

## Examination paper for TTK4130 Modeling and Simulation

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**Examination date: 2019-05-15**

**Examination time (from-to): 0900—1300**

**Permitted examination support material: Code A.**

**Other information:**

**Language: English**

**Number of pages (front page excluded): 4**

**Number of pages enclosed:**

**Informasjon om trykking av eksamensoppgave**

**Originalen er:**

**1-sidig** ☐ **2-sidig** **X**

**sort/hvit** ☐ **farger** **X**

**skal ha flervalgskjema** ☐

**Checked by:**

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Date

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Signature

For questions during the exam:  
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## Exam in TTK4130 Modeling and Simulation

Thursday, May 15 2019

09:00 – 13:00

Permitted aids (code A): All written and handwritten examination support materials are permitted.

Answers in English, Norwegian, or a mixture of the two are accepted.

Grades available: As specified by regulations.

### Problem 1 (11 %)

- (2 %) (a) Explain briefly how the Modelica commands `replaceable` and `redeclare` are connected.
- (6 %) (b) You have available the three models: `RedModel`, `WhiteModel`, and `BlueModel`. You like to implement a flexible model `Austria`, where the `RedModel`, the `WhiteModel` and another `RedModel` are connected and can be exchanged (Fig. 1). The models are connected by just one connector variable.

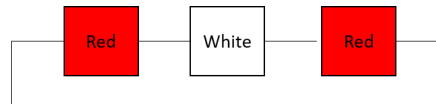


Figure 1: Model `Austria`.

Moreover, you like to set the parameter  $p1$  in one of the `RedModel` to  $p1 = 5$  and in the other to  $p1 = 10$ . Write down the Modelica code of the *flexible* model `Austria`.

- (3 %) (c) A new model `Russia` should be implemented with the structure shown in Fig. 2. Write the Modelica code of the new model by using model `Austria`.

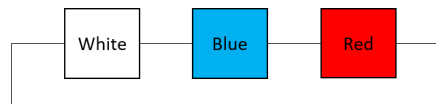


Figure 2: Model `Russia`.

### Problem 2 (10 %)

Consider the following differential-algebraic system with input  $u$

$$\begin{aligned}\dot{x}_1 &= x_3 - x_1 z, \\ \dot{x}_2 &= u - x_2 z, \\ \dot{x}_3 &= -x_1, \\ 0 &= \frac{1}{2}(1 - x_1^2 - x_2^2).\end{aligned}$$

- (4 %) (a) Derive mathematically the differential index of the system.

For the rest of the problem you can neglect the algebraic equation and assume  $z = 1/2$

- (6 %) (b) Is the ODE system passive for input  $u$  and output  $x_2$ ?

**Problem 3 (26 %)**

Consider the following Runge-Kutta method with the Butcher tableau

$$\begin{array}{c|cc} \gamma & \gamma & 0 \\ 1-\gamma & 1-2\gamma & \gamma \\ \hline & 1/2 & 1/2 \end{array}.$$

- (5 %) (a) Write down the equations of the method. What are the limits for  $\gamma$ , and is the method explicit or implicit?
- (7 %) (b) Find the stability function of the method.
- (6 %) (c) Find the limits on  $\gamma$  so the method is B-stable.  
*Hint: Make appropriate assumptions.*
- (4 %) (d) Find the limits on  $\gamma$  for which the method is B- and L-stable.
- (4 %) (e) Explain why the method can be seen as numerical efficient for  $\gamma \neq 0$ .

**Problem 4 (25 %)**

The following system consisting of a slender beam with mass  $m_1$  and length  $l$  and a disc with mass  $m_2$  and radius  $R$  should be modeled. The disc rolls without slipping on the ground (Fig. 3). The system is frictionless.

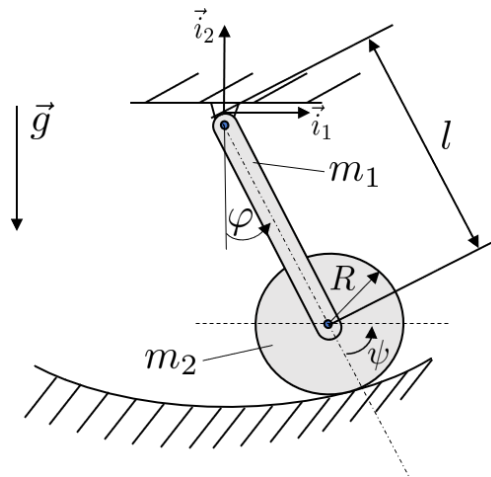


Figure 3: Slender beam with disc.

*Hint: The moment of inertia of a disc:  $I = \frac{1}{2}mR^2$ .*

- (3 %) (a) Find a connection between  $\varphi$  and  $\psi$ . How many degrees of freedom have the system?

