

# Python for control purposes

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August 19, 2019



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# Chapter 1

## Introduction

### 1.1 Install the packages

### 1.2 The simplest way

I prepared a VirtualBox disk image [1] with a Debian distribution and all the required packages. VirtualBox is available for Windows, Linux, OS X and Solaris. All the features described in this document are available.

Please contact me via email to receive the link to the files.

### 1.3 Linux

#### 1.3.1 Required packages

The required modules can be simply installed using the usual package manager of the Linux distribution. It is also possible to install the Anaconda distribution [2] for Linux to get the basic Python modules.

It is important to check the versions of the Python modules, in particular numpy, scipy and sympy. Old versions of these packages don't allow to perform all the tasks described in this document. In case of old versions, it is possible to download the last versions of these packages from the SciPy download page [3], and install them from a Linux shell.

Under Debian we can use the apt manager to install the following packages:

- python-numpy
- python-scipy
- python-matplotlib
- python-sympy
- python-setuptools
- python-psutils
- jupyter

- jupyter-qtconsole

Under Debian and Ubuntu it is possible to check if all the required development packages are correctly installed using the shell command

```
sudo apt-get build-dep python-scipy
```

The following packages are not available as distribution packages and should be installed separately.

- The Python Control toolbox [4]
- The Slycot libraries [5]
- The pysimCoder package [6]

For the second part of the project (code generation etc.) the following packages are required

- python(3)-pyqt5
- python(3)-qwt

This features presented in the second part of this document are at present only interesting under the Linux OS, because the real-time code is generated for a Linux PREEMPT-RT machine.

### 1.3.2 Install the pysimCoder package

The package can be downloaded from the githib page (as zip or using “git clone”). The installation is quite simple. Launch as superuser the command

```
make
```

or

```
make reduced
```

if you don't want to have COMEDI installed.

The installation download the control package, the slycot package and install the full software. As last step it is important to update the “.bashrc” file as normal user with the command

```
make user
```

The system has been tested under “Debian stable”, “Debian testing” with python-2.7, python-3.5, python-3.6 and python-3.7.

## 1.4 Windows

Under Windows it is sufficient to install the “Anaconda” package [2], to have all the python and jupyter modules installed.

The Slycot libraries for Windows can be downloaded from here [7].

At present it is not possible to perform hybrid simulation and code generation under the Windows OS.

## 1.5 Mac OSX

The Anaconda package [2] is available for Mac OSX. The Slycot libraries can be downloaded from here [7].



# Chapter 2

## Python - Some hints for Matlab users

### 2.1 Basics

There are important differences between Matlab and Python. In particular, the Python approach to matrices and to indexed objects is quite different compared to Matlab.

More information about a comparison between Python and Matlab is available online at [8].

The web contains a lot of documentation about Python and its packages. In particular, the book of David Pine [9] gives a good introduction about the features of Python for scientific applications.

Other links present tutorials for **numpy** [10], **scipy** [11], **matplotlib** [12] and **sympy** [13].

### 2.2 The python shell

A Python script can run within a Python shell, but can also be launched as executable.

The basic python shell is similar to the Matlab shell without the java improvements (**matlab -nojvm**).

A better shell is for example **jupyter**. In this interactive form, when started as **jupyter-qtconsole**, jupyter already loads at startup a set of functions and modules.

Another interesting environment, more similar to the Matlab shell, is represented by the **Spyder** application. In this application it is possible to debug scripts and functions like in the Matlab environment.

In this document we are mostly working with **jupyter** launched with the shell commands

```
jupyter-qtconsole
```

Sometimes not all the functions and modules are explicitly loaded at the beginning of the examples. In addition, **jupyter** implements some useful commands like for example **whos** and **run** (for launching scripts).

In the jupyter shell it is possible to start single commands, paste a set of commands or launch a “.py” program using **run**.

```

In [1]: # single command

In [2]: a = 5

In [3]: # paste a set of commands

In [4]: a=5
...: b=7
...: c=a*b
...: print c
...:
35

In [5]: # run a .py file

In [6]: run DCmotorKane.py
Matrix([[ -Dm*w(t) + kt*I(t) ]])
Matrix([[ -J*Derivative(w(t), t) ]])
[[0 1]
 [0 -Dm/J]]
[[0]
 [kt/J]]

```

## 2.3 Python vs. Matlab

Differently from Matlab, Python implements more types of variables

```

In [1]: a=5

In [2]: b=2.7

In [3]: c=[[1,2,3],[4,5,6]]

In [4]: d='Ciao'

In [5]: whos

```

Variable	Type	Data/Info
a	<b>int</b>	5
b	<b>float</b>	2.7
c	<b>list</b>	n=2
d	<b>str</b>	Ciao

## 2.4 List, array and matrix

Python implements three kind of multidimensional objects: **list**, **array** and **matrix**. These objects are handled differently than in Matlab.

## 2.5 List

A Python **list** implements the Matlab **cell**. It represents the simplest and default indexed object.

```
In [1]: a=[[1,2],[3,4]], 'abcd',2]

In [2]: b=[[1,2,3],[4,5,6],[7,8,9]]

In [3]: whos
Variable    Type      Data/Info
-----
a           list      n=3
b           list      n=3
```

## 2.6 Arrays

In Python the **array** is a multidimensional variable that implements sets of values of the same type. Usually the elements of an array are numbers, but can also be booleans, strings, or other objects. An array is the basic instance for most scientific applications.

Operations like `*`, `/`, `**` etc. implement the **dot** operations of the Matlab environment (`.*`, `./` and `.^`). For example, the multiplication of two arrays `a * a` represents the value-by-value multiplication implemented in Matlab with the operation `a.*a`.

```
In [1]: from numpy import mat, matrix, array

In [2]: a=array([[1,2,3],[4,5,6]])

In [3]: b=array([[1],[2]])

In [4]: print a*a
[[ 1  4  9]
 [16 25 36]]

In [5]: print a*b
[[ 1  2  3]
 [ 8 10 12]]
```

## 2.7 Matrices

The **matrix** object is useful in case of linear algebra operations. In this case the variables are instanced using the **mat** or the **matrix** function.

```

In [1]: from numpy import mat, matrix, array

In [2]: a=mat(a)

In [3]: b=array([[1],[2],[3]])

In [4]: a*b
Out[5]:
matrix([[14],
        [32]])

In [6]: a=array(a)

```

```

In [7]: a*b
-----
ValueError                                Traceback (most recent
      call last)
<ipython-input-9-8201c27d19b7> in <module>()
----> 1 a*b

ValueError: operands could not be broadcast together with
      shapes (2,3) (3,1)

In [8]: b=mat(b)

In [9]: a*b
Out[10]:
matrix([[14],
        [32]])

```

## 2.8 Indexing

Indexing in Python is quite different compared with the syntax used in Matlab. Indices start from **0** (and not **1** as in Matlab). In addition, the syntax is different for lists, arrays and matrices.

## 2.9 Lists

1-dimension lists can be accessed using one index (ex.  $a[2]$ ). Multidimensional lists require multiple indices in the form  $[i][j]\dots$



```

In [1]: a=[1,2,3,4,5]

In [2]: %whos
Variable    Type      Data/Info
-----
a           list      n=5

In [3]: a[3]
Out[3]: 4

In [4]: b=[[1,2,3],[4,5,6]]

In [5]: %whos
Variable    Type      Data/Info
-----
a           list      n=5
b           list      n=2

In [6]: b[1][2]
Out[6]: 6

In [7]: b[0]
Out[7]: [1, 2, 3]

```

## 2.10 Arrays

Multidimensional arrays allow the use of indices in the forms  $[i, j]$  and  $[i][j]$ .

```

In [1]: from numpy import array

In [2]: a=array([1,2,3,4,5])

In [3]: b=array([[1,2,3],[4,5,6]])

In [4]: %whos
Variable    Type      Data/Info
-----
a           ndarray  5: 5 elems, type 'int64', 40 bytes
b           ndarray  2x3: 6 elems, type 'int64', 48 bytes

```

```

In [5]: a.shape
Out[5]: (5,)

In [6]: b.shape
Out[6]: (2, 3)

In [7]: a[3]
Out[7]: 4

In [8]: b[0,2]
Out[8]: 3

In [9]: b[0][2]
Out[9]: 3

In [10]: b[:,0]
Out[10]: array([1, 4])

In [11]: b[0,:]
Out[11]: array([1, 2, 3])

In [12]: b[0]
Out[12]: array([1, 2, 3])

```

## 2.11 Matrices

Matrices can be only indexed using the  $[i,j]$  syntax. A matrix has always a minimum of 2 dimensions.

```

In [1]: from numpy import mat

In [2]: a=array([1,2,3,4,5])

In [3]: b=array([[1,2,3],[4,5,6]])

In [4]: %whos
Variable    Type      Data/Info
-----
a           matrix    [[1 2 3 4 5]]
b           matrix    [[1 2 3]\n [4 5 6]]

In [5]: a.shape
Out[5]: (1, 5)

In [6]: b.shape
Out[6]: (2, 3)

```

```

In [7]: a[0,2]
Out[7]: 3

In [8]: b[1,1]
Out[8]: 5

In [9]: b[:,0]
Out[9]:
matrix([[1],
        [4]])

In [10]: b[0,:]
Out[10]: matrix([[1, 2, 3]])

```

## 2.12 Multidimensional arrays and matrices

Matrices and arrays can be defined with more than 2 dimensions.

```

In [1]: from numpy import array, mat

In [2]: a=zeros((3,3,3),int8)

In [3]: a.shape
Out[3]: (3, 3, 3)

In [4]: %whos
Variable      Type           Data/Info
-----
a             ndarray      3x3x3: 27 elems, type 'int8', 27
              bytes

In [5]: a[1,1,1]
Out[5]: 0
In [6]: a[1,1,1]=5

In [7]: a
Out[7]:
array([[[0, 0, 0],
        [0, 0, 0],
        [0, 0, 0]],

       [[0, 0, 0],
        [0, 5, 0],
        [0, 0, 0]],

       [[0, 0, 0],
        [0, 0, 0],
        [0, 0, 0]]], dtype=int8)

```



# Chapter 3

## The Python Control System toolbox

### 3.1 Basics

The Python Control Systems Library, is a package initially developed by Richard Murray at Caltech. This toolbox contains a set of python classes and functions that implement common operations for the analysis and design of feedback control systems. In addition, a MATLAB compatibility package (`control.matlab`) has been integrated in order to provide functions equivalent to the commands available in the MATLAB Control Systems Toolbox.

### 3.2 Models

LTI systems can be described in state-space form or as transfer functions.

### 3.3 Continuous systems

### 3.4 State-space representation

```
In [1]: from control import *
In [2]: a=[[0,1],[-1,-1]]
In [3]: b=[[0],[1]]
In [4]: c=[1,0]
In [5]: d=0
In [6]: sys = ss(a,b,c,d)
In [7]: print sys
A = [[ 0  1]
      [-1 -1]]
B = [[0]
      [1]]
C = [[1 0]]
D = [[0]]
```

### 3.5 Transfer function

```
In [1]: from control import *
In [2]: g=tf(1,[1,1,1])
In [3]: print g
      1
-----
s^2 + s + 1
```

### 3.6 Zeros-Poles-Gain

This method is not implemented in control toolbox yet. It is available in the package **scipy.signal** but it is not completely compatible with the class of LTI objects defined in the Python control toolbox.

### 3.7 Discrete time systems

An additional fields (**dt**) in the **StateSpace** and **TransferFunction** classes is used to differentiate continuous-time and discrete-time systems.

### 3.8 State-space representation

```

In [4]: a=[[0,1],[-1,1]]

In [5]: b=[[0],[1]]

In [6]: c=[1,-1]

In [7]: d=0

In [8]: sysd = ss(a,b,c,d,0.01)

In [9]: print sysd
A = [[ 0  1]
     [-1  1]]
B = [[0]
     [1]]
C = [[ 1 -1]]
D = [[0]]
dt = 0.01

```

### 3.9 Transfer function

```

In [1]: from control import *

In [2]: g=tf([1,-1],[1,-1,1],0.01)

In [3]: print g

      z - 1
      -----
    z^2 - z + 1
dt = 0.01

```

### 3.10 Conversions

The Python control system toolbox only implements conversion from continuous time systems to discrete-time systems (**c2d**) with the methods “zoh”, “tustin” and “matched”. No conversion from discrete to continuous has been implemented yet.

The `supsictrl.ctr_repl` package implements the function **d2c** with the methods “zoh”, “tustin” and “matched”.

```

In [1]: from control import *
In [2]: from control.Matlab import *
In [3]: g=tf(1,[1,1,1])
In [4]  # Matlab compatibility
In [5]: gd = c2d(g,0.01)
In [6]  # control toolbox
In [7]: gd2 = sample_system(g,0.01)
In [8]: print g

```

$$\frac{1}{s^2 + s + 1}$$

```

In [9]: print gd

```

$$\frac{4.983e-05 z + 4.967e-05}{z^2 - 1.99 z + 0.99}$$

```

dt = 0.01

```

```

In [1]: from control import *
In [2]: from supsictrl.ctrl_repl import d2c
In [3]: g=tf(1,[1,1,1])
In [4]: gd =c2d(g,0.01)
In [5]: g2=d2c(gd)
In [6]: print g

```

$$\frac{1}{s^2 + s + 1}$$

```

In [7]: print g2

```

$$\frac{1.729e-14 s + 1}{s^2 + s + 1}$$

### 3.11 Casting

The control.matlab module implements the casting functions to transform LTI systems to a transfer function (**tf**) or to a state-space form (**ss**).



```
In [8]: g = tf(sys)

In [9]: print g
```

$$\frac{1}{s^2 + s + 1}$$

and transfer functions into one of the state-space representation

```
In [10]: sys = ss(g)

In [11]: print sys
A = [[ 0. -1.]
      [ 1. -1.]]
B = [[-1.]
      [ 0.]]
C = [[ 0. -1.]]
D = [[ 0.]]
```

## 3.12 Models interconnection

Commands like **parallel** and **series** are available in order to interconnect systems. The operators **+** and **\*** have been overloaded for the LTI class to perform the same operations. In addition the command **feedback** is implemented exactly as in Matlab.

```
In [1]: from control import *

In [2]: g1=tf(1,[1,1])

In [3]: g2=tf(1,[1,2])

In [4]: print parallel(g1,g2)
```

$$\frac{2s + 3}{s^2 + 3s + 2}$$

```


In [5]: print g1+g2
```

$$\frac{2s + 3}{s^2 + 3s + 2}$$

```
In [6]: print series(g1,g2)
```

$$\frac{1}{s^2 + 3s + 2}$$

```
In [7]: print g1*g2
```

$$\frac{1}{s^2 + 3s + 2}$$

```
In [8]: print feedback(g1,g2)
```

$$\frac{s + 2}{s^2 + 3s + 3}$$

# Chapter 4

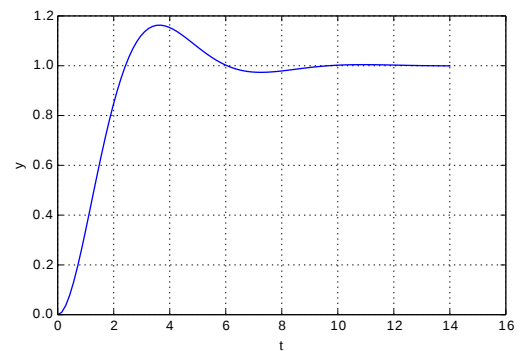
## System analysis

### 4.1 Time response

The Python Control toolbox offers own functions to simulate the time response of systems. For Matlab users, the `control.matlab` module gives the possibility to work with the same syntax as in Matlab. Please take care about the order of the return values!

