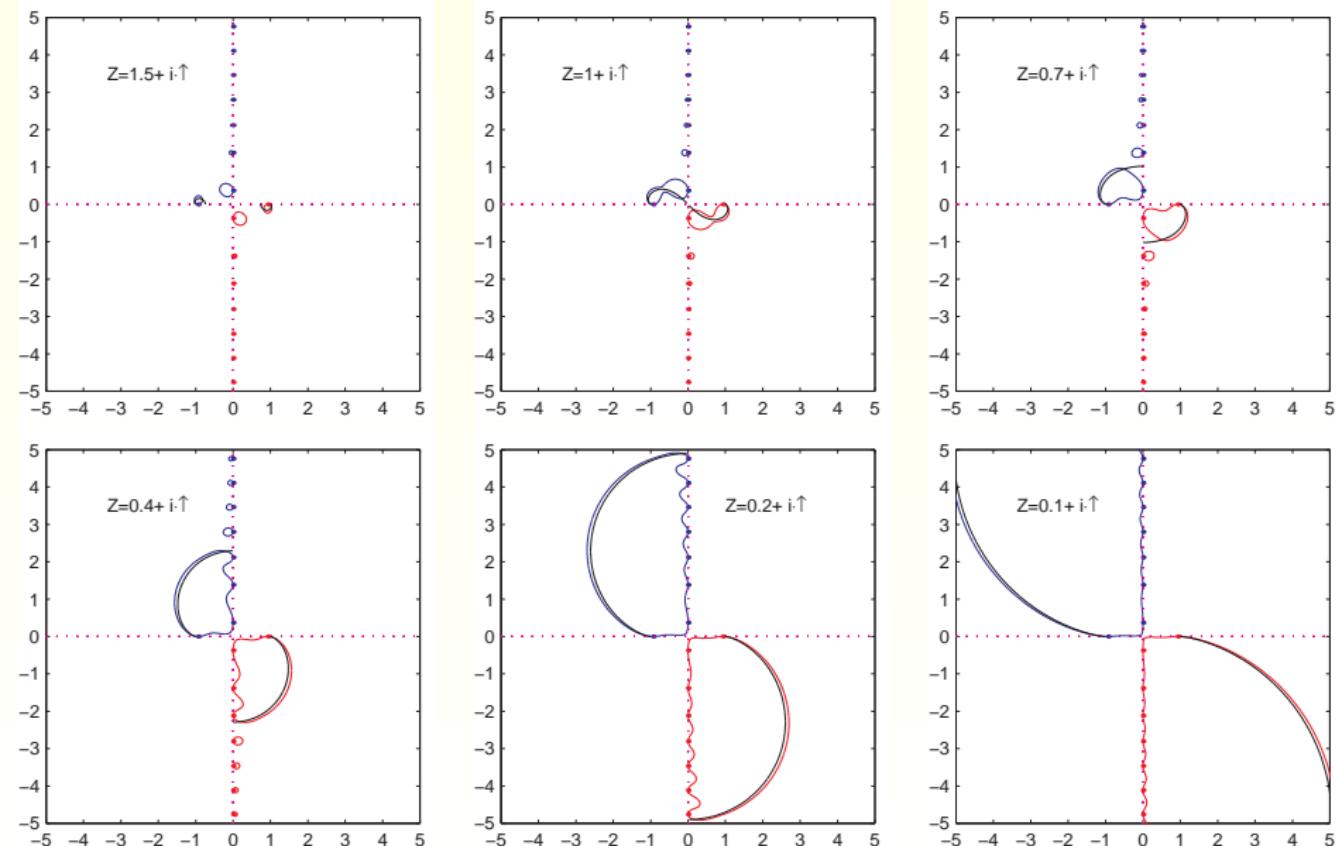
Trajectories of all $\kappa_{m\mu}$ for $\text{Re}(\tilde{Z})$ fixed, $-\infty < \text{Im}(\tilde{Z}) < \infty$:

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Tracing axial wave numbers $\kappa_{m\mu}$

9 left- and right-running modes with $\omega = 5$, $m = 1$, $M = 0$ tracing along $\text{Re } \tilde{Z} = 0.4$

Trajectories of $\kappa_{m\mu}$ for several $\text{Re } \tilde{Z}$



Analysis of 2D eigenvalue equation

ANALYSIS WITH FLOW IS MORE INVOLVED:

- $\operatorname{Re}(\tilde{Z}) = -\operatorname{Re}\left[\frac{(1-M\sigma)^2}{\beta^3\gamma}\right] = 0$, border of possible impedances.
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ANALYSIS WITH FLOW IS MORE INVOLVED:

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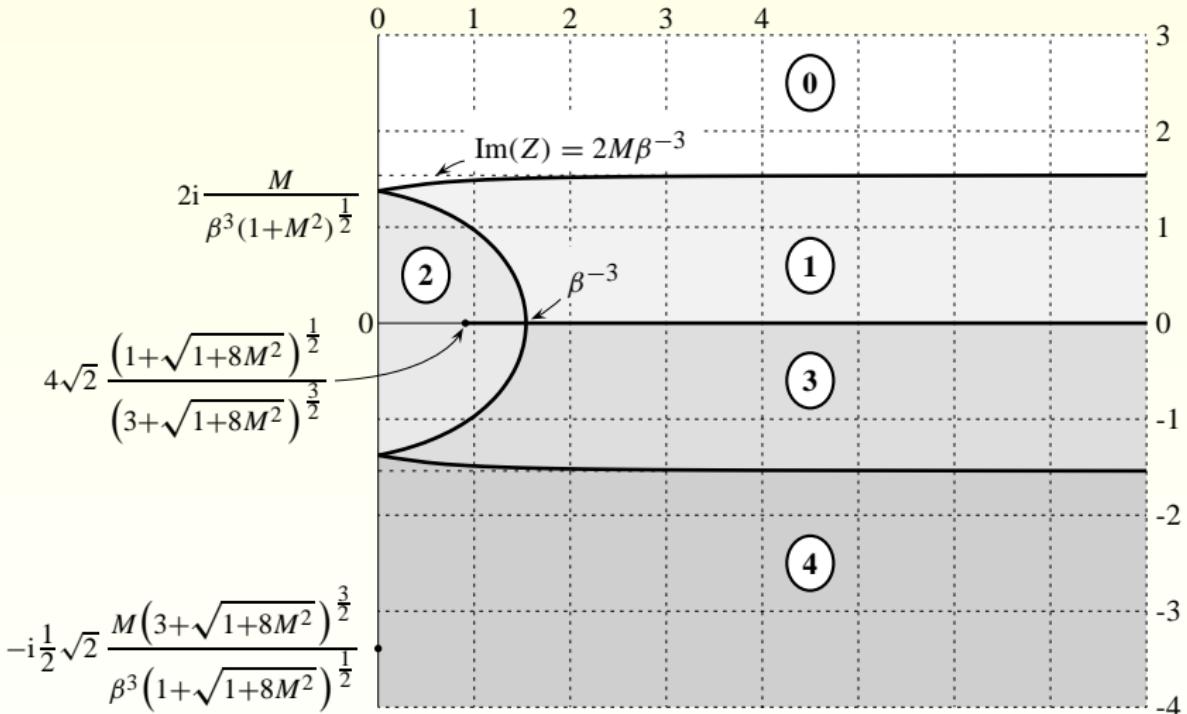
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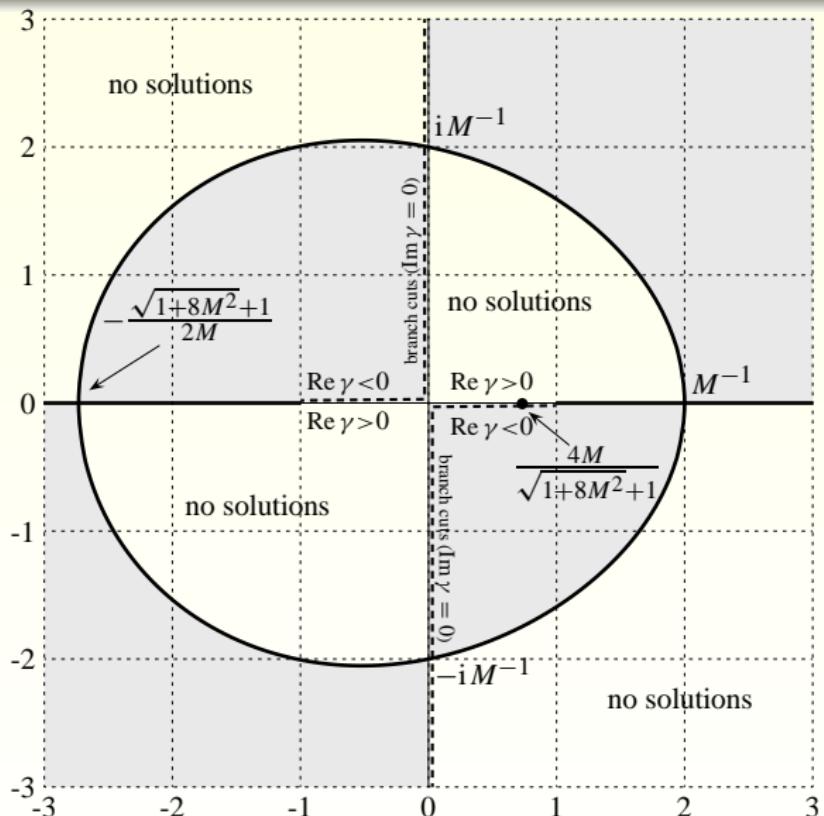
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Regions in \tilde{Z} -plane of occurrence of 2D surface waves



Regions of different numbers of surface waves in complex \tilde{Z} -plane ($M = 0.5$)
Thick lines map to the branch cuts in σ -plane.

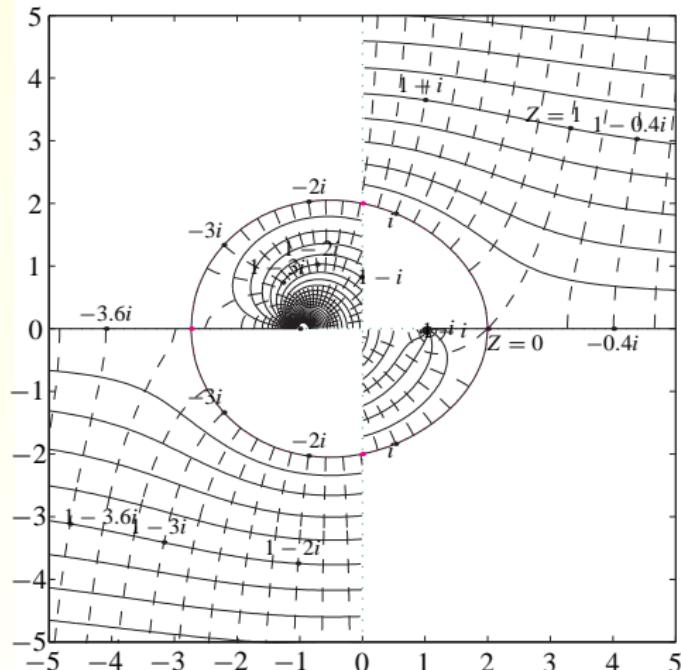
Regions of existence of 2D surface waves in σ -plane



Regions of existence of surface waves σ ($M = 0.5$)

Thick lines map to the imaginary \tilde{Z} -axis.

