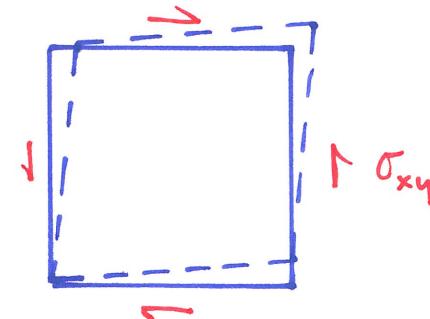


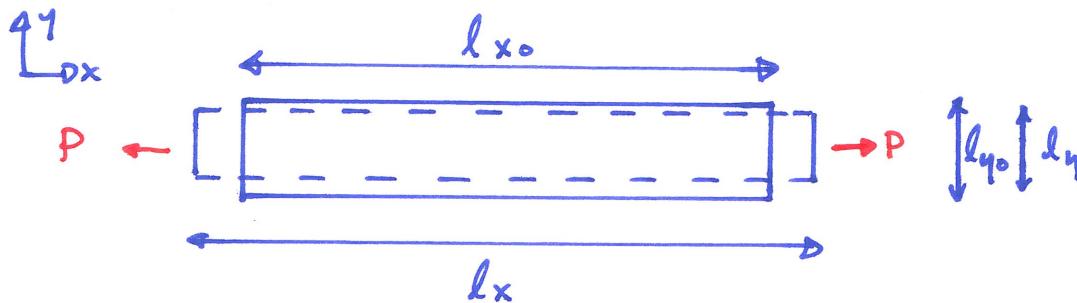
- Shear stress + shear strain related in a similar way, by shear modulus,  $G$

$$\sigma_{xy} = G \gamma_{xy} \quad G \text{ [Pa]}$$



- also define Poisson's ratio,  $\nu = - \frac{\text{lateral strain}}{\text{strain in direction of applied uniaxial stress}}$

rubber band demo



$$\nu = - \frac{\epsilon_y}{\epsilon_x} = - \frac{\frac{l_y - l_{y0}}{l_{y0}}}{\frac{l_x - l_{x0}}{l_{x0}}} = - \frac{\delta_y / l_{y0}}{\delta_x / l_{x0}}$$

$\nu$  - DIMENSIONLESS [-]

- as pull, material contracts sideways
- $\epsilon_y$  opposite sign to  $\epsilon_x$
- also get contraction in z direction
- if material isotropic,  $\epsilon_z = \epsilon_y$

- typical values of  $\nu$

steel    0.28

aluminum 0.33

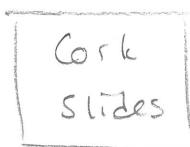
glass    0.25

rubber 0.45-0.49



- cellular materials can have unusual values of  $\nu$  (not isotropic!)

- regular hexagonal honeycomb:  $\nu = 1$
- "inverted" honeycomb:  $\nu < 1$
- cork:  $\nu = 0$



- isotropic solids: often  $\nu \approx 0.3$   
(except rubbers)

- data for elastic moduli: Ashby + Jones p 39-41

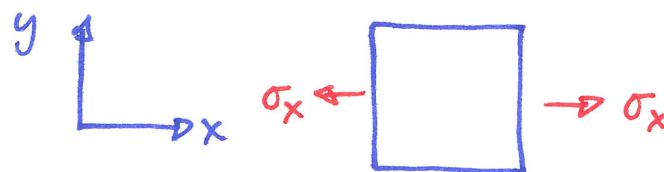
materials databases:

[www.matweb.com](http://www.matweb.com)

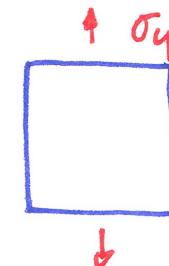
[www.nist.gov/srd/index.cfm](http://www.nist.gov/srd/index.cfm)

## Hooke's law: General state of stress, isotropy

- first, consider  $\epsilon_x$  (one subscript  $\Rightarrow$  normal stress, strain)
- if apply  $\sigma_x$  only
- if apply  $\sigma_y$  only



$$\epsilon_x = \frac{\sigma_x}{E}$$



$$\nu = -\frac{\epsilon_x}{\epsilon_y} = -\epsilon_x \frac{E}{\sigma_y}$$

$$\therefore \epsilon_x = -\nu \frac{\sigma_y}{E}$$

: if apply  $\sigma_x + \sigma_y$  simultaneously:

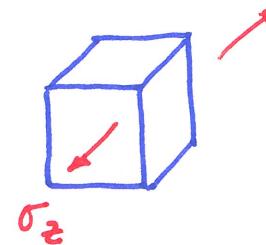
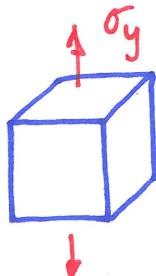
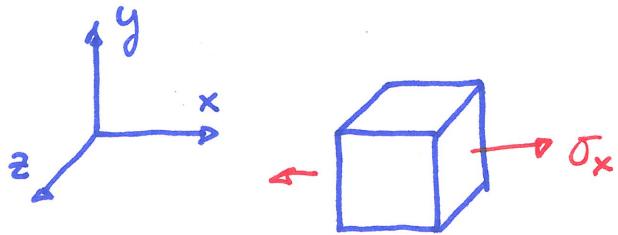
$$\epsilon_x = \frac{\sigma_x}{E} - \nu \frac{\sigma_y}{E}$$

: if apply  $\sigma_x + \sigma_y + \sigma_z$  simultaneously:

$$\epsilon_x = \frac{\sigma_x}{E} - \nu \frac{\sigma_y}{E} - \nu \frac{\sigma_z}{E}$$

Poisson's ratio, isotropic material : remember:

$$\nu = - \frac{\text{lateral strain}}{\text{strain in direction of applied uniaxial stress}}$$



$$\nu = -\frac{\epsilon_y}{\epsilon_x} = -\frac{\epsilon_z}{\epsilon_x}$$

$$\nu = -\frac{\epsilon_x}{\epsilon_y} = -\frac{\epsilon_z}{\epsilon_y}$$

$$\nu = -\frac{\epsilon_x}{\epsilon_z} = -\frac{\epsilon_y}{\epsilon_z}$$

(2)

- for a general state of stress (3 normal, 3 shear), can relate stress to strain by Hooke's law for an isotropic material

$$\epsilon_x = \frac{\sigma_x}{E} - \nu \frac{\sigma_y}{E} - \nu \frac{\sigma_z}{E}$$

$$\gamma_{yz} = \frac{\sigma_{yz}}{G}$$

$$\epsilon_y = -\nu \frac{\sigma_x}{E} + \frac{\sigma_y}{E} - \nu \frac{\sigma_z}{E}$$

$$\gamma_{xz} = \frac{\sigma_{xz}}{G}$$

$$\epsilon_z = -\nu \frac{\sigma_x}{E} - \nu \frac{\sigma_y}{E} + \frac{\sigma_z}{E}$$

$$\gamma_{xy} = \frac{\sigma_{xy}}{G}$$

- or, as matrix:

$$\begin{bmatrix} \epsilon_x \\ \epsilon_y \\ \epsilon_z \\ \gamma_{yz} \\ \gamma_{xz} \\ \gamma_{xy} \end{bmatrix} = \begin{bmatrix} 1/E & -\nu/E & -\nu/E & 0 & 0 & 0 \\ -\nu/E & 1/E & -\nu/E & 0 & 0 & 0 \\ -\nu/E & -\nu/E & 1/E & 0 & 0 & 0 \\ 0 & 0 & 0 & 1/G & 0 & 0 \\ 0 & 0 & 0 & 0 & 1/G & 0 \\ 0 & 0 & 0 & 0 & 0 & 1/G \end{bmatrix} \begin{bmatrix} \sigma_x \\ \sigma_y \\ \sigma_z \\ \sigma_{yz} \\ \sigma_{xz} \\ \sigma_{xy} \end{bmatrix}$$

- this set of equations completely describes the linear elastic behaviour of an isotropic material
- Young's modulus, shear modulus, Poisson's ratio are all elastic moduli
- for an isotropic solid, only 2 are independent

$$G = \frac{E}{2(1+\nu)} \quad \Rightarrow \quad \nu > -1$$

since  $\nu \approx \frac{1}{3}$  typically  $\Rightarrow G \approx \frac{3}{8} E$