Examples of time responses are shown in the figures 4.1, 4.2, 4.3, 4.4 and 4.5.

```
In [1]: from control import *
In [2]: import matplotlib.pyplot as plt
In [3]: g = tf(1,[1,1,1])
In [4]: t,y = step_response(g)
In [5]: plt.plot(t,y)
...: plt.grid()
...: plt.xlabel('t')
...: plt.ylabel('y')
```



or alternatively

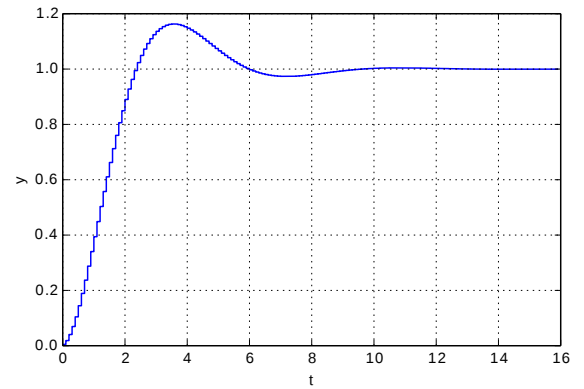
```
In [1]: from control import *
In [2]: from control.matlab import *
In [3]: import matplotlib.pyplot as plt
In [4]: g = tf(1,[1,1,1])
In [5]: y,t = step(g)
In [6]: plt.plot(t,y)
...: plt.xlabel('t')
...: plt.ylabel('y')
...: plt.grid()
```

Figure 4.1: Step response for continuous-time systems

```

In [1]: from control import *
In [2]: from control.matlab import c2d
In [3]: import matplotlib.pyplot as plt
In [4]: g = tf(1,[1,1,1])
In [5]: gz=c2d(g,0.1)
In [6]: t=np.arange(0,16,0.1)
In [7]: t1,y = step_response(gz,t)
In [8]: plt.step(t,y)
...: plt.grid()
...: plt.xlabel('t')
...: plt.ylabel('y')

```



or alternatively

```

In [1]: from control import *
In [2]: from control.matlab import *
In [3]: import matplotlib.pyplot as plt
In [4]: g = tf(1,[1,1,1])
In [5]: gz=c2d(g,0.1)
In [6]: t=np.arange(0,16,0.1)
In [7]: y,t1 = step(gz,t)
In [8]: plt.step(t,y)
...: plt.grid()
...: plt.xlabel('t')
...: plt.ylabel('y')

```

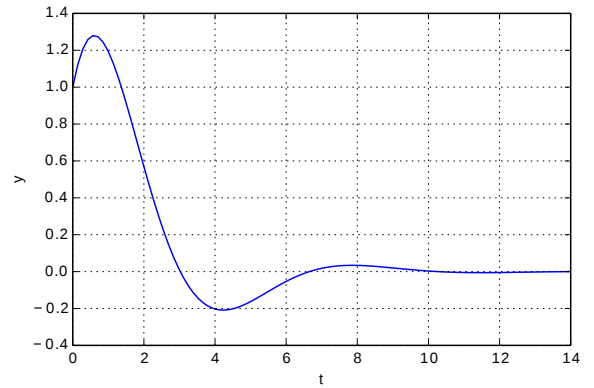
Figure 4.2: Step response for discrete-time systems

```

In [1]: from control import *
In [2]: import matplotlib.pyplot as plt
In [3]: a=[[0,1],[-1,-1]]
In [4]: b=[[0],[1]]
In [5]: c=[1,0]
In [6]: d=[0]
In [7]: sys=ss(a,b,c,d)
In [8]: t,y=initial_response(sys,
                             X0=[1,1])

In [9]: plt.plot(t,y)
...: plt.grid()
...: plt.xlabel('t')
...: plt.ylabel('y')

```



or alternatively

```

In [1]: from control import *
In [2]: from control.matlab import *
In [3]: import matplotlib.pyplot as plt
In [4]: a=[[0,1],[-1,-1]]
In [5]: b=[[0],[1]]
In [6]: c=[1,0]
In [7]: d=[0]
In [8]: sys=ss(a,b,c,d)
In [9]: y,t=initial(sys,X0=[1,1])

In [10]: plt.plot(t,y)
...: plt.xlabel('t')
...: plt.ylabel('y')
...: plt.grid()

```

Figure 4.3: Continuous time systems - Initial condition response

```

In [1]: from control import *
In [2]: import matplotlib.pyplot as plt
In [3]: g = tf(1,[1,1,1])
In [4]: t,y = impulse_response(g)
In [5]: plt.plot(t,y)
...: plt.grid()
...: plt.xlabel('t')
...: plt.ylabel('y')

```

or alternatively

```

In [1]: from control import *
In [2]: from control.matlab import *
In [3]: import matplotlib.pyplot as plt
In [4]: g = tf(1,[1,1,1])
In [5]: y,t = impulse(g)
In [6]: plt.plot(t,y)
...: plt.grid()
...: plt.xlabel('t')
...: plt.ylabel('y')

```

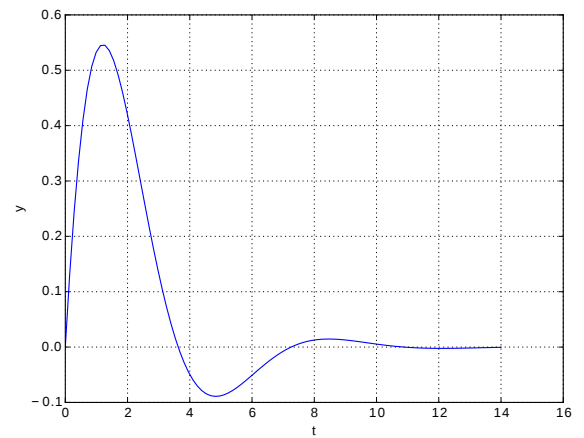
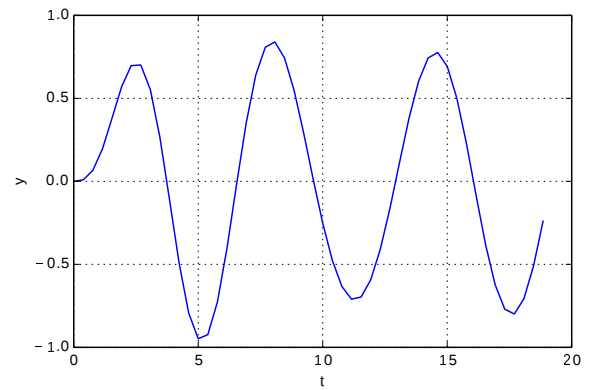


Figure 4.4: Continuous time systems - Impulse response

```

In [1]: from control import *
In [2]: import matplotlib.pyplot as plt
In [3]: g=tf([1,2],[1,2,3,4])
In [4]: t=linspace(0,6*pi)
In [5]: u=sin(t)
In [6]: t,y,x = forced_response(g,t,u)
In [7]: plt.plot(t,y)
...: plt.xlabel('t')
...: plt.ylabel('y')
...: plt.grid()

```



or alternatively

```

In [1]: from control import *
In [2]: from control.matlab import *
In [3]: import matplotlib.pyplot as plt
In [4]: g=tf([1,2],[1,2,3,4])
In [5]: t=linspace(0,6*pi)
In [6]: u=sin(t)
In [7]: y,t,x = lsim(g,u,t)
In [8]: plt.plot(t,y)
...: plt.xlabel('t')
...: plt.ylabel('y')
...: plt.grid()

```

Figure 4.5: Continuous time systems - Generic input

## 4.2 Frequency analysis

The frequency analysis includes some commands like **bode\_response**, **nyquist\_response**, **nichols\_response** and the corresponding Matlab versions **bode**, **nyquist** and **nichols**. (See figures 4.6, 4.7 and 4.8)

```
In [1]: from control import *
In [2]: g=tf([1],[1,0.5,1])
In [3]: bode_plot(g, dB=True);
```

or alternatively

```
In [1]: from control import *
In [2]: from control.matlab import *
In [3]: g=tf([1],[1,0.5,1])
In [4]: bode(g, dB=True);
```

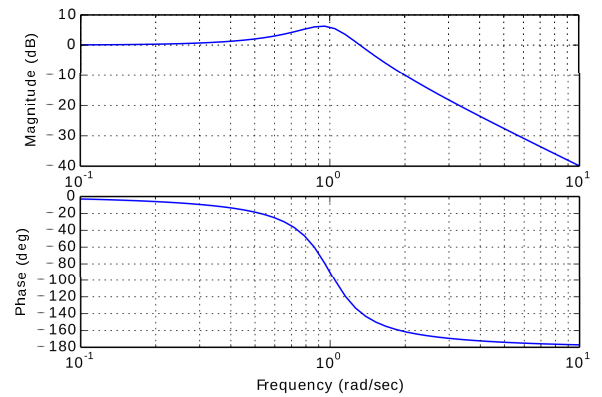


Figure 4.6: Bode plot

The command **margins** returns the gain margin, the phase margin and the corresponding crossover frequencies.

```
In [1]: from control import *
In [2]: g=tf(2,[1,2,3,1])
In [3]: gm, pm, wg, wp = margin(g)
In [4]: gm                                     # Gain, not dB!
Out[4]: 2.5000000000000013
In [5]: pm
Out[5]: 76.274075256921392                     # deg
In [6]: wg
Out[6]: 0.85864877610167201                   # rad/s
In [7]: wp
Out[7]: 1.7320508075688776                     # rad/s
```

In addition, the command **stability\_margins** returns the stability margin and the corresponding frequency. The stability margin values  $w_s$  and  $s_m$ , which correspond to the shortest distance from the Nyquist curve to the critical point  $-1$ , are useful for the sensitivity analysis.



```
In [1]: from control import *
In [2]: import matplotlib.pyplot as plt
In [3]: g=tf([1],[1,2,1])
In [3]: nyquist_plot(g), plt.grid()
```

or alternatively

```
In [1]: from control import *
In [2]: import matplotlib.pyplot as plt
In [3]: from control.matlab import *
In [4]: g=tf(1,[1,2,1])
In [5]: nyquist(g), plt.grid()
```

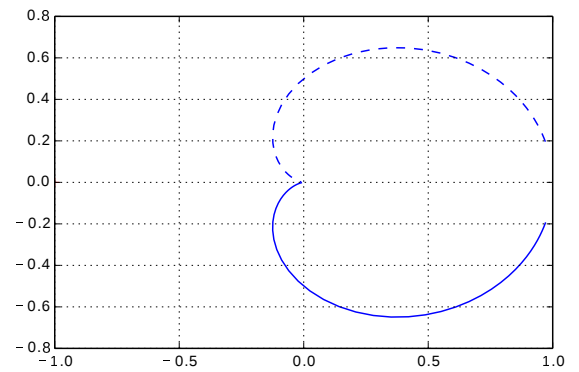


Figure 4.7: Nyquist plot

```
In [1]: from control import *
In [2]: g=tf(1,[1,2,3,4,0])
In [3]: nichols_plot(g)
```

or alternatively

```
In [1]: from control import *
In [2]: g=tf(1,[1,2,3,4,0])
In [3]: nichols(g)
```

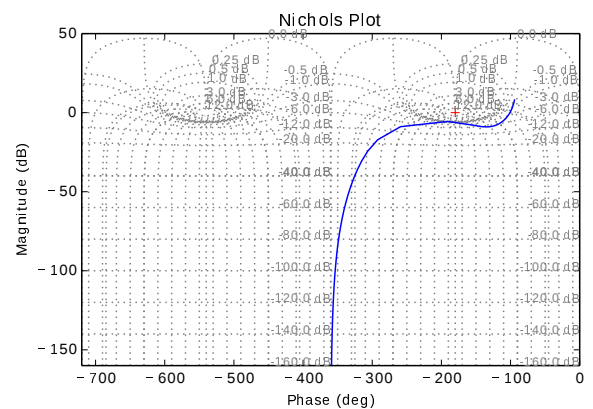


Figure 4.8: Nichols plot

```

In [1]: from control import *
In [2]: g=tf(2,[1,2,3,1])
In [3]: gm, pm, sm, wg, wp, ws = stability_margins(g)
In [4]: gm
Out[4]: 2.50000000000000013          # Gain not dB'
In [5]: pm
Out[5]: 76.274075256921392          # deg
In [6]: wg
Out[6]: 1.7320508075688776          # rad/s
In [7]: wp
Out[7]: 0.85864877610167201          # rad/s
In [8]: sm
Out[8]: 0.54497577553096421          #
In [9]: ws
Out[9]: 1.3669371206538097          # rad/s

```

### 4.3 Poles, zeros and root locus analysis

Poles and zeros of an open loop system can be calculated with the commands **pole**, **zero** or plotted and calculated with **pzmap**.

In addition there are two functions that implement the root locus command: **rlocus** and **root\_locus**. At present no algorithm to automatically choose the values of  $K$  has been implemented: if not provided, the  $K$  vector is calculated in **rlocus** with log values between  $10^{-3}$  and  $10^3$ . For the **root\_locus** function the  $K$  values should be provided.

If in the jupyter shell you set the command **%matplotlib qt**, the root locus is plotted on an external window and it is possible to get the values of gain and damp by clicking with the mouse on the curves.

Clicked at	-0.5724	+1.293j	gain	1.722	damp
0.4048					
Clicked at	-1.119	+0.01874j	gain	2.252	damp
0.9999					
Clicked at	-0.7545	+1.293j	gain	1.114	damp
0.504					

```

In [1]: from control import *
In [2]: from control.pzmap import pzmap
In [3]: g=tf([1,1],[1,2,3,4,0])
In [4]: g.pole()
Out[4]:
array([-1.65062919+0.j           ,
       -0.17468540+1.54686889j ,
       -0.17468540-1.54686889j ,
        0.00000000+0.j          ])
In [5]: g.zero()
Out[5]: array([-1.])
In [6]: poles, zeros = pzmap(g), grid()
In [7]: poles
Out[7]:
array([-1.65062919+0.j           ,
       -0.17468540+1.54686889j ,
       -0.17468540-1.54686889j ,
        0.00000000+0.j          ])
In [8]: zeros
Out[8]: array([-1.])

```

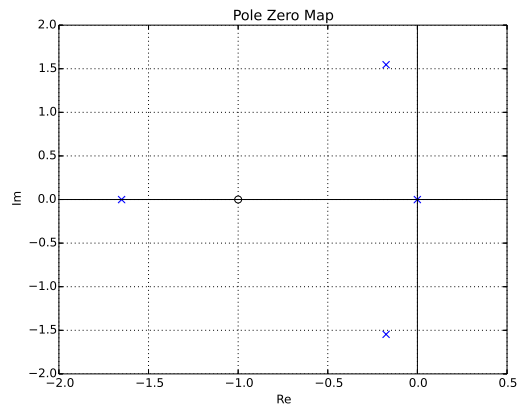


Figure 4.9: Poles and zeros

```

In [1]: from control import *
In [2]: g=tf(1,[1,2,3,0])
In [3]: rlocus(g);

```

or alternatively

```

In [1]: from control import *
In [2]: g=tf(1,[1,2,3,0])
In [3]: k=logspace(-3,3,100)
In [4]: root_locus(g,k);

```

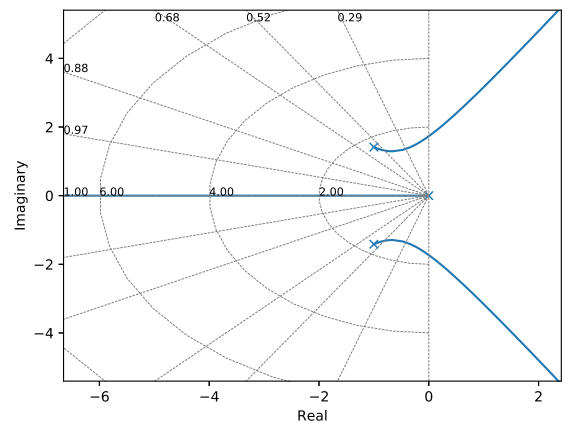


Figure 4.10: Root locus plot



# Chapter 5

## Modeling

The **sympy** module (symbolic python) contains a full set of operations to manage physical systems. In particular, it is possible to find the linearized model of a mechanical system using the Lagrange's method or the Kane's method. More details about the Kane's method are available at [14], [15], [16], [17], [18] and [19].

In the next sections we present the modelling of 3 plants that we can find in our laboratories and that are quite familiar to us.

### 5.1 Model of a DC motor (Lagrange method)

#### 5.1.1 Plant

In this first example we model a DC servo motor with a current input in order to find its state-space representation. The motor is characterized by a torque constant  $k_t$ , an inertia (motor+load)  $J$  and a friction constant  $D_m$ .

The input of the plant is the current  $I$  and the output is the position  $\varphi$ . The rotation center is the point  $\mathbf{O}$ , the main coordinates system is  $\mathbf{N}$  and we add a local reference frame  $\mathbf{Nr}$  which rotates with the load (angle  $\varphi$  and speed  $\omega$ ).

### 5.1.2 Modules and constants

```
In [1]: from sympy import symbols, Matrix, pi
...: from sympy.physics.mechanics import *
...: import numpy as np
...:
...: # Modeling the system with Lagrange method
...:
...: # Signals
...: ph = dynamicsymbols('ph')      # motor angle
...: w = dynamicsymbols('ph', 1)    # motor rot.
...:   speed
...: I = dynamicsymbols('I')        # input current
...:
...: # Constants
...: Dm = symbols('Dm')              # friction
...: M, J = symbols('M,J')          # Mass and inertia
...: t = symbols('t')               # time
...: kt = symbols('kt')             # torque constant
...:
```

### 5.1.3 Reference frames

```
In [2]: # Reference frame for the motor and Load
...: N = ReferenceFrame('N')
...:
...: O = Point('O')                 # center of rotation
...: O.set_vel(N, 0)
...:
...: # Reference frames for the rotating disk
...: Nr = N.orientnew('Nr', 'Axis', [ph, N.x])  #
...:   rotating reference (load)
...:
```

### 5.1.4 Body and inertia of the load

```
In [3]: # Mechanics
...: Io = J*outer(Nr.x, Nr.x)
...:
...: InT = (Io, O)
...:
...: B = RigidBody('B', O, Nr, M, InT)
...: B.potential_energy = 0
...:
```

### 5.1.5 Forces and torques

In order to find the dynamic model of the plant we need some other definitions, in particular the relation between angle  $\varphi$  and angular velocity  $\omega$ , the forces and torques applied to the system and a vector that contains the rigid bodies of the system.

```
In [4]: # Forces and torques
...: forces = [(Nr, (kt*I-Dm*w)*N.x)]
```

### 5.1.6 Model

Using the Lagrange's method is now possible to find the dynamic matrices related to the plant.

```
In [5]: # Lagrange model
...: L = Lagrangian(N, B)      # Lagrange operator
...: LM = LagrangesMethod(L, [ph], forcelist = forces,
...: frame = N)
...: LM.form_lagranges_equations()
...:
...: # symbolically linearize about arbitrary
...: equilibrium
...: MM, linear_state_matrix, linear_input_matrix,
...: inputs = LM.linearize(q_ind = [ph], qd_ind = [w])
...:
```

### 5.1.7 State-space matrices

From the results of the Kane's model identification, we can now extract the matrices  $A$  and  $B$  of the state-space representation.

```
In [6]: print(linear_state_matrix)
...: print(linear_input_matrix)
...:
Matrix([[0, 1], [0, -Dm]])
Matrix([[0], [kt]])
```

## 5.2 Model of a DC motor (Kane method)

### 5.2.1 Plant

In this first example we model a DC servo motor with a current input in order to find its state-space representation. The motor is characterized by a torque constant  $k_t$ , an inertia (motor+load)  $J$  and a friction constant  $D_m$ .

The input of the plant is the current  $I$  and the output is the position  $\varphi$ . The rotation center is the point  $\mathbf{O}$ , the main coordinates system is  $\mathbf{N}$  and we add a local reference frame  $\mathbf{Nr}$  which rotates with the load (angle  $\varphi$  and speed  $\omega$ ).

