

Continuous-Time Dynamics

- Most general / generic for smooth systems:

$$\dot{x} = f(x, u) \quad u \in \mathbb{R}^m \text{ "input"} \\ \begin{matrix} \text{"dynamics"} \\ \text{y} \end{matrix} \quad \begin{matrix} \text{y} \\ x \in \mathbb{R}^n \text{ "state"} \end{matrix}$$

"configuration"
(not always
a vector)

- for a mechanical system $x = \begin{bmatrix} q \\ v \end{bmatrix}$ velocity

- Example (Pendulum):



$$[ml^2\ddot{\theta} + mgl\sin(\theta)] = \ddot{x} \\ q = \theta, \quad v = \dot{\theta}$$

$$x = \begin{bmatrix} \theta \\ \dot{\theta} \end{bmatrix} \Rightarrow \dot{x} = \begin{bmatrix} \dot{\theta} \\ \ddot{\theta} \end{bmatrix} = \begin{bmatrix} \dot{\theta} \\ -\frac{g}{l}\sin(\theta) + \frac{1}{ml^2}u \end{bmatrix}$$

$f(x, u)$

$x \in S^1 \times \mathbb{R}$ (cylinder)

* Control-Affine Systems

$$\dot{x} = \underbrace{f_0(x)}_{\text{"drift"}} + f(x)u \quad \begin{matrix} \text{- most problems can} \\ \text{be put in this form} \end{matrix}$$

- Pendulum: $f_0(x) = \begin{bmatrix} \dot{\theta} \\ -\frac{g}{l}\sin(\theta) \end{bmatrix}, \quad G(x) = \begin{bmatrix} 0 \\ \frac{1}{ml^2} \end{bmatrix}$

* Manipulator Dynamics:

$$\underbrace{M(q) \ddot{v}}_{\text{"Mass matrix"} } + \underbrace{C(q, v)}_{\substack{\text{"Dynamics Bias"} \\ (\text{Coriolis + Gravity})}} = \underbrace{B(q) u}_{\text{"Input Jacobian"}}$$

$$\dot{q} = G(q) v$$

Kinematics

$$\dot{x} = f(x, u) = \begin{bmatrix} G(q) v \\ -M(q)^{-1}(B(q)u - C) \end{bmatrix}$$

- Pendulum:

$$M(q) = ml^2, \quad C(q, v) = g l \sin(\theta), \quad B = I, \quad G = I$$

- All mechanical systems can be written in this form
- This is just a way of re-writing the Euler-Lagrange equation for:

$$L = \frac{1}{2} v^T M(q) v - V(q)$$

(Bonus points for showing this)

* Linear Systems

$$\dot{x} = A(t)x + B(t)u$$

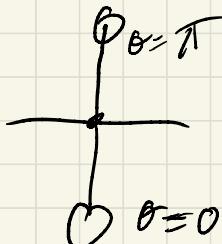
- Called "time invariant" $A(t) = A$, $B(t) = B$
- Called "time varying" otherwise
- Super important in control designs
- We often approximate nonlinear systems with linear ones:

$$\dot{x} = f(x, u) \Rightarrow A \approx \frac{\partial f}{\partial x}, B = \frac{\partial f}{\partial u}$$

Equilibrium:

- A point where system will remain "at rest"
 $\Rightarrow \dot{x} = f(x, u) = 0$
- Algebraically, roots of the dynamics
- Pendulum

$$\dot{x} = \begin{bmatrix} \dot{\theta} \\ -\frac{g}{L} \sin(\theta) \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix} \Rightarrow \begin{array}{l} \dot{\theta} = 0 \\ \theta = 0, \pi \end{array}$$



* First Control Problem:

- Can we move the equilibrium point?

$$\text{Free Fall Model: } \dot{x} = \begin{bmatrix} \dot{\theta} \\ -\frac{g}{l} \sin(\theta) + \frac{1}{m\theta} u \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

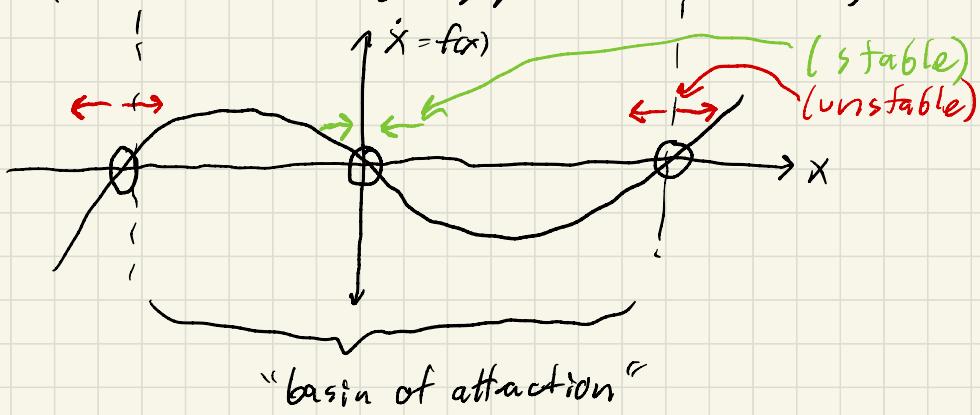
$$m\theta \ddot{u} = -\frac{g}{l} \underbrace{\sin(\theta)}_u \Rightarrow u = mg/l$$

- In general, we get a root-finding problem in u :

$$f(x^*, u) = 0$$

Stability of Equilibria:

- When will a system stay "near" an equilibrium point under perturbations.
- Let's look at a 1D system ($x \in \mathbb{R}$)



$$\frac{\partial f}{\partial x} < 0 \Rightarrow \text{stable}, \quad \frac{\partial f}{\partial x} > 0 \Rightarrow \text{unstable}$$

* To generalize to higher dimensions:

- $\frac{\partial f}{\partial x}$ is a Jacobian matrix
- Take an eigen decomposition \Rightarrow decouple into n 1D systems

$$\operatorname{Re} \left[\operatorname{eig} \left(\frac{\partial f}{\partial x} \right) \right] < 0 \Rightarrow \text{stable}$$

otherwise \Rightarrow unstable

- Pendulum:

$$f(x) = \begin{bmatrix} \dot{\theta} \\ -\frac{g}{l} \sin(\theta) \end{bmatrix} \Rightarrow \frac{\partial f}{\partial x} \begin{bmatrix} 0 & 1 \\ \frac{g}{l} \cos(\theta) & 0 \end{bmatrix}$$

$$\frac{\partial f}{\partial x} \Big|_{\theta=\pi} = \begin{bmatrix} 0 & 1 \\ \frac{g}{l} & 0 \end{bmatrix} \Rightarrow \operatorname{eig} \left(\frac{\partial f}{\partial x} \Big|_{\theta=\pi} \right) = \pm \sqrt{\frac{g}{l}}$$

\Rightarrow unstable

$$\frac{\partial f}{\partial x} \Big|_{\theta=0} = \begin{bmatrix} 0 & 1 \\ -\frac{g}{l} & 0 \end{bmatrix} \Rightarrow \operatorname{eig} \left(\frac{\partial f}{\partial x} \Big|_{\theta=0} \right) = 0 \pm i \sqrt{\frac{g}{l}}$$

- Pure imaginary case is called "marginally stable"
 \Rightarrow undamped oscillations

- Adding damping (e.g. $u = -K_d \ddot{\theta}$) results in strictly negative eigenvalues