- also define bulk modulus,  $K$
- apply  $\sigma_x = \sigma_y = \sigma_z = \sigma_h$  (hydrostatic stress, all shear stresses = 0)

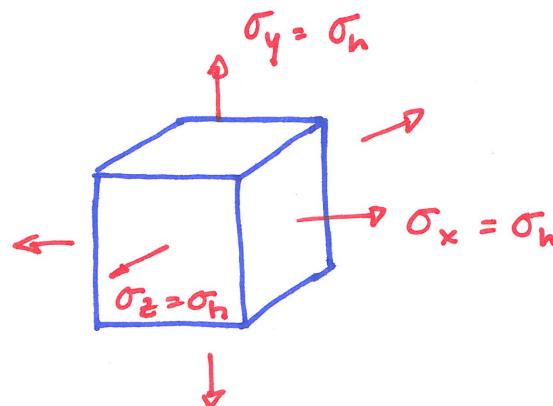
$$K = \frac{\sigma_h}{\Delta V/V_0} \quad \frac{\Delta V}{V_0} = \text{VOLMETRIC STRAIN, ALSO CALLED "DILATATION", } \Delta.$$

$$K = \frac{E}{3(1-2\nu)} \quad \Rightarrow \quad \nu < \frac{1}{2}$$

bounds on Poisson's ratio for isotropic solid:  $-1 < \nu < \frac{1}{2}$

show:  $K = \frac{E}{3(1-2\nu)}$

$$K = \frac{\sigma_h}{\Delta V/V_0}$$



$$\epsilon_x = \epsilon_y = \epsilon_z = \epsilon$$

cube edge length,  $l_0$

$$\frac{\Delta V}{V_0} = ?$$

$$V_0 = l_0^3$$

$$V = (l_0 + \delta)^3 = (l_0 + \epsilon l_0)^3 = l_0^3 (1 + \epsilon)^3$$

$$\frac{\Delta V}{V_0} = \frac{l_0^3 (1 + \epsilon)^3 - l_0^3}{l_0^3} = (1 + \epsilon)^3 - 1 \approx 3\epsilon$$

(neglect  $\epsilon^2, \epsilon^3$  terms  
since  $\epsilon$  small)

(5)

use Hooke's law:

$$\epsilon_x = \frac{\sigma_x}{E} - v \frac{\sigma_y}{E} - v \frac{\sigma_z}{E}$$

$$\sigma_x = \sigma_y = \sigma_z = \sigma_h$$

$$\epsilon_x = \epsilon_y = \epsilon_z = \epsilon$$

$$\epsilon = \left( \frac{1-2v}{E} \right) \sigma_h$$

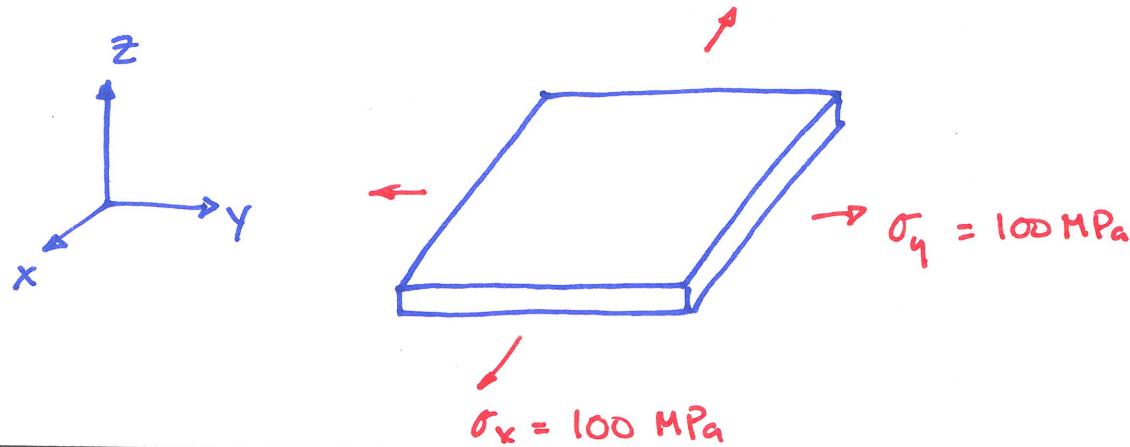
$$K = \frac{\sigma_h}{\Delta V/V_0} = \frac{\sigma_h}{3\epsilon}$$