### 5.2.2 Modules and constants

```
In [1]: from sympy import symbols, Matrix, pi
...: from sympy.physics.mechanics import *
...: import numpy as np
...:
...: # Modeling the system with Kane method
...:
...: # Signals
...: ph = dynamicsymbols('ph')      # motor angle
...: w = dynamicsymbols('w')        # motor rot. speed
...: I = dynamicsymbols('I')        # input current
...:
...: # Constants
...: Dm = symbols('Dm')              # friction
...: M, J = symbols('M J')          # Mass and inertia
...: t = symbols('t')               # time
...: kt = symbols('kt')             # torque constant
...:
```

### 5.2.3 Reference frames

```
In [2]: # Reference frame for the motor and Load
...: N = ReferenceFrame('N')
...:
...: O = Point('O')                # center of rotation
...: O.set_vel(N, 0)
...:
...: # Reference frames for the rotating disk
...: Nr = N.orientnew('Nr', 'Axis', [ph, N.x]) #
...:      rotating reference (load)
...:
...: Nr.set_ang_vel(N, w*N.x)
...:
```

### 5.2.4 Body and inertia of the load

```
In [3]: # Mechanics
...: Io = J*outer(Nr.x, Nr.x)
...:
...: InT = (Io, O)
...:
...: B = RigidBody('B', O, Nr, M, InT)
...:
```

### 5.2.5 Forces and torques

In order to find the dynamic model of the plant we need some other definitions, in particular the relation between angle  $\varphi$  and angular velocity  $\omega$ , the forces and torques applied to the system and a vector that contains the rigid bodies of the system.



```
In [4]: # Forces and torques
...: forces = [(Nr, (kt*I-Dm*w)*N.x)]
...:
...: kindiffs = [(ph.diff(t)-w)]
...:
...: bodies=[B]
...:
```

### 5.2.6 Model

Using the Kane's method is now possible to find the dynamic matrices related to the plant.

```
In [5]: # Model
...: KM = KanesMethod(N, q_ind=[ph], u_ind=[w], kd_eqs=
...:     kindiffs)
...: fr, frstar = KM.kanes_equations(forces, bodies)
...:
...: print fr
...: print frstar
...:
Matrix([[ -Dm*w(t) + kt*I(t) ]])
Matrix([[ -J*Derivative(w(t), t) ]])
```

### 5.2.7 State-space matrices

From the results of the Kane's model identification, we can now extract the matrices  $A$  and  $B$  of the state-space representation.

```
In [6]: # symbolically linearize about arbitrary
...:     equilibrium
...: MM, linear_state_matrix, linear_input_matrix,
...:     inputs =
KM.linearize(new_method=True)
...:
...: # set the the equilibrium point
...: eq_pt = [0, 0]
...: eq_dict = dict(zip([ph,w], eq_pt))
...:
...: f_A_lin = linear_state_matrix.subs(eq_dict)
...: f_B_lin = linear_input_matrix.subs(eq_dict)
...: MM = MM.subs(eq_dict)
...:
...: # compute A and B matrices
...: A = MM.inv() * f_A_lin
...: B = MM.inv() * f_B_lin
```

```

In [6]: print A
...: print B
...:
[[0 1]
 [0 -Dm/J]]
[[0]
 [kt/J]]

```

### 5.3 Model of the inverted pendulum - Lagrange

The second example is represented by the classical inverted pendulum as shown in figure 5.1.

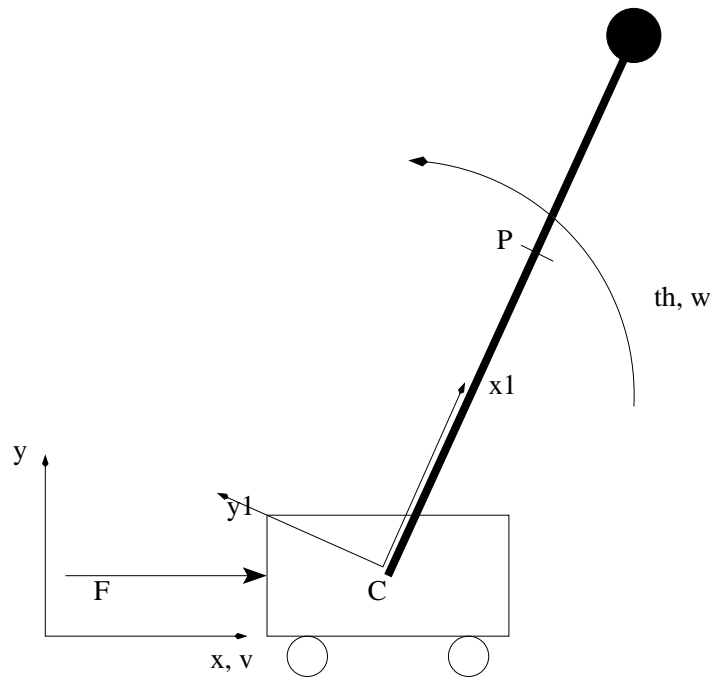


Figure 5.1: Inverted pendulum

The global reference frame is  $\mathbf{Nf}(x, y)$ . The point  $\mathbf{P}$  is the center of mass of the pendulum. The car is moving with speed  $\mathbf{v}$  and position  $\mathbf{C}$ . The pole is rotating with the angle  $\mathbf{th}$  and angular velocity  $\mathbf{w}$ . In addition to the main coordinate frame  $\mathbf{Nf}(x, y)$ , we define a local body-fixed frame to the pendulum  $\mathbf{Npend}(x_1, y_1)$ .



Figure 5.2: Inverted pendulum - Real plant

### 5.3.1 Modules and constants

```
In [1]: from sympy import symbols, Matrix, pi, cos, sin
...: from sympy.physics.mechanics import *
...: import numpy as np
...:
...: # Modeling the system with Kane method
...:
...: # Signals
...: x, th = dynamicsymbols('x_th')
...: v, w = dynamicsymbols('x_th', 1)
...: F = dynamicsymbols('F')
...: d = symbols('d')
...:
...: # Constants
...: m, r = symbols('m_r')
...: M = symbols('M')
...: g, t = symbols('g_t')
...: Ic = symbols('Ic')
...:
```

### 5.3.2 Frames - Car and pendulum

```
In [2]: # Frames and Coord. system
...:
...: # Car
...: Nf = ReferenceFrame('Nf')
...: C = Point('C')
...: C.set_vel(Nf, v*Nf.x)
...: Car = Particle('Car', C, M)
...:
...: # Pendulum
...: A = Nf.orientnew('A', 'Axis', [th, Nf.z])
...: A.set_ang_vel(Nf, w*Nf.z)
...:
...: P = C.locatenew('P', r*A.x)
...: P.v2pt_theory(C, Nf, A)
...: Pa = Particle('Pa', P, m)
...:
```

### 5.3.3 Points, bodies, masses and inertias

```
In [3]: I = outer(Nf.z, Nf.z)
...: Inertia_tuple = (Ic*I, P)
...: Bp = RigidBody('Bp', P, A, m, Inertia_tuple)
...:
...: Bp.potential_energy = m*g*r*sin(th)
...: Car.potential_energy = 0
...:
```

### 5.3.4 Forces, frictions and gravity

```
In [4]: # Forces and torques
...: forces = [(C, F*Nf.x - d*v*Nf.x), (P, 0*Nf.y)]
...:
```

## 5.3.5 Final model and linearized state-space matrices

```

In [5]: # Lagrange operator
...: L = Lagrangian(Nf, Car, Bp)
...:
...: # Lagrange model
...: LM = LagrangesMethod(L, [x, th], forcelist =
...:     forces, frame = Nf)
...: LM.form_lagranges_equations()
...:
...: # Equilibrium point
...: eq_pt = [0.0, pi/2, 0.0, 0.0]
...: eq_dict = dict(zip([x, th, v, w], eq_pt))
...:
...: # symbolically linearize about arbitrary
...:     equilibrium
...: MM, linear_state_matrix, linear_input_matrix,
...:     inputs = LM.linearize(q_ind = [x, th], qd_ind = [v,
...:     w])
...:
...: f_p_lin = linear_state_matrix.subs(eq_dict)
...: f_B_lin = linear_input_matrix.subs(eq_dict)
...:
...: MM = MM.subs(eq_dict)
...:
...: Atmp = MM.inv() * f_p_lin
...: Btmp = MM.inv() * f_B_lin
...:

```

```

In [6]: Atmp
Out[6]:
Matrix([
[0,
0,
0],
[0,
1,
0],
[0,
g*m**2*r**2/(-m**2*r**2 + (Ic + m*r**2)*(M + m)), -d
*(Ic + m*r**2)/(-m**2*r**2 + (Ic + m*r**2)*(M + m)),
0],
[0, g*m*r*(M + m)/(-m**2*r**2 + (Ic + m*r**2)*(M + m)),
-d*m*r/(-m**2*r**2 + (Ic + m*r**2)*(M + m)),
0]])

```

```

In [7]: Btmp
Out[7]:
Matrix([
[0],
[0],
[(Ic + m*r**2)/(-m**2*r**2 + (Ic + m*r**2)*(M + m))],
[m*r/(-m**2*r**2 + (Ic + m*r**2)*(M + m))]])

```

## 5.4 Model of the inverted pendulum - Kane

The global reference frame is  $\mathbf{Nf}(x, y)$ . The point  $\mathbf{P}$  is the center of mass of the pendulum. The car is moving with speed  $\mathbf{v}$  and position  $\mathbf{C}$ . The pole is rotating with the angle  $\mathbf{th}$  and angular velocity  $\mathbf{w}$ . In addition to the main coordinate frame  $\mathbf{Nf}(x, y)$ , we define a local body-fixed frame to the pendulum  $\mathbf{Npend}(x_1, y_1)$ .

### 5.4.1 Modules and constants

```
In [1]: from sympy import symbols, Matrix, pi
...: from sympy.physics.mechanics import *
...: import numpy as np
...:
...: # Modeling the system with Kane method
...:
...: # Signals
...: x, th = dynamicsymbols('x_th')
...: v, w = dynamicsymbols('v_w')
...: F = dynamicsymbols('F')
...:
...: # Constants
...: d = symbols('d') # friction
...: m, r = symbols('m_r')
...: M = symbols('M')
...: g, t = symbols('g_t')
...: J = symbols('J')
...:
```

### 5.4.2 Frames - Car and pendulum

```
In [2]: # Frames and Coord. system
...:
...: # Car - reference x,y
...: Nf = ReferenceFrame('Nf')
...: C = Point('C')
...: C.set_vel(Nf, v*Nf.x)
...: Car = Particle('Car', C, M)
...:
...: # Pendulum - reference x1, y1
...: Npend = Nf.orientnew('Npend', 'Axis', [th, Nf.z])
...: Npend.set_ang_vel(Nf, w*Nf.z)
...:
...: P = C.locatenew('P', r*Npend.x)
...: P.v2pt_theory(C, Nf, Npend)
...: Pa = Particle('Pa', P, m)
...:
```

### 5.4.3 Points, bodies, masses and inertias

```
In [3]: I = outer (Nf.z, Nf.z)
...: Inertia_tuple = (J*I, P)
...: Bp = RigidBody('Bp', P, Npend, m, Inertia_tuple)
...:
```

### 5.4.4 Forces, frictions and gravity

```
In [4]: # Forces and torques
...: forces = [(C,F*Nf.x-d*v*Nf.x),(P,-m*g*Nf.y)]
...: frames = [Nf,Npend]
...: points = [C,P]
...:
...: kindiffs = [x.diff(t)-v, th.diff(t) - w]
...: particles = [Car,Bp]
...:
```

### 5.4.5 Final model and linearized state-space matrices

```
n [5]: # Model
...: KM = KanesMethod(Nf,q_ind=[x,th],u_ind=[v,w],
...: kd_eqs=kindiffs)
...: fr,frstar = KM.kanes_equations(forces,particles)
...:
...: # Equilibrium point
...: eq_pt = [0, pi/2,0,0]
...: eq_dict = dict(zip([x,th,v,w], eq_pt))
...:
...: # symbolically linearize about arbitrary
...: equilibrium
...: MM, linear_state_matrix, linear_input_matrix,
...: inputs =
KM.linearize(new_method=True)
...:
...: # sub in the equilibrium point and the parameters
...: f_A_lin = linear_state_matrix.subs(eq_dict)
...: f_B_lin = linear_input_matrix.subs(eq_dict)
...: MM = MM.subs(eq_dict)
...:
...: # compute A and B
...: A = MM.inv() * f_A_lin
...: B = MM.inv() * f_B_lin
...:
```

```

In [6]: A
Out [6]:
Matrix([
[0, 0, 1, 0],
[0, 0, 0, 1],
[0, g*m**2*r**2/(J*M + J*m + M*m*r**2), -d*(m**2*r**2/((
M + m)*(J*M + J*m
+ M*m*r**2)) + 1/(M + m)), 0],
[0, g*m*r*(M + m)/(J*M + J*m + M*m*r**2),
-d*m*r/(J*M + J*m + M*m*r**2), 0]])

```

```

In [7]: B
Out [7]:
Matrix([
[
0],
[
0],
[m**2*r**2/((M + m)*(J*M + J*m + M*m*r**2)) + 1/(M + m)],
[m*r/(J*M + J*m + M*m*r**2)]]

```

And the results can be written in a better form as

$$A = \begin{bmatrix} 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \\ 0 & \frac{gm^2r^2}{JcM+Jcm+Mmr^2} & -\frac{d(Jc+mr^2)}{JcM+Jcm+Mmr^2} & 0 \\ 0 & \frac{gmr(M+m)}{JcM+Jcm+Mmr^2} & -\frac{dmr}{JcM+Jcm+Mmr^2} & 0 \end{bmatrix}$$

and

$$B = \begin{bmatrix} 0 \\ 0 \\ \frac{Jc+mr^2}{JcM+Jcm+Mmr^2} \\ \frac{mr}{JcM+Jcm+Mmr^2} \end{bmatrix}$$

## 5.5 Model of the Ball-on-Wheel plant - Lagrange

A more complex plant is represented by the Ball-on-Wheel system of figure 5.3, where a ball must be maintained in the unstable equilibrium point on the top of a bike wheel.

In this system we have 4 reference frames. The frame **N** is the main reference frame, **N0** rotates with the line connecting the centers of mass of the wheel (**O**) and of the ball (**CM2**), **N1** ( $x_1, y_1$ ) rotates with the wheel and **N2** ( $x_2, y_2$ ) is body-fixed to the ball.

The radius of the wheel and of the ball are respectively  $R_1$  and  $R_2$ . The non sliding condition is given by

$$R_1 \cdot ph_0 = R_1 \cdot ph_1 + R_2 \cdot ph_2$$

The input of the system is represented by the torque  $T$  applied to the wheel.



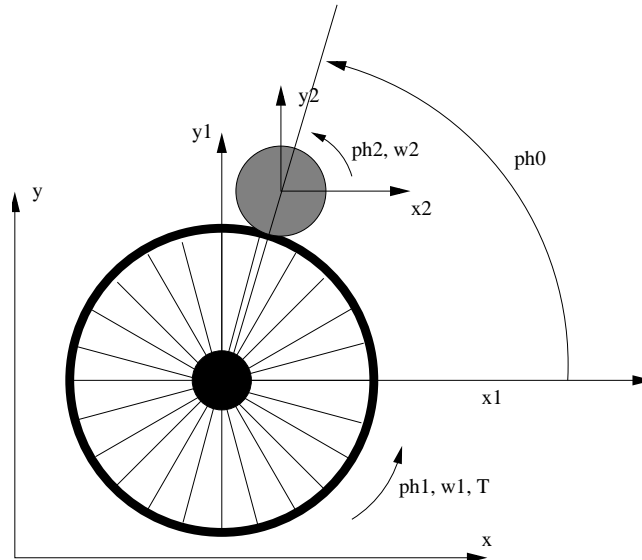


Figure 5.3: Ball-On-Wheel plant

### 5.5.1 Modules and constants

```
In [1]: from sympy import symbols, Matrix, pi, sin, cos
...: from sympy.physics.mechanics import *
...: # Lagrange Model of the system
...: # Index _b: angle between Wheel center and Ball CM
...: # Index _w: Wheel
...: # Index _roll: Ball
...:
...: # Dynamic symbols
...: phi_b, phi_w, phi_roll = dynamicsymbols('phi_b_
...: phi_w_phi_roll')
...: w_b, w_w = dynamicsymbols('phi_b_phi_w', 1)
...: w_roll = dynamicsymbols('w_roll')
...: T = dynamicsymbols('T')
...:
...: # Symbols
...: J_w, J_b = symbols('J_w_J_b')
...: M_w, M_b = symbols('M_w_M_b')
...: R_w, R_b = symbols('R_w_R_b')
...: d_w = symbols('d_w')
...: g = symbols('g')
...: t = symbols('t')
...:
```

### 5.5.2 Reference frames

```

In [2]: # Mechanical system
...: N = ReferenceFrame('N')
...:
...: O = Point('O')
...: O.set_vel(N,0)
...:
...: # Roll conditions
...: phi_roll = -(phi_w*R_w-phi_b*R_w)/R_b
...: w_roll = phi_roll.diff(t)
...:
...: # Rotating axes
...: # Ball rotation
...: # Wheel rotation
...: # Ball position
...: N_b = N.orientnew('N_b','Axis',[phi_b,N.y])
...: N_w = N.orientnew('N_w','Axis',[phi_w,N.y])
...: N_roll = N.orientnew('N_roll','Axis',[phi_roll,N.y])
...:
...:
...: N_w.set_ang_vel(N,w_w*N.y)
...: N_roll.set_ang_vel(N,w_roll*N.y)
...: N_b.set_ang_vel(N,w_b*N.y)
...:

```

### 5.5.3 Centers of mass of the ball

```

In [3]: # Ball Center of mass
...: CM2 = O.locatenew('CM2',(R_w+R_b)*N_b.z)
...: CM2.v2pt_theory(O,N,N_b)
...:
Out[3]: (R_b + R_w)*phi_b'*N_b.x

```

### 5.5.4 Masses and inertias

```

In [4]: # Inertia
...: Iy = outer(N.y,N.y)
...: In1T = (J_w*Iy, O) # Wheel
...: In2T = (J_b*Iy, CM2) # Ball
...:
...: # Bodies
...: B_w = RigidBody('B_w', O, N_w, M_w, In1T)
...: B_r = RigidBody('B_r', CM2, N_roll, M_b, In2T)
...:
...: B_r.potential_energy = (R_w+R_b)*M_b*g*sin(phi_b)
...: B_w.potential_energy = 0
...:

```

### 5.5.5 Forces and torques

```
In [5]: forces = [(N_roll, 0*N.y) , (N_w, T*N.y) ]
```

### 5.5.6 Lagrange's model and linearized state-space matrices

```
In [6]: # Lagrange operator
...: L = Lagrangian(N, B_r, B_w)
...:
...: # Lagrange model
...: LM = LagrangesMethod(L, [phi_b, phi_w], forcelist
...:   = forces, frame = N)
...: LM.form_lagranges_equations()
...:
...: # Equilibrium point
...: eq_pt = [pi/2, 0, 0, 0]
...: eq_dict = dict(zip([phi_b, phi_w, w_b, w_w], eq_pt
...:   ))
...:
...: MM, linear_state_matrix, linear_input_matrix,
...:   inputs = LM.linearize(q_ind=[phi_b, phi_w], qd_ind
...:   = [w_b, w_w])
...:
...: f_p_lin = linear_state_matrix.subs(eq_dict)
...: f_B_lin = linear_input_matrix.subs(eq_dict)
...:
...: MM = MM.subs(eq_dict)
...:
...: Atmp = MM.inv() * f_p_lin
...: Btmp = MM.inv() * f_B_lin
...:
```

```

In [7]: Atmp
Out [7]:
Matrix([
[
0, 0, 1, 0],
[
0, 0, 0, 1],
[M_b*g*(R_b + R_w)*(J_b*R_w**2/R_b**2 + J_w)/(-J_b**2*R_w
**4/R_b**4 + (J_b*R_w**2/R_b**2 + J_w)*(J_b*R_w**2/R_b
**2 + M_b*(R_b + R_w)**2)), 0, 0, 0],
[J_b*M_b*R_w**2*g*(R_b + R_w)/(R_b**2*(-J_b**2*R_w
**4/R_b**4 + (J_b*R_w**2/R_b**2 + J_w)*(J_b*R_w**2/R_b
**2 + M_b*(R_b + R_w)**2))), 0, 0, 0]])

In [8]: Btmp
Out [8]:
Matrix([
[
0],
[
0],
[J_b*R_w**2/(R_b**2*(-J_b**2*R_w**4/
R_b**4 + (J_b*R_w**2/R_b**2 + J_w)*(J_b*R_w**2/R_b**2
+ M_b*(R_b + R_w)**2))),
[(J_b*R_w**2/R_b**2 + M_b*(R_b + R_w)**2)/(-J_b**2*R_w**4/
R_b**4 + (J_b*R_w**2/R_b**2 + J_w)*(J_b*R_w**2/R_b**2
+ M_b*(R_b + R_w)**2))]])

```

or as formula

$$A = \begin{bmatrix} 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \\ \frac{J_2 M_2 R_1^2 g}{J_1 J_2 R_1 + J_1 J_2 R_2 + J_1 M_2 R_1 R_2^2 + J_1 M_2 R_2^3 + J_2 M_2 R_1^3 + J_2 M_2 R_1^2 R_2} & \frac{J_2 M_2 R_1 R_2 g}{J_1 J_2 R_1 + J_1 J_2 R_2 + J_1 M_2 R_1 R_2^2 + J_1 M_2 R_2^3 + J_2 M_2 R_1^3 + J_2 M_2 R_1^2 R_2} & 0 & 0 \\ \frac{J_1 M_2 R_1 R_2 g}{(R_1 + R_2)(J_1 J_2 + J_1 M_2 R_2^2 + J_2 M_2 R_1^2)} & \frac{J_1 M_2 R_2^2 g}{(R_1 + R_2)(J_1 J_2 + J_1 M_2 R_2^2 + J_2 M_2 R_1^2)} & 0 & 0 \end{bmatrix}$$

and

$$B = \begin{bmatrix} 0 \\ 0 \\ \frac{M_2^2 R_1^2 R_2^2}{(J_1 + M_2 R_1^2)(J_1 J_2 + J_1 M_2 R_2^2 + J_2 M_2 R_1^2)} + \frac{1}{J_1 + M_2 R_1^2} \\ -\frac{M_2 R_1 R_2}{J_1 J_2 + J_1 M_2 R_2^2 + J_2 M_2 R_1^2} \end{bmatrix}$$

## 5.6 Model of the Ball-on-Wheel plant - Kane

In this system we have 4 reference frames. The frame **N** is the main reference frame, **N0** rotates with the line connecting the centers of mass of the wheel (**O**) and of the ball (**CM2**), **N1** ( $x_1$ ,  $y_1$ ) rotates with the wheel and **N2** ( $x_2$ ,  $y_2$ ) is body-fixed to the ball.

The radius of the wheel and of the ball are respectively  $R_1$  and  $R_2$ . The non sliding condition is given by

$$R_1 \cdot ph_0 = R_1 \cdot ph_1 + R_2 \cdot ph_2$$

The input of the system is represented by the torque  $T$  applied to the wheel.

### 5.6.1 Modules and constants

```
In [1]: from sympy import symbols, Matrix, pi
...: from sympy.physics.mechanics import *
...: import numpy as np
...:
...: ph0, ph1, ph2 = dynamicsymbols('ph0_ph1_ph2')
...: w1, w2 = dynamicsymbols('w1_w2')
...:
...: T = dynamicsymbols('T')
...:
...: J1, J2 = symbols('J1_J2')
...: M1, M2 = symbols('M1_M2')
...: R1, R2 = symbols('R1_R2')
...: d1 = symbols('d1')
...: g = symbols('g')
...: t = symbols('t')
...:
```

### 5.6.2 Reference frames

```
In [2]: N = ReferenceFrame('N')
...:
...: O = Point('O')
...: O.set_vel(N, 0)
...:
...: ph0 = (R1*ph1+R2*ph2)/R1
...:
...: N0 = N.orientnew('N0', 'Axis', [ph0, N.z])
...: N1 = N.orientnew('N1', 'Axis', [ph1, N.z])
...: N2 = N.orientnew('N2', 'Axis', [ph2, N.z])
...: N1.set_ang_vel(N, w1*N.z)
...: N2.set_ang_vel(N, w2*N.z)
...:
```

### 5.6.3 Centers of mass of the ball

```
In [3]: CM2 = O.locatenew('CM2', (R1+R2)*N0.y)
...: CM2.v2pt_theory(O, N, N0)
...:
Out [3]: (-R1*ph1 - R2*ph2)*N0.x
```

### 5.6.4 Masses and inertias

```
In [4]: Iz = outer(N.z,N.z)
...: In1T = (J1*Iz, O)
...: In2T = (J2*Iz, CM2)
...:
...: B1 = RigidBody('B1', O, N1, M1, In1T)
...: B2 = RigidBody('B2', CM2, N2, M2, In2T)
...:
```

### 5.6.5 Forces and torques

```
In [5]: #forces = [(N1, (T-d1*w1)*N.z), (CM2,-M2*g*N.y)]
...: forces = [(N1, T*N.z), (CM2,-M2*g*N.y)]
...:
...: kindiffs = [ph1.diff(t)-w1, ph2.diff(t)-w2]
...:
```

### 5.6.6 Kane's model and linearized state-space matrices

```
In [6]: KM = KanesMethod(N, q_ind=[ph1, ph2], u_ind=[w1, w2],
...: kd_eqs=kindiffs)
...: fr, frstar = KM.kanes_equations(forces, [B1, B2])
...:

In [7]: # Equilibrium point
...: eq_pt = [0, 0, 0, 0, 0]
...: eq_dict = dict(zip([ph1, ph2, w1, w2, T], eq_pt))
...:

In [8]: # symbolically linearize about arbitrary equilibrium
...: MM, linear_state_matrix, linear_input_matrix,
...: inputs =
KM.linearize(new_method=True)
...:
...: # sub in the equilibrium point and the parameters
...: f_A_lin = linear_state_matrix.subs(eq_dict)
...: f_B_lin = linear_input_matrix.subs(eq_dict)
...: MM = MM.subs(eq_dict)
...:
...: # compute A and B
...: A = MM.inv() * f_A_lin
...: B = MM.inv() * f_B_lin
```

```

In [9]: A
Out [9]:
Matrix([
[0, 0, 1, 0],
[0, 0, 0, 1],
[-M2**2*R1**2*R2**2*g/((R1 + R2)*(J1*J2 + J1*M2*R2**2 + J2
*M2*R1**2)) +
M2*R1**2*g*(M2**2*R1**2*R2**2/((J1 + M2*R1**2)*(J1*J2 + J1
*M2*R2**2 +
J2*M2*R1**2)) + 1/(J1 + M2*R1**2))/(R1 + R2), -M2**2*R1*R2
**3*g/((R1 +
R2)*(J1*J2 + J1*M2*R2**2 + J2*M2*R1**2)) + M2*R1*R2*g*(M2
**2*R1**2*R2**2/((J1 +
M2*R1**2)*(J1*J2 + J1*M2*R2**2 + J2*M2*R1**2)) + 1/(J1 +
M2*R1**2))/(R1 + R2),
0, 0],
[
-M2**2*R1**3*
R2*g/((R1 + R2)*(J1*J2
+ J1*M2*R2**2 + J2*M2*R1**2)) + M2*R1*R2*g*(J1 + M2*R1**2)
/((R1 + R2)*(J1*J2 +
J1*M2*R2**2 + J2*M2*R1**2)),
-M2**2*R1**2*R2**2*g/((R1 + R2)*(J1*J2 + J1*M2*R2**2 + J2*
M2*R1**2)) +
M2*R2**2*g*(J1 + M2*R1**2)/((R1 + R2)*(J1*J2 + J1*M2*R2**2
+ J2*M2*R1**2)), 0,
0]])
In [10]: B
Out [10]:
Matrix([
[
0],
[
0],
[M2**2*R1**2*R2**2/((J1 + M2*R1**2)*(J1*J2 + J1*M2*R2**2 +
J2*M2*R1**2)) +
1/(J1 + M2*R1**2)],
[
-M2*R1*R2/(
J1*J2 + J1*M2*R2**2 +
J2*M2*R1**2)]]])

```





# Chapter 6

## Control design

### 6.1 PI+Lead design example

#### 6.1.1 Define the system and the project specifications

In this first example we design a controller for a plant with the transfer function

$$G(s) = \frac{1}{s^2 + 6 \cdot s + 5}$$

The requirements for the control are

$$e_{\infty} = 0$$

for a step input

$$PM \geq 60^\circ$$

and

$$\omega_{gc} = 10 \text{rad/s}$$

The controller can be written in the form

$$C(s) = K \cdot \frac{1 + s \cdot T_i}{s \cdot T_i} \cdot \frac{1 + \alpha \cdot T_D \cdot s}{1 + s \cdot T_D}$$

with a PI and a lead part.

We have to design the controller and find the values of  $\mathbf{T_i}$ ,  $\alpha$ ,  $\mathbf{T_D}$  and  $\mathbf{K}$ . The full design is performed using the bode diagram.

After installing the required modules, we can define the plant transfer function and the requirements of the project.

```

In [1]: # Modules
In [2]: from matplotlib.pyplot import *
In [3]: from control import *
In [4]: from numpy import pi, linspace
In [5]: from scipy import sin, sqrt
In [6]: from supsisim.RCPblk import *
In [7]: from supsictrl.ctrl_utils import *
In [8]: from supsictrl.ctrl_repl import *
In [9]: g=tf([1],[1,6,5])
In [10]: bode(g,dB=True);
In [11]: legend(['G(s)'],prop={'size':10})
Out[11]:
(<matplotlib.axes.AxesSubplot at 0x7f85b5193550>,
 <matplotlib.legend.Legend at 0x7f85b47e6950>)
In [12]: wgc = 10          # Desired Bandwidth
In [13]: desiredPM = 60    # Desired Phase margin

```

Figure 6.1 shows the bode diagram of the plant.

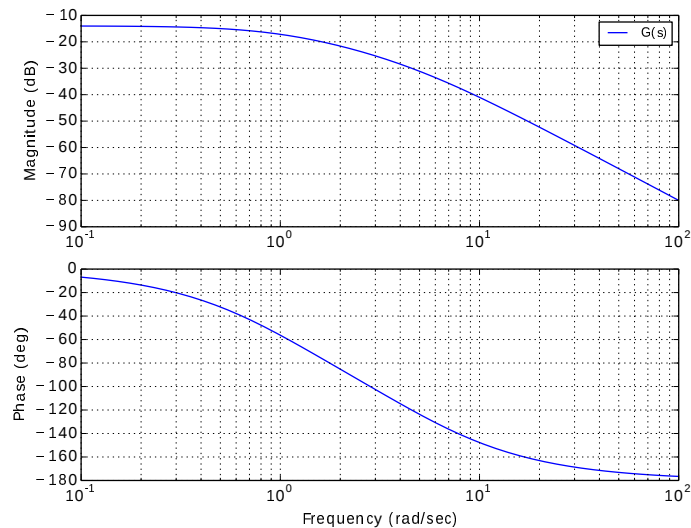


Figure 6.1: Bode diagram of the plant

### 6.1.2 PI part

Now we choose the integration time for the PI part of the controller. In this example we set

$$T_i = 0.15s$$

```

In [14]: # PI part
In [15]: Ti=0.15
In [16]: Gpi=tf([Ti,1],[Ti,0])
In [17]: print "PI part is:", Gpi
PI part is:
0.15 s + 1
-----
0.15 s

In [18]: figure()
Out[18]: <matplotlib.figure.Figure at 0x7f85b47eaa10>
In [19]: bode(g,dB=True,linestyle='dashed');
In [20]: bode(Gpi*g,dB=True);
In [21]: legend((['G(s)', 'Gpi(s)*G(s)']),prop={'size':10})
Out[21]:
(<matplotlib.axes.AxesSubplot at 0x7f85b4806250>,
 <matplotlib.legend.Legend at 0x7f85b4303850>)

```

Figure 6.2 shows the bode plot of the plant with and without the PI controller part.

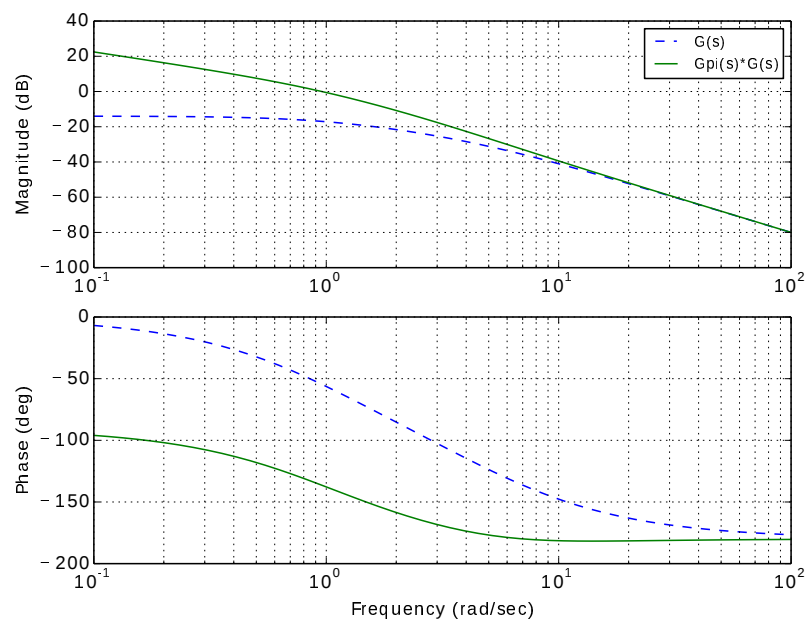


Figure 6.2: Bode diagram:  $G$  (dashed) and  $G_{pi}*G$

### 6.1.3 Lead part

Now we can get the  $PM$  at the frequency  $\omega_{gc}$  in order to calculate the additional phase contribution of the lead part of the controller.

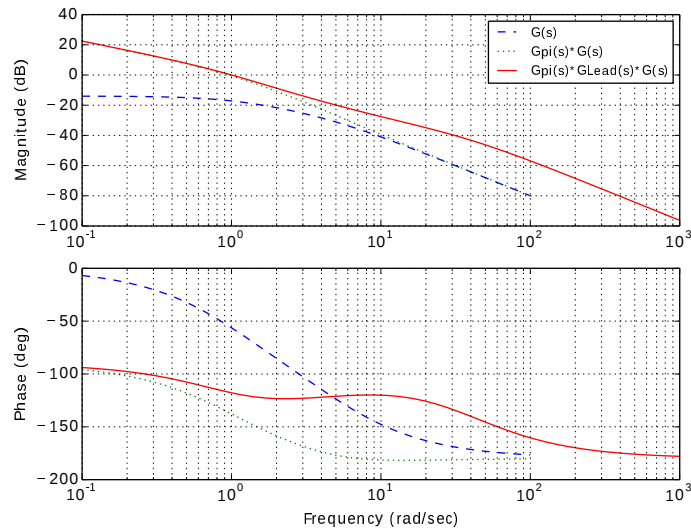
```
In [22]: mag, phase, omega = bode(Gpi*g, [wgc], Plot=False)
In [23]: ph = phase[0]
In [24]: if ph>=0:
...:     ph = phase[0]-360;
...:
In [2]: Phase = -180+desiredPM
In [26]: dPM = Phase-ph
In [27]: print "Additional phase from Lead part:", dPM
Additional phase from Lead part: 61.4144232114
```

Now it is possible to calculate the lead controller by finding the values of  $\alpha$  and  $T_D$ .

```
In [28]: # Lead part
In [29]: dPMrad = dPM/180*pi
In [30]: alfa = (1+sin(dPMrad))/(1-sin(dPMrad));
In [31]: print "Alpha is:", alfa
Alpha is: 15.4073552425
In [32]: TD = 1/(sqrt(alfa)*wgc);
In [33]: Glead = tf([alfa*TD,1],[TD,1])
In [34]: print "Lead part is:", Glead
Lead part is:
  0.3925 s + 1
-----
 0.02548 s + 1

In [35]: figure()
Out[35]: <matplotlib.figure.Figure at 0x7f85b43462d0>
In [36]: bode(g, dB=True, linestyle='dashed');
In [37]: bode(Gpi*Glead*g, dB=True);
In [38]:
legend((['G(s)', 'Gpi(s)*G(s)', 'Gpi(s)*GLead(s)*G(s)']),
       prop={'size':10})
Out[38]:
(<matplotlib.axes.AxesSubplot at 0x7f85b43736d0>,
 <matplotlib.legend.Legend at 0x7f85b3b1f450>)
```

Figure 6.3 shows now the bode plot of the plant, the plant with the PI part and the plant with PI and Lead part

Figure 6.3: Bode diagram -  $G$  (dashed),  $G_{pi} \cdot G$  (dotted) and  $G_{pi} \cdot G_{Lead} \cdot G$ 

### 6.1.4 Controller Gain

The last step is to find the amplification  $K$  of the controller which move up the bode gain plot in order to obtain the required crossover frequency  $\omega_{gc}$ .

```
In [39]: mag, phase, omega = bode(Gpi*Glead*g, [wgc], Plot=False)

In [40]: print "Phase at wgc is: ", phase[0]
Phase at wgc is: -120.0

In [41]: K=1/mag[0]

In [42]: print "Gain to have MAG at gwc 0dB: ", K
Gain to have MAG at gwc 0dB: 23.8177769548

In [43]: figure()
Out[43]: <matplotlib.figure.Figure at 0x7f85b3a703d0>

In [44]: bode(g, dB=True, linestyle='dashed');

In [45]: bode(Gpi*Glead*g, dB=True, linestyle='-.');

In [46]: bode(K*Gpi*Glead*g, dB=True);

In [47]:
legend((['G(s)', 'Gpi(s)*G(s)', 'Gpi(s)*GLead(s)*G(s)',
'K*Gpi(s)*GLead(s)*G(s)'], prop={'size':10})
Out[47]:
(<matplotlib.axes.AxesSubplot at 0x7f85b3a76690>,
<matplotlib.legend.Legend at 0x7f85b33e6f90>)
```

In the figure 6.4 we see now that the gain plot has been translated up to get  $0dB$  at the gain crossover frequency  $\omega_{gc}$ .