Trajectories of 2D surface waves (with flow):



Trajectories of σ for varying $\tilde{Z} = R + iX; M = 0.5$

Fixed R & $X = -\infty : 0.2 : \infty$ —————

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Trajectories of modes $\kappa_{m\mu}$ with mean flow for several $\text{Re } \tilde{Z}$

THE FULL MODAL PICTURE WITH MEAN FLOW

Trajectories of modes $\kappa_{m\mu}$ with mean flow for several $\text{Re } \tilde{Z}$

THE FULL MODAL PICTURE WITH MEAN FLOW

Remember the Lorentz transform to make the pictures clean:

$$\kappa_{m\mu}^{\pm} = \frac{k(\pm\sigma_{m\mu} - M)}{\beta^2}, \quad \sigma_{m\mu} = \sqrt{1 - \gamma_{m\mu}^2}, \quad \gamma_{m\mu} = \frac{\beta\alpha_{m\mu}}{k}$$

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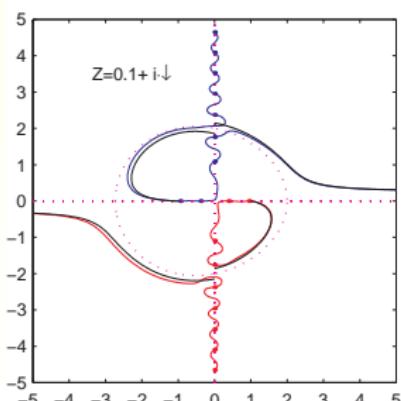
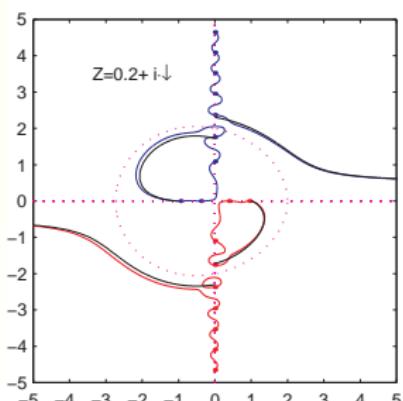
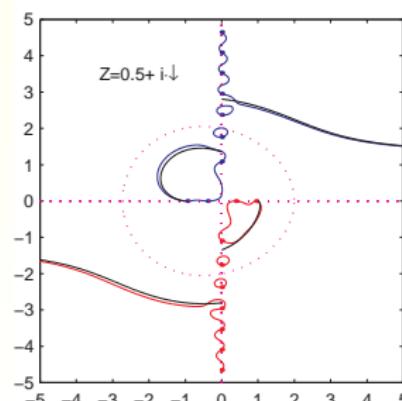
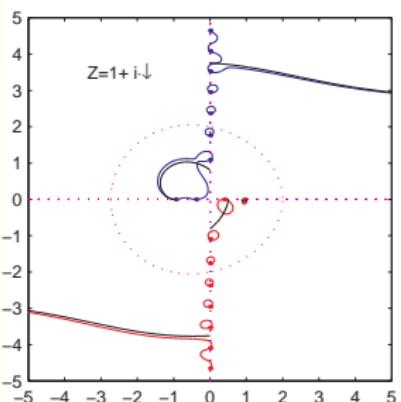
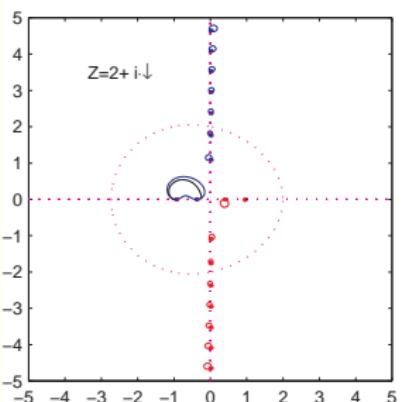
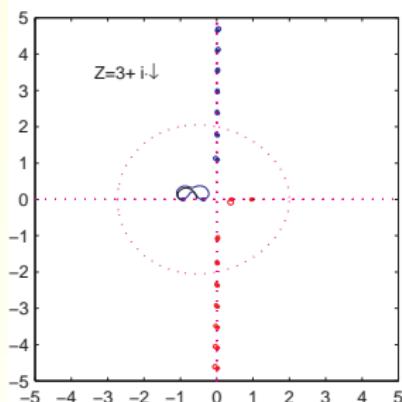
$$m = 1, \omega = 5, M = 0.5$$

Note: 4 surface modes!

