

Stability Analysis Case Study: Simplex Architecture Controlled Inverted Pendulum

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Inverted Pendulum

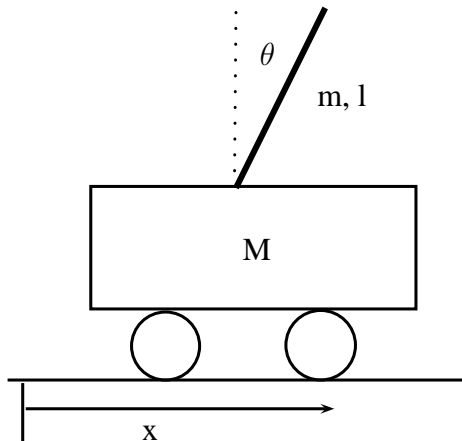


Figure: Inverted Pendulum System

Nonlinear System Model

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- ▶ Equations of Motion for Plant

$$(m + M) \ddot{x} + \frac{1}{2}ml \cos(\theta) \ddot{\theta} - \frac{1}{2}ml \sin(\theta) \dot{\theta}^2 = F - f_c$$

$$\frac{1}{2}ml\theta\ddot{x} + \frac{1}{3}ml^2\ddot{\theta} - \frac{1}{2}mgl \sin(\theta) = -f_p$$

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- Full System Model (including Motor Dynamics)

$$\ddot{x} = \frac{1}{D} \left[\frac{1}{3} ml^2 (B_l V_a - f_c - C_1) + \frac{1}{2} ml \cos(\theta) (f_p + C_2) \right]$$

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 where $D = \frac{1}{3} \bar{M} ml^2 - \frac{1}{4} m^2 l^2 \cos^2(\theta)$

$$\bar{M} = \frac{m + M + (K_g * J_m)}{r^2}$$

$$C_1 = \bar{B} \dot{x} - \frac{1}{2} ml \sin(\theta) \dot{\theta}^2$$

$$C_2 = -\frac{1}{2} mgl \sin(\theta)$$

$$\bar{B} = \frac{K_g B_m}{r^2} + \frac{K_g^2 K_i K_b}{r^2 R_a}$$

$$B_l = \frac{K_g K_i}{r R_a}$$

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$$A = \begin{bmatrix} 0 & 1 & 0 & 0 \\ 0 & -a_{22} & -a_{23} & a_{24} \\ 0 & 0 & 0 & 1 \\ 0 & a_{42} & a_{43} & -a_{44} \end{bmatrix},$$

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- Notable linearizations are $\sin(\theta) \approx \theta$ for small θ and armature inductance $L_a = 0$ to remove the armature current I_a state variable, reducing the order of the system, thus leaving only armature voltage V_a for control ($u = V_a$)

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- ▶ First-order approximations for \dot{x} and $\dot{\theta}$

$$\dot{\theta}(t) = \frac{[\theta(t) - \theta(t - mT_s)]}{mT_s}$$

$$\dot{x}(t) = \frac{[x(t) - x(t - mT_s)]}{mT_s}$$

where m is an integer greater than one (chosen as 2 by experimentation)

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- ▶ Stabilizable Region

This is a subset of the feasible region, determined by solving a linear-matrix inequality (LMI) problem to find controller gains that maximize the region

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- ▶ Switching Controller

Choose $u_{\sigma} \in [u_{sc}, u_{bc}, u_{ec}]$ based on the current (and future) stabilizable region(s) the system is within

