





Lecture 5_1: Dynamics: Newton-Euler

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Outlines

- * Introduction
- * Acceleration of a Rigid Body
- * Mass Distribution
- * Newton's Equation, Euler's Equation
- * <u>Iterative Newton-Euler Dynamic Formulation</u>
- * The Structure of a Manipulator's Dynamic Equations
- Dynamic Equations in Cartesian Space
- Inclusion of Non-rigid Body Effects

Introduction

- Kinematics: Study of positions, static forces, and velocities.
- Dynamics: Study of forces required to cause motion.
- Motion of the manipulator arises from:
 - > Torques applied by the actuators
 - **External forces** applied to the manipulator
- Dynamics of mechanisms is an extensive field.
 - Here, we consider **certain formulations** of the dynamics problem seem particularly well suited to application to **serial manipulators**.
- **Two problems** related to the dynamics:
 - Fiven a trajectory point, θ , $\dot{\theta}$ and $\ddot{\theta}$, find the required vector of joint torques, τ (Control).
 - Fiven a torque vector, τ , calculate the resulting motion of the manipulator, θ , $\dot{\theta}$ and $\ddot{\theta}$ (Simulation).

• The linear and angular velocity vectors have <u>derivatives</u> that are called the linear and angular accelerations.

$${}^{B}\dot{V}_{Q} = \frac{d}{dt} {}^{B}V_{Q} = \lim_{\Delta t \to 0} \frac{{}^{B}V_{Q}(t + \Delta t) - {}^{B}V_{Q}(t)}{\Delta t}$$

$${}^{A}\dot{\Omega}_{B} = \frac{d}{dt}{}^{A}\Omega_{B} = \lim_{\Delta t \to 0} \frac{{}^{A}\Omega_{B}(t + \Delta t) - {}^{A}\Omega_{B}(t)}{\Delta t}$$

• When the reference frame of the differentiation is universal reference frame, {U}:

$$\dot{v}_A = {}^U \dot{V}_{AORG}$$

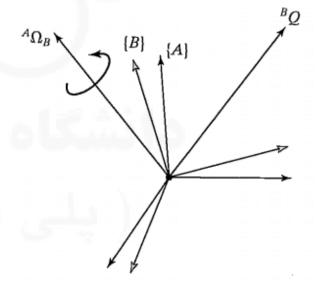
$$\dot{\omega}_A = {}^U\dot{\Omega}_A$$

□ Linear Acceleration

Remark: The velocity of a vector BQ as seen from frame $\{A\}$ when the origins are coincident:

$$^{A}Q = {}^{A}R_{B}{}^{B}Q$$

$${}^AV_Q = {}^AR_B{}^BV_Q + {}^A\Omega_B \times {}^AR_B{}^BQ$$



Linear Acceleration

Remark:

$${}^{A}Q = {}^{A}R_{B}{}^{B}Q$$

$${}^{A}V_{Q} = {}^{A}R_{B}{}^{B}V_{Q} + {}^{A}\Omega_{B} \times {}^{A}R_{B}{}^{B}Q$$

$$(1)$$

The LHS describes how ${}^{A}Q$ is changing in time, so

$${}^{A}V_{Q} = \frac{d}{dt}({}^{A}Q) = \frac{d}{dt}({}^{A}R_{B}{}^{B}Q) = {}^{A}R_{B}{}^{B}V_{Q} + {}^{A}\Omega_{B} \times {}^{A}R_{B}{}^{B}Q$$

(2)

Differentiating Eq. (1):

$${}^{A}\dot{V}_{Q} = \frac{d}{dt} \left({}^{A}R_{B} {}^{B}V_{Q} \right) + {}^{A}\dot{\Omega}_{B} \times {}^{A}R_{B} {}^{B}Q + {}^{A}\Omega_{B} \times \frac{d}{dt} \left({}^{A}R_{B} {}^{B}Q \right)$$

$$\tag{3}$$

Applying Eq. (2) twice (to the first & last term), the RHS of Eq. (3) becomes

$${}^{A}\dot{V}_{Q} = \left\{ {}^{A}R_{B}{}^{B}\dot{V}_{Q} + {}^{A}\Omega_{B} \times {}^{A}R_{B}{}^{B}V_{Q} \right\} + {}^{A}\dot{\Omega}_{B} \times {}^{A}R_{B}{}^{B}Q + \left\{ {}^{A}\Omega_{B} \times {}^{A}R_{B}{}^{B}V_{Q} + {}^{A}\Omega_{B} \times {}^{A}\Omega_{B} \times {}^{A}R_{B}{}^{B}Q \right\}$$

(4)

□ Linear Acceleration

Combining two similar terms in Eq. (4),

$${}^{A}\dot{V}_{Q} = {}^{A}R_{B}{}^{B}\dot{V}_{Q} + 2 {}^{A}\Omega_{B} \times {}^{A}R_{B}{}^{B}V_{Q} + {}^{A}\dot{\Omega}_{B} \times {}^{A}R_{B}{}^{B}Q + {}^{A}\Omega_{B} \times ({}^{A}\Omega_{B} \times {}^{A}R_{B}{}^{B}Q)$$

Generalize to the case in which the origins are not coincident:

$${}^{A}\dot{V}_{Q} = {}^{A}\dot{V}_{BORG} + {}^{A}R_{B}{}^{B}\dot{V}_{Q} + 2 {}^{A}\Omega_{B} \times {}^{A}R_{B}{}^{B}V_{Q} + {}^{A}\dot{\Omega}_{B} \times {}^{A}R_{B}{}^{B}Q + {}^{A}\Omega_{B} \times ({}^{A}\Omega_{B} \times {}^{A}R_{B}{}^{B}Q)$$

• For a particular case that BQ is constant:

$${}^{B}V_{Q} = {}^{B}\dot{V}_{Q} = 0$$

$${}^{A}\dot{V}_{Q} = {}^{A}\dot{V}_{BORG} + {}^{A}\dot{\Omega}_{B} \times {}^{A}R_{B}{}^{B}Q + {}^{A}\Omega_{B} \times ({}^{A}\Omega_{B} \times {}^{A}R_{B}{}^{B}Q)$$

- For a prismatic joint: Eq. (6)
- For a revolute joint: Eq. (7)

7

(5)

(6)

(7)

☐ Angular Acceleration

- Assume $\{B\}$ is rotating relative to $\{A\}$ with ${}^A\Omega_B$ and $\{C\}$ is rotating relative to $\{B\}$ with ${}^B\Omega_C$.
- Therefore

$${}^{A}\Omega_{C} = {}^{A}\Omega_{B} + {}^{A}R_{B} {}^{B}\Omega_{C}$$

$$(8)$$

By differentiating, we obtain

$${}^{A}\dot{\Omega}_{C} = {}^{A}\dot{\Omega}_{B} + \frac{d}{dt} ({}^{A}R_{B} {}^{B}\Omega_{C})$$

$$\tag{9}$$

• Applying Eq. (2) to the last term of Eq. (9):

$${}^{A}\dot{\Omega}_{C} = {}^{A}\dot{\Omega}_{B} + {}^{A}R_{B} {}^{B}\dot{\Omega}_{C} + {}^{A}\Omega_{B} \times {}^{A}R_{B} {}^{B}\Omega_{C}$$

$$\tag{10}$$

☐ Angular Acceleration

$${}^{A}\dot{\Omega}_{C} = {}^{A}\dot{\Omega}_{B} + {}^{A}R_{B} {}^{B}\dot{\Omega}_{C} + {}^{A}\Omega_{B} \times {}^{A}R_{B} {}^{B}\Omega_{C}$$

