1 General Considerations

$$\frac{D\rho}{Dt} = \frac{\partial\rho}{\partial t} + u_i \frac{\partial\rho}{\partial x_i} \neq 0 \tag{1}$$

- · Wave propagation
- · Convective flows with buoancy
- Flows with variable temperature, friction, sources of heat
- High speed flows with Mach numbers $Ma \ge 1$

Compressible flows can still be described through the continuum model and conservation laws. The assumption is also that the thermodynamic state of the fluid is in a local equilibrium.

Assumptions

- Length scale of flows $\underline{\text{large}}$ compared to molecular scales (mean free path λ)
- Length scale of flows $\underline{\text{small}}$ compared to the geometric scales (length L)
- Time scale τ_F of the flow <u>long</u> compared to the molecular process (relaxation) time constants τ_B

Description of the "Continuum" Flow State

- Three components of flow velocity u(x,t)
- The fluid density $\rho(x,t)$
- The fluid pressure p(x,t)
- The energy e(x,t)

The required equations are the conservation laws for mass, momentum and energy together with suitable thermodynamic equations of state. With corresponding initial and boundary conditions, the evolution can then be computed.

2 Thermodynamic Relations

State Variables

- Density: $\rho = \rho(p, T)$
- Pressure: $p = p(\rho, T)$

- Temperature: $T = T(\rho, p)$
- Internal energy: $e = e(\rho, T) [e] = J/kg$
- Enthalpy: h = h(p, T)
- Entropy: $s = s(\rho, T)$

Van der Waals Gas

$$(p+a\rho^2)\left(\frac{1}{\rho}-b\right) = RT\tag{2}$$

Incompressible Fluid

$$\rho = const. \neq \rho(p, T) \tag{3}$$

3 Conservation Laws for Continuum Flows

$$\frac{Dm}{Dt} = \frac{D}{Dt} \int_{\tilde{V}} \rho d\tilde{V} = 0 \ (material \ volume) \tag{4}$$

$$\int_{V} \frac{\partial \rho}{\partial t} dV + \int_{S} \rho(\mathbf{u} \cdot \mathbf{n}) dS = 0 \ (Eulerian \ Volume)$$
 (5)

$$\frac{\partial \rho}{\partial t} + \frac{\partial}{\partial x_i} (\rho u_i) = 0 \ (material \ volume \ / \ index) \tag{6}$$

$$\frac{D\rho}{Dt} = -\rho \frac{\partial u_i}{\partial x_i} \left(Eulerian \ Volume / \ index \right) \tag{7}$$

Mass Conservation

Material Volume

$$\frac{Dm}{Dt} = \frac{D}{Dt} \int_{\vec{V}} \rho d\vec{V} = 0 \tag{8}$$

$$\frac{\partial \rho}{\partial t} + \frac{\partial \rho}{\partial x_i}(\rho u_i) = 0 \tag{9}$$

Eulerian Volume

$$\int_{V} \frac{\partial \rho}{\partial t} dV + \int_{S} \rho(\vec{u} \cdot \vec{n}) dS = 0$$
 (10)

$$\frac{D\rho}{Dt} = -\rho \frac{\partial u_i}{\partial x_i} \tag{11}$$

Momentum Conservation

$$\frac{\partial}{\partial t}(\rho u_i) + \frac{\partial}{\partial x_j}(\rho u_i u_j) = \frac{\partial}{\partial x_j}\sigma_{ij} + \rho f_i \tag{12}$$

$$\rho \frac{Du_i}{Dt} = \frac{\partial}{\partial x_j} \sigma_{ij} + \rho f_i \tag{13}$$

$$\sigma_{ij} = -p\delta_{ij} + \tau_{ij} \tag{14}$$

$$\tau_{ij} = \mu \left(\frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right) + \left(\mu_v - \frac{2}{3} \mu \right) \delta_{ij} \frac{\partial u_k}{\partial x_k}$$
 (15)

$$\rho \frac{Du_i}{Dt} = -\frac{\partial p}{\partial x_i} + \frac{\partial}{\partial x_j} \left[\mu \left(\frac{\partial u_i}{x_j} + \frac{\partial u_j}{\partial x_i} \right) + \left(\mu_v - \frac{2}{3} \mu \right) \delta_{ij} \frac{\partial u_k}{\partial x_k} \right] + \rho f_i$$
(16)