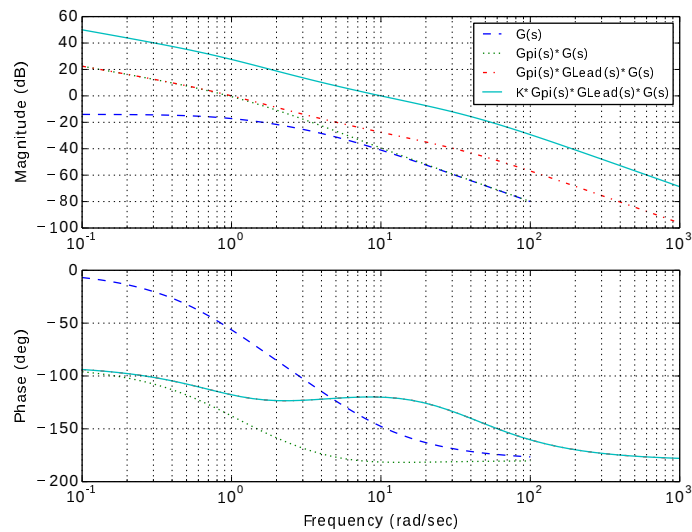


Figure 6.4: Bode diagram -  $G$  (dashed),  $G_{pi} \cdot G$  (dotted),  $G_{pi} \cdot G_{Lead} \cdot G$  (dot-dashed) and  $K \cdot G_{pi} \cdot G_{Lead} \cdot G$

### 6.1.5 Simulation of the controlled system

Now it is possible to simulate the controlled system after closing the loop.

```
In [48]: Contr = K*Gpi*Glead

In [49]: print "Full controller:", Contr
Full controller:
1.402 s^2 + 12.92 s + 23.82
-----
0.003821 s^2 + 0.15 s

In [50]: mag, phase, omega = bode(K*Gpi*Glead*g, [wgc], Plot=False)

In [51]: print "Data at wgc:", omega[0], "Magnitude:", mag[0], "Phase:"
", phase[0]
Data at wgc - wgc: 10 Magnitude: 1.0 Phase: -120.0

In [52]: gt=feedback(K*Gpi*Glead*g,1)

In [53]: t=linspace(0,1.5,300)

In [54]: y,t = step(gt,t)

In [55]: figure()
Out[55]: <matplotlib.figure.Figure at 0x7f85b3514290>

In [56]: plot(t,y), xlabel('t'), ylabel('y'), title('Step-
response of the
controlled plant')
Out[56]:
([<matplotlib.lines.Line2D at 0x7f85b34252d0>],

In [57]: grid()
```

The simulation of the controlled plant with a step input is shown in figure 6.5.

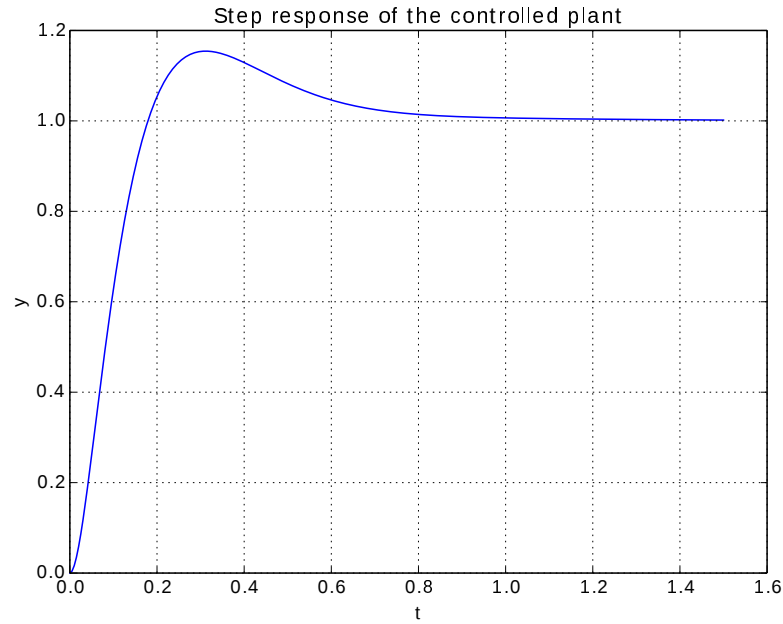


Figure 6.5: Step response of the controlled plant

## 6.2 Discrete-state feedback controller design

### 6.2.1 Plant and project specifications

In this example we design a discrete-state feedback controller for a DC servo motor.

We want to have a controlled system with a maximum of 4% overshooting and an error  $e_\infty = 0$  with a step input. In addition we desire a bandwidth of the controlled system of at least 6 rad/s.

The step response of the motor with the current input of  $I_{in} = 500mA$ ) has been saved into the file "MOT".

### 6.2.2 Motor parameters identification

We try to find the parameters of the srvo motor using a least square identification from the collected data.

The transfer function of the DC motor from input current  $I(s)$  to output angle  $\Phi(s)$  can be represented as

$$G(s) = \frac{\Phi(s)}{I_{in}(s)} = \frac{K_t/J}{s^2 + s \cdot D/J}$$

### 6.2.3 Required modules

```
In [1]: from scipy.optimize import leastsq
In [2]: from scipy.signal import step2
In [3]: import numpy as np
In [4]: import scipy as sp
In [5]: from control import *
In [6]: from control.Matlab import *
In [7]: from supsisim.RCPblk import *
...: from supsictrl.ctrl_utils import *
...: from supsictrl.ctrl_repl import *
...:
```

### 6.2.4 Function for least square identification

We define now the function **residuals** which returns the error between the collected and the simulated data. Using this function we can try to minimize the error using a least square approach.

```
In [8]: # Motor response for least square identification
In [9]: def residuals(p, y, t):
...:     [k, alpha] = p
...:     g = tf(k,[1,alpha,0])
...:     Y,T = step(g,t)
...:     err=y-Y
...:     return err
...:
```

### 6.2.5 Parameter identification

We load the collected data to perform the parameter identification of the numerator  $K = K_t/J$  and the denominator value  $\alpha = D/J$ .



```

In [10]: # Identify motor
In [11]: x = np.loadtxt('MOT');
In [12]: t = x[:,0]
In [13]: y = x[:,2]
In [14]: Io = 1000
In [15]: y1 = y/Io
In [16]: p0 = [1,4]
In [17]: plsq = leastsq(residuals, p0, args=(y1, t))
In [18]: kt = 0.0000382          # Motor torque constant
In [19]: Jm=kt/plsq[0][0]        # Motor Inertia
In [20]: Dm=plsq[0][1]*Jm        # Motor friction
In [21]: g=tf([kt/Jm],[1,Dm/Jm,0]) # Transfer function

```

### 6.2.6 Check of the identified parameters

The next step is to check how good our parameters have been identified by comparing the simulated function with the measured data (see figure 6.6)

```

In [22]: Y,T = step(g,t)
In [23]: plot(T,Y,t,y1), legend(('Identified_transfer_
        function','Collected
        data'),prop={'size':10},loc=2), xlabel('t'), ylabel('y'),
        title('Step
        response'), grid()
Out[23]:
([<matplotlib.lines.Line2D at 0x7fb9a1b6b590>,
  <matplotlib.lines.Line2D at 0x7fb9a1b6b710>],
 <matplotlib.legend.Legend at 0x7fb9a1b6bb10>,
 <matplotlib.text.Text at 0x7fb9a3cec310>,
 <matplotlib.text.Text at 0x7fb9a1b8b910>,
 <matplotlib.text.Text at 0x7fb9a1b3cbd0>,
 None)

```

### 6.2.7 Continuous and discrete model

For the state controller design we need to model our motor in the state-space form. We define the continuous-state and the discrete-state space model

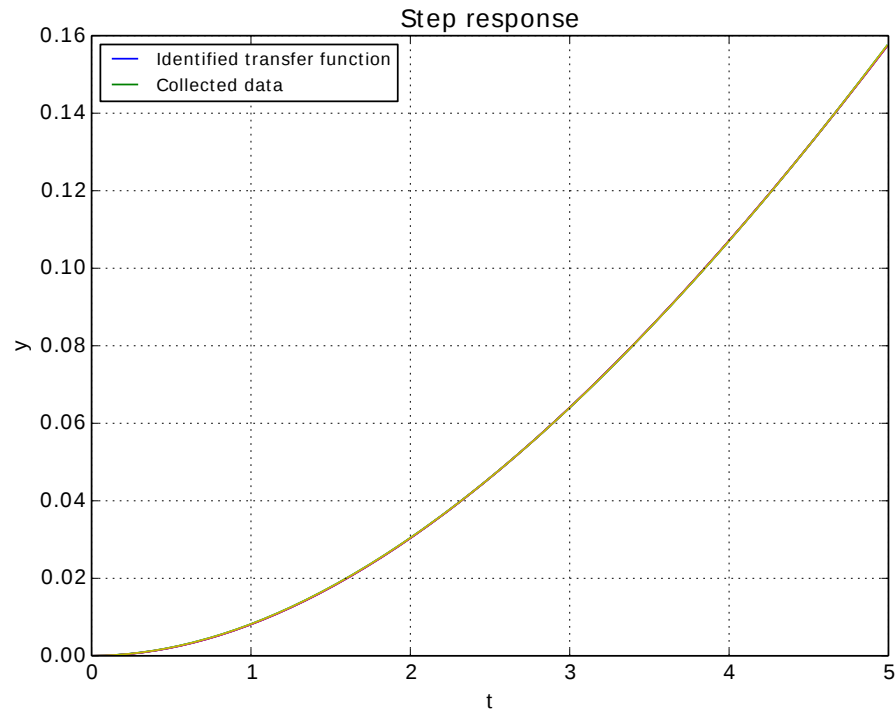


Figure 6.6: Step response and collected data

```

In [24]: # Controller Design Motor 1
In [25]: a=[[0,1],[0,-Dm/Jm]]
In [26]: b=[[0],[1]]
In [27]: c=[[kt/Jm,0]];
In [28]: d=[0];
In [29]: sysc=ss(a,b,c,d)           # Continuous
state-space form
In [30]: Ts=0.01                    # Sampling time
In [31]: sys = c2d(sysc,Ts,'zoh')   # Discrete ss
form

```

### 6.2.8 Controller design

For the controller we set a bandwidth to 6 rad/s with a damping factor of  $\xi = \sqrt{2}/2$ .

```

In [32]: # Control system design

In [33]: print rank(ctrb(sys.A,sys.B))==2    #
          Controllability check
True

In [34]: # State feedback with integral part

In [35]: wn=6

In [36]: xi=np.sqrt(2)/2

In [37]: angle = np.arccos(xi)

```

We add a discrete integral part to eliminate the steady state error and we obtain an additional state for the error between reference and output signal. The two matrices  $\Phi$  and  $\Gamma$  required by the pole placement routine must be extended with the additional state.

```

In [38]: cl_poles = -wn*array([1, exp(1j*angle), exp(-1j*
          angle)]) # three poles

In [39]: cl_polesd=sp.exp(cl_poles*Ts)    # Desired
          discrete poles

In [40]: sz1=sp.shape(sys.A);

In [41]: sz2=sp.shape(sys.B);

In [42]: # Add discrete integrator for steady state zero
          error

In [43]: Phi_f=np.vstack((sys.A,-sys.C*Ts))

In [44]: Phi_f=np.hstack((Phi_f,[[0],[0],[1]]))

In [45]: G_f=np.vstack((sys.B,zeros((1,1))))

In [46]: k=place(Phi_f,G_f,cl_polesd)

```

### 6.2.9 Observer design

Now we can implement the observer: in this example we choose a reduced-order observer and we can use the function provided by the yottalab module to obtain it.

```

In [47]: #Reduced order observer

In [48]: print rank(observ(sys.A,sys.C))==2      #
          Observability check
True

In [49]: p_oc=-10*max(abs(cl_poles))

In [50]: p_od=sp.exp(p_oc*Ts);

In [51]: T=[0,1]

In [52]: r_obs=red_obs(sys,T,[p_od])

```

### 6.2.10 Controller in compact form

The yottalab function **comp\_form\_i** allows to integrate the controller gains and the observer into an unique block.

```

In [53]: # Controller + integral + observer in compact
          form

In [54]: contr_I=comp_form_i(sys,r_obs,k)

```

### 6.2.11 Anti windup

The last operation consists in dividing the controller into an input part and a feedback part in order to realize the anti-windup mechanism and considering the saturation block.

```

In [55]: # Anti windup

In [56]: [gss_in , gss_out]=set_aw(contr_I,[0,0])

```

### 6.2.12 Simulation of the controlled plant

The block diagram of the final controlled system is represented in figure 6.7. It is not possible to simulate the resulting system in Python because of:

- The controller is discrete and the plant is continuous. At present it is not possible to perform hybrid simulation using the control package. In some cases we can substitute the plant with the discrete-time system and perform a discrete simulation. Hybrid simulation is possible using the pysimCoder application described in the next chapter.
- The block “CTRIN” has two inputs. The step function can only find the output from a single input.

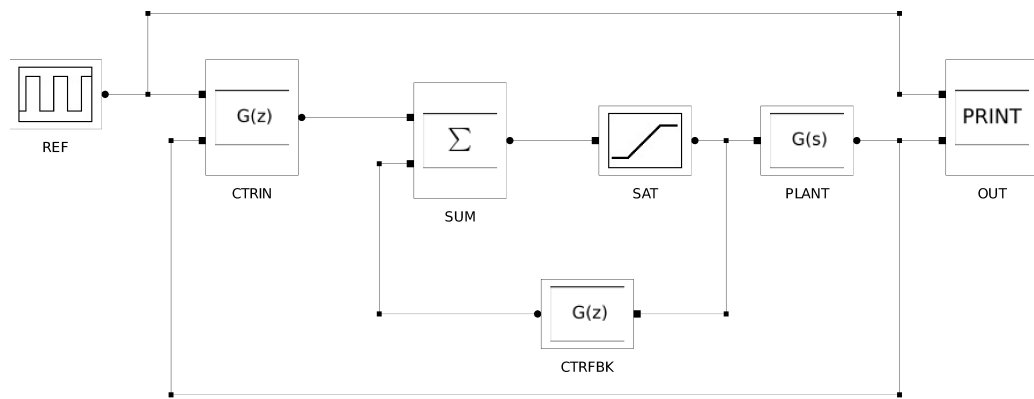


Figure 6.7: Block diagram of the controlled system

- The control toolbox can handle only linear system (and there is a saturation in the final system).



# Chapter 7

## Hybrid simulation and code generation

### 7.1 Basics

CACSD environments usually offer a graphical editor to perform the hybrid simulation (Matlab $\leftrightarrow$ Simulink, Scioslab $\leftrightarrow$ Scicos, Scilab $\leftrightarrow$ xCos etc.).

The “pysimCoder.py” application should cover this task for the Python Control environment. In the following we’ll explain how it is possible, from the pysimCoder schematics, to generate code for the hybrid simulation. Code for the RT controller can be generated in the same way: users should only replace the mathematical model of the plant with the blocks interfacing the sensors and the actuators of the real system.

### 7.2 pysimCoder

#### 7.2.1 The editor

The application “pysimCoder“ is a block diagram editor to design schematics for simulation and code generation.

Starting points for the pysimCoder application were the PySimEd project ([20]) and the qtnodes-develop project ([21]).

PyEdit offers the most used blocks in control design. A little set of these blocks is shown in figure 7.1.

#### 7.2.2 The first example

Using the editor we want to create the block diagram of figure 7.2.

We open a shell and we give the command

```
pysimCoder
```

The application opens 2 windows as shown in figure 7.3

The window on the left shows the library with the available blocks and on the right we have the diagram window. Now we can start to draw our block diagram.

From the library window we can choose the tab “input“ and using “drag and drop“ we can get the block “Step“ and move it into the editor window. We can do the same operation with the “LTI continuous“ (from tab “linear“) and the “Plot“ (from tab “output“) blocks.