Tracing axial wave numbers $\kappa_{m\mu}$ with mean flow

9 left- and right-running modes with $\omega = 5, m = 1, M = 0.5$ tracing along $\text{Re } \tilde{Z} = 0.4$

Trajectories of $\sigma_{m\mu}$ with mean flow

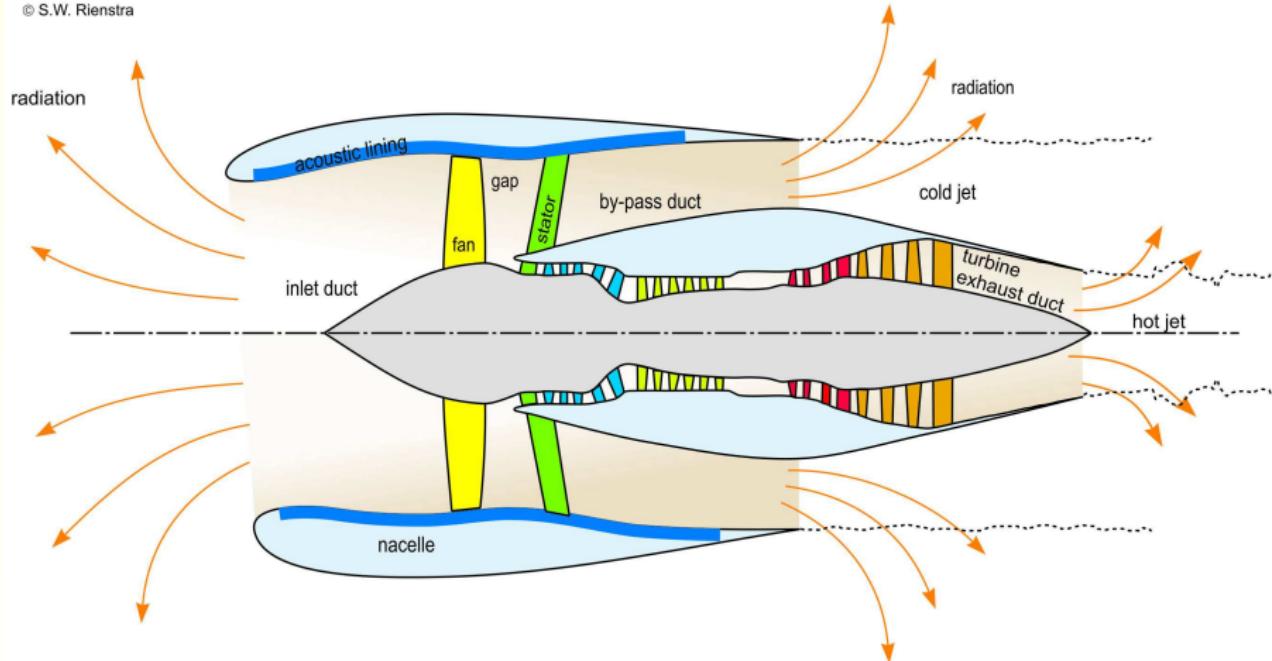


Outline

- 1 General Introduction
- 2 Equations and Boundary Conditions
- 3 Duct Modes, Cut-off, Flow Effects
- 4 Circular Ducts: Specific Details
- 5 Surface Waves and Other Behaviour
- 6 Some Applications
- 7 Slowly Varying Modes
- 8 Summary

Important applications of basic* duct acoustics

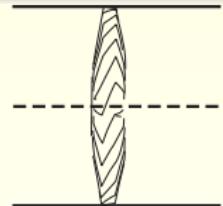
© S.W. Rienstra



*relatively...

Propeller in duct

B blades, B -periodic with period $\Delta\vartheta = 2\pi/B$:

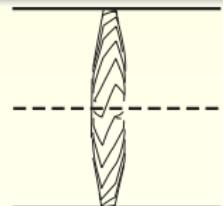


Propeller in duct

B blades, B -periodic with period $\Delta\vartheta = 2\pi/B$:

Any periodic function can be written as a Fourier series.

$$p(0, r, \vartheta, 0) = \sum_{m=-\infty}^{\infty} q_m(r) e^{-im2\pi \frac{\vartheta}{\Delta\vartheta}} = \sum_{m=-\infty}^{\infty} q_m(r) e^{-imB\vartheta}$$

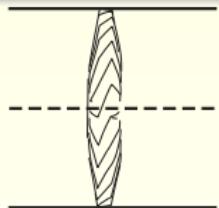


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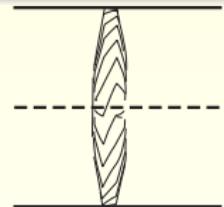
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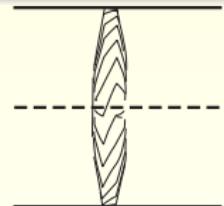


$$\begin{aligned} p(0, r, \vartheta, 0) &= \sum_{m=-\infty}^{\infty} q_m(r) e^{-im2\pi \frac{\vartheta}{\Delta\vartheta}} = \sum_{m=-\infty}^{\infty} q_m(r) e^{-imB\vartheta} \\ p(0, r, \vartheta, t) &= p(0, r, \vartheta - \Omega t, 0) = \sum_{m=-\infty}^{\infty} q_m(r) e^{-imB\vartheta + imB\Omega t} \\ &\quad \Downarrow \text{(in whatever way ...)} \\ p(x, r, \vartheta, t) &= \sum_{m=-\infty}^{\infty} \sum_{\mu=1}^{\infty} Q_{m\mu} J_{mB}(\alpha_{mB,\mu} r) e^{-i\kappa_{mB,\mu} x - imB\vartheta + imB\Omega t} \end{aligned}$$

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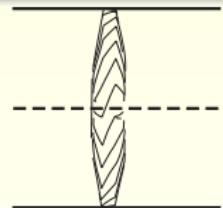
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Cut-on if: $\kappa_{mB,1}^2 = \left(\frac{mB\Omega}{c_0}\right)^2 - \alpha_{mB,1}^2 = \left(\frac{mB\Omega}{c_0}\right)^2 - \left(\frac{j'_{mB,1}}{R}\right)^2 > 0$

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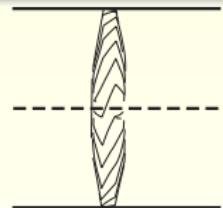
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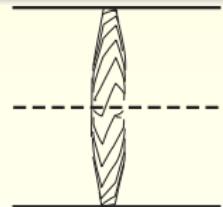


Ideally: subsonic tip speed \rightarrow no cut-on modes ! In reality: blades **and** vanes...

