

Governing Equation

For, *one-dimensional, steady state, conductive* heat transfer,

$$\frac{d}{dx} \left(k \frac{dT}{dx} \right) + q = 0$$

Parameters

$$k = 1000 \text{ W (m. K)}^{-1}$$

$$q = 5095 \text{ kW m}^{-3}$$

$$L = 0.5 \text{ m}$$

$$A = 10 \times 10^{-3} \text{ m}^2$$

$$T_A = 100^\circ\text{C}$$

$$T_B = 500^\circ\text{C}$$

Discretisation

Dividing L in to 5 equal control volumes, results in nodal distances, δx , of 0.1 m.

Now, over a control volume, for nodes 2,3 & 4, the governing equation:

$$\begin{aligned} \int_{\Delta V} \frac{d}{dx} \left(k \frac{dT}{dx} \right) dV + \int_{\Delta V} q dV &= \int_A \left(k \frac{dT}{dx} \right) dA + \int_{\Delta V} q dV = 0 \\ \left[\left(kA \frac{dT}{dx} \right)_e - \left(kA \frac{dT}{dx} \right)_w \right] + q \Delta V &= 0 \\ \left[\left(kA \frac{T_E - T_P}{\delta x} \right)_e - \left(kA \frac{T_P - T_W}{\delta x} \right)_w \right] + qA\delta x &= 0 \\ \frac{kA}{\delta x} T_E - \frac{kA}{\delta x} T_P - \frac{kA}{\delta x} T_P + \frac{kA}{\delta x} T_W + qA\delta x &= 0 \\ \left(2 \frac{kA}{\delta x} \right) T_P &= \left(\frac{kA}{\delta x} \right) T_W + \left(\frac{kA}{\delta x} \right) T_E + qA\delta x \end{aligned} \quad (1)$$

So that,

a_W	a_E	a_P	S_P	S_u
$\frac{kA}{\delta x}$	$\frac{kA}{\delta x}$	$2 \frac{kA}{\delta x}$	0	$qA\delta x$
100	100	200	0	5,095

For $i = 2,3$ or 4

$$200T_i = 100T_{i-1} + 100T_{i+1} + 5095 \quad (2)$$

Now, over a control volume, for node 1, equation 1 becomes:

$$\begin{aligned} \left[\left(kA \frac{T_E - T_P}{\delta x} \right)_e - \left(kA \frac{T_P - T_A}{0.5\delta x} \right)_w \right] + qA\delta x &= 0 \\ \frac{kA}{\delta x} T_E - \frac{kA}{\delta x} T_P - \frac{kA}{0.5\delta x} T_P + \frac{kA}{0.5\delta x} T_A + qA\delta x &= 0 \\ \left(\frac{kA}{\delta x} + \frac{kA}{0.5\delta x} \right) T_P &= \left(\frac{kA}{\delta x} \right) T_E + qA\delta x + \frac{kA}{0.5\delta x} T_A \end{aligned}$$

So that,

a_W	a_E	a_P	S_P	S_u
0	$\frac{kA}{\delta x}$	$3\frac{kA}{\delta x}$	$-2\frac{kA}{\delta x}$	$qA\delta x + 2\frac{kA}{\delta x}T_A$
0	100	300	-200	25,095

$$300T_1 = 100T_2 + 5095 + 200T_A \quad (3)$$

Now, over a control volume, for node 5, equation 1 becomes:

$$\begin{aligned} & \left[\left(kA \frac{T_B - T_P}{0.5\delta x} \right)_e - \left(kA \frac{T_P - T_W}{\delta x} \right)_w \right] + qA\delta x = 0 \\ & \frac{kA}{0.5\delta x}T_B - \frac{kA}{0.5\delta x}T_P - \frac{kA}{\delta x}T_P + \frac{kA}{\delta x}T_W + qA\delta x = 0 \\ & \left(\frac{kA}{\delta x} + \frac{kA}{0.5\delta x} \right) T_P = \left(\frac{kA}{\delta x} \right) T_W + qA\delta x + \frac{kA}{0.5\delta x}T_B \end{aligned}$$

So that,

a_W	a_E	a_P	S_P	S_u
$\frac{kA}{\delta x}$	0	$3\frac{kA}{\delta x}$	$-2\frac{kA}{\delta x}$	$qA\delta x + 2\frac{kA}{\delta x}T_B$
100	0	300	-200	105,095

