

1 General Considerations

$$\frac{D\rho}{Dt} = \frac{\partial\rho}{\partial t} + u_i \frac{\partial\rho}{\partial x_i} \neq 0 \quad (1)$$

- Wave propagation
- Convective flows with buoyancy
- Flows with variable temperature, friction, sources of heat
- High speed flows with Mach numbers $Ma \geq 1$

Compressible flows can still be described through the continuum model and conservation laws. The assumption is also that the thermodynamic state of the fluid is in a local equilibrium.

Assumptions

- Length scale of flows large compared to molecular scales (mean free path λ)
- Length scale of flows small compared to the geometric scales (length L)
- Time scale τ_F of the flow long compared to the molecular process (relaxation) time constants τ_R

Description of the “Continuum” Flow State

- Three components of flow velocity $\underline{u}(\underline{x}, t)$
- The fluid density $\rho(\underline{x}, t)$
- The fluid pressure $p(\underline{x}, t)$
- The energy $e(\underline{x}, t)$

The required equations are the conservation laws for mass, momentum and energy together with suitable thermodynamic equations of state. With corresponding initial and boundary conditions, the evolution can then be computed.

2 Thermodynamic Relations

State Variables

- Density: $\rho = \rho(p, T)$
- Pressure: $p = p(\rho, T)$
- Temperature: $T = T(\rho, p)$
- Internal energy: $e = e(\rho, T)$ [e] = J/kg
- Enthalpy: $h = h(p, T)$
- Entropy: $s = s(\rho, T)$

Van der Waals Gas

$$(p + a\rho^2) \left(\frac{1}{\rho} - b \right) = RT \quad (2)$$

Incompressible Fluid

$$\rho = \text{const.} \neq \rho(p, T) \quad (3)$$

3 Conservation Laws for Continuum Flows

$$\frac{Dm}{Dt} = \frac{D}{Dt} \int_{\tilde{V}} \rho d\tilde{V} = 0 \text{ (material volume)} \quad (4)$$

$$\int_V \frac{\partial\rho}{\partial t} dV + \int_S \rho(\mathbf{u} \cdot \mathbf{n}) dS = 0 \text{ (Eulerian Volume)} \quad (5)$$

$$\frac{\partial\rho}{\partial t} + \frac{\partial\rho}{\partial x_i} (\rho u_i) = 0 \text{ (material volume / index)} \quad (6)$$

$$\frac{D\rho}{Dt} = -\rho \frac{\partial u_i}{\partial x_i} \text{ (Eulerian Volume / index)} \quad (7)$$

Mass Conservation

Material Volume

$$\frac{Dm}{Dt} = \frac{D}{Dt} \int_{\tilde{V}} \rho d\tilde{V} = 0 \quad (8)$$

$$\frac{\partial\rho}{\partial t} + \frac{\partial\rho}{\partial x_i} (\rho u_i) = 0 \quad (9)$$

Eulerian Volume

$$\int_V \frac{\partial\rho}{\partial t} dV + \int_S \rho(\tilde{\mathbf{u}} \cdot \tilde{\mathbf{n}}) dS = 0 \quad (10)$$

$$\frac{D\rho}{Dt} = -\rho \frac{\partial u_i}{\partial x_i} \quad (11)$$

Momentum Conservation

$$\frac{\partial}{\partial t} (\rho u_i) + \frac{\partial}{\partial x_j} (\rho u_i u_j) = \frac{\partial}{\partial x_j} \sigma_{ij} + \rho f_i \quad (12)$$

$$\rho \frac{Du_i}{Dt} = \frac{\partial}{\partial x_j} \sigma_{ij} + \rho f_i \quad (13)$$

$$\sigma_{ij} = -p\delta_{ij} + \tau_{ij} \quad (14)$$

$$\tau_{ij} = \mu \left(\frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right) + \left(\mu_v - \frac{2}{3}\mu \right) \delta_{ij} \frac{\partial u_k}{\partial x_k} \quad (15)$$

$$\rho \frac{Du_i}{Dt} = -\frac{\partial p}{\partial x_i} + \frac{\partial}{\partial x_j} \left[\mu \left(\frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right) + \left(\mu_v - \frac{2}{3}\mu \right) \delta_{ij} \frac{\partial u_k}{\partial x_k} \right] + \rho f_i \quad (16)$$

Energy Conservation

$$\rho \frac{D}{Dt} \left(e + \frac{1}{2} u_1^2 \right) = \frac{\partial}{\partial x_j} (\sigma_{ij} u_i) + \rho f_i u_i - \frac{\partial q_i}{\partial x_i} + \rho q_v \quad (17)$$

$$\rho \frac{D}{Dt} \left(e + \frac{1}{2} u_1^2 \right) = -\frac{\partial}{\partial x_i} (p u_i) + \frac{\partial}{\partial x_j} (\tau_{ij} u_i) + \rho f_i u_i - \frac{\partial q_i}{\partial x_i} + \rho q_v \quad (18)$$

$$\begin{aligned} \rho u_i \frac{Du_i}{Dt} &= \rho \frac{D}{Dt} \left(\frac{u_i^2}{2} \right) = -u_i \frac{\partial p}{\partial x_i} + u_i \frac{\partial}{\partial x_j} \tau_{ij} + \rho f_i u_i - \rho \frac{D e}{Dt} = \\ &= \rho \frac{D}{Dt} \left(e + \frac{1}{2} u_i^2 \right) - \rho \frac{D}{Dt} \left(\frac{u_i^2}{2} \right) = \\ &= -p \frac{\partial u_i}{\partial x_i} + \tau_{ij} \frac{\partial u_i}{\partial x_j} + \rho q_v - \frac{\partial q_i}{\partial x_i} \end{aligned}$$