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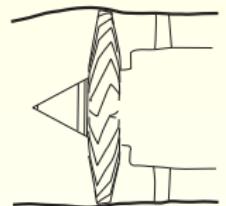
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Supersonic tip speed: **buzz-saw noise**

Tyler & Sofrin's selection rule

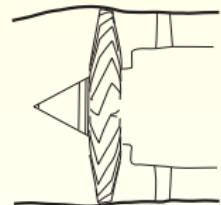
Ingenious manipulation of periodicity in B rotor blades and V stator vanes:



Tyler & Sofrin's selection rule

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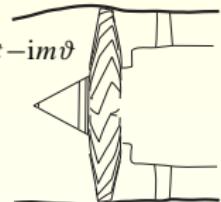
$$p(r, \vartheta, t) = \sum_{n=-\infty}^{\infty} P_n(r, \vartheta) e^{inB\Omega t}$$



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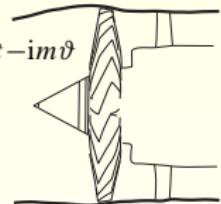
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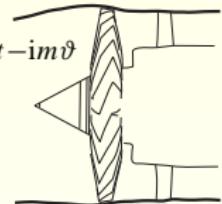


with $\Delta\vartheta = 2\pi/V$ in a time step $\Delta t = \Delta\vartheta/\Omega$ because vanes are fixed.

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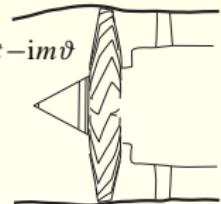
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$$e^{-inB\Omega\Delta t + im\Delta\vartheta} = 1 = e^{2\pi ik}, \quad \text{or: } \boxed{m = kV + nB, \quad k \in \mathbb{Z}}.$$

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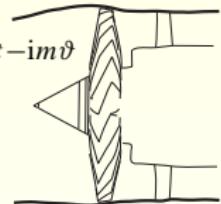
m -modes cut-off if

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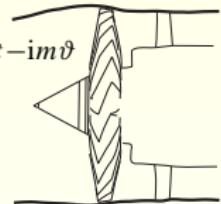
Since (usually) tip speed $\Omega R < c_0$, this is if

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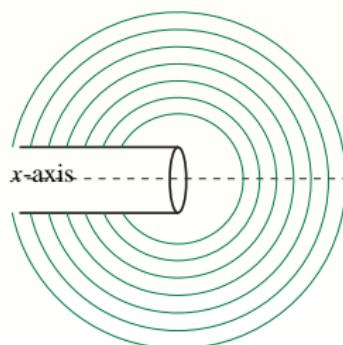
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(and this is how real engines are designed!)

Reflection at/radiation from open end

SOLUTION IS CLASSIC EXAMPLE OF WIENER-HOPF TECHNIQUE

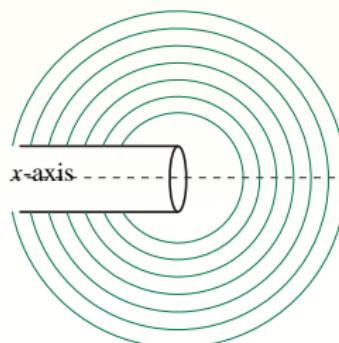


Reflection at/radiation from open end

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Inside duct: *incident mode + reflected field* $p(x, r, \vartheta) = p_{m\mu}(x, r) e^{-im\vartheta}$

$$p_{m\mu}(x, r) = U_{m\mu}(r) e^{-i\kappa_{m\mu}x} + \sum_{v=1}^{\infty} R_{m\mu v} U_{mv}(r) e^{i\kappa_{mv}x}.$$