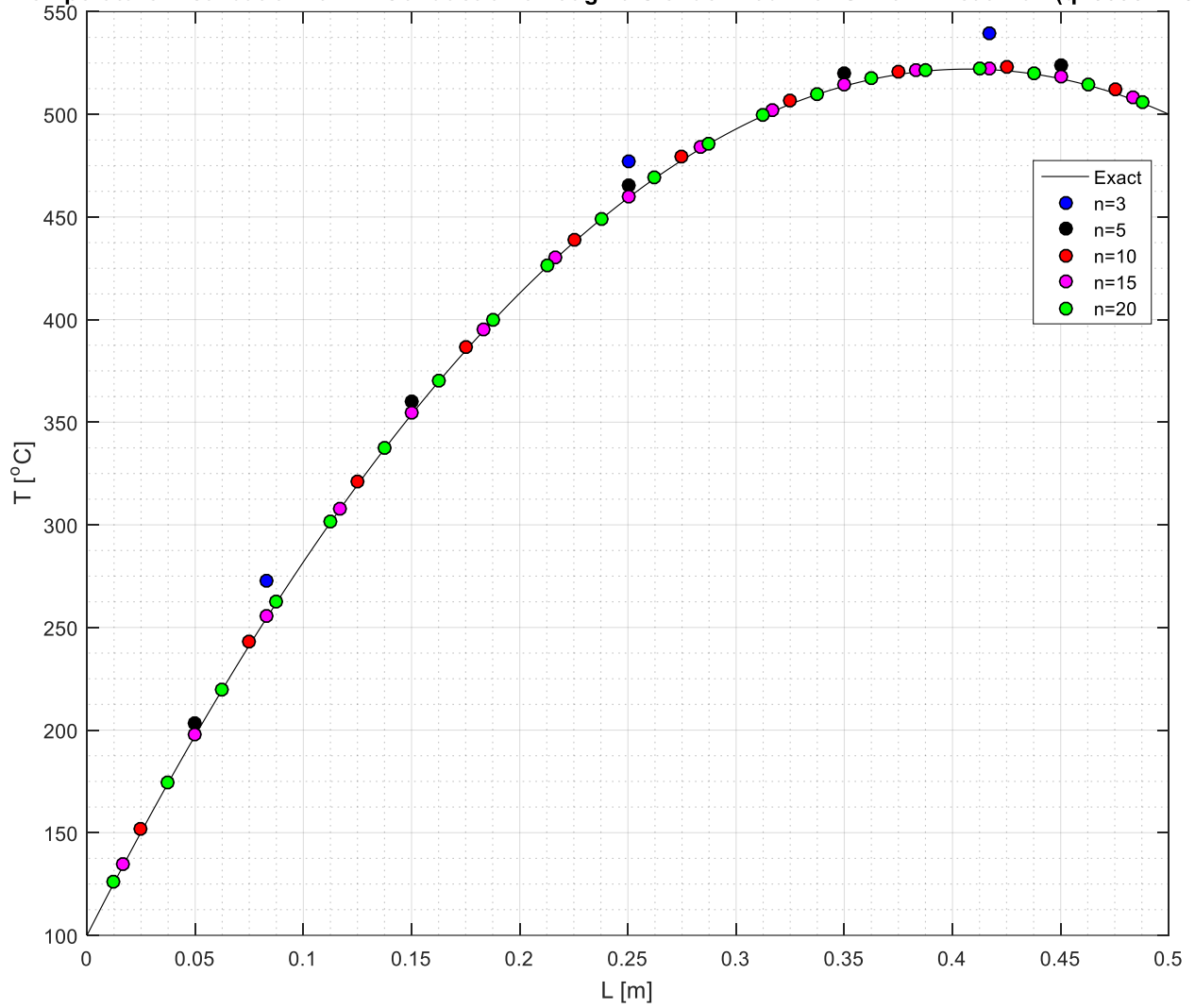
$$300T_5 = 100T_4 + 5095 + 200T_B \quad (4)$$