$$= \frac{\sigma_h E}{3(1-2v)\sigma_h}$$

$$K = \frac{E}{3(1-2v)}$$

## Example: Hooke's law

- Al plate sees biaxial stress,  $\sigma_x = \sigma_y = 100 \text{ MPa}$
- What are  $\epsilon_x, \epsilon_y, \epsilon_z$ ? Al:  $E = 70 \text{ GPa}$   $\nu = 0.33$



$$\epsilon_x = \frac{\sigma_x - \nu \sigma_y}{E} = \frac{(1 - 0.33)(100 \text{ MPa})}{(70,000 \text{ MPa})} = 9.52 \times 10^{-4}$$

$$\epsilon_y = \epsilon_x \quad (\text{symmetry})$$

$$\epsilon_z = -\nu \frac{\sigma_x}{E} - \nu \frac{\sigma_y}{E} = -\frac{(0.33)(2)(100 \text{ MPa})}{70,000 \text{ MPa}} = -9.52 \times 10^{-4}$$

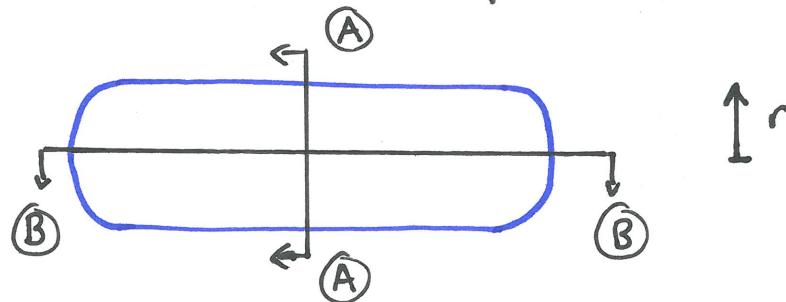
Note:

- strains small
- transverse strain can be of same order as strains in direction of applied load

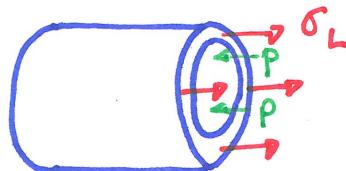
## Example: Pressure vessels (for lab 2)

Consider a thin-walled cylinder (radius  $r \gg$  thickness,  $t$ )

with an internal pressure,  $P$ . Stresses in wall = ?



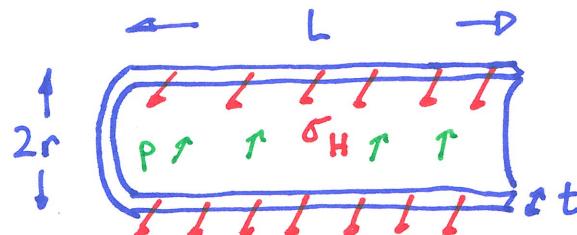
cut A-A, free body diagram



$$\sigma_L 2\pi r t = p \pi r^2$$

$$\sigma_L = \frac{pr}{2t}$$

cut B-B, free body diagram



$$\sigma_H 2L t = p 2r L$$

$$\sigma_H = \frac{pr}{t}$$

and radial stress,  $\sigma_R$ :