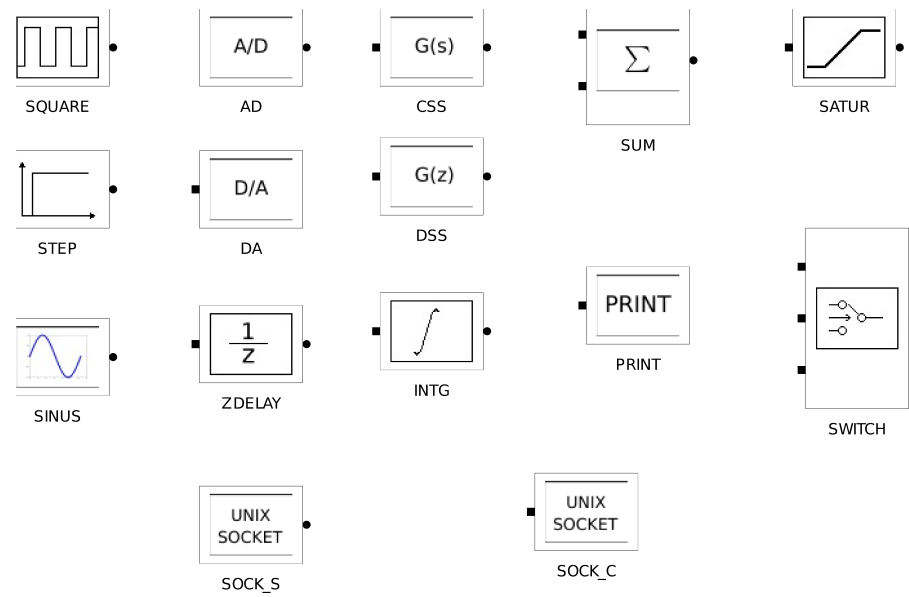


Figure 7.1: Some pysimCoder blocks for control design

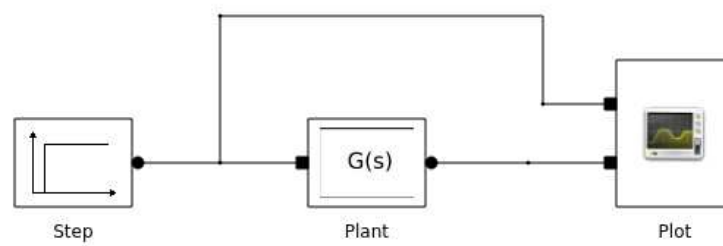


Figure 7.2: The first example



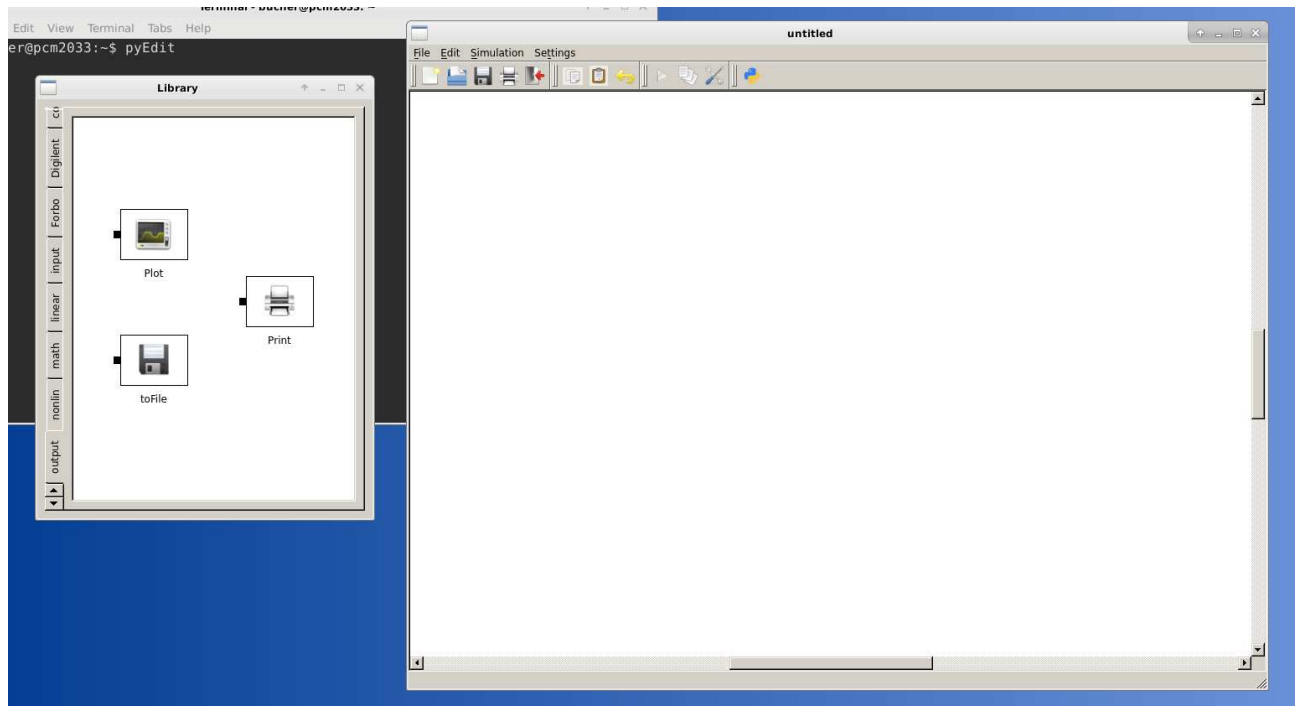


Figure 7.3: The pysimCoder environment

Now we should obtain the diagram shown in figure 7.4

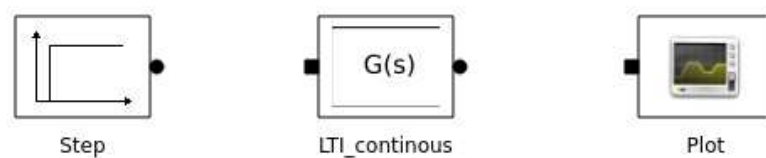


Figure 7.4: Result from the drag and drop operations

Before starting with the connection, we set some parameters to the blocks.

- Double click with the mouse on the block "LTI continuous". In the dialog windows set the System to  $\mathbf{tf(1,[1,1])}$
- Click the right mouse on the LTI continuous block. In the new menu choose "Change Name" and rename it as **Plant**.
- Click the right mouse on the Plot block. In the new menu choose "Block I/Os" and set the number of inputs to **2**.

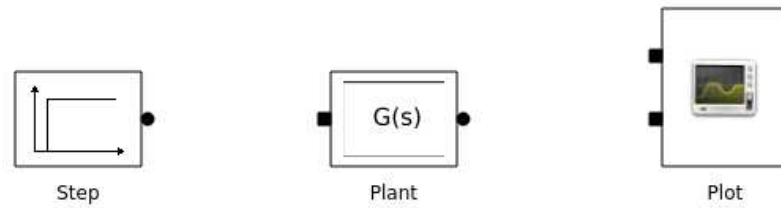


Figure 7.5: Result after parametrization

Figure 7.5 shows the new diagram.

Now we can proceed with the connections.

- Move the mouse on the output of the block “Step”: the mouse pointer should become a “cross”. Click and release the left mouse button.
- Now we can move the mouse to the input of the block “Plant”: the mouse pointer should become a “cross”. Click and release the left mouse button.
- Do the same operation from the output of the block “Plant” to the second input of the block “Plot”
- Now move to the node (the little circle) between the “Step” and the block “Plant”: the mouse pointer should become a “cross”. Click and release the left mouse button.
- move the mouse up, click, and continue to move left the mouse. Left of the position of the block “Plot”, click and release again the left mouse button and then finish the connection on the first input of the block “Plot” (click and release the left mouse button)

You should obtain the diagram of figure 7.2

Now we are able to simulate the diagram.

- From the menu “Simulation” choose “Simulate” or click on the button “Simulate” on the toolbar (the button with the triangle).
- Double click with the mouse on the block “Plot” to get the graphical output of the simulation (see figure 7.6).

### 7.2.3 Some remarks

- the simulation result (Plot) **is available only after the simulation**. Please be sure to restart the simulation before opening the plot result. The simulation creates a file with the name of the block in “/tmp” folder: this file is overwritten by every new simulation.
- For the simulation, the application creates and compile a C-executable. The sources are written in the folder “xxxxxx\_gen”, where “xxxxxx” is the name of the diagram.

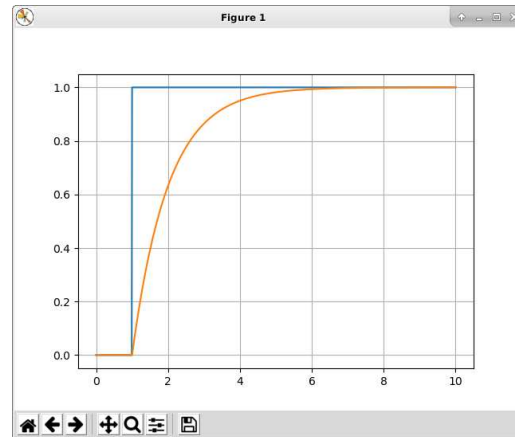


Figure 7.6: Result (plot) of the simulation

### 7.2.4 Defining new blocks

The user can define new blocks and integrate them into the pysimCoder application. Two applications help the user to define a new block.

- defBlocks
- xblk2Blk

The first application (defBlocks) is used to generate the “.xblk” file, with the default values of the block, by simply filling the different fields and adding the parameters on the bottom (see figure 7.7).

The parameters in the window represent:

**Library** is the name of the “tab” window in the pysimCoder library

**Name** is the name of the block which appears under the block in the editor

**Icon** is the name of the icon file (located under “resources/blocks/Icons” without the extension (“.svg”))

**Function** is the name of the “.py” block which translates the block into the RCPBlk class objects (see code generation)

**Inputs** : number of the input ports

**Outputs** : number of the output ports

**input settable** is a flag which indicates if the number of input ports can be changed or not

**output settable** is a flag which indicates if the number of output ports can be changed or not

**Bottom window** is a grid which contains the parameters of the block (Label+default value)

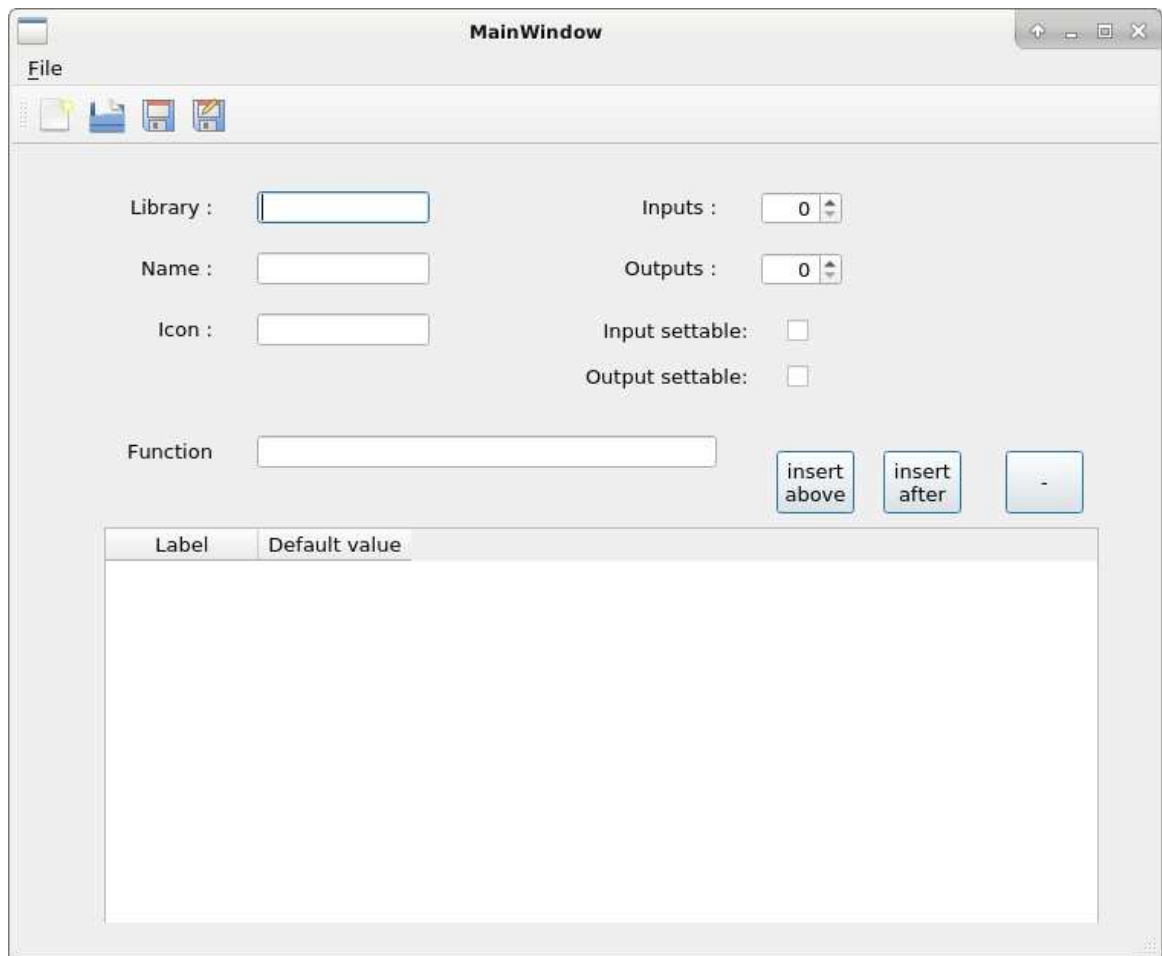


Figure 7.7: The “defBlocks” application

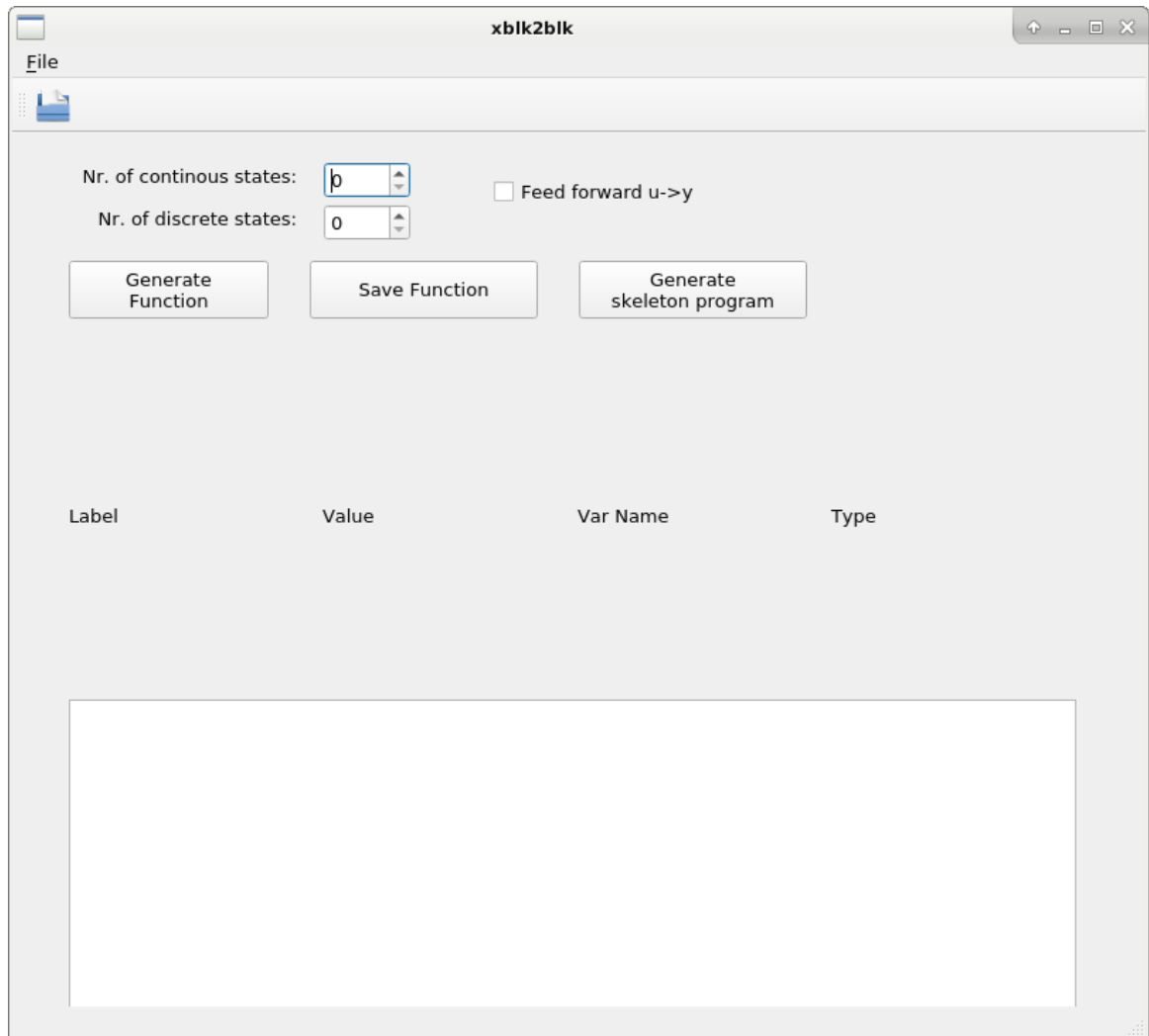


Figure 7.8: The “xblk2Blk” application

The “Save” or “Save as” operation generates the “.xblk” file. This file must be placed under “resources/blocks/blocks”

The second step is to call the application “xblk2Blk” (see figure 7.8).

After opening the “.xblk” file, it is possible to set a name and a type of each parameters of the block.

These informations are used to generate the “.py” which can be modified and saved, and the “\*.c\*” skeleton, which should be modified for the specific block tasks.

The “.py” file must be moved in the folder “resources/blocks/rcpBlk”, the “\*.c\*” file must be edited and stored under “CodeGen/devices”.

### 7.2.5 Personalize the most used blocks

A special file “common.blks” is defined in the “resources/block/block” folder. This file contains a list of blocks, that the user can modify to have its own most used blocks. The blocks are shown in the library both in the “common” tab and in the specific tab.

## 7.3 The editor window

### 7.3.1 The toolbar

The application offers set of operations in the toolbar as shown in the figure 7.9.

