

Hooke's law: anisotropic materials

- new notation

$$\sigma_{xx} \rightarrow \sigma_{11} \rightarrow \sigma_1 \quad \sigma_{yz} \rightarrow \sigma_{23} \rightarrow \sigma_4$$

$$\sigma_{yy} \rightarrow \sigma_{22} \rightarrow \sigma_2 \quad \sigma_{xz} \rightarrow \sigma_{13} \rightarrow \sigma_5$$

$$\sigma_{zz} \rightarrow \sigma_{33} \rightarrow \sigma_3 \quad \sigma_{xy} \rightarrow \sigma_{12} \rightarrow \sigma_6$$

$$\epsilon_{xx} \rightarrow \epsilon_{11} \rightarrow \epsilon_1 \quad \gamma_{yz} \rightarrow \epsilon_4$$

$$\epsilon_{yy} \rightarrow \epsilon_{22} \rightarrow \epsilon_2 \quad \gamma_{xz} \rightarrow \epsilon_5$$

$$\epsilon_{zz} \rightarrow \epsilon_{33} \rightarrow \epsilon_3 \quad \gamma_{xy} \rightarrow \epsilon_6$$

- matrix notation: most anisotropic material possible

$$\begin{bmatrix} \epsilon_1 \\ \epsilon_2 \\ \epsilon_3 \\ \epsilon_4 \\ \epsilon_5 \\ \epsilon_6 \end{bmatrix} = \begin{bmatrix} S_{11} & S_{12} & S_{13} & S_{14} & S_{15} & S_{16} \\ & S_{22} & S_{23} & S_{24} & S_{25} & S_{26} \\ & & S_{33} & S_{34} & S_{35} & S_{36} \\ & & & S_{44} & S_{45} & S_{46} \\ & & & & S_{55} & S_{56} \\ & & & & & S_{66} \end{bmatrix} \begin{bmatrix} \sigma_1 \\ \sigma_2 \\ \sigma_3 \\ \sigma_4 \\ \sigma_5 \\ \sigma_6 \end{bmatrix}$$

↓

Compliance matrix, S

21 independent components

can also write:

$$[\sigma] = [C][\epsilon]$$



Stiffness matrix, C

meaning of components

(a) if apply σ_1 only

$$\epsilon_1 = S_{11} \sigma_1 \Rightarrow S_{11} = 1/E_1$$

if apply σ_2 only

$$\epsilon_2 = S_{22} \sigma_2 \Rightarrow S_{22} = 1/E_2$$

if apply σ_3 only

$$\epsilon_3 = S_{33} \sigma_3 \Rightarrow S_{33} = 1/E_3$$

(b) if apply σ_4 only (shear in 2-3 plane)

$$\epsilon_4 = S_{44} \sigma_4 \Rightarrow S_{44} = 1/G_{23}$$

similarly,

$$S_{55} = 1/G_{13}$$

$$S_{66} = 1/G_{12}$$

(c) if apply σ_1 only

$$\epsilon_2 = S_{21} \sigma_1$$

$$\nu_{ij} = -\frac{\epsilon_j}{\epsilon_i} \leftarrow \begin{array}{l} \text{transverse} \\ \text{direction of applied stress} \end{array}$$

$$\nu_{12} = -\frac{\epsilon_2}{E_1} = -\frac{S_{21} \sigma_1}{S_{11} \sigma_1} = -\frac{S_{21}}{S_{11}}$$

$$S_{21} = -\frac{\nu_{12}}{E_1}$$

$$\text{similarly, } S_{31} = -\frac{\nu_{13}}{E_1}$$

$$S_{32} = -\frac{\nu_{23}}{E_2}$$

reciprocal relation:

$$S_{21} = S_{12} \Rightarrow -\frac{\nu_{12}}{E_1} = -\frac{\nu_{21}}{E_2}$$

(3)

① Consider elements S_{14} S_{15} S_{16}

S_{14} relates ϵ_1 to σ_4
normal strain ↓
 shear stress

typically, S_{14} S_{15} S_{16}

S_{24} S_{25} S_{26} all zero
 S_{34} S_{35} S_{36}

② Consider elements S_{45} , S_{46} , S_{56}

S_{45} relates ϵ_4 to σ_5
shear strain ↓
 shear stress
in 2-3 plane

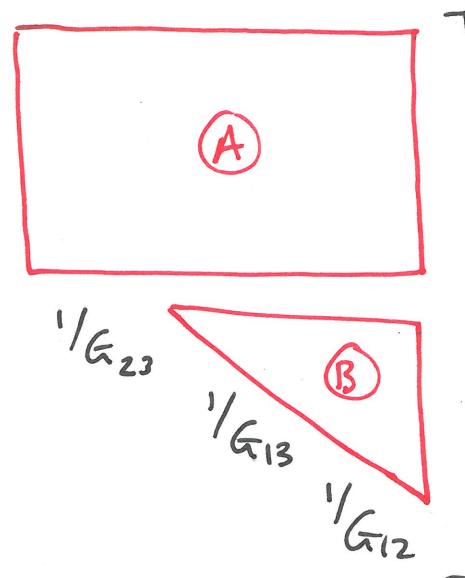
S_{46} , S_{56} typically zero
in 1-3 plane