*If you were not able to find a connection between  $\varphi$  and  $\psi$  use  $\varphi = \psi$  [which is not necessarily the correct answer to (a)]*

- (6 %) (b) Find the position, velocity and acceleration vector of the center of mass of both body parts.
- (13 %) (c) Find the equation of motion of the system as a function of the given variables (Fig. 3) with the Newton-Euler approach.

*Caution: Solutions using the Lagrange approach will give no points in this task.*

*Hint 1: Given variables:  $\vec{g}, m_1, m_2, l, R, \psi, \varphi$ , including their derivatives.*

*Hint 2: The equation of motion does not necessary contain all given variables.*

If you were not able to find the EoM use in the following  $\ddot{\varphi} t^2 (2m_1 + m_2) + 1/3 l g (3m_1 + 1/2 m_2) \sin \varphi = 0$  [which is not necessarily the correct answer to (c)]

- (3%) (d) Find the period of oscillation.

*Hint: Make appropriate assumptions.*

**Problem 5 (12%)**

Consider a thin plate (Fig. 6) of height  $h$  with a constant density  $\rho$  and the mass  $m = \pi/4 abh\rho$ . The outer contour of the cross-section can be described by  $z^2 = b^2 - b^2 y^2/a^2$ , where  $a \gg h$  and  $b \gg h$ .

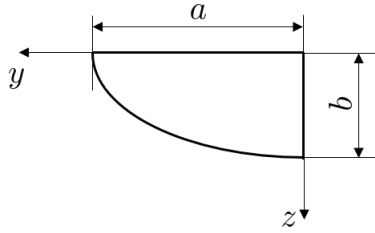


Figure 6: Thin plate

The following may be useful

$$\begin{aligned} \int \sqrt{X} dx &= \frac{1}{2} (x\sqrt{X} + c^2 \arcsin \frac{x}{c}) \\ \int x\sqrt{X} dx &= -\frac{1}{3} \sqrt{X}^3 \\ \int x^2 \sqrt{X} dx &= -\frac{x}{4} \sqrt{X}^3 + \frac{c^2}{8} (x\sqrt{X} + c^2 \arcsin \frac{x}{c}), \end{aligned}$$

with  $X = (c^2 - x^2)$ .

- (7%) (a) Find the moment of inertia  $I_{y,o}$  of the plate for the given coordinate system as a function of mass  $m$ .

If you were not able to find  $I_{y,o}$  use in the following  $I_{y,o} = 1/6 a^2 b^2 m$  [which is not necessarily the correct answer to (a)]

- (5%) (b) Find the moment of inertia  $I_{y,c}$  around the center of mass of the plate as a function of mass  $m$ .

**Problem 6 (16 %)**

In an open wind channel of diameter  $D$  the air flow around a wind turbine with diameter  $d$  is investigated (Fig. 7). The air flow has a constant density  $\rho$ . The upstream wind velocity is  $u_\infty$ , the following velocity field over the cross-section of the channel is measured at a specific downstream position  $x = \text{const}$ .

$$u_e(r, \phi) = \begin{cases} u_\infty ((1 - c) - c \cos(\pi r/R)), & 0 \leq r \leq R \\ u_\infty, & R < r \leq D/2 \end{cases}$$

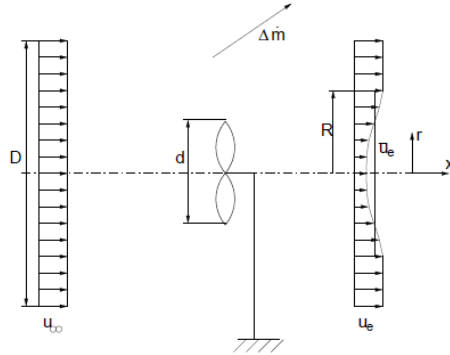


Figure 7: Open wind channel with wind turbine

Assume that the wind channel is in steady-state,  $d \ll D$ ,  $d/2 < R < D/2$  and  $0 < c < 1$ . Moreover, friction forces can be neglected.

The following may be useful

$$\int x \cos(ax) dx = \frac{\cos(ax)}{a^2} + \frac{x \sin(ax)}{a}.$$

- (7 %) (a) Determine the displaced mass flow  $\Delta \dot{m}$  in the wind channel.  
*Hint: As usually for these problems you first have to define the control volume.*

*If you were not able to find the displaced mass flow continue with  $\Delta \dot{m} = \rho u_\infty c (1 - \frac{2}{\pi})$  [which is not necessarily the correct answer to (a)]*

- (3 %) (b) The velocity profile  $u_e(r, \phi)$  should be replaced with an piecewise constant velocity  $u_e(r, \phi) = \bar{u}_e$ ,  $0 \leq r \leq R$  and  $u_e(r, \phi) = u_\infty$ ,  $R < r \leq D/2$ . Determine the average velocity  $\bar{u}_e$  such that the displaced mass flow  $\Delta \dot{m}$  stays the same as in a).

For the following task assume the average velocity  $\bar{u}_e$  is known.

- (6 %) (c) Determine the force  $F$  produced from the wind turbine using the piecewise velocity profile.