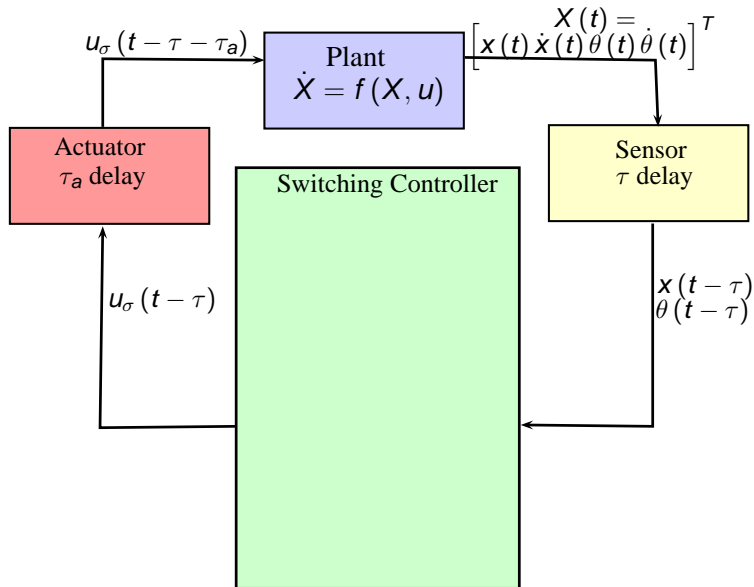
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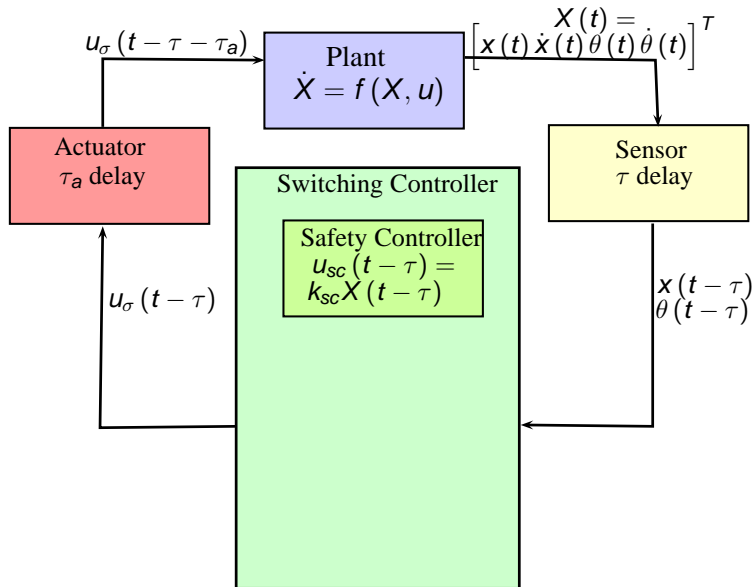
Choose $u_{\sigma} \in [u_{sc}, u_{bc}, u_{ec}]$ based on the current (and future) stabilizable region(s) the system is within
- ▶ Entire system modeled using HIOA (not shown here due to time-constraints)

Simplex Architecture

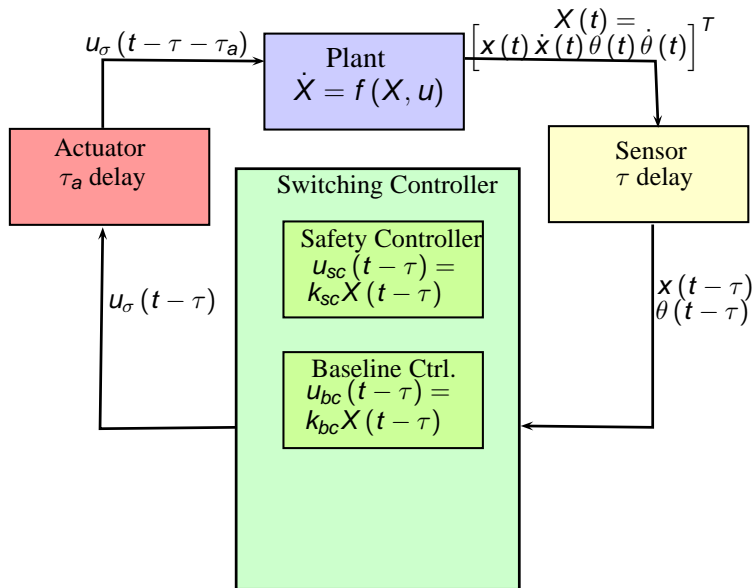
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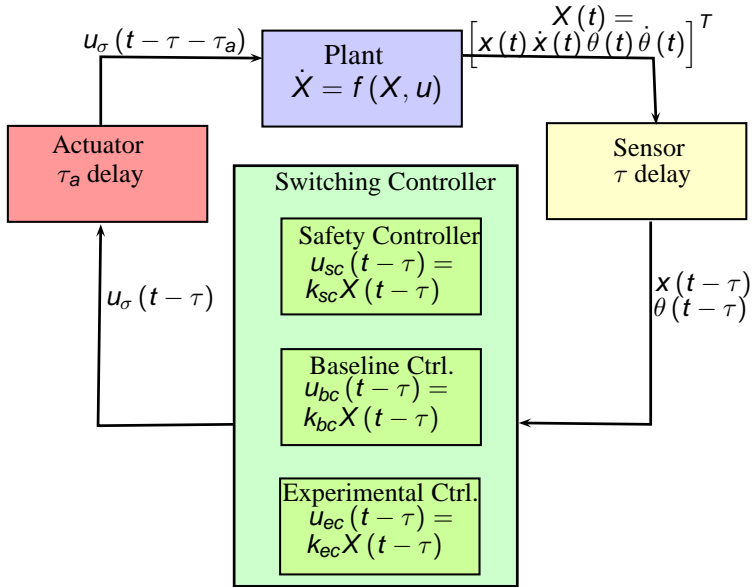
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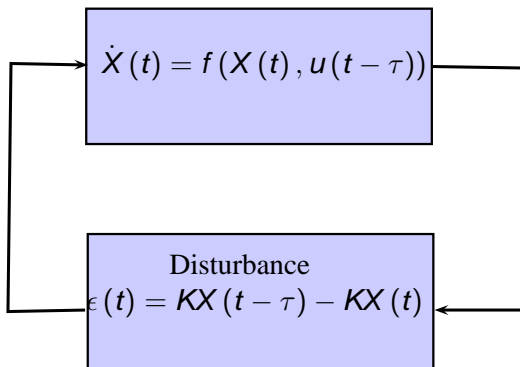
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- ▶ In the case of a disturbance (error) caused by delay, this allows one to bound the delay by bounding the error caused by the delay, allowing one to determine a tolerable (not maximal) delay to ensure ISS.