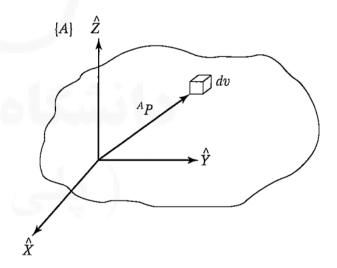
$$\tag{10}$$

• For a particular case that ${}^B\Omega_C = {}^B\dot{\Omega}_C = 0$:

$${}^{A}\dot{\Omega}_{C} = {}^{A}\dot{\Omega}_{B} \tag{11}$$

- Use these results to calculate the angular acceleration of the links of a manipulator.
- For a revolute joint: Eq. (10)
- For a prismatic joint: Eq. (11)

- For a single DOF system:
 - Mass of a rigid body is considered for linear motions.
 - Moment of inertia is considered for rotational motion.
- For a free rigid body in 3D space, there are *infinitely* many possible rotation axes.
- In the case of rotation about an arbitrary axis, it is necessary to characterize the mass distribution of a rigid body, i.e. the **inertia tensor**.
- A rigid body with an attached frame is considered.

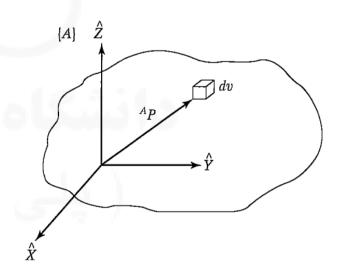


- Inertia tensors can be defined relative to any frame, but always consider a frame attached to the rigid body (*Why?*)
- The inertia tensor relative to frame $\{A\}$ is expressed in the matrix form:

$${}^{A}I = \begin{bmatrix} I_{xx} & -I_{xy} & -I_{xz} \\ -I_{xy} & I_{yy} & -I_{yz} \\ -I_{xz} & -I_{yz} & I_{zz} \end{bmatrix}$$

where

$$\begin{split} I_{xx} &= \iiint_V (y^2 + z^2) \rho dv, \qquad I_{xy} = \iiint_V xy \rho dv, \\ I_{yy} &= \iiint_V (x^2 + z^2) \rho dv, \qquad I_{xz} = \iiint_V xz \rho dv, \\ I_{zz} &= \iiint_V (x^2 + y^2) \rho dv, \qquad I_{yz} = \iiint_V yz \rho dv, \end{split}$$



- The elements $I_{\chi\chi}$, $I_{\gamma\gamma}$ and I_{zz} are called the **mass moments of inertia.**
- Note: Integrating the mass elements, ρdv , times the squares of the perpendicular distances from the corresponding axis.
- The elements with mixed indices are called the mass products of inertia.
- This set of six independent quantities depend on the position and orientation of the frame in which they are defined.
- If it is free to choose the orientation of the reference frame, So for any specified position, it is possible to <u>find an orientation</u>, which cause the products of inertia to be zero.
- The axes of the mentioned reference frame are called the **principal axes**.
- The corresponding mass moments are the principal moments of inertia.

Example 1:

• Find the inertia tensor for the rectangular body of uniform density ρ with respect to the coordinate system? \hat{z}

Moments of inertia

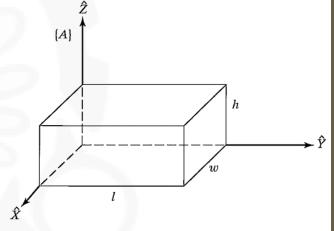
$$I_{xx} = \int_0^h \int_0^l \int_0^\omega (y^2 + z^2) \rho \, dx \, dy \, dz$$

$$= \int_0^h \int_0^l (y^2 + z^2) \omega \rho \, dy \, dz$$

$$= \int_0^h \left(\frac{l^3}{3} + z^2 l \right) \omega \rho \, dz$$

$$= \left(\frac{h l^3 \omega}{3} + \frac{h^3 l \omega}{3} \right) \rho$$

$$= \frac{m}{3} (l^2 + h^2),$$



$$I_{yy} = \frac{m}{3}(\omega^2 + h^2)$$

$$I_{zz} = \frac{m}{3}(l^2 + \omega^2).$$

Example 1:

Products of inertia

$$I_{xy} = \int_0^h \int_0^l \int_0^\omega xy\rho \, dx \, dy \, dz \qquad I_{xz} = \frac{m}{4}h\omega$$

$$= \int_0^h \int_0^l \frac{\omega^2}{2} y\rho \, dy \, dz \qquad I_{yz} = \frac{m}{4}hl.$$

$$= \int_0^h \frac{\omega^2 l^2}{4} \rho dz$$

$$= \frac{m}{4}\omega l.$$

Hence, the inertia tensor for this object is

$${}^{A}I = \begin{bmatrix} \frac{m}{3}(l^{2} + h^{2}) & -\frac{m}{4}\omega l & -\frac{m}{4}h\omega \\ -\frac{m}{4}\omega l & \frac{m}{3}(\omega^{2} + h^{2}) & -\frac{m}{4}hl \\ -\frac{m}{4}h\omega & -\frac{m}{4}hl & \frac{m}{3}(l^{2} + \omega^{2}) \end{bmatrix}$$

□ Parallel-axis theorem

- The inertia tensor is a function of the **position** and **orientation** of the reference frame.
- Parallel-axis theorem describes how the inertia tensor changes under translations of the reference coordinate system.
- It relates the inertia tensor in a frame with origin at the center of mass to the inertia tensor with respect to another reference frame:

$${}^{A}I_{zz} = {}^{C}I_{zz} + m(x_c^2 + y_c^2),$$

 ${}^{A}I_{xy} = {}^{C}I_{xy} - mx_cy_c,$

• where $\{C\}$ is located at the center of mass of the body, and $\{A\}$ is an arbitrarily translated frame.

□ Parallel-axis theorem

$${}^{A}I_{zz} = {}^{C}I_{zz} + m(x_c^2 + y_c^2),$$

 ${}^{A}I_{xy} = {}^{C}I_{xy} - mx_cy_c,$

- Assume $P_c = [x_c, y_c, z_c]^T$ locates the center of mass $\{C\}$ relative to $\{A\}$.
- It may be stated in vector-matrix form as:

$$^{A}I = ^{C}I + m[P_{c}^{T}P_{c}I_{3} - P_{c}P_{c}^{T}]$$

• where I_3 is the 3 × 3 identity matrix.

