

# MECE-606 Systems Modeling

## Computer Project #3: DC Motor & Inertia Modeling

**Goal:** Develop a dynamic model of the DC Motor connected to a steel shaft that is simply supported by two ball bearing mounts. Estimate the moment of inertia of the shaft and the linear viscous damping coefficient of the bearings. First, derive the second order model and use the parameters provided by the motor manufacturer to fit the experimental data. Then repeat the simulation with the reduced order model when the armature inductance is assumed zero. Compare both models to the modeling and validation data sets.

**Measurements:** Angular position from a rotary encoder (500 CPR) & Armature current from a non-contact hall-effect sensor (LEM HAS-50). The current sensor may have a small DC bias at zero voltage command that can be subtracted.

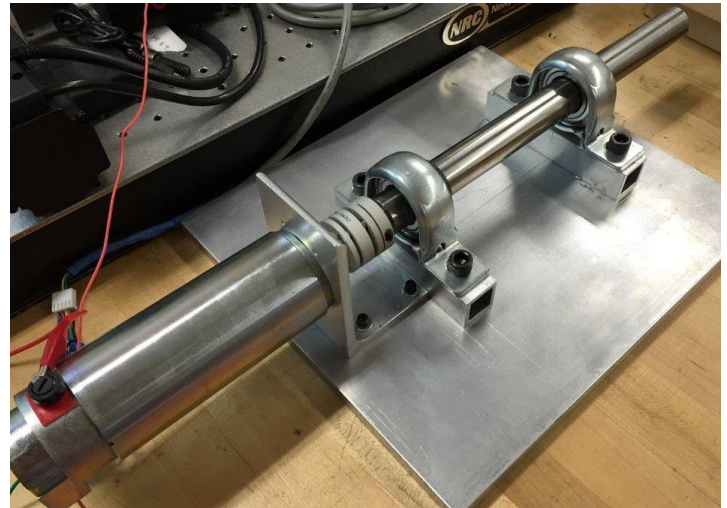


Figure 1 – DC Motor with Inertia

**System:** A Pittman Brushed DC motor is connected through a coupling to a steel shaft ( $\rho = 7850\text{-kg/m}^3$ ). See attached data sheet. The shaft is supported by two standard ball bearings and the motor is driven by a standard DC power supply.

**Considerations:** For both the first and second order models use the specifications provided by the motor manufacture. First analyze the motor only data set (inertia disconnected). How well does the reported specification match the result? Can you modify any parameters to get a better fit? Why? Second, analyze the system when

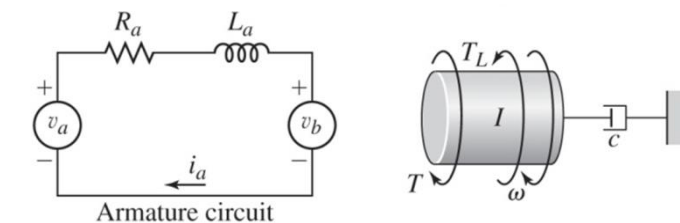


Figure 2 - Armature Controlled DC Motor Schematic

connected to the shaft. For this case the inertia and damping will have to be estimated. Use a 2D parameter sweep with a cost function utilizing motor speed only and again using both speed and current. (*Note:* you will need a weight factor to scale the two measurements  $J=e^T e + e^T W e$ .) Redo both cases using Matlab's "*fminsearch*" command. Document all optimizations and discuss the differences. Make sure to have quantifiable measures of the model fits (RMS, etc.). Is there another model form that is better? Simulate the identified models versus the validation data sets using the measured motor command voltage as the input. How does the model fit (RMS) compare versus the validation data sets? Lastly, determine a first-order plus dead-time (FOPDT) model ( $\tau, K, \theta$ ) from a single step change in the data. How does this simplified model match the full modeling data set? The validation sets? The full physics-based model?

**Deliverable:** A concise three page (max.) report on the modeling approach with full derivation of all equations from first principles (i.e.  $F=ma$ ) through final model. A discussion of the final results related to quality of the model fit of the data is critical including a quantification of the error (RMS, etc.). The number of plots should be kept to a minimum but be of high quality and description (legends, captions). A portion of the grade is reserved for the quality of the written report. Please submit all of your Matlab/Simulink code separately and consolidate it to as few pages as possible.