Energy Conservation

$$\rho \frac{D}{Dt}(e + \frac{1}{2}u_1^2) = \frac{\partial}{\partial x_j}(\sigma_{ij}u_i) + \rho f_i u_i - \frac{\partial q_i}{\partial x_i} + \rho q_v \quad (17)$$

$$\rho \frac{D}{Dt}(e + \frac{1}{2}u_1^2) = -\frac{\partial}{\partial x_i}(pu_i) + \frac{\partial}{\partial x_j}(\tau_{ij}u_i) + \rho f_i u_i - \frac{\partial q_i}{\partial x_i} + \rho q_v$$
(18)

$$\begin{split} \rho u_i \frac{D u_i}{D t} &= \rho \frac{D}{D t} \left(\frac{u_i^2}{2} \right) = -u_i \frac{\partial p}{\partial x_i} + u_i \frac{\partial}{\partial x_j} \tau_{ij} + \rho f_i u_i \ \rho \frac{D e}{D t} &= \\ \rho \frac{D}{D t} \left(e + \frac{1}{2} u_i^2 \right) - \rho \frac{D}{D t} \left(\frac{u_i^2}{2} \right) = \\ &= -p \frac{\partial u_i}{\partial x_i} + \tau_{ij} \frac{\partial u_i}{\partial x_j} + \rho q_v - \frac{\partial q_i}{\partial x_i} \end{split}$$

Dissipation Function Φ

Insert $h=e+\frac{p}{\rho}$ to obtain Enthalpy equation, introduce $h_t=h+\frac{u_i^2}{2}$ and add kinetic energy (p. 15). For perfect gasses, $h=c_pT$, $q_i=-k\frac{dT}{dx}$, derive the temperature equation.

Entropy Equation

$$\rho T \frac{Ds}{Dt} = \Phi + \rho q_v - \frac{\partial q_i}{\partial x_i} \tag{19}$$

Vorticity Equation

$$\rho \frac{D}{Dt} \left(\frac{\vec{\omega}}{\rho} \right) = (\vec{\omega} \cdot \nabla) \vec{u} + \frac{1}{\rho^2} \nabla \rho \times \nabla p + \nabla \times \left(\frac{1}{\rho} \nabla \cdot \vec{\tau} \right)$$
 (20)

Crocco Theorem (rewritten momentum equation using Enthalpy and Entropy)

$$\frac{\partial u}{\partial t} + \nabla \left(\frac{1}{2} \vec{u}^2 + h + \psi \right) = \vec{u} \times \vec{\omega} + T \nabla s + \frac{1}{\rho} \nabla \cdot \vec{\tau}$$
 (21)

Compressible Bernoulli

equation (integrate momentum equation law along particle path). Clasical not feasible

$$\rho \left(\frac{Dh_t}{Dt} - f_i u_i \right) = 0 \tag{22}$$

$$f_i = -\frac{\partial \psi}{\partial x_i} \tag{23}$$

$$\psi \neq \psi(t) \tag{24}$$

$$\frac{D}{Dt}\left(h_t + \psi\right) = 0\tag{25}$$

Between 2 points along stream line

$$h_t + \psi = e + \frac{p}{\rho} + \frac{u_i^2}{2} + \psi = const.$$
 (26)

4 Simplification Strategies (p.20)

- Unsteady \rightarrow steady (no wave propagation) (no time dependence)
- $3D \rightarrow 2D \rightarrow quasi 1-D$
- Viscous, heat conduction → inviscid, adiabatic (isentropic, homentropic)
- Subsonic → transonic → supersonic → hypersonic (Elliptic → hyperbolic)
- Full nonlinear → linearised (solve for small pertubations around predefined flow state unique solvable problem, separation of influencing factors facilitated)

5 Conservation Laws for Stream Tubes (p. 22)

Quasi 1D, separate for environment. Outer surface formed by instantaneous streamlines, no flow across boundaries. Inlet + outlet. Shape (t). For small enough A, flow properties can be treated constant in any cross section.

