# Modeling 1DoF System

ARO (@\_art\_of\_electronics\_)
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### 1 Introduction

This document presents the modeling of a simple 1 degree-of-freedom (DoF) system using Lagrangian mechanics and co-energy analysis. It also provides a method for including a friction model. A Simulink block diagram is also provided for simulation purposes.

# 2 1DoF System Description

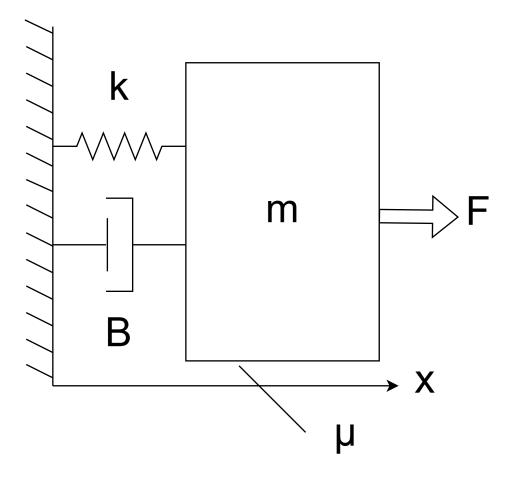


Figure 1: Overview of the 1DoF system

We consider a 1DoF system in 2 main cases: first, where a plain, frictionless model is presented; second, where friction submodel is included into 1DoF system.

### 3 Lagrangian Formulation - General Case

The kinetic and potential energies of 1DoF system are defined as:

$$E_k = \frac{1}{2}m\dot{x}^2$$

$$E_p = k \int_0^x x dx = \frac{1}{2}kx^2$$

The system has a damper, which is a dissipative element:

$$D_B = B \int_0^{\dot{x}} \dot{x} dx = \frac{1}{2} B \dot{x}^2$$

Now, we can include the friction model, as it is also a dissipative element:

$$D_{\mu} = B_{\mu}\dot{x}$$

In that case, the total dissipation in 1DoF system can be described as:

$$D = \frac{1}{2}B\dot{x}^2 + B_{\mu}\dot{x}$$

With this approach, we can either define the friction submodel, or equate it to 0. The Lagrangian of 1DoF system is:

$$\mathcal{L}(x, \dot{x}) = E_k - E_p = \frac{1}{2}m\dot{x}^2 - \frac{1}{2}kx^2$$

The Rayleigh dissipation function can be written as:

$$\mathcal{D} = D - P = \left(\frac{1}{2}B\dot{x}^2 + B_{\mu}\dot{x}\right) - F\dot{x}$$

Using Lagrange's equation with a non-conservative force (damper), the equation expands into the following form:

$$\frac{d}{dt}\left(\frac{\partial \mathcal{L}}{\partial \dot{q}}\right) - \frac{\partial \mathcal{L}}{\partial q} + \frac{\partial \mathcal{D}}{\partial \dot{q}} = 0$$

This leads to the following differential equation:

$$\frac{d}{dt} \left( \frac{\partial \left( \frac{1}{2} m \dot{x}^2 - \frac{1}{2} k x^2 \right)}{\dot{x}} \right) - \frac{\partial \left( \frac{1}{2} m \dot{x}^2 - \frac{1}{2} k x^2 \right)}{x} + \frac{\partial \left( \left( \frac{1}{2} B \dot{x}^2 + B_{\mu} \dot{x} \right) - F \dot{x} \right)}{\dot{y}} = 0$$

$$\frac{d}{dt} \left( \frac{\partial \left( \frac{1}{2} m \dot{x}^2 - 0 \right)}{\dot{x}} \right) - \frac{\partial \left( 0 - \frac{1}{2} k x^2 \right)}{x} + \frac{\partial \left( \left( \frac{1}{2} B \dot{x}^2 + B_{\mu} \dot{x} \right) - F \dot{x} \right)}{\dot{y}} = 0$$

$$\frac{d}{dt}\left(2 \cdot \frac{1}{2}m\dot{x}^{2-1}\right) - \left(-\left(2 \cdot \frac{1}{2}kx^{2-1}\right)\right) + \left(\left(2 \cdot \frac{1}{2}B\dot{x}^{2-1} + \cdot B_{\mu}\dot{x}^{1-1}\right) - F\dot{x}^{1-1}\right) = 0$$

$$\frac{d}{dt}\left(m\dot{x}\right) - \left(-\left(kx\right)\right) + \left(\left(B\dot{x} + B_{\mu}\right) - F\right) = 0$$

$$m\ddot{x} + kx + B\dot{x} + B_{\mu} - F = 0$$

Sorting yields the equation's final form:

$$m\ddot{x} + B\dot{x} + kx = F - B_{\mu}$$

# 4 Co-energy and Energy Expressions

#### Mass

The general inertia co-energy for mass is:

$$T = \int_0^v mv' \, dv' = \frac{1}{2} mv^2 = \frac{1}{2} m\dot{x}^2$$

In linear systems, energy and co-energy are equal.