at inner wall  $\sigma_R = p$

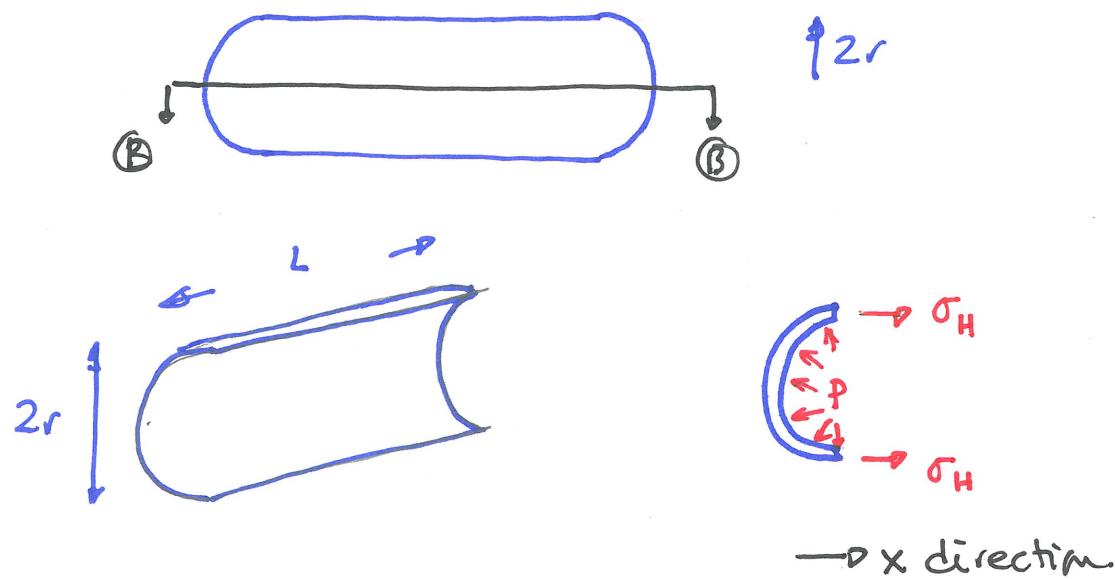
at outer wall  $\sigma_R = 0$

$\sigma_R \ll \sigma_H$  or  $\sigma_L$

(since  $r/t$  large)

$\Rightarrow$  neglect  $\sigma_R$

Last time, cylindrical pressure vessel.



- fluid cannot resist shear stress  
⇒ flows under shear
- pressure acts normal to surface
- x component of force from the pressure acts on projected area with normal in x direction

Force balance:  $\Sigma F_x = 0 \rightarrow$

$$\sigma_H L 2t = p L 2r$$

$$\sigma_H = \frac{pr}{t}$$

## Lab 2: Pressure vessels

- measure strain on pressure vessel with strain gauge
- ratio of  $\epsilon_H / \epsilon_L = ?$

$$\epsilon_H = \frac{\sigma_H}{E} - \nu \frac{\sigma_L}{E}$$

$$\epsilon_L = \frac{\sigma_L}{E} - \nu \frac{\sigma_H}{E}$$

$$\frac{\epsilon_H}{\epsilon_L} = \frac{(1-\nu/2)}{(\frac{1}{2}-\nu)}$$

$$\epsilon_H = \frac{Pr}{Et} \left(1 - \frac{\nu}{2}\right)$$

$$= \frac{Pr}{Et} \left(\frac{1}{2} - \nu\right)$$

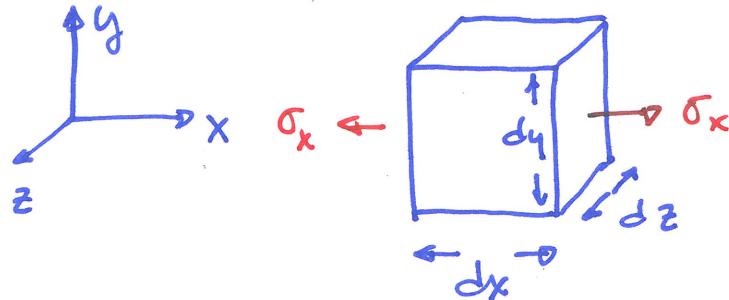
e.g.  $\nu = 0.3$

$$\epsilon_H / \epsilon_L = 4.25$$

(not 2!)

## Elastic strain energy, $U$

- Work done = force  $\times$  distance force moves through, in direction of force
- Consider normal stress,  $\sigma_x$ , acting on element  $dx dy dz$



$$\text{Force} = \sigma_x dy dz$$

If  $F=0$  initially,

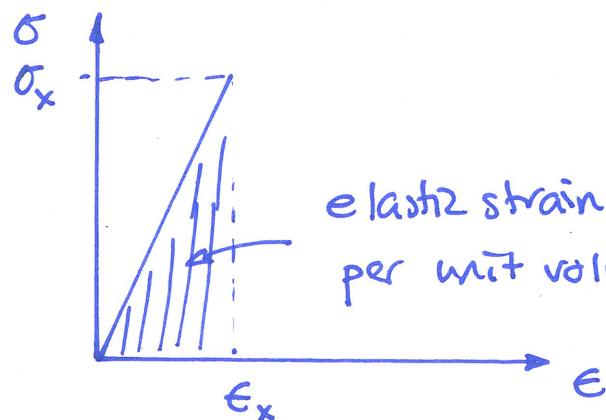
$$\text{average force} = \frac{\sigma_x dy dz}{2} \quad (\text{linear elastic})$$