Small-Gain Theorem - Coupling with Disturbance from Delay

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Bounding Our System for ISS and Small-Gains

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$$|X| > \frac{||PB||}{\lambda_{\min}(Q)} \cdot |\epsilon|$$

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$$|x| > \frac{\|PB\|}{\lambda_{\min}(Q)} \cdot |\epsilon|$$

Define $\rho(r) = cr$ where $c = \frac{\|PB\|}{\lambda_{\min}(Q)}$

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- ▶ For this system, the following tolerable delays were found for each controller $\tau_{sc} = 9\text{ms}$, $\tau_{bc} = 12.3\text{ms}$, $\tau_{ec} = 18.6\text{ms}$.

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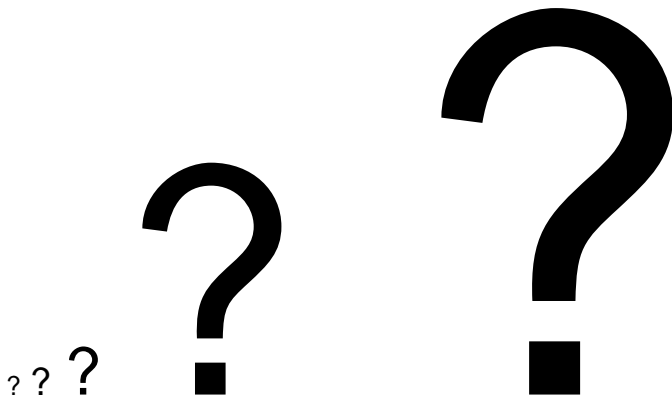
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- ▶ Control period is $T_s = 20\text{ms}$, so this method cannot guarantee stability with respect to the actual delay experienced, so model-based state projection is necessary.
- ▶ Note that small-gain is only a sufficient condition, so it could be the case that it is stable without state projection. A future line of work could be to define and solve an optimization problem to find the maximum tolerable τ .
- ▶ When perfect state projection is considered, stability is then guaranteed for a delay $T_s + \tau$. State projection is not perfect here however, due primarily to the first-order approximation used to reconstruct \dot{x} and $\dot{\theta}$ since they are unobservable, in addition to other errors like quantization not explicitly shown here. Further work could be done here as well.

Questions



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- ▶ LTI Case

If $\exists P = P^T > 0$ for some $Q = Q^T > 0$ that satisfies Lyapunov equation $A^T P + P A + Q = 0$, then the system is asymptotically stable, and the Lyapunov function is $V = X^T P X$

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$$\|\varphi(t, x, w)\| \leq \max\{\beta(\|x\|, t), \gamma(\|w\|_\infty)\}$$

Questions: Nonlinear System Model Derived from Energy

- Coordinates of Small Portion of Bar with Mass dm

$$x_{dm} = x + q \sin(\theta) \Rightarrow \dot{x}_{dm} = \dot{x} + q \cos(\theta) \dot{\theta}$$

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- ▶ Euler-Lagrange Equations for Forces on System

$$\frac{d}{dt} \frac{\partial L}{\partial \dot{x}} - \frac{\partial L}{\partial x} = F - f_c \text{ and } \frac{d}{dt} \frac{\partial L}{\partial \dot{\theta}} - \frac{\partial L}{\partial \theta} = -f_p$$