Mass Conservation

$$\int_{1}^{2} \frac{\partial}{\partial t} \left[\rho(s, t) A(s, t) \right] ds + \rho_{2} A_{2} u_{2} - \rho_{1} A_{1} u_{1} = 0$$
 (27)

$$\dot{m} = \rho A u = const. \tag{28}$$

Momentum Conservation

$$\int_{1}^{2} \frac{\partial}{\partial t} \left[\rho(s, t) A(s, t) \right] ds + \rho_{2} A_{2} u_{2} \vec{u}_{2} - \rho_{1} A_{1} u_{1} \vec{u}_{1} =$$
(29)
$$= -p_{2} A_{2} \vec{n}_{2} + p_{1} A_{1} \vec{n}_{1} + F_{r}|_{1}^{2} + F_{S}$$
(30)

Steady, frictionless

$$\rho_2 u_2^2 + p_2 = \rho_1 u_1^2 + p_1 \tag{31}$$

Energy Conservation (p.20)

Steady, frictionless

$$e_2 + \frac{u_2^2}{2} + \frac{p_2}{\rho_2} = e_1 + \frac{u_1^2}{2} + \frac{p_1}{\rho_1}$$
 (32)

Enthalpy substitution $h=e+\frac{p}{\rho}\to h_{t1}=h_{t2}=const.$

6 Steady one-dimensional Flow without Friction and Heat

Assumptions:

- No friction (inviscid)
- No heat source or transport
- · No flow through mantle
- · Perfect gas

$$Ma = -\frac{u}{a} \tag{33}$$

$$a^2 = \gamma RT \tag{34}$$

Stagnation properties, when u=0:

$$\frac{h_0}{h} = \frac{T_0}{T} = \left(\frac{a_0^2}{a^2}\right) = 1 + \frac{\gamma - 1}{2}Ma^2 \tag{35}$$

Isentropic flow:

$$\frac{p_0}{p} = \left(\frac{T_0}{T}\right)^{\frac{\gamma}{\gamma - 1}} = \left[1 + \frac{\gamma - 1}{2}Ma^2\right]^{\frac{\gamma}{\gamma - 1}} \tag{36}$$

$$\frac{\rho_0}{\rho} = \left(\frac{T_0}{T}\right)^{\frac{1}{\gamma - 1}} = \left[1 + \frac{\gamma - 1}{2}Ma^2\right]^{\frac{1}{\gamma - 1}} \tag{37}$$

When Ma < 0.3, density changes < 4.5%: Assumption is: incompressible. The critical state is then (Ma = 1), superscript *

$$\frac{h^*}{h_0} = \frac{T^*}{T_0} = \left(\frac{a^{*2}}{a_0^2}\right) = \left[1 + \frac{\gamma - 1}{2}\right]^{-1} = \frac{2}{\gamma + 1} = 0.8333(\gamma = 1.4)$$

$$\frac{p^*}{p_0} = \left(\frac{2}{\gamma + 1}\right)^{\frac{\gamma}{\gamma - 1}} = 0.5283(\gamma = 1.4) \tag{38}$$

$$\frac{\rho^*}{\rho_0} = \left(\frac{2}{\gamma + 1}\right)^{\frac{1}{\gamma - 1}} = 0.6339(\gamma = 1.4) \tag{39}$$

Critical Ma^* :

$$Ma^* = \frac{u}{a^*} = \frac{u}{a(Ma=1)} = \frac{u}{a} \frac{a}{a_0} \frac{a_0}{a^*}$$
 (40)

$$= Ma\sqrt{\frac{T}{T_0}}\sqrt{\frac{T_0}{T^*}} = \sqrt{\frac{\frac{\gamma+1}{2}Ma^2}{1 + \frac{\gamma-1}{2}Ma^2}}$$
 (41)

$$Ma^* \to \sqrt{\frac{\gamma + 1}{\gamma - 1}} \ (Ma \to \infty) = 2.4495 \ (\gamma = 1.4)$$
 (42)