(some woven fabrics have non-zero values
of these components)

Summary

$$\begin{bmatrix} \frac{1}{E_1} & -\frac{\nu_{21}}{E_2} & -\frac{\nu_{31}}{E_3} \\ -\frac{\nu_{12}}{E_1} & \frac{1}{E_2} & -\frac{\nu_{32}}{E_3} \\ -\frac{\nu_{13}}{E_1} & -\frac{\nu_{23}}{E_2} & \frac{1}{E_3} \end{bmatrix}$$

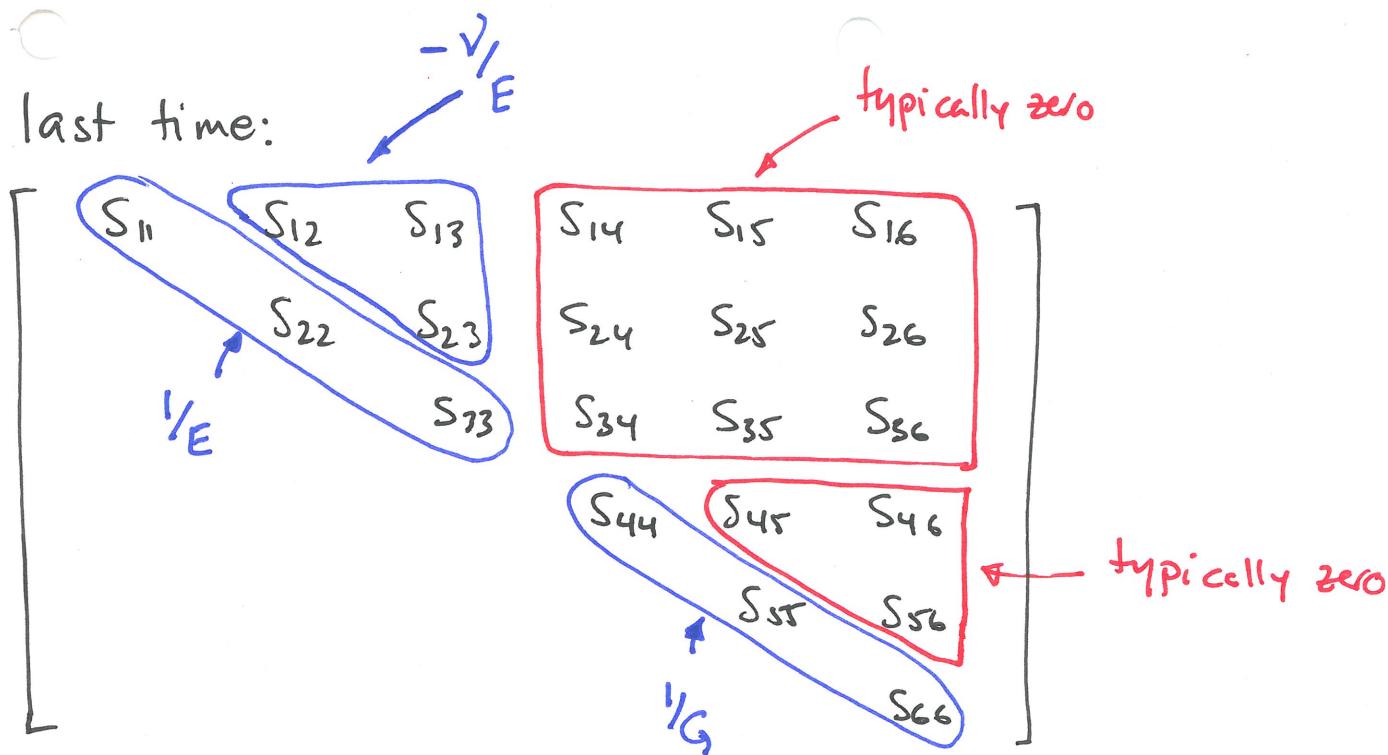
Symmetric



Ⓐ relates normal strain
to shear stress

Ⓑ relates shear strain in one
plane to shear stress in a
different plane

last time:



Anisotropic materials: Symmetry

Orthotropic materials

- rotate 180° about each of 3 mutually perpendicular axes & structure remains unchanged
- 9 independent elastic constants
e.g. wood, some fiber composites

| | | | | | |
|----------|----------|----------|----------|----------|---|
| S_{11} | S_{12} | S_{13} | 0 | 0 | 0 |
| S_{22} | S_{23} | 0 | 0 | 0 | |
| | S_{33} | 0 | 0 | 0 | |
| | | S_{44} | 0 | 0 | |
| | | | S_{55} | 0 | |
| | | | | S_{66} | |

Sym

Transversely isotropic materials

- isotropic in one plane
- e.g. matrix: x_1-x_2 plane isotropic
- 5 independent elastic constants
- e.g. extruded materials

| | | | | | |
|----------|----------|----------|----------|---|---|
| S_{11} | S_{12} | S_{13} | 0 | 0 | 0 |
| S_{11} | S_{13} | 0 | 0 | 0 | |
| | S_{33} | 0 | 0 | 0 | |
| | | S_{44} | 0 | 0 | |
| | | | S_{44} | 0 | |

Sym

$$\text{Note: } 2(S_{11} - S_{12}) = 2 \left(\frac{1}{E_1} + \frac{\nu_{12}}{E_1} \right)$$

$$= \frac{2(1 + \nu_{12})}{E_1} = \frac{1}{G_{12}}$$

$$2(S_{11} - S_{12})$$



cubic materials

- $E_1 = E_2 = E_3 = 1/S_{11}$
- $G_{23} = G_{13} = G_{12} = 1/S_{44}$
- all ν equal = $-S_{12}/S_{11}$
- 3 independent elastic constants

$$\left[\begin{array}{cccccc} S_{11} & S_{12} & S_{12} & 0 & 0 & 0 \\ S_{12} & S_{11} & S_{12} & 0 & 0 & 0 \\ S_{12} & S_{12} & S_{11} & 0 & 0 & 0 \\ 0 & 0 & 0 & S_{44} & 0 & 0 \\ 0 & 0 & 0 & 0 & S_{44} & 0 \\ 0 & 0 & 0 & 0 & 0 & S_{44} \end{array} \right]$$

Symmetric

isotropic materials

- $E_1 = E_2 = E_3 = 1/S_{11}$
- all $G = \frac{E}{2(1+\nu)} = \frac{1}{2(S_{11}-S_{12})}$
- all ν equal.

$$\left[\begin{array}{cccccc} S_{11} & S_{12} & S_{12} & 0 & 0 & 0 \\ S_{12} & S_{11} & S_{12} & 0 & 0 & 0 \\ S_{12} & S_{12} & S_{11} & 0 & 0 & 0 \\ 0 & 0 & 0 & 2(S_{11}-S_{12}) & 0 & 0 \\ 0 & 0 & 0 & 0 & 2(S_{11}-S_{12}) & 0 \\ 0 & 0 & 0 & 0 & 0 & 2(S_{11}-S_{12}) \end{array} \right]$$

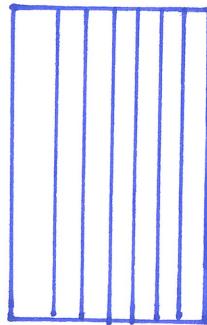
SCALING + STRESS:

WOODPECKER TALK

Composite materials:

- combination of 2 or more materials

e.g. fibre reinforced materials

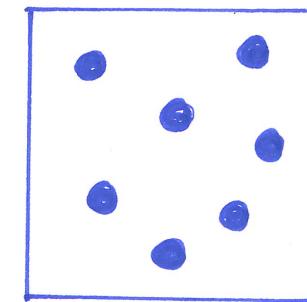


e.g. glass fibers /

polyester resin

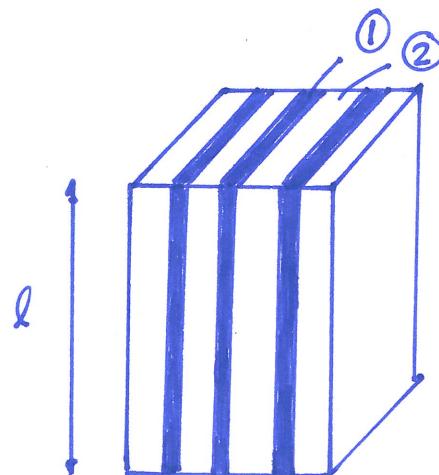
metal matrix
composites

particulate composites



e.g. rubber
toughened
resins.