Example 2:

- Find the inertia tensor described in a coordinate system with origin at the body's center of mass?
- Applying the parallel-axis theorem,

$$\begin{bmatrix} x_c \\ y_c \\ z_c \end{bmatrix} = \frac{1}{2} \begin{bmatrix} \omega \\ l \\ h \end{bmatrix} \qquad CI_{zz} = \frac{m}{12} (\omega^2 + l^2)$$
$$CI_{xy} = 0.$$

• The resulting inertia tensor written in the frame at the center of mass is:

$${}^{C}I = \begin{bmatrix} \frac{m}{12}(h^{2} + l^{2}) & 0 & 0\\ 0 & \frac{m}{12}(\omega^{2} + h^{2}) & 0\\ 0 & 0 & \frac{m}{12}(l^{2} + \omega^{2}) \end{bmatrix}$$

• The result is diagonal, so frame $\{C\}$ must represent the **principal axes** of this body.

□ Properties of inertia tensors

- 1) If two axes of the reference frame form a plane of symmetry for the mass distribution of the body, the products of inertia having as an index that is normal to the plane of symmetry will be zero.
- 2) Moments of inertia must always be positive. Products of inertia may have either sign.
- 3) The sum of the three moments of inertia is invariant under orientation changes in the reference frame.
- 4) The eigenvalues of an inertia tensor are the principal moments for the body. The associated eigenvectors are the principal axes.

Note:

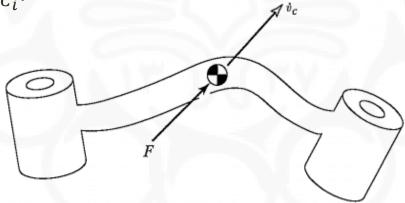
- Most manipulators have links with complicated geometry and composition. So, calculating the inertia tensor is difficult in practice.
- An alternative approach is using a measuring device (e.g., an *inertia* pendulum).

Newton's Equation, Euler's Equation

• Newton's equation, along with its rotational analog, Euler's equation, describes how forces, inertias, and accelerations relate.

☐ Newton's Equation

• Assume a rigid body whose center of mass is accelerating with acceleration \dot{v}_{C_i} .



• The force, F_i , acting at the center of mass and causing this acceleration is as follows:

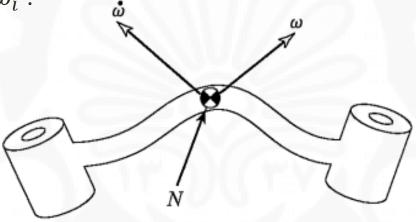
$$F_i = m_i \dot{v}_{C_i}$$

• m_i is the total mass of the body.

Newton's Equation, Euler's Equation

□ Euler's Equation

• Assume a rigid body rotating with angular velocity ${}^{i}\omega_{i}$ and with angular acceleration ${}^{i}\dot{\omega}_{i}$.



• The moment N_i , which must be acting on the body to cause this motion, is as follows:

$$N_i = {^{C_i}I_i}^i \dot{\omega}_i + {^i}\omega_i \times {^{C_i}I_i}^i \omega_i$$

• ${}^{C_i}I_i$ is the inertia tensor of the body written in a frame, $\{C\}$, whose origin is located at the center of mass.

• **Problem:** Computing the torques that correspond to a given trajectory of a manipulator.

Inputs:

- \triangleright Known position, velocity, and acceleration of the joints, $(\Theta, \dot{\Theta}, \ddot{\Theta})$
- Knowledge of the kinematics
- Mass-distribution information

Output:

- The joint torques required to cause this motion
- The method includes three steps:
 - 1) Outward iterations to compute velocities and accelerations.
 - 2) Compute the inertial forces and torques acting on the links.
 - 3) Inward iterations to compute forces and torques.

- ☐ Outward Iterations to Compute Velocities and Accelerations
- Newton's equation

$$F_i = m_i \, \dot{v}_{C_i}$$

Euler's equation

$$N_i = {^{C_i}I_i}^i \dot{\omega}_i + {^i}\omega_i \times {^{C_i}I_i}^i \omega_i$$

- To compute inertial forces, it is necessary to compute the rotational velocity (ω) and linear and rotational acceleration $(\dot{v}_C, \dot{\omega})$ of the <u>center of mass</u> of each link.
- It will be done in an iterative way, starting with link 1 and moving successively, outward to link n.

- **☐** Outward Iterations to Compute Velocities and Accelerations
- Rotational Velocity
- The "propagation" of rotational velocity:
- > Revolute joint

$$^{i+1}\omega_{i+1} = {}^{i+1}R_i{}^i\omega_i + \dot{\theta}_{i+1}{}^{i+1}\hat{Z}_{i+1}$$

Prismatic joint (!)

$$^{i+1}\omega_{i+1} = ^{i+1}R_i{}^i\omega_i$$

- **☐** Outward Iterations to Compute Velocities and Accelerations
- ☐ Angular Acceleration
- The "propagation" of angular acceleration:
- > Revolute joint

$${}^{i+1}\dot{\omega}_{i+1} = {}^{i+1}R_i{}^i\dot{\omega}_i + {}^{i+1}R_i{}^i\omega_i \times \dot{\theta}_{i+1}{}^{i+1}\hat{Z}_{i+1} + \ddot{\theta}_{i+1}{}^{i+1}\hat{Z}_{i+1}$$

- ➤ How?
- 1) Direct Differentiation of rotational velocity:

$$^{i+1}\omega_{i+1} = ^{i+1}R_i{}^i\omega_i + \dot{\theta}_{i+1}{}^{i+1}\hat{Z}_{i+1}$$

Or

• 2) Remember:

$${}^{A}\dot{\Omega}_{C} = {}^{A}\dot{\Omega}_{B} + {}^{A}R_{B} {}^{B}\dot{\Omega}_{C} + {}^{A}\Omega_{B} \times {}^{A}R_{B} {}^{B}\Omega_{C}$$

• Assume:

$$C = i + 1$$

$$B = i$$

$$A = 0$$

Prismatic joint (!)

$$^{i+1}\dot{\omega}_{i+1} = ^{i+1}R_i{}^i\dot{\omega}_i$$

- **☐** Outward Iterations to Compute Velocities and Accelerations
- ☐ Linear Acceleration
- The "propagation" of linear acceleration of each link-frame origin:
- Revolute joint (!)

$$^{i+1}\dot{v}_{i+1} = ^{i+1}R_i \left(^i\dot{\omega}_i \times ^iP_{i+1} + ^i\omega_i \times \left(^i\omega_i \times ^iP_{i+1} \right) + ^i\dot{v}_i \right)$$

- ➤ How?
- Remember:

$$^{A}\dot{V}_{O} = ^{A}\dot{V}_{BORG} + ^{A}\dot{\Omega}_{B} \times ^{A}R_{B}{}^{B}Q + ^{A}\Omega_{B} \times (^{A}\Omega_{B} \times ^{A}R_{B}{}^{B}Q)$$

• Assume:

$$Q = i + 1$$
$$B = i$$
$$A = 0$$

Prismatic joint

$$\dot{v}_{i+1} = \dot{v}_{i+1} = \dot{v}_{i} \left(\dot{\omega}_{i} \times \dot{v}_{i+1} + \dot{\omega}_{i} \times \left(\dot{\omega}_{i} \times \dot{v}_{i+1} \right) + \dot{v}_{i} \right) + 2 \dot{v}_{i+1} \\
\times \dot{d}_{i+1} \dot{v}_{i+1} + \ddot{d}_{i+1} \dot{v}_{i+1} \dot{z}_{i+1} \\
\times \dot{d}_{i+1} \dot{v}_{i+1} \dot{v}_{i+1} \dot{z}_{i+1} + \ddot{d}_{i+1} \dot{v}_{i+1} \dot{z}_{i+1}$$