Area velocity relation

A velocity increase \rightarrow density decrease. If Ma << 1, then the density changes are small compared to the velocity changes.

$$Ma^2 \frac{1}{u} \frac{du}{dx} = -\frac{1}{\rho} \frac{d\rho}{dx} \tag{43}$$

$$(Ma^2 - 1)\frac{1}{u}\frac{du}{dx} = \frac{1}{A}\frac{dA}{dx} \tag{44}$$

If Ma < 1, then an area increase will result in a velocity reduction. If Ma > 1, then opposite applies. If Ma = 1, then a change has no effect (chocked flow)

Stationary normal shock

$$\frac{u_2}{u_1} = \frac{\rho_1}{\rho_2} = 1 - \frac{2}{\gamma + 1} \left(1 - \frac{1}{Ma_1^2} \right) = \frac{1}{Ma^{*2}}$$
 (45)

$$\frac{p_2}{p_1} = 1 + \frac{2\gamma}{\gamma + 1} \left(Ma_1^2 - 1 \right) \tag{46}$$

$$\frac{T_2}{T_1} = \left[1 + \frac{2\gamma}{\gamma + 1} \left(Ma_1^2 - 1\right)\right] \left[1 - \frac{2}{\gamma + 1} \left(1 - \frac{1}{Ma_1^2}\right)\right]$$
(47)

$$\begin{split} \frac{\Delta s}{R} &= \frac{1}{\gamma - 1} \left[\ln \left(\frac{p_2}{p_1} \right) - \gamma \ln \left(\frac{\rho_2}{\rho_1} \right) \right] = \\ \frac{1}{\gamma - 1} \left\{ \left[1 + \frac{2\gamma}{\gamma + 1} \left(Ma_1^2 - 1 \right) \right] \left[1 - \frac{2}{\gamma + 1} \left(1 - \frac{1}{Ma_1^2} \right) \right] \right\} \end{split}$$

 $h_{01} = h_{02}$, $T_{01} = T_{02}$, and total enthalpy conserved (however stagnation pressure not constant, $p_{01} \neq p_{02}$):

$$\begin{split} \frac{p_{02}}{p_{01}} &= \frac{p_{02}}{p_2} \frac{p_2}{p_1} \frac{p_1}{p_{01}} = \frac{p_2}{P-1} \left(\frac{T_{02}}{T_2}\right)^{\frac{\gamma}{\gamma-1}} \left(\frac{T_1}{T_{01}}\right)^{\frac{\gamma}{\gamma-1}} = \\ \left[1 + \frac{2\gamma}{\gamma+1} (Ma_1^2 - 1)\right]^{\frac{1}{\gamma-1}} \left[1 - \frac{2}{\gamma+1} \left(1 - \frac{1}{Ma_1^2}\right)\right]^{\frac{-\gamma}{\gamma-1}} \end{split}$$

As s increases, u decreases. Ma_2 is always < 1, when $Ma_1 \to \infty$:

$$Ma_2 \to \sqrt{\frac{\gamma - 1}{2\gamma}} = 0.38 \ (\gamma = 1.4)$$
 (48)

$$Ma_2^2 = \left(\frac{u_2}{a_2}\right)^2 = \left(\frac{u_2}{u_1}\right)^2 \left(\frac{u_1}{a_1}\right)^2 \left(\frac{a_1}{a_2}\right)^2 = \left(\frac{u_2}{u_1}\right)^2 Ma_1^2 \left(\frac{T_1}{T_2}\right)^2$$

$$Ma_2 = \sqrt{\frac{1 + \frac{\gamma - 1}{\gamma + 1} \left(Ma_1^2 - 1 \right)}{1 + \frac{2\gamma}{\gamma + 1} \left(Ma_1^2 - 1 \right)}} \tag{49}$$

Rankine Hugoniot (p.32) - Adiabatic Shock

$$\frac{p_2}{p_1} = 1 + \frac{2\gamma \left(\frac{\rho_2}{\rho_1} - 1\right)}{\gamma + 1 - (\gamma - 1)\frac{\rho_2}{\rho_1}}$$

