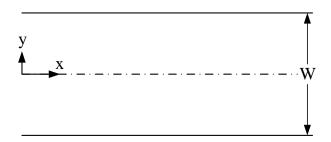
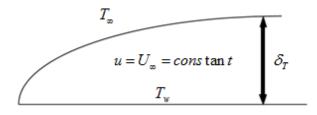
- I. Explain the following terms: (15%)
 - (1) Prandtl number
 - (2) Laminer sublayer
 - (3) Bulk fluid temperature
 - (4) Natural convection
 - (5) Similarity conditions
- II. 簡答題: (33%)
- 1. In what conditions is there no convection heat transfer?
- 2. What's relationship between Prandtl number and boundary layer thickness?
- 3. For a thermal flow in a tube, why is the bulk fluid temperature used instead of the average temperature?
- 4. In a convection-heat-transfer problem, in what condition can the effect of free convection be neglected?
- 5. For a thermal flow in a tube does $\partial T/\partial z = 0$ mean "fully-developed" for the temperature field? Why? or why not?
- 6. If the fluid is melted material, which boundary layer thickness is larger, the momentum or thermal boundary layer thickness? Why?
- 7. Is the boundary layer theory useful for a flow field of low Reynolds number? Why? or Why not?
- 8. What problem will be created when the viscous term is neglected in the momentum equation?
- 9. How do people judge whether natural convection is important in a convection problem?
- 10. In natural convection, the density is changed with temperature. This would make the problem of natural convection very difficult

- to solve. How do Boussineq propose to make the problem easier?
- 11. For a uniform flow of liquid metal passing a flat plate, it can be assumed that $u=U_{\infty}$. Why?
- III. Consider the heat transfer in a parallel plate duct with constant wall heat flux.
- (a) Prove $\frac{\partial T}{\partial z} = \frac{\partial T_{w}}{\partial z} = \frac{\partial T_{m}}{\partial z} = constant$, where T_{m} is the bulk fluid temperature and T_{w} is the wall temperature.(7%)
- (b) Derive the expression of Nu for the case of constant wall heat flux in the fully-developed region if it is assumed that $u = U_{\infty}$ and v = 0(10%)



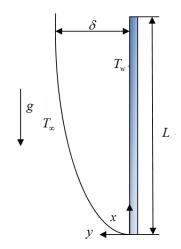
IV. A steady uniform flow, whose velocity is U_{∞} , passes over a flat plate. The fluid is at uniform temperature T_{∞} and the temperature of the plate is T_{w} . Assume that $u = U_{\infty} = \text{constant}$ and v = 0.



Derive the integral equation of energy.

$$\frac{\mathrm{d}}{\mathrm{d}x} \int_{0}^{\delta_{t}} U_{\infty} \left(T_{\infty} - T \right) \mathrm{d}y = \alpha \frac{\partial T}{\partial y} \bigg|_{w} (12\%)$$

V. A vertical plate with a uniform temperature $T_{\rm w} \ \ \text{in an environment at temperature} \ \ T_{\rm \infty}.$ Assume Pr=1.



- (a) In the boundary layer, what are the two forces, which determine the velocity prefile in the layer? How do these two forces determine the velocity profile? (6%)
- (b) Derive the momentum equation in the boundary layer,

$$\rho \left(u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} \right) = \rho \beta g \left(T - T_{\infty} \right) + \mu \frac{\partial^2 u}{\partial y^2},$$

From the equation,

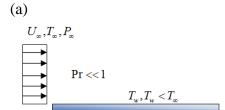
$$\rho \left(u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} \right) = -\frac{dp}{dx} - \rho g + \mu \frac{\partial^2 u}{\partial y^2} \quad (8\%)$$

Hint:
$$\beta = \frac{1}{V} \left(\frac{\partial V}{\partial T} \right)_{p}$$

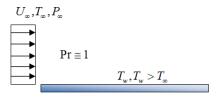
(c) Derive the floolwing dimensionless equation,

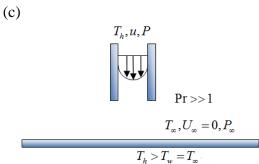
$$u^* \frac{\partial u^*}{\partial x^*} + v^* \frac{\partial u^*}{\partial y^*} = \theta + \frac{1}{\sqrt{Gr}} \left(\frac{L^2}{\delta^2}\right) \frac{\partial^2 u^*}{\partial y^{*2}} (9\%)$$

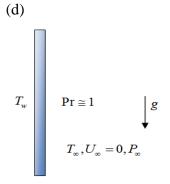
VI. 根據下列流動狀況來手繪流體邊界層、 熱邊界層以及溫度、速度分布圖。(20%)



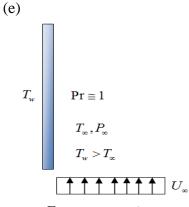
(b)







 $T_w > T_\infty$, Natural _convection



Force _convection