**Due Date:** Two weeks after the assigned date.

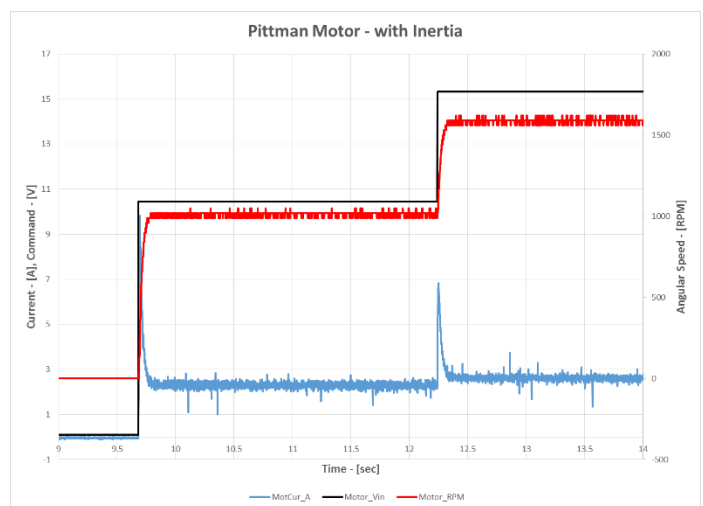


Figure 3 - Angular Velocity, Armature Current, & Voltage Command vs. Time

## 14207S008

Lo-Cog® DC Servo Motor



Assembly Data	Symbol	Units	Value
Reference Voltage	E	V	24
No-Load Speed	$S_{NL}$	rpm (rad/s)	3,211 (336)
Continuous Torque (Max.) <sup>1</sup>	$T_C$	oz-in (N-m)	50 (3.5E-01)
Peak Torque (Stall) <sup>2</sup>	$T_{PK}$	oz-in (N-m)	410 (2.9E+00)
Weight	$W_M$	oz (g)	58 (1630)
Motor Data			
Torque Constant	$K_T$	oz-in/A (N-m/A)	10.0 (7.06E-02)
Back-EMF Constant	$K_E$	V/krpm (V/rad/s)	7.39 (7.06E-02)
Resistance	$R_T$	$\Omega$	0.59
Inductance	L	mH	0.87
No-Load Current	$I_{NL}$	A	0.30
Peak Current (Stall) <sup>2</sup>	$I_P$	A	40.4
Motor Constant	$K_M$	oz-in/ $\sqrt{W}$ (N-m/ $\sqrt{W}$ )	13.1 (9.25E-02)
Friction Torque	$T_F$	oz-in (N-m)	2.2 (1.6E-02)
Rotor Inertia	$J_M$	oz-in-s <sup>2</sup> (kg-m <sup>2</sup> )	6.7E-03 (4.7E-05)
Electrical Time Constant	$\tau_E$	ms	1.50
Mechanical Time Constant	$\tau_M$	ms	5.5
Viscous Damping	D	oz-in/krpm (N-m-s)	0.25 (1.7E-05)
Damping Constant	$K_D$	oz-in/krpm (N-m-s)	127 (8.6E-03)
Maximum Winding Temperature	$\theta_{MAX}$	°F (°C)	311 (155)
Thermal Impedance	$R_{TH}$	°F/watt (°C/watt)	41.0 (4.98)
Thermal Time Constant	$\tau_{TH}$	min	32.3
Gearbox Data			
Encoder Data			
Channels			3
Resolution		CPR	500

<sup>1</sup> - Specified at max. winding temperature at 25°C ambient without heat sink. <sup>2</sup> - Theoretical values supplied for reference only.

### Included Features

2-Pole Stator  
Ceramic Magnets  
Heavy-Gauge Steel Housing  
11-Slot Armature  
Silicon Steel Laminations  
Stainless Steel Shaft  
Copper-Graphite Brushes  
Diamond Turned Commutator  
Motor Ball Bearings

### Customization Options

Alternate Winding  
Sleeve or Ball Bearings  
Modified Output Shaft  
Custom Cable Assembly  
Special Brushes  
EMI/RFI Suppression  
Spur or Planetary Gearbox  
Special Lubricant  
Optional Encoder  
Fail-Safe Brake