Reflection at/radiation from open end

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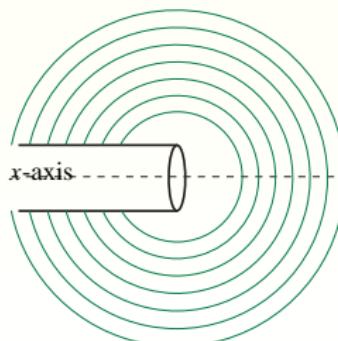
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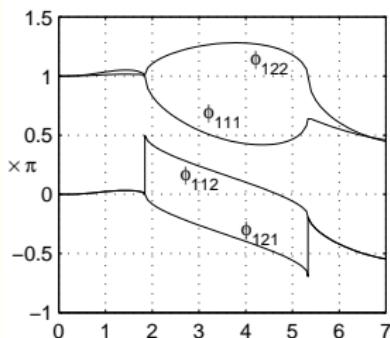
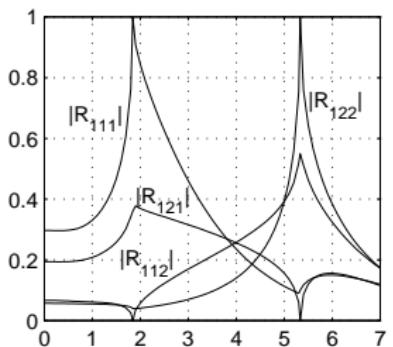
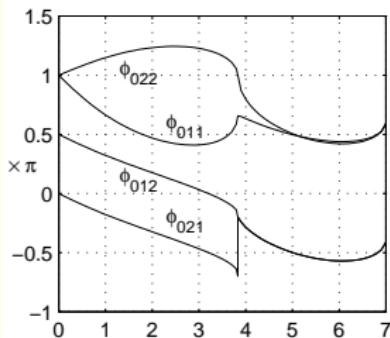
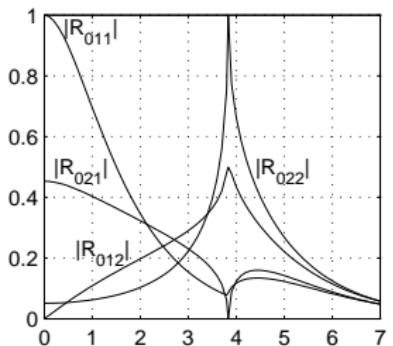
Outside far field: for $k\varrho \rightarrow \infty$

$$p_{m\mu}(x, r) \simeq D_{m\mu}(\xi) \frac{e^{-ik\varrho}}{k\varrho}, \quad (x, r) = \varrho(\cos \xi, \sin \xi)$$

$D_{m\mu}(\xi)$ is: *directivity function*,
 $|D_{m\mu}(\xi)|$ is: *radiation pattern*.

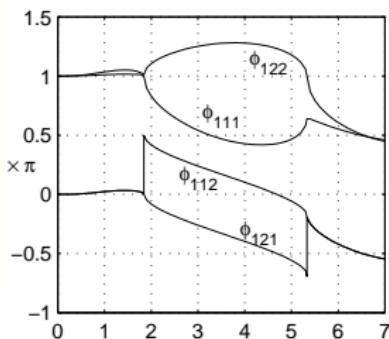
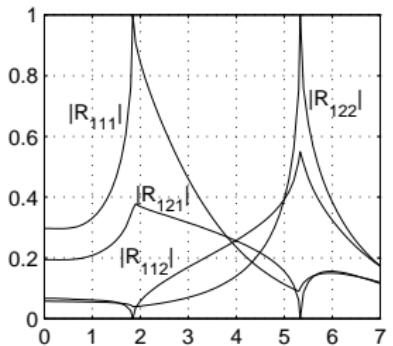
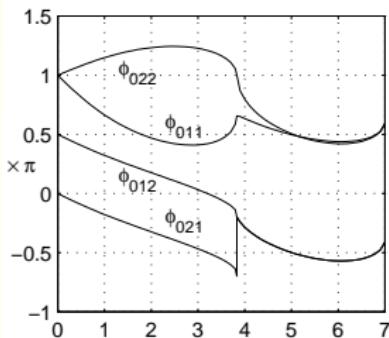
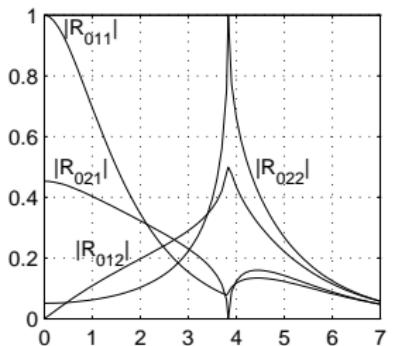


Reflection matrix



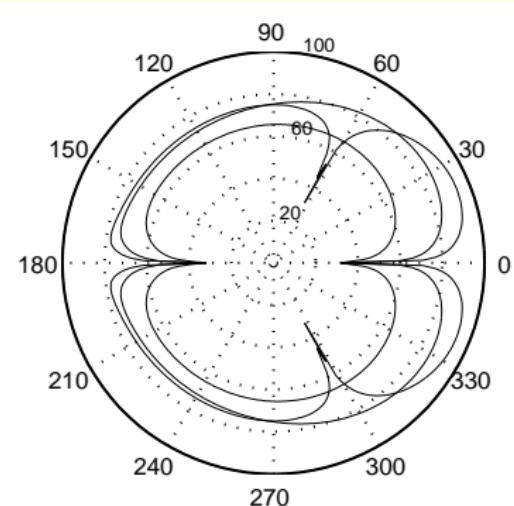
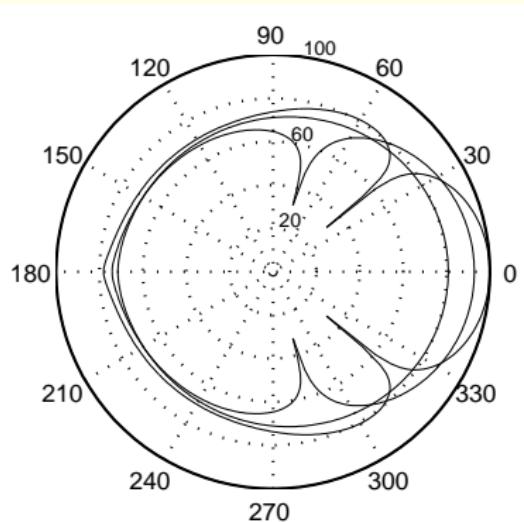
Modulus and phase of reflection coefficients $R_{m\mu\nu}$ for $m = 0 \dots 2$, $\mu, \nu = 1, 2$, as a function of $kR = 0 \dots 7$.