- deformation,  $\delta = \epsilon_x dx$
- linear elastic,  $\sigma_x \propto \epsilon_x$
- work done =  $F_{\text{avg}} \delta = \frac{\sigma_x}{2} \epsilon_x dx dy dz = \text{internal elastic strain energy, } dU.$
- elastic strain energy per unit volume,  $U = \frac{dU_0}{dV} = \frac{\sigma_x \epsilon_x}{2}$

## Elastic strain energy, U

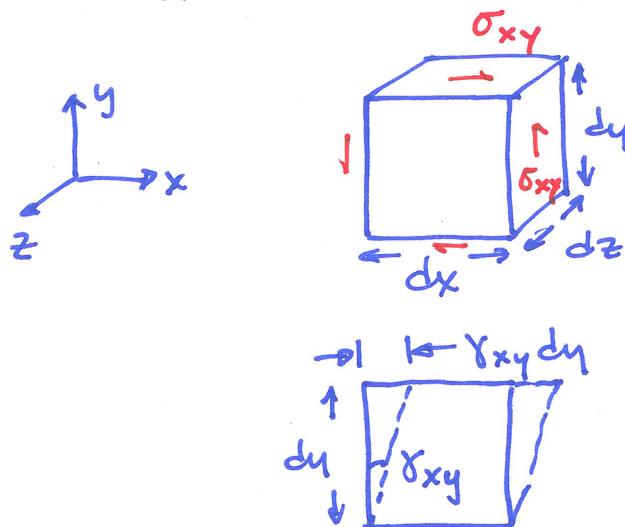
$$U = \frac{1}{2} \sigma_x \epsilon_x = \frac{\sigma_x^2}{2E} = \frac{1}{2} E \epsilon_x^2$$

analogous expressions for  
 $\sigma_y, \sigma_z$



elastic strain energy per unit volume = area under  $\sigma$ - $\epsilon$  curve

- Consider shear stress,  $\sigma_{xy}$ , applied to element

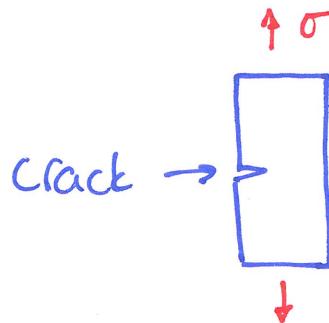


- average force on top plane =  $\frac{\sigma_{xy}}{2} dx dz$
- distance force moves =  $\gamma_{xy} dy$
- shear strain energy in element =  $\frac{\sigma_{xy} \gamma_{xy}}{2} dx dy dz$
- shear strain energy per unit volume

$$U = \frac{\sigma_{xy} \gamma_{xy}}{2} = \frac{\sigma_{xy}^2}{2G} = \frac{\gamma_{xy}^2 G}{2}$$

## Strain energy

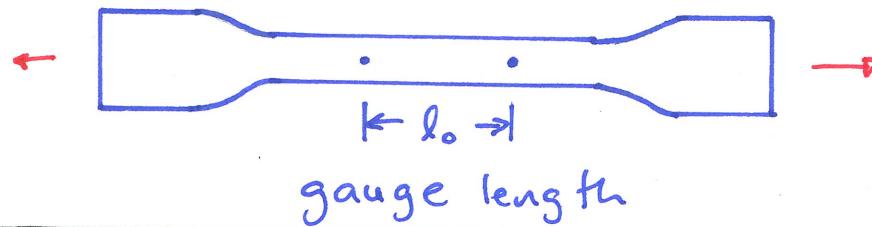
- in plasticity, dislocations create a stress field in the crystal lattice
- we will calculate the strain energy associated with the dislocation
- in fracture mechanics



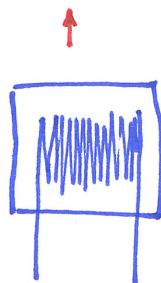
- for crack to grow, surface energy required to form new surfaces
- that energy is supplied by release of strain energy of material around the growing crack