Dissipation Function Φ

Insert $h = e + \frac{p}{\rho}$ to obtain Enthalpy equation, introduce $h_t = h + \frac{u_i^2}{2}$ and add kinetic energy (p. 15). For perfect gasses, $h = c_p T$, $q_i = -k \frac{dT}{dx}$, derive the temperature equation.

Entropy Equation

$$\rho T \frac{Ds}{Dt} = \Phi + \rho q_v - \frac{\partial q_i}{\partial x_i} \quad (19)$$

Vorticity Equation

$$\rho \frac{D}{Dt} \left(\frac{\vec{\omega}}{\rho} \right) = (\vec{\omega} \cdot \nabla) \vec{u} + \frac{1}{\rho^2} \nabla \rho \times \nabla p + \nabla \times \left(\frac{1}{\rho} \nabla \cdot \vec{\tau} \right) \quad (20)$$

Crocco Theorem (rewritten momentum equation using Enthalpy and Entropy)

$$\frac{\partial u}{\partial t} + \nabla \cdot \left(\frac{1}{2} \vec{u}^2 + h + \psi \right) = \vec{u} \times \vec{\omega} + T \nabla s + \frac{1}{\rho} \nabla \cdot \vec{\tau} \quad (21)$$

Compressible Bernoulli

equation (integrate momentum equation law along particle path). Classical not feasible

$$\rho \left(\frac{Dh_t}{Dt} - f_i u_i \right) = 0 \quad (22)$$

$$f_i = - \frac{\partial \psi}{\partial x_i} \quad (23)$$

$$\psi \neq \psi(t) \quad (24)$$

$$\frac{D}{Dt} (h_t + \psi) = 0 \quad (25)$$

Between 2 points along stream line

$$h_t + \psi = e + \frac{p}{\rho} + \frac{u_i^2}{2} + \psi = \text{const.} \quad (26)$$

4 Simplification Strategies

- Unsteady \rightarrow steady (no wave propagation)
 - 3D \rightarrow 2D \rightarrow quasi 1-D
 - Viscous, heat conduction \rightarrow inviscid, adiabatic (isentropic, homentropic)
 - Subsonic \rightarrow transonic \rightarrow supersonic \rightarrow hypersonic (Elliptic \rightarrow hyperbolic)
 - Full nonlinear \rightarrow linearised (solve for small perturbations around predefined flow state unique solvable problem, separation of influencing factors facilitated)
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5 Conservation Laws for Stream Tubes

Quasi 1D, separate for environment. Outer surface formed by instantaneous streamlines, no flow across boundaries. Inlet + outlet. Shape (t). For small enough A , flow properties can be treated constant in any cross section.

Mass Conservation

$$\int_1^2 \frac{\partial}{\partial t} [\rho(s, t) A(s, t)] ds + \rho_2 A_2 u_2 - \rho_1 A_1 u_1 = 0 \quad (27)$$

$$\dot{m} = \rho A u = \text{const.} \quad (28)$$

Momentum Conservation

$$\int_1^2 \frac{\partial}{\partial t} [\rho(s, t) A(s, t)] ds + \rho_2 A_2 u_2 \vec{u}_2 - \rho_1 A_1 u_1 \vec{u}_1 = \quad (29)$$

$$= -p_2 A_2 \vec{n}_2 + p_1 A_1 \vec{n}_1 + F_\tau|_1^2 + F_S \quad (30)$$

Steady, frictionless

$$\rho_2 u_2^2 + p_2 = \rho_1 u_1^2 + p_1 \quad (31)$$

Energy Conservation (p.20)

Steady, frictionless

$$e_2 + \frac{u_2^2}{2} + \frac{p_2}{\rho_2} = e_1 + \frac{u_1^2}{2} + \frac{p_1}{\rho_1} \quad (32)$$

Enthalpy substitution $h = e + \frac{p}{\rho} \rightarrow h_{t1} = h_{t2} = \text{const.}$

6 Steady one-dimensional Flow without Friction and Heat

Assumptions:

- No friction (inviscid)
 - No heat source or transport
 - No flow through mantle
 - Perfect gas
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7 Unsteady one-dimensional Flows

8 Two-dimensional steady supersonic Flow

9 Method Characteristics for planar homentropic supersonic Flows

10 Homentropic Flow around slender Wings

11 Homentropic Flow around axisymmetric slender Bodies

12 Similarity Relations