- ☐ Outward Iterations to Compute Velocities and Accelerations
- ☐ Linear Acceleration

$$^{i+1}\dot{v}_{i+1}=^{i+1}R_i\left(^i\dot{\omega}_i\times^iP_{i+1}+^i\omega_i\times\left(^i\omega_i\times^iP_{i+1}\right)+^i\dot{v}_i\right)$$

- Note: a frame, $\{C_i\}$, attached to each link, having its origin located at the center of mass of the link and having the same orientation as the link frame, $\{i\}$.
- The "propagation" of linear acceleration of the center of mass of each link (!)

$$C_{i}\dot{v}_{C_{i}} = C_{i}R_{i} \left({}^{i}\dot{\omega}_{i} \times {}^{i}P_{C_{i}} + {}^{i}\omega_{i} \times \left({}^{i}\omega_{i} \times {}^{i}P_{C_{i}} \right) + {}^{i}\dot{v}_{i} \right)$$

$$C_{i}\dot{v}_{C_{i}} = C_{i}R_{i}{}^{i}\dot{v}_{C_{i}} = {}^{i}\dot{v}_{C_{i}} = {}^{i}\dot{\omega}_{i} \times {}^{i}P_{C_{i}} + {}^{i}\omega_{i} \times \left({}^{i}\omega_{i} \times {}^{i}P_{C_{i}} \right) + {}^{i}\dot{v}_{i}$$

$$(*)$$

- Eq. (*) doesn't involve joint motion and is valid for joint *i*, regardless of whether it is revolute or prismatic. (*Why?*)
- Apply the equations to link 1 by ${}^0\omega_0 = {}^0\dot{\omega}_0 = 0$.

☐ The Force and Torque Acting on a Link

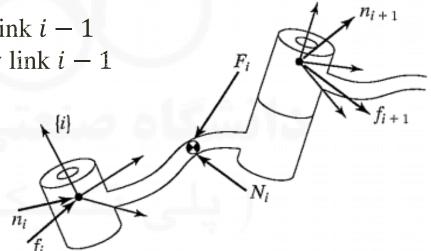
• Apply the Newton—Euler equations to compute the inertial force and torque acting at the center of mass of each link.

$$F_i = m_i \dot{v}_{C_i}$$

$$N_i = {^{C_i}I_i}^i \dot{\omega}_i + {^i}\omega_i \times {^{C_i}I_i}^i \omega_i$$

☐ Inward Iterations to Compute Forces and Torques

- Calculate the joint torques that will result in these net forces and torques being applied to each link.
- Writing a force-balance and moment-balance equation based on a freebody diagram of a typical link.
- Each link has forces and torques exerted on it by
 - 1) Its neighbors
 - 2) Inertial force and torque
- f_i = force exerted on link i by link i − 1
 n_i = torque exerted on link i by link i − 1



 $\{i+1\}$

☐ Inward Iterations to Compute Forces and Torques

Force-balance relationship

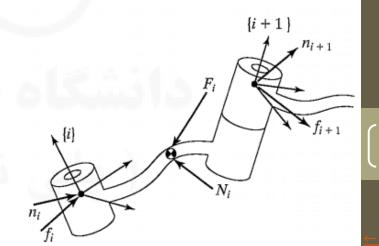
$${}^{i}F_{i} = {}^{i}f_{i} - {}^{i}R_{i+1}{}^{i+1}f_{i+1}$$

Torque-balance equation (about the center of mass)

$${}^{i}N_{i} = {}^{i}n_{i} - {}^{i}n_{i+1} + (-{}^{i}P_{C_{i}}) \times {}^{i}f_{i} - ({}^{i}P_{i+1} - {}^{i}P_{C_{i}}) \times {}^{i}f_{i+1}$$

 Using the result from the force-balance relation and adding a few rotation matrices

$${}^{i}N_{i} = {}^{i}n_{i} - {}^{i}R_{i+1}{}^{i+1}n_{i+1} + (-{}^{i}P_{C_{i}}) \times {}^{i}F_{i} - {}^{i}P_{i+1} \times {}^{i}R_{i+1}{}^{i+1}f_{i+1}$$



☐ Inward Iterations to Compute Forces and Torques

• Rearrange the equations to be appeared as iterative relationships (higher to lower numbered neighbor).

$${}^{i}f_{i} = {}^{i}R_{i+1}{}^{i+1}f_{i+1} + {}^{i}F_{i}$$

$${}^{i}n_{i} = {}^{i}N_{i} + {}^{i}R_{i+1}{}^{i+1}n_{i+1} + {}^{i}P_{C_{i}} \times {}^{i}F_{i} + {}^{i}P_{i+1} \times {}^{i}R_{i+1}{}^{i+1}f_{i+1}$$

- These equations are evaluated link by link, starting from link n going inward toward the base of the robot.
- The method is analogous to the static force iterations, except that inertial forces and torques are now considered at each link.
- The required joint torques are found by taking the Z_i component of the torque (force) applied i.e. ${}^in_i ({}^if_i)$ (As like as the static case).

☐ Inward Iterations to Compute Forces and Torques

Revolute joint

$$\tau_i = {}^i n_i^T \hat{Z}_i$$

Prismatic joint

$$\tau_i = {}^i f_i^T \hat{Z}_i$$

- The symbol τ_i is used for a linear actuator force as like as rotary actuator torque.
- For a robot moving in free space, $^{N+1}f_{N+1}$ and $^{N+1}n_{N+1}$ are set equal to zero.
- If the robot is in contact with the environment, the forces and torques due to this contact can be included in the force balance.

☐ The Iterative Newton—Euler Dynamics Algorithm

- Summary:
 - 1) Link velocities and accelerations are iteratively computed from link 1 out to link n.
 - 2) The Newton-Euler equations are applied to each link.
 - 3) Forces and torques of interaction and joint actuator torques are computed recursively from link *n* back to link 1.