## Stress-strain curves: test methods

- material behaviour characterized by stress-strain curves
- gives material properties independent of specimen dimensions (area, length)
- typical test: uniaxial tension on waisted specimen
  - e.g. "dog bone" or "waisted cylinder"

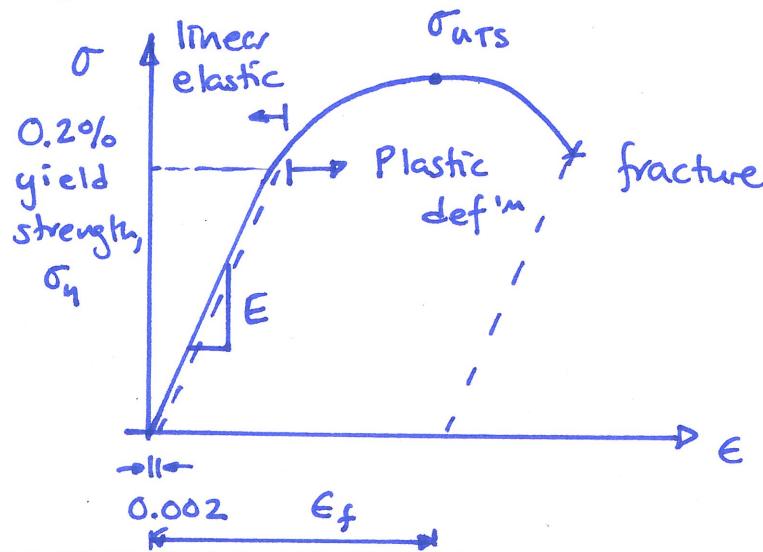


- deformation measured by extensometer, attached to specimen
- alternatively, can use strain gauge, glued onto specimen
  - as pull, wire stretches, area changes
  - resistance changes  $\rightarrow$  output voltage changes
  - strain gauge linear:  $\Delta V \propto \delta$
  - also used in load cells
- or, crudely, can use cross head displacement



## Typical stress-strain behaviour

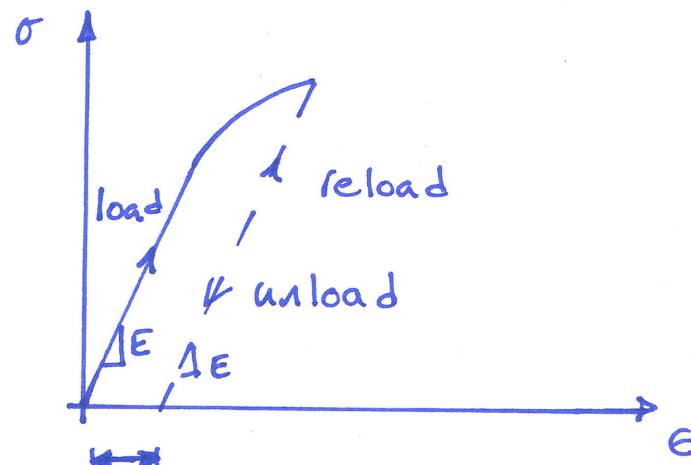
### (a) Metals



### Characteristic features:

- Young's modulus - linear elasticity
- $0.2\%$  offset yield strength,  $\sigma_y$
- $0.2\%$  permanent strain
- at yield, deformation is irrecoverable "plastic"
- ductile = material can deform to large plastic strains

