Plastic Heat Tests

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1 Plastic deformation heating a porous skeleton

In this section, plastic deformation causes a heat energy-density rate $(J.m^{-3}.s^{-1})$ of

$$c(1-\phi)\sigma_{ij}\epsilon_{ij}^{\text{plastic}}$$
, (1.1)

where c is a coefficient (s⁻¹), ϕ is the porosity, σ is the stress, and $\epsilon^{\text{plastic}}$ is the plastic strain.

There is no fluid, and no heat flow is studied: the heat energy released simply heats up the porous skeleton:

$$\frac{\partial}{\partial t}(1-\phi)c_R\rho_R T = c(1-\phi)\sigma_{ij}\epsilon_{ij}^{\text{plastic}}.$$
(1.2)

The porosity (ϕ) and volumetric heat capacity of the rock grains $(c_R \rho_R)$ are chosen to be constant.

Perfect capped weak-plane plasticity is used, so that the admissible zone is defined by

$$\sigma_{zz} \leq S_T , \qquad (1.3)$$

$$\sigma_{zz} \geq -S_C \tag{1.4}$$

$$\sqrt{\sigma_{zx}^2 + \sigma_{zy}^2} + \sigma_{zz} \tan \Phi \leq C. \tag{1.5}$$

Here S_T is the tensile strength, S_C is the compressive strength, C is the cohesion and Φ is the friction angle. The elastic tensor is chosen to be

$$E_{ijkl} = \lambda \delta_{ij} \delta_{kl} + \mu (\delta_{ik} \delta_{jl} + \delta_{il} \delta_{jk}) . \tag{1.6}$$

The parameters in these expressions are chosen to be: $S_T = 1$, $S_C = 1$, C = 1, $\Phi = \pi/4$, $\lambda = 1/2$, and $\mu = 1/4$ (all in consistent units).

In each experiment a single finite-element is used.

1.1 Tensile failure

The top of the finite element is pulled upwards with displacement:

$$u_z = zt (1.7)$$

while the displacement in the x and y directions is chosen to be zero. This implies the only non-zero component of total strain is

$$\epsilon_{zz}^{\text{total}} = t$$
 . (1.8)

The constitutive law implies

$$\sigma_{zz} = t . (1.9)$$

This stress is admissible for $t \leq 1$, while for t > 1 the system yields in tension:

$$\sigma_{zz} = 1 \quad \text{for } t > 1 , \qquad (1.10)$$

and the plastic strain is,

$$\epsilon_{zz}^{\text{plastic}} = t - 1 \quad \text{for } t > 1 .$$
(1.11)

This means that the material's temperature should increase as

$$c_R \rho_R \dot{T} = c \quad \text{for } t > 1 , \qquad (1.12)$$

while the right-hand side is zero for $t \leq 1$.

1.2 Compressive failure

The top of the finite element is pushed downwards with displacement:

$$u_z = -zt (1.13)$$

while the displacement in the x and y directions is chosen to be zero. This implies only non-zero component of total strain is

$$\epsilon_{zz}^{\text{total}} = -t$$
 . (1.14)

The constitutive law implies

$$\sigma_{zz} = -t \ . \tag{1.15}$$

This stress is admissible for $t \leq 1$, while for t > 1 the system yields in compression

$$\sigma_{zz} = -1 \quad \text{for } t > 1 , \qquad (1.16)$$

and the plastic strain is,

$$\epsilon_{zz}^{\text{plastic}} = -(t-1) \quad \text{for } t > 1 .$$
(1.17)

This means that the material's temperature should increase as

$$c_R \rho_R \dot{T} = c \quad \text{for } t > 1 , \qquad (1.18)$$

while the right-hand side is zero for $t \leq 1$.

1.3 Shear failure

The top of the finite element is sheared with displacement:

$$u_x = zt (1.19)$$

while the displacement in the y and z directions is chosen to be zero. This implies only non-zero component of total strain is

$$\epsilon_{xz}^{\text{total}} = t$$
 . (1.20)

The constitutive law implies

$$\sigma_{xz} = t/4 \ . \tag{1.21}$$

This stress is admissible for $t \leq 4$, while for t > 4 the system yields in shear:

$$\sigma_{xz} = 1 \quad \text{for } t > 4 , \qquad (1.22)$$

and the plastic strain is,

$$\epsilon_{xz}^{\text{plastic}} = t - 4 \quad \text{for } t > 4 \ .$$
(1.23)

This means that the material's temperature should increase as

$$c_R \rho_R \dot{T} = c \quad \text{for } t > 4 \;, \tag{1.24}$$

while the right-hand side is zero for $t \leq 4$.