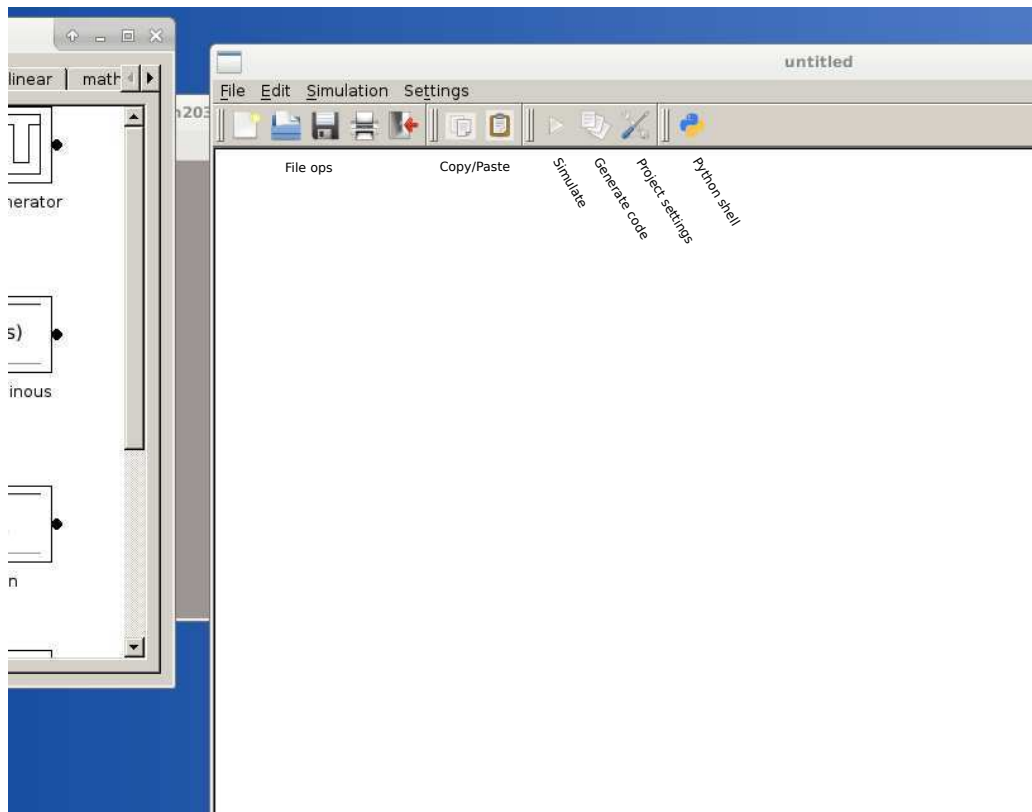


Figure 7.9: The pysimCoder application

### 7.3.2 Operations with the right mouse button

Depending on the position of the mouse, clicking and releasing the right mouse button leads to different behaviours.

### 7.3.3 Operations with the right mouse button on a block

Clicking with the right mouse button on a block opens a popup menu with the following commands:

**Block I/Os** to modify (if possible) the number of input and output ports of the block

**Flip block** Flip left/right the block

**Change name** Each block in the diagram must have a **unique name**

**Block parameters** to modify the parameters: this operation is available with a double click tool

**Clone block** to get a copy of the selected block

### 7.3.4 Operations with the right mouse button on a connection

Moving the mouse on a connection, change the pointer to a pointing hand and by clicking with the right mouse button a popup menu is opened with the following commands:

**Start connection** Insert a node and start a new connection

**Delete connection** deletes the pointed connection

**Insert node** inserts a new node on the connection. This is needed for examples if we have to draw a new connection.

### 7.3.5 Operations with the right mouse button on a node

Moving the mouse on a node, change the pointer to a cross and by clicking with the right mouse button a popup menu is opened with the following commands:

**Delete node** deletes the node and the connections associated with this node.

**Bind node** eliminate the node without eliminate the connection.

### 7.3.6 Behaviour of the right mouse button by drawing a connection

Clicking the right mouse button by drawing a connection, put a new node in the mouse position and exit the drawing mode.

## 7.4 Basic editor operations

### 7.4.1 Inserting a block

Get a block from a library and drag it into the main window.

### 7.4.2 Connecting blocks

- Move to the output port of a block or to a node.
- Click and release the left button of the mouse.
- Move the mouse to draw the connection.
- Click again the left mouse on an input port of a block to finish the connection or click the mouse to obtain a "node" and to continue to draw the connection.

### 7.4.3 Inserting a node

- Move to a connection and click the right mouse button
- Select the "insert node" menu.

If a new "node" is needed into a connection simply click on it with the right mouse button.

### 7.4.4 Deleting a block or a node

- Move to a block or node and click with the right mouse button.
- Choose the submenu "delete"

## 7.5 Remove a node

- Move to the node.
- Click with the right mouse button on the node.
- Choose the submenu "Bind node" The connection is maintained but the node is cleared.



# Chapter 8

## Simulation and Code generation

Each element of a block diagram is defined with three or four functions:

**The interface function** that describes how the block must be drawn in the block diagram

**The Implementation function** that contains the code to be executed to perform the tasks related with this block.

The translation of the block into the RCPblk class described in the RCPblk.py module

A dlg function to implement a special dialog box for the block parameters (only if required)

In addition we need to know all the nodes connected to the inputs and to the outputs of each block.

### 8.1 Interface functions

Each block is defined into a file with extension “.xblk”, stored in the “resources/blocks/blocks” folder. The file is defined as a Python dictionary:

```
{"lib": "math", "name": "Sum", "ip": 2, "op": 1, "stin": 1, "stout": 0, "icon": "SUM", "params": "sumBlk|Gains: [1,-1]"}
```

using the following fields:

“**lib**” the name of the tab for the block library (example “tab”:“linear”)

“**name**” the default name of the block

“**ip**” number of inputs

“**op**” number of outputs

“**stin**” flag which indicates if the number of inputs can be modified

“**stout**” flag which indicates if the number of outputs can be modified

“**icon**” the name of the “.svg” file with the icon of the block

“**param**” the parameters of the block

The first string in the param field is used as name of the Python function used to prepare the block to be translated into C-Code.

The block libraries are loaded after launching the pysimCoder application as shown in figure 8.1

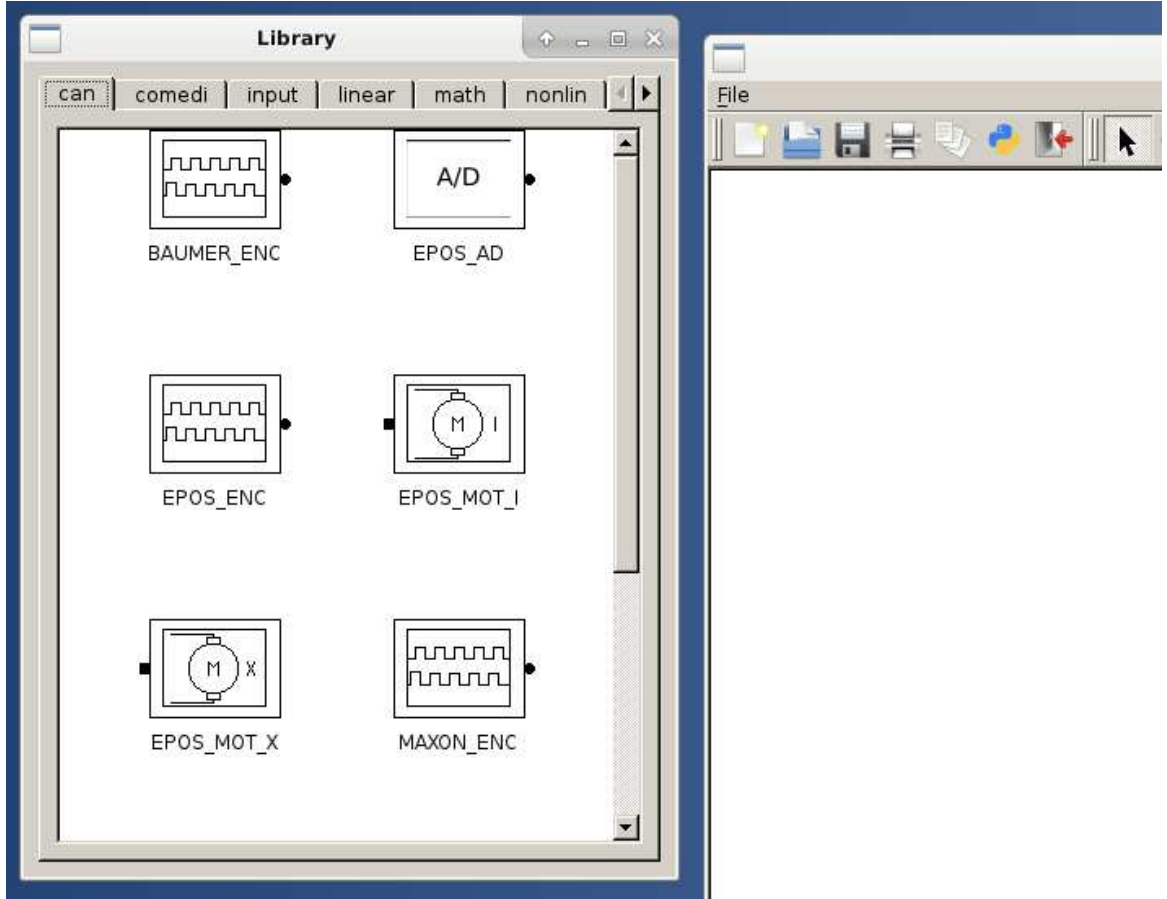


Figure 8.1: Window with the block libraries

Each block must be renamed with a unique name (popup menu “Change name”), and its parameters can be modified directly in the pysimCoder application with a double click.

## 8.2 The implementation functions

In a schematic, each block can be described with the functions (8.1) for continuous-time systems or (8.2) for discrete-time systems.

$$\begin{aligned} \mathbf{y} &= \mathbf{g}(\mathbf{x}, \mathbf{u}, t) \\ \dot{\mathbf{x}} &= \mathbf{f}(\mathbf{x}, \mathbf{u}, t) \end{aligned} \quad (8.1)$$

$$\begin{aligned} \mathbf{y}_k &= \mathbf{g}(\mathbf{x}_k, \mathbf{u}_k, k) \\ \mathbf{x}_{k+1} &= \mathbf{f}(\mathbf{x}_k, \mathbf{u}_k, k) \end{aligned} \quad (8.2)$$

The **g(...)** function represents the static part of the block. This function is used to read inputs, read sensors, write actuators or update the outputs of the block.

The second function (**f(...)**) is only required if the block has internal states, and it is only used by dynamic systems. In addition, each block implements two other functions, one for the block initialization and one to cleanly terminate it.

All these functions are programmed as C-files, compiled and archived into a library.

## 8.3 Translating the block into the RCPblk class

Before generating the C-Code, each block in the diagram must be translated into an element of the RCPblk class (see section 8.7 for more details). For each block, the corresponding function (the name is given by the 1. string in the parameters line) must exist and should be declared with the required parameters. This function is responsible to fill all the RCPblk fields.

## 8.4 Special dialog box for the block parameters

Usually, the graphic editor build a simple dialog box to enter the block parameters.

In special cases, it is possible to write a special function to enter the parameters.. In this case, the user should provide this function in the RCPDlg.py file. The name of this function is built using the first string of the parameter line, by substituting the last 3 letters “Blk” with “Dlg”. This new function must receive as input:

- Number of inputs
- Number of outputs
- The parameters line

This function returns a modified parameters line. An example is the “PlotDlg” function in the file “toolbox/supsim/src/RCPGDlg.py”.

## 8.5 Example

We can show with an example what happens with a block in the different phases from block to RCPblk class.

The “Pulse generator” input block is stored in the “PulseGenerator.xblk” file with the following infos

```
{ "lib": "input", "name": "PulseGenerator", "ip": 0, "op": 1, "stin": 0, "stout": 0, "icon":
"square", "params": "squareBlk|Amplitude: 1|Period: 4|Width: 2|Bias: 0|Delay:
0" }
```

The block has no inputs, 1 output, the I/O are not modifiable (settable=0).

After a double click on the block, the “params” field is parsed and translated into the dialog box shown in figure 8.2.

By generating the element of the class RCPblk, the function “squareBlk” is called with the following parameters:

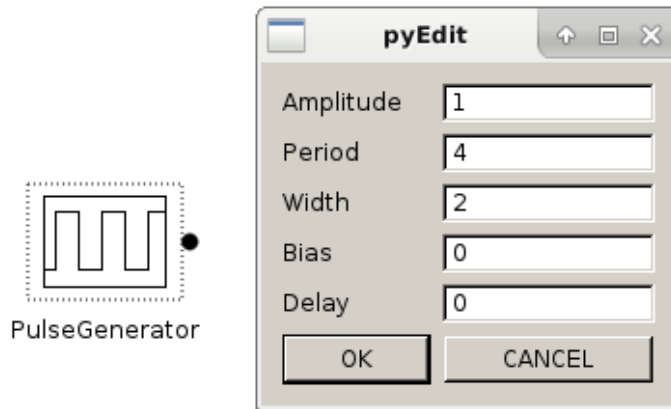


Figure 8.2: Dialog box for the Pulse generator block

```
SQUARE = squareBlk(pout, Amp, Period, Width, Bias, Delay)
```

where

**pout** is the matrix with the id of the inputs (connections)

**Amp** is the signal amplitude

**Period** is the period of the signal

**width** is the duration where the signal has value “Amp-bias”

**bias** is an offset for the signal

**delay** represent the time wenn the signal start

The function translate the block into the following object of the RCPblk class

```
Function          : square
Input ports       : []
Output ports      : [2]
Nr. of states     : [0 0]
Relation u->y     : 0
Real parameters   : [ 4  8  3  0 12]
Integer parameters : []
```

## 8.6 The parameters for the code generation

Before clicking on the “code generation” tool on the toolbar, the user should fill some parameters in a dialog box (see figure 8.3).

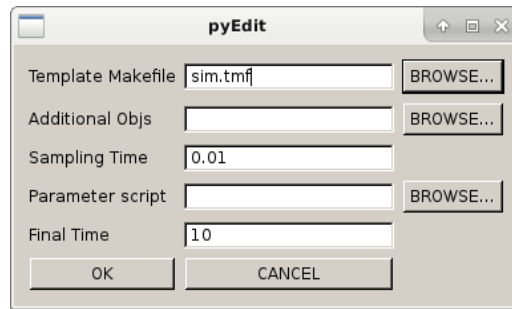


Figure 8.3: Dialog for code generation

In this dialog it is possible to choose the “template makefile” for simulation or real-time execution, the sampling time of the system and some additional libraries, required by special blocks.

## 8.7 Translating the diagram into elements of the RCPdlg class

After this first setup it is possible to translate the block diagram into a list of elements of the class **RCPblk** provided by the **suspisim** package. This class contains all the information required for the code generation.

This class contains the following fields:

**fcn:** the name of the C-Function to be used to handle this block

**pin:** an array containing the id of the input nodes

**pout:** an array containing the id of the output nodes

**nx:** the number of internal states (continuous or discrete)

**uy:** a flag which indicates a direct dependency between input and output signals (feed-through flag).

**realPar:** an array containing the real parameters of the block

**intPar:** an array containing the integer parameters of the block

**str:** a string related to the block

For example, the diagram in figure 8.4 is translated into the following code

```

from supsisim.RCPblk import *

STEP = stepBlk([1], 1, 1)
PM = sumBlk([1,3],[2], [1,-1])
CSS = cssBlk([2],[3], sys, 0)
PRINT = printBlk([1,3])

blks = [STEP,PM,CSS,PRINT,]
fname = 'step'
genCode(fname, 0.01, blks)
genMake(fname, 'sim.tmf', addObj = '')

```

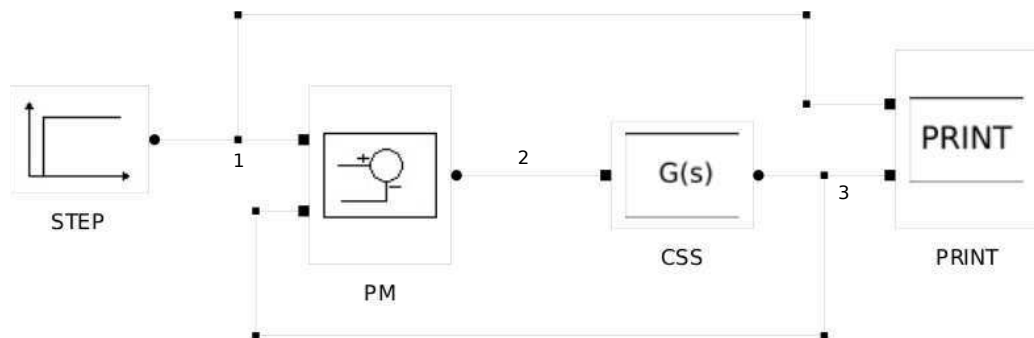


Figure 8.4: Simple block diagram

The block **CSS** has one input connected to node ② and one output connected to node ③, it is a continuous transfer function (`cssBlk`,  $1/(s+1)$ ) with zero initial conditions. The **PM** block has 2 inputs connected to node ① and ③, one output connected to node ② and performs a subtraction of the output from the input signals.

## 8.8 Translating the block list into C-code

### 8.8.1 Finding the right execution sequence

Before starting with the translation of the block diagram into C-code, we need to find the correct sequence of execution of the blocks. This task can be performed by analyzing the *uy* flag of the block object. When in a block the *uy* flag is set to 1, we need the output of the blocks connected at his input before starting to update his output. This means that we have to generate a dependency tree of all the blocks and then we must rearrange the order of the block list for code generation.

In linear blocks for examples, the *uy* flag is set if the *D* matrix is not null.

In the blockdiagram of figure 8.4, the **PM** and the **PRINT** blocks require to know their inputs before update their outputs.

```

In [5]: NrofNodes = 3

In [6]: ordered_list = detBlkSeq(NrofNodes, blks)

In [7]: for n in ordered_list:
...:     print n
...:
Function          : css
Input ports       : [2]
Outputs ports     : [3]
Nr. of states     : [2 0]
Relation u->y     : 0
Real parameters   : [[ 0.  0. -1.  1. -1. -1.
                       0.  0. -1.  0.  0.  0.]]
Integer parameters : [ 2  1  1  1  5  7  9 10]
String Parameter  :

Function          : step
Input ports       : []
Outputs ports     : [1]
Nr. of states     : [0 0]
Relation u->y     : 0
Real parameters   : [1 1]
Integer parameters : []
String Parameter  :

Function          : print
Input ports       : [1 3]
Outputs ports     : []
Nr. of states     : [0 0]
Relation u->y     : 1
Real parameters   : []
Integer parameters : []
String Parameter  :

Function          : sum
Input ports       : [1 3]
Outputs ports     : [2]
Nr. of states     : [0 0]
Relation u->y     : 1
Real parameters   : [ 1 -1]
Integer parameters : []
String Parameter  :

```

If the block diagram contains algebraic loops it is not possible to find a solution for the **det-BlkSeq** function and an error is raised.