### Spring

The general energy stored in the spring  $k_1$  is:

$$U = \int_0^x kx' \, dx' = \frac{1}{2} kx^2$$

Since the spring is conservative (no energy loss), co-energy equals energy.

### Damper and Friction

The general damper dissipates energy as friction:

$$W_d = \int_0^t Bv^2(\tau) d\tau = \frac{1}{2}Bv^2 = \frac{1}{2}B\dot{x}^2$$

The general friction dissipates energy in the following manner:

$$W_f = \frac{d}{d\dot{x}} \left[ \int_0^{\dot{x}} B_{\mu}(v) v \, dv \right] = B_{\mu}(\dot{x}) \dot{x} = B_{\mu} \dot{x}$$

Dampers, as friction, do not store energy, so they do not have co-energy.

Using the co-energy equation with a non-conservative force, we can write:

$$\frac{d}{dt} \left[ \frac{\partial T}{\dot{y}} \right] - \frac{\partial T}{\dot{y}} + \frac{\partial U}{\dot{y}} + \frac{\partial D}{\dot{y}} = f_i$$

Adjusting the general co-energy expressions to 1DoF systems speciffic configuration gives:

$$T = \frac{1}{2}m\dot{x}^2$$

$$U = \frac{1}{2}kx^2$$

$$D = \frac{1}{2}B\dot{x}^2 + B_{\mu}\dot{x}$$

$$f_i = F$$

By substituting the coefficients into the equation, we obtain the following:

$$\frac{d}{dt} \left[ \frac{\partial \left( \frac{1}{2} m \dot{x}^2 \right)}{\dot{x}} \right] - \frac{\partial \left( \frac{1}{2} m \dot{x}^2 \right)}{x} + \frac{\partial \left( \frac{1}{2} k x^2 \right)}{x} + \frac{\partial \left( \frac{1}{2} B \dot{x}^2 + B_{\mu} \dot{x} \right)}{\dot{x}} = F$$

$$\frac{d}{dt} \left[ \frac{\partial \left( \frac{1}{2} m \dot{x}^2 \right)}{\dot{x}} \right] - 0 + \frac{\partial \left( \frac{1}{2} k x^2 \right)}{x} + \frac{\partial \left( \frac{1}{2} B \dot{x}^2 + B_{\mu} \dot{x} \right)}{\dot{x}} = F$$

$$\frac{d}{dt} \left[ m \dot{x} \right] + k x + B \dot{x} + B_{\mu} = F$$

$$m \ddot{x} + B \dot{x} + k x = F - B_{\mu}$$

## 5 Simulink Model for Lagrangian and Co-energy Expression

In order to model the equation properly, we must translate the dynamic equation into a form understandable for Simulink:

$$m\ddot{x} + B\dot{x} + kx = F - B_{\mu}$$

$$\ddot{x} = \frac{1}{m} \left( F - B\dot{x} - kx - B_{\mu} \right)$$

Before coding the solution in Simulink, we should consider the friction submodel. The most versatile friction model is the Coulomb function:

$$B_{\mu} = \mu mq \cdot siqn(\dot{x}) + c\dot{x}$$

where:

 $\mu$  – dry friction coefficient;

c – viscous friction coefficient;

q – gravitational force;

In the following, the Simulink model is presented.