Young's modulus



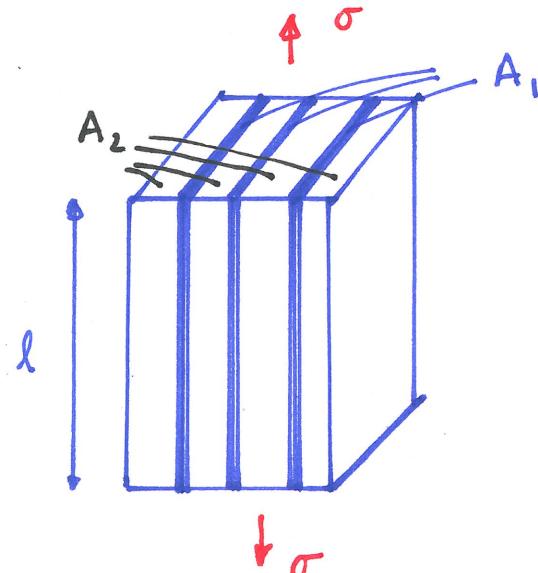
phases 1 & 2 : E_1, E_2

composite: no subscript (σ, ϵ, E)

→ load \parallel layers : deformation same in all layers

→ load \perp layers : load same in all layers.

Young's modulus: parallel to layers. (E_{\parallel})



$$\delta = \delta_1 = \delta_2$$

$$l_1 = l_2 = l$$

$$E_1 = E_2 = E$$

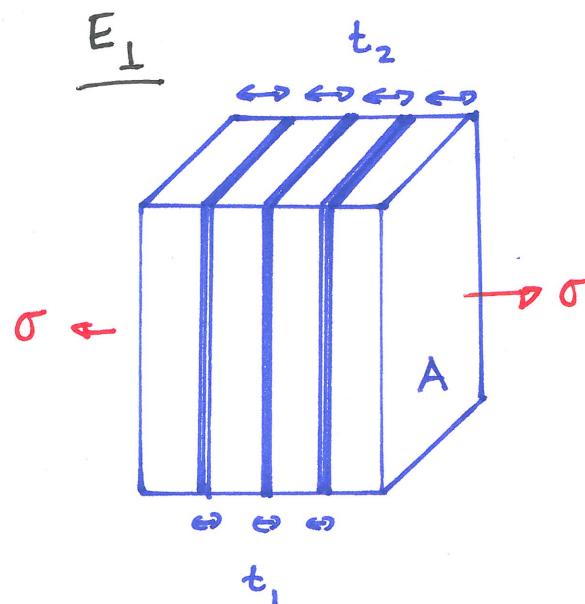
$$P = P_1 + P_2$$

$$= E_1 \epsilon A_1 + E_2 \epsilon A_2$$

$$\sigma = \left(\frac{E_1 A_1 + E_2 A_2}{A} \right) \epsilon$$

$$E_{\parallel} = E_1 V_1 + E_2 V_2$$

V_1, V_2 = volume fraction
rule of mixtures **upper bound** of 1, 2



$$P_1 = P_2 = P$$

$$A_1 = A_2 = A$$

$$\sigma_1 = \sigma_2 = \sigma$$

$$\delta = \delta_1 + \delta_2$$

$$= \frac{P t_1}{A E_1} + \frac{P t_2}{A E_2}$$

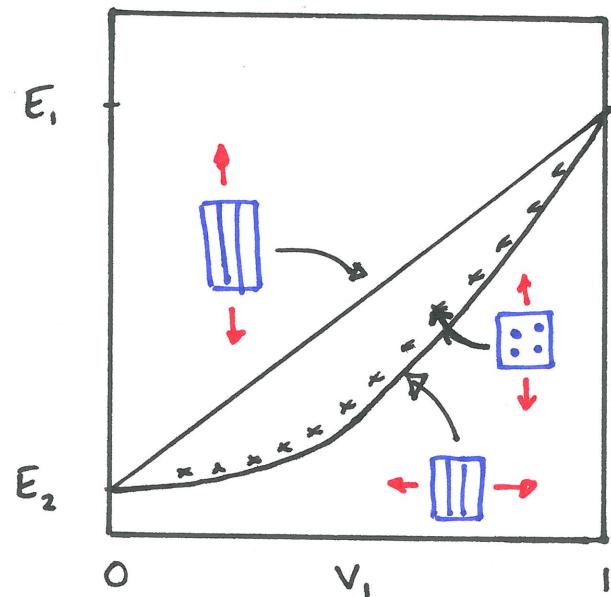
$$= \sigma \left(\frac{t_1}{E_1} + \frac{t_2}{E_2} \right)$$

$$\epsilon = \frac{\delta}{t} = \sigma \left(\frac{t_1}{t E_1} + \frac{t_2}{t E_2} \right) = \sigma \left(\frac{V_1}{E_1} + \frac{V_2}{E_2} \right)$$

$$E_{\perp} = \frac{E_1 E_2}{E_1 V_2 + E_2 V_1}$$

lower bound

Composites



unidirectional fiber composites

- loaded parallel to fibers - close to upper bound
- loaded \perp to fibers - close to lower bound

particulate composites - close to lower bound