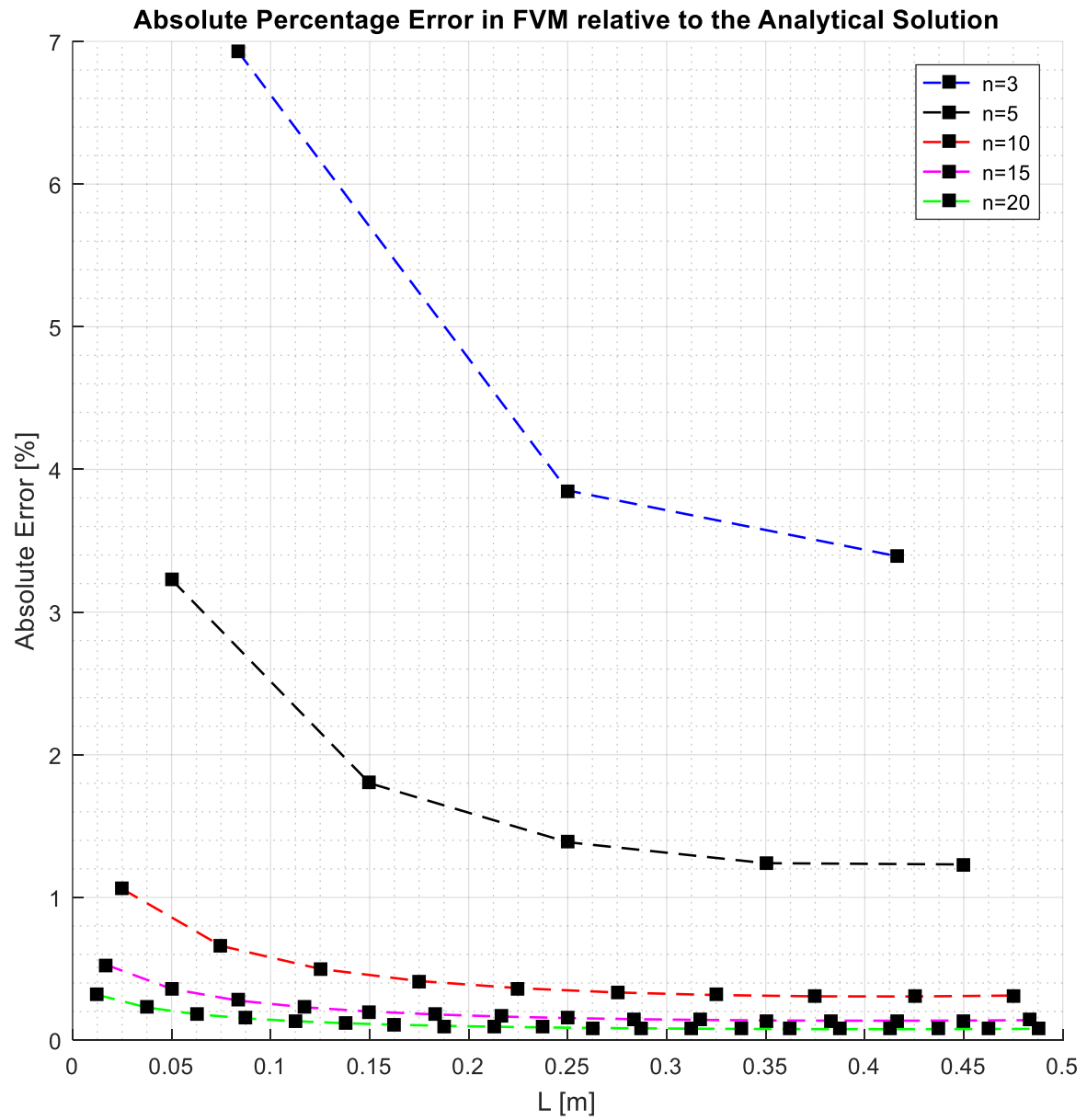
Temperatures at nodes 1-5 can be evaluated by solving the following equations, where the coefficient matrix is formed from noting the respective coefficients in equations 2-4;

$$\begin{bmatrix} 300 & -100 & 0 & 0 & 0 \\ -100 & 200 & -100 & 0 & 0 \\ 0 & -100 & 200 & -100 & 0 \\ 0 & 0 & -100 & 200 & -100 \\ 0 & 0 & 0 & -100 & 300 \end{bmatrix} \begin{bmatrix} T_1 \\ T_2 \\ T_3 \\ T_4 \\ T_5 \end{bmatrix} = \begin{bmatrix} 25,095 \\ 5,095 \\ 5,095 \\ 5,095 \\ 105,095 \end{bmatrix}$$

A similar procedure for 3, 10, 15 and 20 nodes is done and the solutions are plotted against the exact (analytical) solution, as well as the absolute percentage errors for each case.

Temperature Distribution for 1D Conduction through a Slender Rod with Uniform Heat Flux ($q=5095 \text{ kW/m}^3$)





The error is seen to decrease with the increase in the number of nodes and the numerical solution approaches the exact analytical solution.