☐ The Iterative Newton—Euler Dynamics Algorithm

- Summary: (Revolute joints)
- Outward iterations: $i: 0 \rightarrow 5$

$$\triangleright$$
 $i^{+1}\omega_{i+1} = i^{+1}R_i{}^i\omega_i + \dot{\theta}_{i+1}{}^{i+1}\hat{Z}_{i+1}$

$$\qquad \qquad ^{i+1}\dot{\omega}_{i+1} = {}^{i+1}R_i{}^i\dot{\omega}_i + {}^{i+1}R_i{}^i\omega_i \times \dot{\theta}_{i+1}{}^{i+1}\hat{Z}_{i+1} + \ddot{\theta}_{i+1}{}^{i+1}\hat{Z}_{i+1}$$

$$\succ \ ^{i+1}\dot{v}_{C_{i+1}} = {}^{i+1}\dot{\omega}_{i+1} \times {}^{i+1}P_{C_{i+1}} + {}^{i+1}\omega_{i+1} \times \left({}^{i+1}\omega_{i+1} \times {}^{i+1}P_{C_{i+1}}\right) + {}^{i+1}\dot{v}_{i+1}$$

$$ightharpoonup i^{i+1}F_{i+1} = m_{i+1}^{i+1}\dot{v}_{C_{i+1}}$$

$$> {}^{i+1}N_{i+1} = {}^{C_{i+1}}I_{i+1}{}^{i+1}\dot{\omega}_{i+1} + {}^{i+1}\omega_{i+1} \times {}^{C_{i+1}}I_{i+1}{}^{i+1}\omega_{i+1}$$

■ Inward iterations: $i: 6 \rightarrow 1$

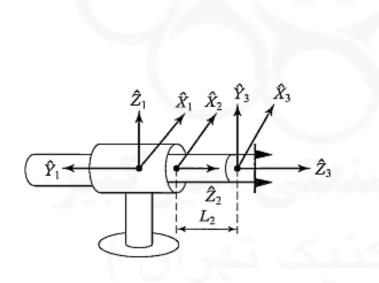
$$\rightarrow i f_i = i R_{i+1}^{i+1} f_{i+1} + i F_i$$

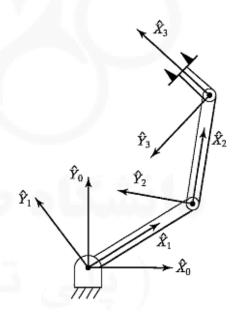
$$\rightarrow i n_i = i N_i + i R_{i+1}^{i+1} n_{i+1} + i P_{C_i} \times i F_i + i P_{i+1} \times i R_{i+1}^{i+1} f_{i+1}$$

$$\succ \tau_i = {}^i n_i^T \hat{Z}_i$$

☐ Inclusion of Gravity in the Dynamics Algorithm

- The effect of gravity can be included by setting ${}^0\dot{v}_0 = G$.
- G has the magnitude of the gravity vector but points in the opposite direction.
- It is equivalent to saying that the base of the robot is accelerating upward with 1 g acceleration.





☐ Iterative Vs. Closed Form

- This algorithm gives a computational scheme (given $(\theta, \dot{\theta}, \ddot{\theta})$, compute the required joint torques), as like as development of equations to compute the Jacobian.
- It can be used in two ways:
 - > As a Numerical Algorithm
 - Once ${}^{C}I_{i}$, m_{i} , ${}^{i}P_{C_{i}}$ and ${}^{i+1}R_{i}$ are specified, the equations can be applied directly to compute the joint torques corresponding to any motion.
 - > As an Analytical Algorithm
 - To analyze the structure of the equations (e.g. the gravity effects, the inertial effects & ...), it is useful to apply the recursive Newton-Euler equations **symbolically** to develop **closed form equations** (analogous to derive the symbolic form of the Jacobian).

Example:

- Compute the closed-form dynamic equations for the two-link RR planar manipulator
- **Assumption**: Point masses at the end of each link $(m_1 \text{ and } m_2)$
- Center of mass vectors:

$$^{1}P_{C_{1}}=l_{1}\hat{X}_{1},$$

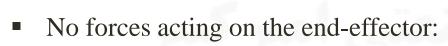
$${}^{1}P_{C_{1}} = l_{1}\hat{X}_{1},$$

 ${}^{2}P_{C_{2}} = l_{2}\hat{X}_{2}.$

Due to the point-mass assumption:

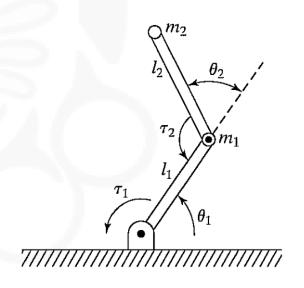
$$^{C_1}I_1=0,$$

$$^{C_2}I_2=0.$$



$$f_3 = 0$$
,

$$n_3 = 0.$$



Example:

The base of the robot is not rotating, so:

$$\omega_0 = 0,$$

$$\dot{\omega}_0 = 0.$$

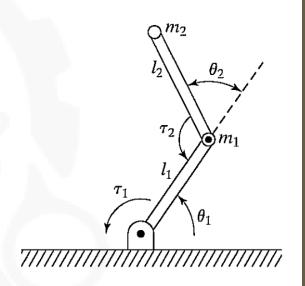
To include gravity forces:

$${}^0\dot{v}_0 = g\hat{Y}_0.$$

■ The rotation between successive link frames:

$$\begin{array}{l}
 i \\
 i_{i+1}R = \begin{bmatrix}
 c_{i+1} & -s_{i+1} & 0.0 \\
 s_{i+1} & c_{i+1} & 0.0 \\
 0.0 & 0.0 & 1.0
 \end{array}$$

$$i^{i+1}R = \begin{bmatrix} c_{i+1} & s_{i+1} & 0.0 \\ -s_{i+1} & c_{i+1} & 0.0 \\ 0.0 & 0.0 & 1.0 \end{bmatrix}$$



Example:

• The outward iterations for link 1:

$$^{1}\omega_{1}=\dot{\theta}_{1}\ ^{1}\hat{Z}_{1}=\left[egin{array}{c}0\\0\\\dot{\theta}_{1}\end{array}
ight],$$

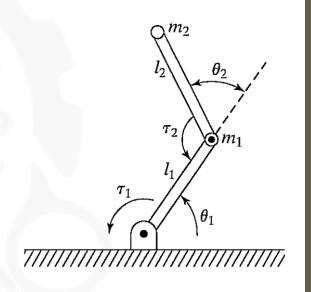
$${}^1\dot{\omega}_1=\ddot{ heta}_1{}^1\hat{Z}_1=\left[egin{array}{c}0\\0\\\ddot{ heta}_1\end{array}
ight],$$

$${}^{1}\dot{v}_{1} = \left[\begin{array}{ccc} c_{1} & s_{1} & 0 \\ -s_{1} & c_{1} & 0 \\ 0 & 0 & 1 \end{array} \right] \left[\begin{array}{c} 0 \\ g \\ 0 \end{array} \right] = \left[\begin{array}{c} gs_{1} \\ gc_{1} \\ 0 \end{array} \right],$$

$${}^{1}\dot{v}_{C_{1}} = \begin{bmatrix} 0 \\ l_{1}\ddot{\theta}_{1} \\ 0 \end{bmatrix} + \begin{bmatrix} -l_{1}\dot{\theta}_{1}^{2} \\ 0 \\ 0 \end{bmatrix} + \begin{bmatrix} gs_{1} \\ gc_{1} \\ 0 \end{bmatrix} = \begin{bmatrix} -l_{1}\dot{\theta}_{1}^{2} + gs_{1} \\ l_{1}\ddot{\theta}_{1} + gc_{1} \\ 0 \end{bmatrix},$$

$${}^{1}F_{1} = \begin{bmatrix} -m_{1}l_{1}\dot{\theta}_{1}^{2} + m_{1}gs_{1} \\ m_{1}l_{1}\ddot{\theta}_{1} + m_{1}gc_{1} \\ 0 \end{bmatrix},$$