Reflection matrix



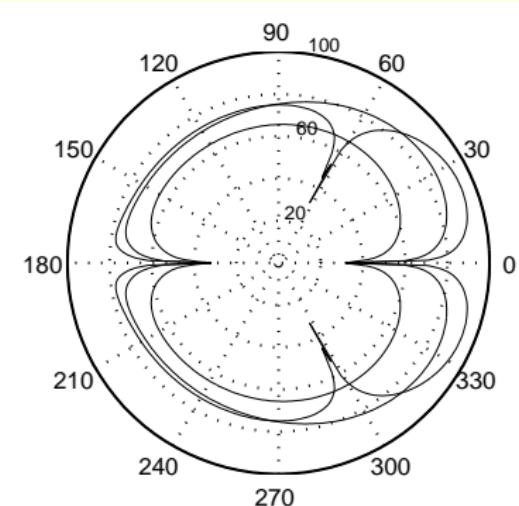
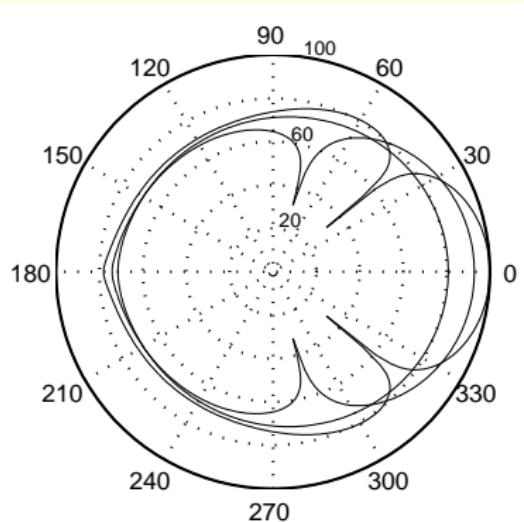
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Note: $|R_{m\mu\nu}| = 1$: total reflection.

Far field radiation pattern



Radiation pattern $20 \log_{10} |D_{m\mu}|$
for $m\mu = 01, = 11$ and $kR = 2, 4, 6$.

Far field radiation pattern

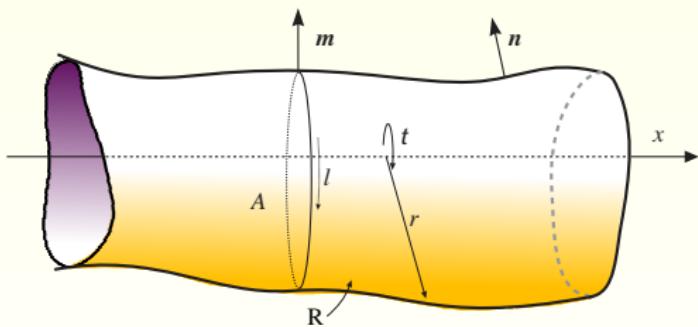


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Note: lobes!

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Multiple scales (WKB) approximation:

Slowly varying ducts



$$\nabla \cdot (c_0^2 \nabla p) + \omega^2 p = 0$$

A canonical model problem

Model equation

$$\frac{\partial^2}{\partial t^2}y - \frac{\partial}{\partial x}\left(c(\varepsilon x)^2 \frac{\partial}{\partial x}y\right) = 0, \quad \varepsilon \ll 1$$

A canonical model problem

Model equation

$$\frac{\partial^2}{\partial t^2}y - \frac{\partial}{\partial x}\left(c(\varepsilon x)^2 \frac{\partial}{\partial x}y\right) = 0, \quad \varepsilon \ll 1$$

Locally

$$y(x, t) \simeq A e^{i\omega t - ikx}, \quad k = \omega/c.$$

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yielding: $O(1): \gamma(X)^2 = \omega^2/c^2(X)$

$$O(\varepsilon): (c^2 \gamma A_0^2)' = 0 \rightarrow A_0(X) = \frac{\text{const}}{c(X) \sqrt{\gamma(X)}}$$

Classical straight-duct solution

Classical straight-duct solution as a modal sum:

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$$J'_m(\alpha_{m\mu} R) = 0 \rightarrow \alpha_{mn} = j'_{m\mu}/R.$$

$$\kappa_{mn}^2 + \alpha_{mn}^2 = \frac{\omega^2}{c_0^2}, \quad \kappa_{mn} \in \begin{cases} \text{IV} & \text{right running} \\ \text{II} & \text{left running} \end{cases}$$

Acoustic field: slowly varying mode

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Use $\frac{\partial}{\partial x} \sim -i\gamma(X) + \varepsilon \frac{\partial}{\partial X}$, and substitute:

$$\boxed{\mathcal{L}(A) = i\varepsilon(\gamma_X A + 2\gamma A_X) + O(\varepsilon^2) = \frac{i\varepsilon}{A} \frac{\partial}{\partial X} (\gamma A^2) + O(\varepsilon^2)}$$

where $\mathcal{L} = \frac{\partial^2}{\partial r^2} + \frac{1}{r} \frac{\partial}{\partial r} + \frac{\omega^2}{c_0^2} - \gamma^2 - \frac{m^2}{r^2}$.

(Similar for the boundary conditions.)