Moving Shock Wave (p.33)

Switch to reference frame (from frame fixed with moving shock front into a frame moving with shock)

$$u_1 = u_s$$
, $p_1 = p_e$, $\rho_1 = \rho_0$

Flow behind

$$u_2 = u_s - u_d$$
, $p_2 = p_d$, $\rho_2 = \rho_d$

Shock u_d

$$u_d = u_s - u_2 = u_1 - u_2 = u_1 \left(1 - \frac{u_2}{u_1} \right) = u_1 \frac{2}{\gamma + 1} \left(1 - \frac{1}{Ma_1^2} \right)$$

$$Ma_{d} = \frac{u_{d}}{a_{d}} = \frac{u_{1} - u_{2}}{a_{d}} = \frac{u_{1}}{a_{1}} \frac{a_{1}}{a_{d}} \left(1 - \frac{u_{2}}{u_{1}} \right) = Ma_{1} \sqrt{\frac{T_{1}}{T_{2}}} \left(1 - \frac{u_{2}}{u_{1}} \right)$$
$$u_{d} = \frac{a_{0}}{\gamma} \frac{\frac{\Delta p}{p_{0}}}{\sqrt{1 + \frac{\gamma + 1}{2} \frac{\Delta p}{p_{0}}}}$$

Pressure increase

$$\frac{\Delta p}{p_0} = \frac{p_d - p_0}{0_0} = \frac{2\gamma}{\gamma + 1} \left(Ma_S^2 - 1 \right)$$

The ratio (Pressure increase) has an asymptotic limit. For high Ma_s , the function becomes limited.

Detonations ($Ma_2 > 1$) and Deflagrations ($Ma_2 < 1$) (p.36)

Assumption: Ignore adiabatic flow, include however heat release Rayleigh line: $\frac{p_1}{p_0}=1+\frac{\rho_0}{p_0}u_0^2-\frac{\rho_0}{p_0}\frac{\rho_1}{\rho_0}u_1^2=1+\gamma Ma_0^2\left(1-\frac{\rho_0}{\rho_1}\right)$, Rankine Hugeniot with heat: $\frac{p_2}{p_0}=\frac{(\gamma+1)-(\gamma-1)\frac{\rho_0}{\rho_2}+2\gamma\hat{q}}{(\gamma+1)\frac{\rho_0}{\rho_2}-(\gamma-1)}\;,\;\hat{q}=\frac{q_{heat}}{c_0T_1}$

Chapman Jouget Point

...is the intersection where Ma=1, so $Ma_2=1=Ma_0\sqrt{\frac{\rho_0}{\rho_2}}\sqrt{\frac{\rho_0}{\rho_2}}$ The limiting case for shock cycle:

$$\frac{\rho_0}{\rho_2}|_c = \frac{u_2}{u_0}|_c = \frac{\gamma M a_0^2 + 1}{M a_0^2 (\gamma + 1)}$$

Behind the shock, the flow is subsonic \leftrightarrow strong detonation. There is a weak deflagration if the density ratio $\frac{\rho_1}{\rho_2}>1$ is large. The reaction front propagates at subsonic speed

Laval Nozzle (p. 39)

Varying cross-section:

$$\begin{split} \frac{p(x)}{p_0} &= \left[1 + \frac{\gamma - 1}{2} M a^2(x)\right]^{\frac{-\gamma}{\gamma - 1}} \\ \frac{A^*}{A(x)} &= M a(x) \left[\frac{2}{\gamma + 1} + \frac{\gamma - 1}{\gamma + 1} M a^2(x)\right] \\ u(x) &= M a(x) a_0 \frac{a(x)}{a_0} = M a(x) a_0 \sqrt{\frac{T(x)}{T_0}} = \frac{a_0 \cdot M a(x)}{\sqrt{1 + \frac{\gamma - 1}{2} M a^2(x)}} \\ u^* &= a^* \text{, if } M a^* = 1 \end{split}$$

In order to increase the Ma_{exit} , reduce the area ration (tune A^*). Different flow regimes are shown on p. 41. A variable exit area is in practice not possible

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