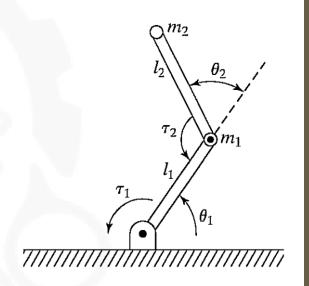
$$^{1}N_{1} = \begin{bmatrix} 0 \\ 0 \\ 0 \end{bmatrix}.$$



Example:

• The outward iterations for link 2:

$$\begin{split} ^{2}\omega_{2} &= \begin{bmatrix} & 0 \\ & 0 \\ & \dot{\theta}_{1} + \dot{\theta}_{2} \end{bmatrix}, \\ ^{2}\dot{\omega}_{2} &= \begin{bmatrix} & 0 \\ & 0 \\ & \ddot{\theta}_{1} + \ddot{\theta}_{2} \end{bmatrix}, \\ ^{2}\dot{v}_{2} &= \begin{bmatrix} & c_{2} & s_{2} & 0 \\ & -s_{2} & c_{2} & 0 \\ & 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} & -l_{1}\dot{\theta}_{1}^{2} + gs_{1} \\ & l_{1}\ddot{\theta}_{1} + gc_{1} \end{bmatrix} = \begin{bmatrix} & l_{1}\ddot{\theta}_{1}s_{2} - l_{1}\dot{\theta}_{1}^{2}c_{2} + gs_{12} \\ & l_{1}\ddot{\theta}_{1}c_{2} + l_{1}\dot{\theta}_{1}^{2}s_{2} + gc_{12} \end{bmatrix}, \\ ^{2}\dot{v}_{C_{2}} &= \begin{bmatrix} & 0 \\ & l_{2}(\ddot{\theta}_{1} + \ddot{\theta}_{2}) \end{bmatrix} + \begin{bmatrix} & -l_{2}(\dot{\theta}_{1} + \dot{\theta}_{2})^{2} \\ & 0 \\ & 0 \end{bmatrix} \\ &+ \begin{bmatrix} & l_{1}\ddot{\theta}_{1}s_{2} - l_{1}\dot{\theta}_{1}^{2}c_{2} + gs_{12} \\ & l_{1}\ddot{\theta}_{1}c_{2} + l_{1}\dot{\theta}_{1}^{2}s_{2} + gc_{12} \end{bmatrix}, \\ ^{2}F_{2} &= \begin{bmatrix} & m_{2}l_{1}\ddot{\theta}_{1}s_{2} - m_{2}l_{1}\dot{\theta}_{1}^{2}c_{2} + m_{2}gs_{12} - m_{2}l_{2}(\dot{\theta}_{1} + \dot{\theta}_{2})^{2} \\ & m_{2}l_{1}\ddot{\theta}_{1}c_{2} + m_{2}l_{1}\dot{\theta}_{1}^{2}s_{2} + m_{2}gc_{12} + m_{2}l_{2}(\ddot{\theta}_{1} + \ddot{\theta}_{2}) \end{bmatrix}, \\ ^{2}N_{2} &= \begin{bmatrix} & 0 \\ & 0 \\ & & 0 \end{bmatrix}. \end{split}$$



Example:

• The inward iterations for link 2:

$${}^{2}f_{2} = {}^{2}F_{2},$$

$${}^{2}n_{2} = \begin{bmatrix} & 0 & & & \\ & 0 & & & \\ & m_{2}l_{1}l_{2}c_{2}\ddot{\theta}_{1} + m_{2}l_{1}l_{2}s_{2}\dot{\theta}_{1}^{2} + m_{2}l_{2}gc_{12} + m_{2}l_{2}^{2}(\ddot{\theta}_{1} + \ddot{\theta}_{2}) \end{bmatrix}$$

Example:

• The inward iterations for link 1:

$$\begin{array}{l} 1 & \text{first first and tertations for first fir$$

Example:

• Extracting the \hat{Z}_i components of the in_i to find the joint torques:

$$\begin{split} \tau_1 &= m_2 l_2^2 (\ddot{\theta}_1 + \ddot{\theta}_2) + m_2 l_1 l_2 c_2 (2 \ddot{\theta}_1 + \ddot{\theta}_2) + (m_1 + m_2) l_1^2 \ddot{\theta}_1 - m_2 l_1 l_2 s_2 \dot{\theta}_2^2 \\ &- 2 m_2 l_1 l_2 s_2 \dot{\theta}_1 \dot{\theta}_2 + m_2 l_2 g c_{12} + (m_1 + m_2) l_1 g c_1, \\ \tau_2 &= m_2 l_1 l_2 c_2 \ddot{\theta}_1 + m_2 l_1 l_2 s_2 \dot{\theta}_1^2 + m_2 l_2 g c_{12} + m_2 l_2^2 (\ddot{\theta}_1 + \ddot{\theta}_2). \end{split}$$

- As shown, dynamic equations relates the (acceleration of manipulator joints) to (torques acting at it).
- The structure of dynamic equations can be expressed as:
 - ➤ The State-Space Equations
 - ➤ The Configuration-Space Equations

☐ The State-Space Equation

A dynamic equation can be written in the form:

$$\tau = M(\Theta)\ddot{\Theta} + V(\Theta, \dot{\Theta}) + G(\Theta)$$

- $M(\theta)$ is the $n \times n$ mass matrix of the manipulator.
- $V(\Theta, \dot{\Theta})$ is an $n \times 1$ vector of centrifugal and Coriolis terms.
- $G(\Theta)$ is an $n \times 1$ vector of gravity terms.
- It is called the state-space equation because the term $V(\theta, \dot{\theta})$ has both position and velocity dependence.
- Each element of $M(\theta)$ and $G(\theta)$ is a complex functions of θ (the position of all joints).
- Each element of $V(\Theta, \dot{\Theta})$ is a complex function of both Θ and $\dot{\Theta}$.

☐ The State-Space Equation

$$\tau = M(\Theta)\ddot{\Theta} + V(\Theta, \dot{\Theta}) + G(\Theta)$$

- Notes:
- $M(\theta)$ is composed of all those terms which multiply $\ddot{\theta}$, i.e. inertia forces.
- Any manipulator mass matrix, $M(\Theta)$, is symmetric and positive definite, therefore, always invertible (*Why?*).
- The velocity term, $V(\theta, \dot{\theta})$, contains all those terms that have any dependence on joint velocity, i.e. centrifugal and Coriolis forces.
- The gravity term, $G(\Theta)$, contains all those terms in which the gravitational constant, g, appears i.e. potential forces.