### 8.8.2 Generating the C-code

Starting from the ordered list of blocks, it is possible to generate C-code. The code contains 3 functions:

- The initialization function
- The termination function
- The periodic task

### 8.8.3 The init function

In this function each block is translated into a `python_block` structure defined as follows:

```
typedef struct {
    int nin;           /* Number of inputs */
    int nout;          /* Number of outputs */
    int nx;            /* Cont. and Discr states */
    void **u;          /* inputs */
    void **y;          /* outputs */
    double *realPar;   /* Real parameters */
    int *intPar;        /* Int parameters */
    char *str;         /* String */
    void *ptrPar;       /* Generic pointer */
}python_block;
```

The nodes of the block diagram are defined as “double” variables and the inputs and outputs of the blocks are defined as vectors of pointers to them.

```
...
/* Nodes */
static double Node_1[] = {0.0};
static double Node_2[] = {0.0};
static double Node_3[] = {0.0};

/* Input and outputs */
static void *inptr_0[] = {0};
static void *outptr_0[] = {0};
static void *outptr_1[] = {0};
static void *inptr_2[] = {0,0};
static void *inptr_3[] = {0,0};
static void *outptr_3[] = {0};
...
    inptr_0[0] = (void *) Node_2;
    outptr_0[0] = (void *) Node_3;
..
    block_test[0].nin = 1;
    block_test[0].nout = 1;
    block_test[0].nx = nx_0;
    block_test[0].u = inptr_0;
    block_test[0].y = outptr_0;
...

```

After this initialization phase, the implementation functions of the blocks are called with the flag **INIT**.

```
css(INIT, &block_test[0]);
step(INIT, &block_test[1]);
print(INIT, &block_test[2]);
sum(INIT, &block_test[3]);
```

### 8.8.4 The termination function

This procedure calls the implementation functions of the blocks with the flag **END**.



### 8.8.5 The ISR function

This procedure represents the periodic task of the RT execution. First of all, the implementation functions are called with the flag **OUT**, in order to perform the output update of each blocks. As a second step, the implementation functions of the block containing internal states ( $nx \neq 0$ ) are called with the flag **STUPD** (state update).

```
...
css(OUT, &block_test[0]);
step(OUT, &block_test[1]);
print(OUT, &block_test[2]);
sum(OUT, &block_test[3]);
...
css(OUT, &block_test[0]);
css(STUPD, &block_test[0]);
...
```

## 8.9 The main file

The core of the RT execution is represented by the “python\_main\_rt.c” file. During the RT execution, the main procedure starts a high priority thread for handling the RT behavior of the system. The following main file, for example, is used to launch the executable in a Linux preempt\_rt environment.

```
void *rt_task(void *p)
{
    ...
    param.sched_priority = prio;
    if(sched_setscheduler(0, SCHED_FIFO, &param)==-1){
        perror("sched_setscheduler _failed");
        exit(-1);
    }

    ...
    double Tsamp = NAME(MODEL, _get_tsamp)();

    ...
    NAME(MODEL, _init)();

    while(!end){
        /* wait untill next shot */
        clock_nanosleep(CLOCK_MONOTONIC,
                        TIMER_ABSTIME, &t, NULL);

        ...
        /* periodic task */
        NAME(MODEL, _isr)(T);
        ...
    }
    NAME(MODEL, _end)();
}
```



# Chapter 9

## Example

### 9.1 The plant

One of the educational plants available at the SUPSI laboratory is the system shown in figure 9.1. This example is located in to the “pycontrol/Tests/ControlDesign/DisksAndSpring” folder,

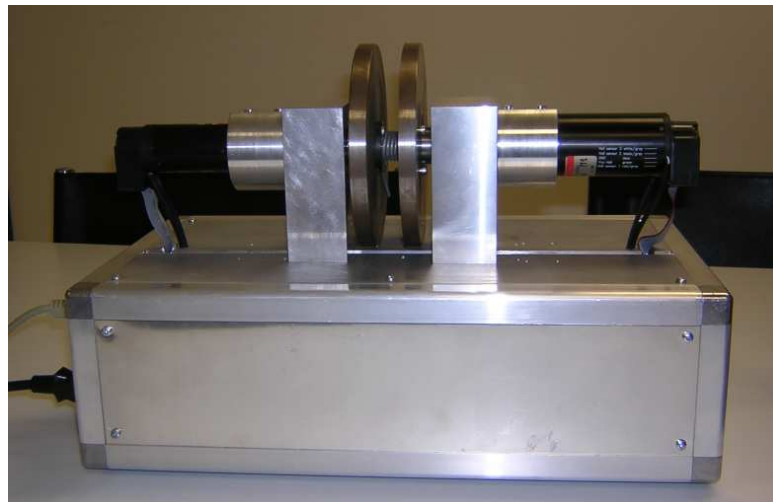


Figure 9.1: The disks and spring plant

Two disks are connected by a spring. The goal for the students is to control the angle of the disk on the right by applying an appropriate torque to the disk on the left.

The physical model of this plant can be directly calculated in python using for example the **sympy** toolbox. Sympy can deliver a symbolic description of the system and through a python *dictionary* it is possible to easily obtain the numerical matrices of the state-space representation of the plant.

```

In [1]: # Real plants parameters
...: # Motor 1
...: jm1 = 0.0000085 # inertia [kg*m2]
...: kt1 = 0.0000382 # Torque constant
...: d1 = 0.0002953 # Damp
...:
...: # Last motor 1
...: rho_ac = 7900 # density [g/m3]
...: rv1 = 0.065 # radius [m]
...: hv1 = 0.01 # thickness [m]
...: mv1 = ((rho_ac*(rv1**2))*np.pi)*hv1 # mass [kg]
...: jv1 = (mv1*(rv1**2))/2 # inertia [kg*m2]
...: J1 = jm1+jv1 # total inertia [kg*m2]
...:

```

```

In [2]: # Motor 2
...: jm2 = 0.000003 # inertia [kg*m2]
...: kt2 = 0.0000205 # Torque constant
...: d2 = 0.0004001 # damp
...:
...: # Last motor 2
...: rho_ac = 7900 # density [kg/m3]
...: rv2 = 0.065 # radius [m]
...: hv2 = 0.01 # thickness [m]
...: mv2 = ((rho_ac*(rv2**2))*np.pi)*hv2 # mass [kg]
...: jv2 = (mv2*(rv2**2))/2 # inertia [kg*
m2]
...: J2 = jv2+jm2 # total inertia [km
*m2]
...:

```

```

In [3]: # Spring
...: d = 0.0027836 # damp
...: c = 0.4797954 # spring factor
...:

```

```

In [4]: A
Out[4]:
matrix([[0, 0, 1, 0],
        [0, 0, 0, 1],
        [-c/J1, -c/J1, (-d - d1)/J1, -d/J1],
        [-c/J2, -c/J2, -d/J2, (-d - d2)/J2]])

In [5]: B1
Out[5]:
matrix([[0, 0],
        [0, 0],
        [kt1/J1, 0],
        [0, kt2/J2]])

In [6]: B = B1[:,0]

In [7]: C
Out[7]: [[1, 0, 0, 0], [0, 1, 0, 0]]

In [8]: C2
Out[8]: [0, 1, 0, 0]

In [9]: D
Out[9]: [[0], [0]]

In [10]: D2
Out[10]: [0]

```

The control system toolbox and the additional “yottalab.py” package contain all the functions required for the design of the controller. In this case we design a discrete-state feedback controller with integral part for eliminating steady-state errors. The states are estimated with a reduced-order observer. In addition, an anti-windup mechanism has been implemented. The sampling time is set to 10 ms.

The yottalab module offers 3 functions that facilitate the controller design:

- The function **red\_obs**(sys, T, poles) which implements the reduced-order observer for the system **sys**, using the submatrix **T** (required to obtain the estimator C-matrix and the desired state-estimator poles **poles**).

$$P = [C; T] \rightarrow C^* = C \cdot P^{-1} = [I_q, O_{(n-q)}]$$

- The function **comp\_form\_i**(sys, obs, K, Cy) that transforms the observer **obs** with the state-feedback gains **K** and the integrator part into a single dynamic block with the reference signal and the two positions  $\varphi_1$  and  $\varphi_2$  as inputs and the control current  $I_1$  as output. The vector **Cy** is used to select  $\varphi_2$  as the output signal that is compared with the reference signal for generating the steady-state error for the integral part of the controller.
- The function **set\_aw**(sys, poles) that transforms the previous controller ( $Contr(s) = N(s)/D(s)$ ) in an input state-space system and a feedback state-space system, implementing the anti-windup mechanism. The vector **poles** contains the desired poles of the two new systems ( $D_{new}(s)$ ) (see figure 9.2).

$$sys_{in}(s) = \frac{N(s)}{D_{new}(s)}$$

$$sys_{fbk}(s) = 1 - \frac{D(s)}{D_{new}(s)}$$

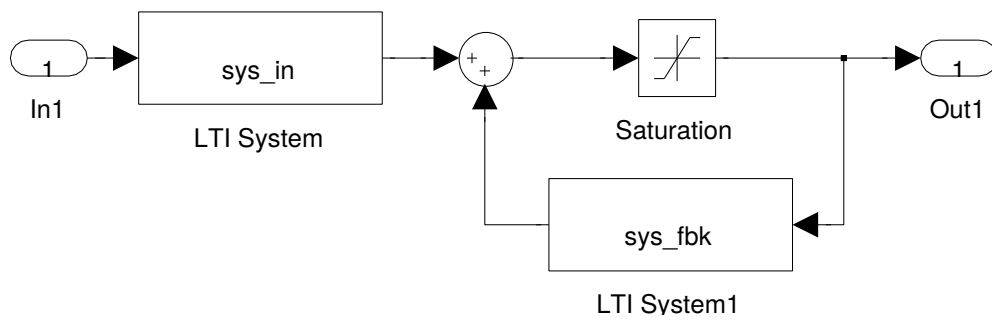


Figure 9.2: Anti windup

## 9.2 The plant model

```
# Sampling time
ts = 10e-3

gss1 = ss(A,B,C,D)
gss = ss(A,B,C2,D2)
gz = c2d(gss,ts,'zoh')
```

## 9.3 Controller design

```
# Control design
wn = 10
xi1 = np.sqrt(2)/2
xi2 = 0.85

cl_p1 = [1, 2*xi1*wn, wn**2]
cl_p2 = [1, 2*xi2*wn, wn**2]
cl_p3 = [1, wn]
cl_poly1 = sp.polymul(cl_p1, cl_p2)
cl_poly = sp.polymul(cl_poly1, cl_p3)
cl_poles = sp.roots(cl_poly)      # Desired continuous
                                   poles
cl_polesd = sp.exp(cl_poles*ts)  # Desired discrete poles

# Add discrete integrator for steady state zero error
Phi_f = np.vstack((gz.A, -gz.C*ts))
Phi_f = np.hstack((Phi_f, [[0], [0], [0], [0], [1]]))
G_f = np.vstack((gz.B, zeros((1,1))))

# Pole placement
k = placep(Phi_f, G_f, cl_polesd)
```

## 9.4 Observer design

```
# Observer design - reduced order observer
poli_o = 5*cl_poles[0:2]
poli_oz = sp.exp(poli_o*ts)

disks = ss(A,B,C,D)
disksz = StateSpace(gz.A, gz.B, C, D, ts)
T = [[0, 0, 1, 0], [0, 0, 0, 1]]

# Reduced order observer
r_obs = red_obs(disksz, T, poli_oz)

# Controller and observer in the same matrix - Compact
form
contr_I = comp_form_i(disksz, r_obs, k, [0, 1])

# Implement anti windup
[gss_in, gss_out] = set_aw(contr_I, [0.1, 0.1, 0.1])
```

## 9.5 Simulation

We can perform the simulation of the discrete-time controller with the continuous-time mathematical plant model using the block diagram of figure 9.3

This diagram is stored as “disks\_sim.dgm” in the folder.

The plant is represented by a continuous-time state-space block with 1 input and 2 outputs. The controller implements the state-feedback gains and the state observer and it has been split into a CTRIN block and a CTRFBK block in order to implement the anti-windup mechanism.

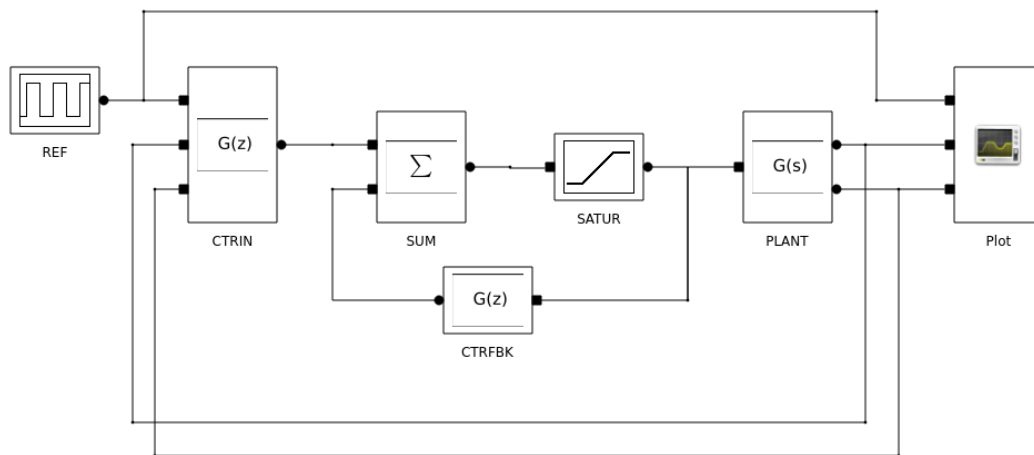


Figure 9.3: Block diagram for the simulation

Now we can launch the simulation with the command “Simulate” from the toolbar or from the menu.

A double click on the ‘block “Plot” show the result of the simulation (see figure 9.4)

## 9.6 Real-time controller

In order to generate the RT controller for the real plant, we first have to substitute the plant with the interfaces for sensors and actuators using blocks that send and receive CAN message using a USB dongle of Peak System. The template makefile for this system is now `rt.tmf`, that allows to generate code with real-time behaviour.

The block diagram for the real-time controller is represented in figure 9.5.

The motor position can be plotted in python at the end of the execution (see figure 9.6).



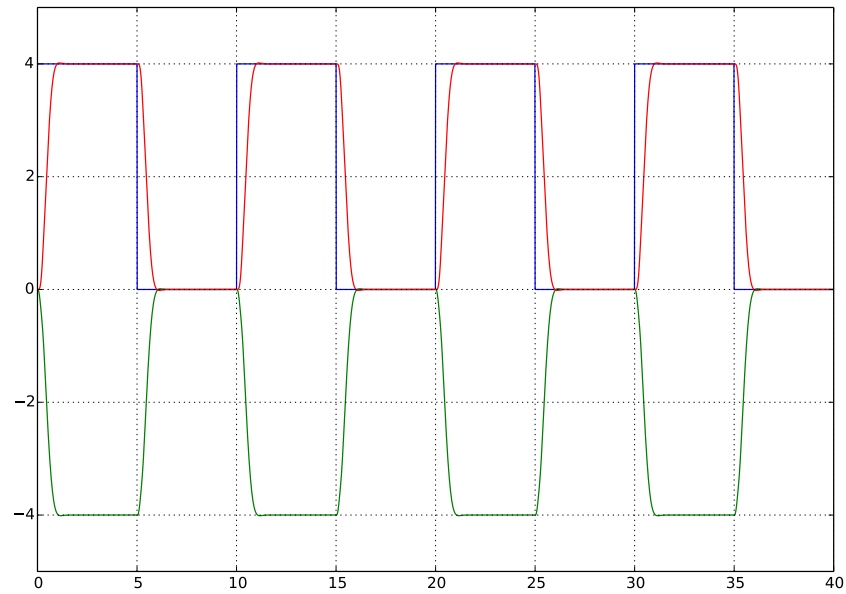


Figure 9.4: Simulation of the plant

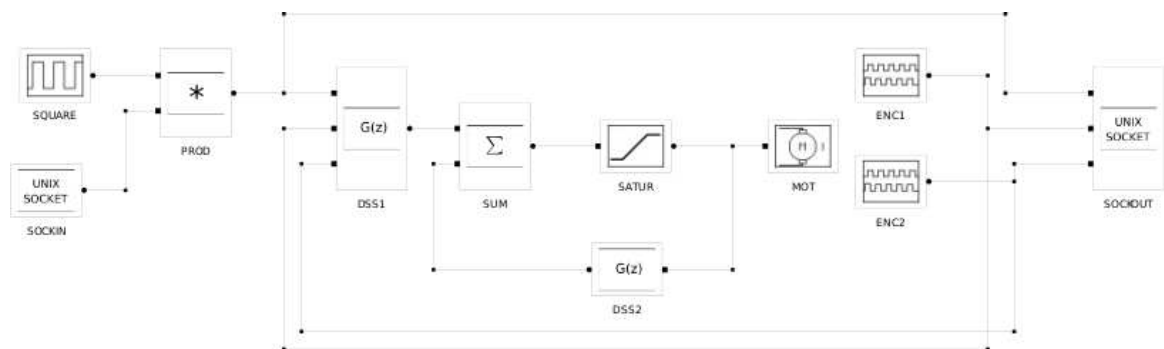


Figure 9.5: Block diagram for the RT implementation

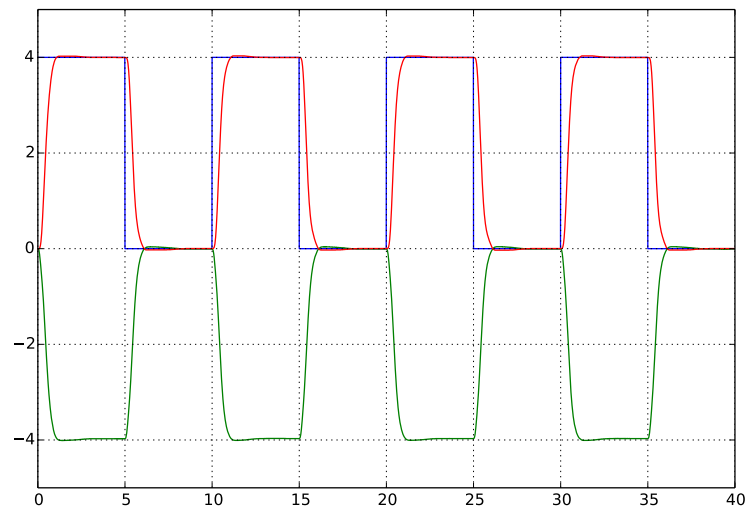


Figure 9.6: RT execution

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