Expand: $A(X, r; \varepsilon) = A_0(X, r) + \varepsilon A_1(X, r) + \dots,$

$$O(1) : \quad \mathcal{L}(A_0) = 0$$

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$$O(1) : A_0(X, r) = N(X) J_m(\alpha(X)r)$$

$$\text{Dispersion relation:} \quad \alpha^2 + \gamma^2 = \frac{\omega^2}{c_0^2}.$$

$$\text{Boundary condition: } \frac{\partial}{\partial r} A_0 = N(X) \alpha J'_m(\alpha R) = 0$$

$$\rightarrow \text{determines } \alpha(X) = \frac{j'_{m\mu}}{R(X)}$$

$$\rightarrow \text{determines } \gamma(X) = \sqrt{\frac{\omega^2}{c_0^2} - \alpha^2}$$

Finally determine $N(X)$

Solvability condition for A_1 : manipulate

$$\int_0^R \left[A_0 \mathcal{L}(A_1) - A_1 \mathcal{L}(A_0) \right] r \, dr = \int_0^R i \frac{\partial}{\partial X} (\gamma A_0^2) r \, dr$$

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Solvability condition for A_1 : manipulate

$$\left[rA_0 A_{1,r} - rA_1 A_{0,r} \right]_0^R = \int_0^R i \frac{\partial}{\partial X} (\gamma A_0^2) r \, dr$$

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$$-\mathrm{i} R R_X \gamma A_0^2(X, R) = \int_0^R \mathrm{i} \frac{\partial}{\partial X} (\gamma A_0^2) r \, dr$$

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$$N(X) = \frac{N_0}{R(X) \sqrt{\gamma(X)}}$$

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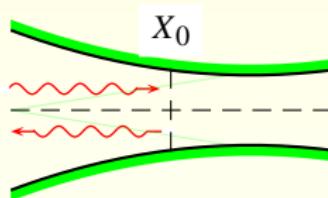
(Note possible cut-off or *turning point* $X = X_0$ where $\gamma(X) = 0$.)

Slowly varying modes in engine inlet: $m = 10, M = -0.5$

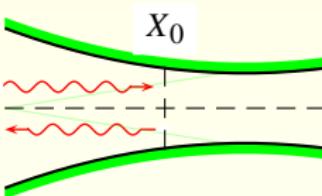
$\omega = 16$ and $\omega = 30$ (hard wall)

$\omega = 20$ (soft wall), well cut-on and near cut-off

Turning point reflection



Turning point reflection



$X < X_0 :$

$$p = \frac{n(X)}{\sqrt{\gamma(X)}} J_m(\alpha r) \left[e^{-\frac{i}{\varepsilon} \int_{X_0}^X \gamma(z) dz} + \mathcal{R} e^{\frac{i}{\varepsilon} \int_{X_0}^X \gamma(z) dz} \right]$$

$X > X_0 :$

$$p = \mathcal{T} \frac{n(X)}{\sqrt{\gamma(X)}} J_m(\alpha r) e^{-\frac{1}{\varepsilon} \int_{X_0}^X |\gamma(z)| dz}$$

$$\gamma(X_0) = 0, \quad \alpha_0 = \alpha(X_0), \quad \alpha'_0 = \alpha'(X_0), \quad n(X) = N_0/R(X)$$

$$\frac{d}{dX} \gamma(X_0)^2 < 0 \Leftrightarrow \frac{\alpha'_0}{\alpha_0} > 0$$

Local analysis

Rescale

$$X = X_0 + \varepsilon^{\frac{2}{3}} \lambda^{-1} \xi, \quad \text{where} \quad \lambda = \frac{\omega}{c_0} \left(\frac{2\alpha'_0}{\alpha_0} \right)^{\frac{1}{3}}$$

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$$p = \psi(\xi) J_m(\alpha(X_0 + \dots) r)$$

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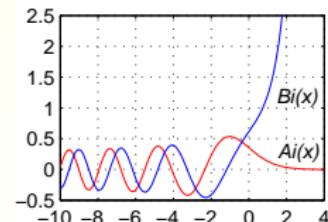
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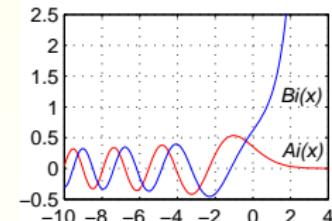
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Matching:

$$\mathcal{R} = i, \quad \mathcal{T} = 1, \quad b = 0, \quad a = \frac{n_0 2\sqrt{\pi} e^{\frac{1}{4}\pi i}}{\varepsilon^{\frac{1}{6}} \lambda^{\frac{1}{2}}}$$

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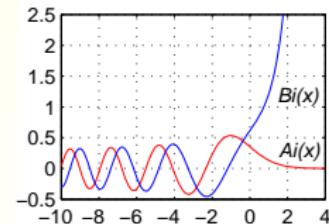
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(Note the $\frac{1}{2}\pi$ phase change by reflection.)

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- 2 Equations and Boundary Conditions
- 3 Duct Modes, Cut-off, Flow Effects
- 4 Circular Ducts: Specific Details
- 5 Surface Waves and Other Behaviour
- 6 Some Applications
- 7 Slowly Varying Modes
- 8 Summary

- Modes are solutions, self-similar in one direction
- Derivation of parallel flow acoustic equations:
shear flow, uniform mean and no mean flow
- Boundary conditions: hard walls, soft walls without and with mean flow
- Duct modes: hard/soft walls, cut-on cut-off, group and phase velocity
- Circular duct: Bessel functions, source expansion, modal power
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- Open end radiation, reflection coefficient
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