☐ The State-Space Equation

- **Example 1:**
- Calculate $M(\theta)$, $V(\theta, \dot{\theta})$, and $G(\theta)$ for two-link RR planar manipulator?
- Remember:

$$\begin{split} \tau_1 &= m_2 l_2^2 (\ddot{\theta}_1 + \ddot{\theta}_2) + m_2 l_1 l_2 c_2 (2 \ddot{\theta}_1 + \ddot{\theta}_2) + (m_1 + m_2) l_1^2 \ddot{\theta}_1 - m_2 l_1 l_2 s_2 \dot{\theta}_2^2 \\ &- 2 m_2 l_1 l_2 s_2 \dot{\theta}_1 \dot{\theta}_2 + m_2 l_2 g c_{12} + (m_1 + m_2) l_1 g c_1, \\ \tau_2 &= m_2 l_1 l_2 c_2 \ddot{\theta}_1 + m_2 l_1 l_2 s_2 \dot{\theta}_1^2 + m_2 l_2 g c_{12} + m_2 l_2^2 (\ddot{\theta}_1 + \ddot{\theta}_2). \end{split}$$

■ The manipulator mass matrix:

$$M(\Theta) = \begin{bmatrix} m_2 l_2^2 + 2 m_2 l_1 l_2 c_2 + (m_1 + m_2) l_1^2 & m_2 l_2^2 + m_2 l_1 l_2 c_2 \\ m_2 l_2^2 + m_2 l_1 l_2 c_2 & m_2 l_2^2 \end{bmatrix}$$

• $M(\Theta)$ is a function of Θ .

- ☐ The State-Space Equation
- **Example 1:**

$$V(\Theta, \dot{\Theta}) = \begin{bmatrix} -m_2 l_1 l_2 s_2 \dot{\theta}_2^2 - 2m_2 l_1 l_2 s_2 \dot{\theta}_1 \dot{\theta}_2 \\ m_2 l_1 l_2 s_2 \dot{\theta}_1^2 \end{bmatrix}$$

- A term like $-m_2l_1l_2s_2\dot{\theta}_2^2$ is caused by a centrifugal force, because it depends on the square of a joint velocity.
- A term such as $-2m_2l_1l_2s_2\dot{\theta}_1\dot{\theta}_2$ is caused by a Coriolis force, because it always contain the product of two different joint velocities.

$$G(\Theta) = \begin{bmatrix} m_2 l_2 g c_{12} + (m_1 + m_2) l_1 g c_1 \\ m_2 l_2 g c_{12} \end{bmatrix}$$

• The gravity term depends only on Θ and not on its derivatives.

☐ The Configuration-Space Equation

$$\tau = M(\Theta)\ddot{\Theta} + V(\Theta, \dot{\Theta}) + G(\Theta)$$

• Writing the velocity-dependent term, $V(\theta, \dot{\theta})$, in a different form:

$$\tau = M(\theta)\ddot{\theta} + B(\theta)[\dot{\theta}\dot{\theta}] + C(\theta)[\dot{\theta}^2] + G(\theta)$$

- $B(\theta)$: $n \times n(n-1)/2$ matrix of Coriolis coefficients.
- $[\dot{\Theta}\dot{\Theta}]$: $n(n-1)/2 \times 1$ vector of joint velocity products given by: $[\dot{\Theta}\dot{\Theta}] = [\dot{\theta}_1\dot{\theta}_2 \quad \dot{\theta}_1\dot{\theta}_3 \quad ... \quad \dot{\theta}_{n-1}\dot{\theta}_n]^T$
- $C(\Theta)$: $n \times n$ matrix of centrifugal coefficients.
- $\left[\dot{\Theta}^2\right]: n \times 1$ vector given by $\left[\dot{\Theta}^2\right] = \left[\dot{\theta}_1^2 \quad \dot{\theta}_2^2 \quad \dots \quad \dot{\theta}_n^2\right]^T$

☐ The Configuration-Space Equation

$$\tau = M(\theta)\ddot{\theta} + B(\theta)[\dot{\theta}\dot{\theta}] + C(\theta)[\dot{\theta}^2] + G(\theta)$$

- It is called the configuration-space equation, because the matrices are functions only of manipulator position.
- It is useful in applications in which the dynamic equations must be updated as the manipulator moves. (e.g. control of a manipulator)

- **☐** The Configuration-Space Equation
- **Example 2:**
- Calculate $B(\theta)$ and $C(\theta)$ for two-link RR planar manipulator
- Remember

$$V(\Theta, \dot{\Theta}) = \begin{bmatrix} -m_2 l_1 l_2 s_2 \dot{\theta}_2^2 - 2m_2 l_1 l_2 s_2 \dot{\theta}_1 \dot{\theta}_2 \\ m_2 l_1 l_2 s_2 \dot{\theta}_1^2 \end{bmatrix}$$

For the two-link manipulator

$$\left[\dot{\Theta}\dot{\Theta}\right] = \left[\dot{\theta}_1\dot{\theta}_2\right]$$

$$\begin{bmatrix} \dot{\Theta}^2 \end{bmatrix} = \begin{bmatrix} \dot{\theta}_1^2 & \dot{\theta}_2^2 \end{bmatrix}^T$$

So,

$$B(\Theta) = \begin{bmatrix} -2m_2l_1l_2s_2\\ 0 \end{bmatrix}$$

$$C(\Theta) = \begin{bmatrix} 0 & -m_2 l_1 l_2 s_2 \\ m_2 l_1 l_2 s_2 & 0 \end{bmatrix}$$

Remember

$$\tau = M(\Theta)\ddot{\Theta} + V(\Theta, \dot{\Theta}) + G(\Theta)$$

- It has been developed in joint space because the serial-link nature of the mechanism was utilized in deriving the equations.
- It relates the (acceleration of manipulator joints) to (torques acting at it).
- Now, find the formulation that relate (acceleration of the end-effector expressed in Cartesian space) to (joints torques) or (Cartesian forces and moments acting at the end-effector).
- It can be expressed as:
 - 1) The Cartesian State-Space Torque Equations
 - 2) The Cartesian State-Space Force Equations
 - 3) The Cartesian Configuration-Space Torque Equations
 - 4) The Cartesian Configuration-Space Force Equations

1) The Cartesian State-Space Torque Equations

• The dynamics of a manipulator with respect to Cartesian variables:

$$\tau = M_X(\Theta)\ddot{X} + V_X(\Theta, \dot{\Theta}) + G_X(\Theta)$$

X is an appropriate Cartesian vector representing position and orientation of the end-effector.

$$\dot{X} = \begin{bmatrix} \dot{d}(\theta) \\ \dot{\Theta}(\theta) \end{bmatrix} \quad or \quad \dot{X} = \begin{bmatrix} \dot{d}(\theta) \\ \omega(\theta) \end{bmatrix}$$

- $M_X(\Theta)$ is the Cartesian mass matrix.
- $V_X(\Theta, \dot{\Theta})$ is a vector of velocity terms in Cartesian space.
- $G_X(\Theta)$ is a vector of gravity terms in Cartesian space.

1) The Cartesian State-Space Torque Equations

$$\tau = M(\Theta)\ddot{\Theta} + V(\Theta, \dot{\Theta}) + G(\Theta)$$

Develop a relationship between joint space and Cartesian acceleration,

$$\dot{X} = J(\theta)\dot{\theta}
\ddot{X} = J(\theta)\ddot{\theta} + \dot{J}(\theta)\dot{\theta}
\ddot{\theta} = J^{-1}(\theta)\ddot{X} - J^{-1}(\theta)\dot{J}(\theta)\dot{\theta}$$

(*)

Substituting

$$\tau = M(\Theta) \left(J^{-1} \ddot{X} - J^{-1} \dot{J} \dot{\Theta} \right) + V(\Theta, \dot{\Theta}) + G(\Theta)$$

Therefore

$$\tau = M_X(\Theta)\ddot{X} + V_X(\Theta, \dot{\Theta}) + G_X(\Theta)$$

$$M_X(\Theta) = M(\Theta)J^{-1}(\Theta)$$

$$V_X(\Theta, \dot{\Theta}) = V(\Theta, \dot{\Theta}) - M(\Theta)J^{-1}(\Theta)\dot{J}(\Theta)\dot{\Theta}$$

$$G_Y(\Theta) = G(\Theta)$$

• Note: the Jacobian, $J(\theta)$, is written in the same frame as \ddot{X} . The choice of this frame is **arbitrary** but is usually the tool frame, $\{T\}$.