Questions: Plant HIOA

automaton Plant($A : \text{Real}^{4 \times 4}, B : \text{Real}^{4 \times 1}$)

variables

input $u : \text{Real};$

output $\theta : \text{Real}; x : \text{Real};$

internal $\theta_h : \text{Real} := 0; \dot{\theta}_h : \text{Real} := 0;$

$x_h : \text{Real} := 0; \dot{x}_h : \text{Real} := 0;$

trajectories

trajdef plantDynamics

evolve $d(x_h) = \dot{x}_h;$

$d(\dot{x}_h) = -a_{22}\dot{x}_h - a_{23}\theta_h + a_{24}\dot{\theta}_h + b_2u;$

$d(\theta_h) = \dot{\theta}_h;$

$d(\dot{\theta}_h) = a_{42}\dot{x}_h + a_{43}\theta_h - a_{34}\dot{\theta}_h - b_4u;$

$\theta = \theta_h; x = x_h;$

Figure: Linearized Plant Model

Questions: Sensor HIOA

automaton $\text{Sensor}(T_s : \text{Real})$ **where** $T_s > 0$

signature

output $\text{sample}(\theta', x' : \text{Real})$

variables

input $\theta : \text{Real}; x : \text{Real};$

internal $\theta_s : \text{Real}; x_s : \text{Real};$

$\text{now}_s : \text{Real} := 0;$

$\text{next_sample} : \text{AugmentedReal} := T_s;$

let $\text{time_left} := \text{next_sample} - \text{now}_s$

transitions

output $\text{sample}(\theta', x')$

pre $\text{now}_s = \text{next_sample} \wedge \theta' = \theta_s \wedge x' = x_s;$

eff $\text{next_sample} := \text{next_sample} + T_s;$

trajectories

trajdef periodicSample

stop when $\text{now}_s = \text{next_sample}$

evolve $d(\text{now}_s) = 1; \theta_s = \theta; x_s = x;$

Figure: Sensor

Questions: Actuator HIOA

automaton Actuator($T_a : \text{Real}$) **where** $T_a > 0$

signature

input controllerOutput($u' : \text{Real}$)

variables

output $u : \text{Real}$;

internal $u_a : \text{Discrete Real} := 0$;

$\text{ready}_a : \text{Bool} := \text{false}$;

$\text{now}_a : \text{Real} := 0$;

transitions

input switchingOutput(u')

eff $u_a = u'$;

$\text{ready}_a := \text{true}$;

trajectories

trajdef hold

evolve $d(\text{now}_a) = 1$; $u = u_a$;

Figure: Actuator

Questions: Safety, Baseline, Experimental Controller HIOAs

automaton SafetyController($K_{sc} : \text{Real}^{4 \times 1}$, T_s , $T_{safety} : \text{Real}$, $m : \text{Int}$)

signature

input sample($\theta', x' : \text{Real}$)

output safetyOutput($u'_{sc} : \text{Real}$)

variables

internal $\theta_{sc} : \text{Real} := 0$; $\dot{\theta}_{sc} : \text{Real} := 0$;

$x_{sc} : \text{Real} := 0$; $\dot{x}_{sc} : \text{Real} := 0$;

$u_{sc} : \text{Real} := 0$; $rt : \text{Real} := 0$;

$next_cycle : \text{AugmentedReal} := T_{safety}$;

$buffer : \text{Seq}[prevTheta : \text{Real}, prevX : \text{Real}] := \{\}$;

let $time_left := next_cycle - rt$;

let $length := length(buffer)$

Figure: Safety Controller

Questions: Safety, Baseline, Experimental Controller HIOAs (cont)

transitions

input $\text{sample}(\theta', x')$

eff $\text{buffer} := \text{buffer} \vdash [\theta_{\text{sc}}, x_{\text{sc}}]$

$\theta_{\text{sc}} := \theta'; x_{\text{sc}} := x';$

output $\text{safetyOutput}(u'_{\text{sc}})$

pre $rt = \text{next_cycle} \wedge u'_{\text{sc}} = u_{\text{sc}}$

eff $\text{next_cycle} := \text{next_cycle} + T_{\text{safety}};$

$\dot{\theta}_{\text{sc}} := [\theta_{\text{sc}} - \text{head}(\text{buffer}).\text{prevTheta}] / (mT_s);$

$\dot{x}_{\text{sc}} := [x_{\text{sc}} - \text{head}(\text{buffer}).\text{prevX}] / (mT_s);$

$u'_{\text{sc}} := K_{\text{sc1}} * x_{\text{sc}} + K_{\text{sc2}} * \dot{x}_{\text{sc}} + K_{\text{sc3}} * \theta_{\text{sc}} + K_{\text{sc4}} * \dot{\theta}_{\text{sc}};$

$\text{buffer} := \text{tail}(\text{buffer});$

trajectories

trajdef periodicControl

stop when $rt = \text{next_cycle}$

evolve $d(rt) = 1;$

Figure: Safety Controller