

**B. Printing Engineering Examination, 2018**

**( 2<sup>nd</sup> Year, 2<sup>nd</sup> Semester )**

**PRINTING MACHINE DESIGN**

Time :3 hrs.

Full marks : 100.

**Attempt any one from (a) and (b) in Question-1.**

1. (a) (i) What do you **understand** by the nominal size and basic size? (2)  
  
(ii) What are the various **types** of fits according to Indian Standard? **Explain** these with the help of neat sketches. (8)
- (b) (i) According to Indian Standard Specification what do you mean by **100 H 8 / g5**. (2)  
  
(ii) **Calculate** the tolerance, fundamental deviations and limits of sizes for the shaft Designation as **40 H8 / f7**. Given :  $i = 0.45\sqrt[3]{D} + 0.001D$  microns and steps diameter **30mm to 50mm** and fundamental deviation (upper deviation) for shaft '**f**' is  $es = - 5.5(D)^{0.41}$ . (8)
2. (a) (i) **Define** the following terms : **major diameter; minor diameter; pitch; lead**. (6)  
  
(ii) **Derive** an expression for the maximum load in a bolt when a bracket with circular base is bolted to a wall by means of four bolts. (8)  
  
(iii) A flanged bearing as shown in **FIGURE-1**, is fastened to a frame by means of **four bolts** spaced equally on **φ500mm** bolt circle. The diameter of flange is **φ650mm** and a load of **400kN** acts at a distance of **250mm** from the frame. Take safe tensile stress  $\sigma_t = 60\text{N/mm}^2$  for the material of the bolts. **Determine the size of the bolts**. (6)

**Attempt any two from (a), (b) and (c) in Question-3.**

3. (a) **Discuss the different types** of pulleys used in flat belt drives with neat **sketches**. (10)

- (b) **Discuss** the procedure used in designing a cast iron pulley. (10)
- (c) (i) **Indicate** what type of coupling is used under the following conditions: 5.
- (A) with shafts having **collinear axis**;
- (B) shafts having **intersecting axis**;
- (C) shafts having parallel axis with a **small distance** apart. (1X3)
- (ii) **Sketch** a Protective type **Flange Coupling** and indicate there on its **leading dimensions** for shafts size of 'd'. (7)

**Attempt any one from (a), (b) and (c) in Question-4.**

1. (a) A pulley transmits **35kW at 240 r.p.m.** The belt drive is vertical and the angle of wrap may be taken as **180°**. The distance of the pulley centre line from the nearest bearing is **350mm**, co-efficient of friction( $\mu$ ) = **0.25**, No. of arms(**n**) = **6**, assuming the section Of the arm may be taken as **elliptical**, the major axis being **twice** the minor axis. The following stresses may be taken for **design purpose**:
- Shaft – tension and compression** : **80 N/mm<sup>2</sup>**;
- Key - shear** : **50 N/mm<sup>2</sup>**;
- Belt - tension** : **2.5 N/mm<sup>2</sup>**;
- Pulley rim -** : **4.5 N/mm<sup>2</sup>**;
- Pulley arms -** : **15 N/mm<sup>2</sup>**.
- Design the following : (i) **diameter of the pulley**; (ii) **width of the Belt**( assuming Thickness of 10mm); (iii) **diameter of the shaft**; (iv) **size of the arms**. (20)
- (b) (i) **Explain** with the help of a neat **sketch**, the construction of **roller chain**. (10)
- (ii) Write the **design procedure** for a **chain drive**. (10)
- (c) (i) What are the various forces acting on spur gears, **explain** with **neat sketch**. (7)
- (ii) Why tangential component of gear tooth force is **called** 'useful component'? (3)
- (iii) **Discuss** the design procedure of spur gears. (10)

**Attempt any one from (a),(b) and (c) in Question-5.**

5. (a) The following particulars of a single reduction spur gear are given: **Gear ratio = 10:1**;  
Distance between centres = **660mm** approximately; Pinion transmits **500kW** at  
**1800 r.p.m.**; Involute teeth of standard proportions (addendum =  $m$ ) with pressure  
angle between teeth = **175N per mm of width**. Find :
- (i) the nearest standard **module** if no interference is to occur;
- (ii) the number of **teeth** on each wheel;
- (iii) the necessary **width** of the pinion;
- (iv) the **load** on the bearings of the wheels due to power transmitted. (20)
- (b) The dimensions of a pair of Bevel gears are shown in **FIGURE-2**, the gear 'G' delivers  
**5kW** power at **500 r.p.m.**, to the output shaft. The **bearings A and B** are mounted on  
The output shaft in such a way that the **bearing B** can take radial as well as thrust load,  
While the **bearing A** can take only radial load and **pressure angle( $\phi$ ) = 20°**.  
Determine: (i) **components of tooth force on pinion**; (ii) **components of tooth force**  
**on gear**; (iii) **draw a space free-body-diagram of this drive**; (iv) **reactions at bearings**  
**A and B**. (20)
- (c) (i) Name the **various types of ball and roller bearings**. What are the **applications** of these  
bearings? (6)
- (ii) what is  **$L_{10}$  life** and  **$L_{10h}$  life** of a bearing? Find their **relation**. (4)
- (iii) A single-row deep groove ball bearing is subjected to a pure radial force of **3kN**  
from a shaft that rotates at **600 r.p.m.** The **expected life** of the bearing is **30,000 hrs**.  
The minimum acceptable diameter of the shaft is  **$\phi 40\text{mm}$** . **Select** a suitable ball  
bearing for this **application**.  
Given: six different bearings are available for the shaft diameter  **$\phi 40\text{mm}$**  are : **61808**,  
**16008**, **6008**, **6208**, **6308** and **6408** for dynamic loads **4160N**, **13300N**, **16800N**, **30700N**,

41000N and 63700N.

Attempt any one from (a) and (b) in Question-6.

(a) Name the various rollers used in inking system of a OFF-SET printing machines, and mention their dimensions, types of covering use. (10)

(b) A hollow circular cylinder is to transmit 5.65kW power at 80 r.p.m. If the shear stress is not to exceed  $60\text{N/mm}^2$  and internal diameter is 0.6 times of the external diameter, Find the internal and external diameters of the cylinder. Assume that the maximum Torque is 1.4 times the mean torque. (10)