2) The Cartesian State-Space Force Equations

The dynamics of a manipulator with respect to Cartesian variables

$$F = M_X(\Theta)\ddot{X} + V_X(\Theta, \dot{\Theta}) + G_X(\Theta)$$

- F is a force-torque vector acting on the end-effector
- **Remark:** The acting end-effector forces, F, could in fact be applied by the actuators at the joints:

$$\tau = J^T(\Theta) F$$

- It means that F is the Cartesian expression of joints torques
- Note: the Jacobian, $J(\Theta)$, is written in the same frame as F and \ddot{X} . The choice of this frame is **arbitrary** but is usually the tool frame, $\{T\}$
- **? Q**: What does this dynamic represent?

2) The Cartesian State-Space Force Equations

$$\tau = M(\Theta)\ddot{\Theta} + V(\Theta, \dot{\Theta}) + G(\Theta)$$

Premultiplying by the inverse of the Jacobian transpose

$$J^{-T}\tau = J^{-T}M(\Theta)\ddot{\Theta} + J^{-T}V(\Theta,\dot{\Theta}) + J^{-T}G(\Theta)$$

or

$$F = J^{-T}M(\Theta)\ddot{\Theta} + J^{-T}V(\Theta,\dot{\Theta}) + J^{-T}G(\Theta)$$

Using Θ as Equ. (*)

$$F = J^{-T}M(\Theta) J^{-1}\ddot{X} - J^{-T}M(\Theta) J^{-1}\dot{J}\dot{\Theta} + J^{-T}V(\Theta,\dot{\Theta}) + J^{-T}G(\Theta)$$

Therefore

$$F = M_X(\Theta)\ddot{X} + V_X(\Theta, \dot{\Theta}) + G_X(\Theta)$$

$$M_X(\theta) = J^{-T}(\theta)M(\theta)J^{-1}(\theta)$$

$$V_X(\theta,\dot{\theta}) = J^{-T}(V(\theta,\dot{\theta}) - M(\theta)J^{-1}(\theta)\dot{J}(\theta)\dot{\theta})$$

$$G_X(\theta) = J^{-T}(\theta)G(\theta)$$

2) The Cartesian State-Space Force Equations

- **Example:**
- Derive the Cartesian-space dynamic equations for the two-link RR planar arm in terms of a frame attached to the end of the second link
- the Jacobian

$$J(\Theta) = \begin{bmatrix} l_1 s_2 & 0 \\ l_1 c_2 + l_2 & l_2 \end{bmatrix}$$

the inverse Jacobian

$$J^{-1}(\theta) = \frac{1}{l_1 l_2 s_2} \begin{bmatrix} l_2 & 0\\ -l_1 c_2 - l_2 & l_1 s_2 \end{bmatrix}$$

• the time derivative of the Jacobian:

$$\dot{J}(\theta) = \begin{bmatrix} l_1 c_2 \dot{\theta}_2 & 0 \\ -l_1 s_2 \dot{\theta}_2 & 0 \end{bmatrix}$$

2) The Cartesian State-Space Force Equations

- **Example:**
- Therefore

$$F = M_X(\Theta)\ddot{X} + V_X(\Theta, \dot{\Theta}) + G_X(\Theta)$$

$$M_{x}(\Theta) = \begin{bmatrix} m_{2} + \frac{m_{1}}{s_{2}^{2}} & 0\\ 0 & m_{2} \end{bmatrix}$$

$$V_{x}(\theta,\dot{\theta}) = \begin{bmatrix} -(m_{2}l_{1}c_{2} + m_{2}l_{2})\dot{\theta}_{1}^{2} - m_{2}l_{2}\dot{\theta}_{2}^{2} - \left(2m_{2}l_{2} + m_{2}l_{1}c_{2} + m_{1}l_{1}\frac{c_{2}}{s_{2}^{2}}\right)\dot{\theta}_{1}\dot{\theta}_{2} \\ m_{2}l_{1}s_{2}\dot{\theta}_{1}^{2} + m_{2}l_{1}s_{2}\dot{\theta}_{1}\dot{\theta}_{2} \end{bmatrix}$$

$$G_{x}(\Theta) = \begin{bmatrix} m_{1}g \frac{c_{1}}{s_{2}} + m_{2}g s_{12} \\ m_{2}g c_{12} \end{bmatrix}$$

3) The Cartesian Configuration-Space Torque Equations

$$\tau = M_X(\theta) \ddot{X} + B_X(\theta) \left[\dot{\theta} \dot{\theta} \right] + C_X(\theta) \left[\dot{\theta}^2 \right] + G_X(\theta)$$

- $B_X(\Theta) \neq B(\Theta)$
- $C_X(\theta) \neq C(\theta)$
- $G_X(\Theta) = G(\Theta)$

4) The Cartesian Configuration-Space Force Equations

$$F = M_X(\Theta) \ddot{X} + B_X(\Theta) \left[\dot{\Theta} \dot{\Theta} \right] + C_X(\Theta) \left[\dot{\Theta}^2 \right] + G_X(\Theta)$$

- $B_X(\Theta) \neq B(\Theta)$
- $C_X(\Theta) \neq C(\Theta)$
- $G_X(\Theta) \neq G(\Theta)$

Inclusion of Non-rigid Body Effects

- Derived dynamic equations do not encompass all the effects acting on a manipulator.
- The most important source of forces that are not included is friction.
- **Viscous friction:** the torque due to friction is proportional to the velocity of joint motion.

$$\tau_{friction} = v\dot{\theta}$$

- v is a viscous-friction constant.
- Coulomb friction: the torque due to friction is constant except for a sign dependence on the joint velocity.

$$\tau_{friction} = c \, sgn(\dot{\theta})$$

- c is a Coulomb-friction constant.
- c is often taken at one value when $\dot{\theta} = 0$, i.e. the **static coefficient**, but at a lower value, when $\dot{\theta} \neq 0$, i.e. the **dynamic coefficient**.

Inclusion of Non-rigid Body Effects

To include both

$$\tau_{friction} = c \, sgn(\dot{\theta}) + v\dot{\theta}$$

- In many joints, friction displays a dependence on the joint position.
- A major cause might be not perfectly round gears (Also eccentricity).
- A fairly complex friction model

$$\tau_{friction} = F(\theta, \dot{\theta})$$

Adding to the dynamic terms derived from the rigid-body model

$$\tau = M(\Theta)\ddot{\Theta} + V(\Theta, \dot{\Theta}) + G(\Theta) + F(\Theta, \dot{\Theta})$$

• In addition **bending effects** (which give rise to resonances) are neglected (Flexible links and joints).

The END

• References:

1) ..