At  $\dot{x} = 0$ , the sign function has a discontinuity (jump from -1 to +1), and Simulink's solver repeatedly attempts to find the exact zero-crossing time, failing to progress. This causes the 1000 consecutive zero-crossings error. In order to solve the issue, use either:

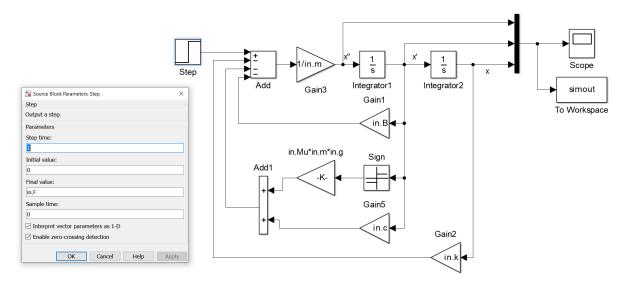


Figure 2: Simulink model of 1DoF system - general form

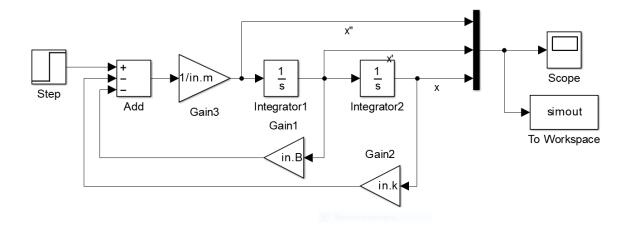


Figure 3: Simulink model of 1DoF system - explicitly frictionless

- smoothing approximation  $sign(\dot{x}) \approx \tanh\left(\frac{\dot{x}}{\epsilon}\right)$
- switch function instead of sign.

### 6 Matlab and Python script

For script-based system modeling, we must take a slightly different approach to translating mathematical equations into code. Let x(t) = x(1),  $\dot{x}(t) = x(2)$ , in that case the matrix of the 1DoF equation will be written as:

$$\frac{d}{dt} \begin{bmatrix} x(1) \\ x(2) \end{bmatrix} = \begin{bmatrix} x(2) \\ (F - B \cdot x(2) - k \cdot x(1) - friction) / m \end{bmatrix}$$

Now, the equation can be coded into Matlab:

$$Dx = [x(2); (F-B * x(2) - k * x(1) - friction) / m];$$

Python counts matrix elements from 0, so the equivalent Python code can be written as:

```
def sindle_dof(x, t):
    friction = Mu * m * g * np.sign(x[1]) + c * x[1]
    return x[1], (F - B * x[1] - k * x[0] - friction) / m
```

# 7 Simulation Results

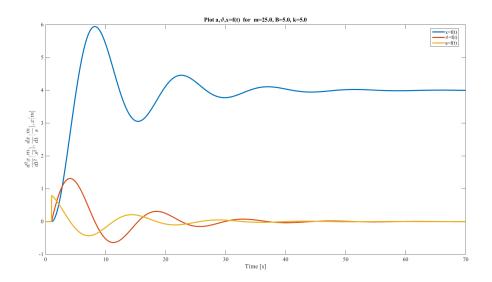


Figure 4: Result without friction

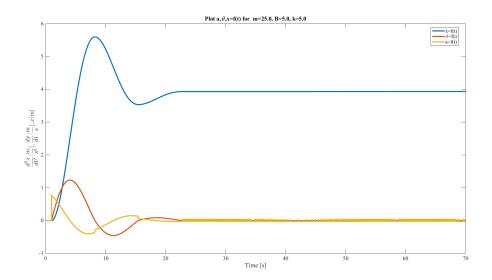


Figure 5: Result with friction

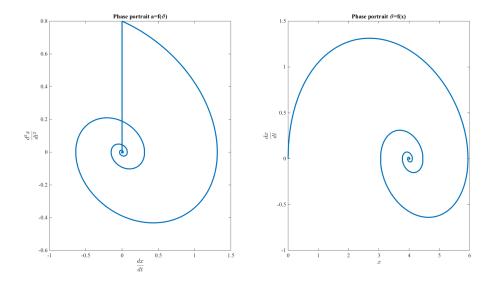


Figure 6: Phase portrait without friction

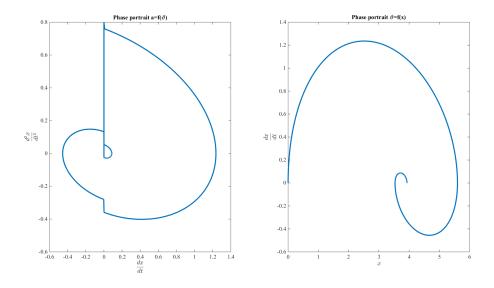


Figure 7: Phase portrait with friction

### 8 Conclusion

This document demonstrates the Lagrangian and co-energy approach to modeling a 1DoF system. It also provides a visual block diagram suitable for simulation in Simulink and basic code for scripting.