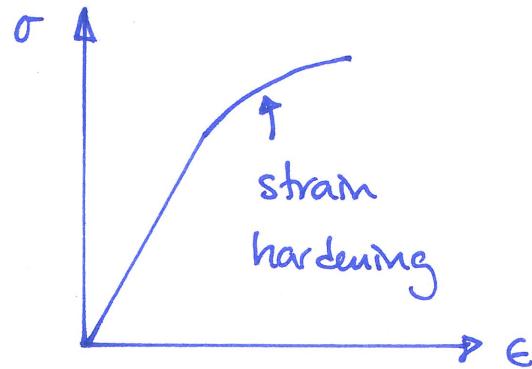
- plastic strain



$$\epsilon_p = \text{plastic strain} \\ (\text{permanent, irrecoverable})$$

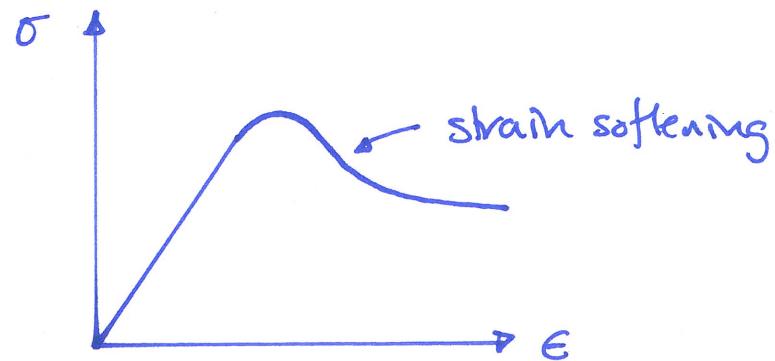
- on unloading, do not recover plastic strain
- recover elastic strain (with same slope as  $E$ )

### Strain hardening



- as strain increases in plastic regime, stress increases

### Strain softening



- as strain increases in plastic regime, stress decreases

### Ultimate tensile strength, $\sigma_{UTS}$

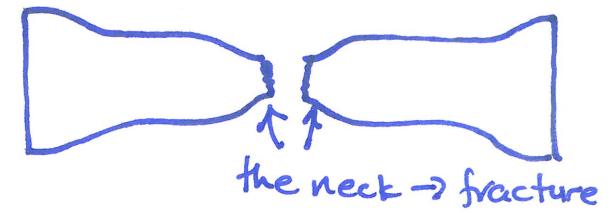
- Maximum stress reached in a uniaxial test

### Ductility, $\epsilon_f$

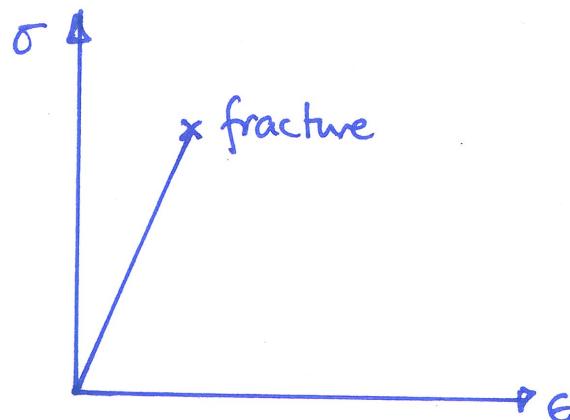
- plastic strain at fracture

## Necking

- as pull on some materials (e.g. ductile metals), plastic deformation becomes localized
- one section weaker, or has smaller area
  - stress increases, strain increases, area further reduced
  - stress increases again  $\Rightarrow$  instability
  - eventually, fracture
- called "necking"



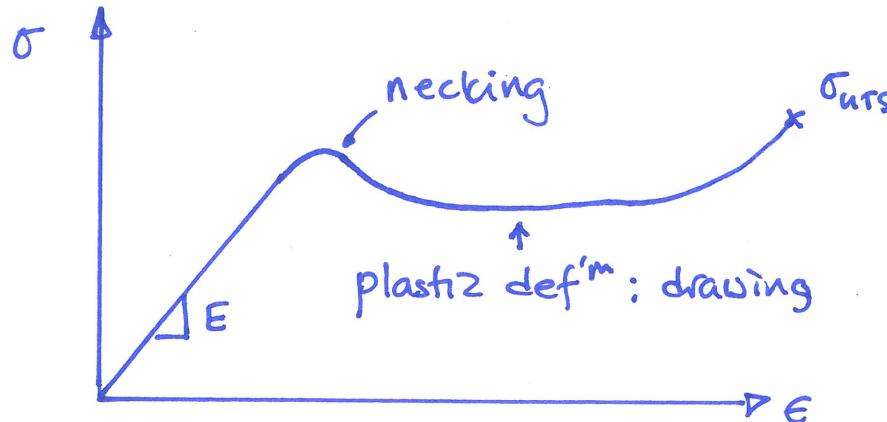
## Ceramics



- $E$  generally higher than metals
- brittle - fail suddenly at small strain (e.g. 0.001)

## Polymers

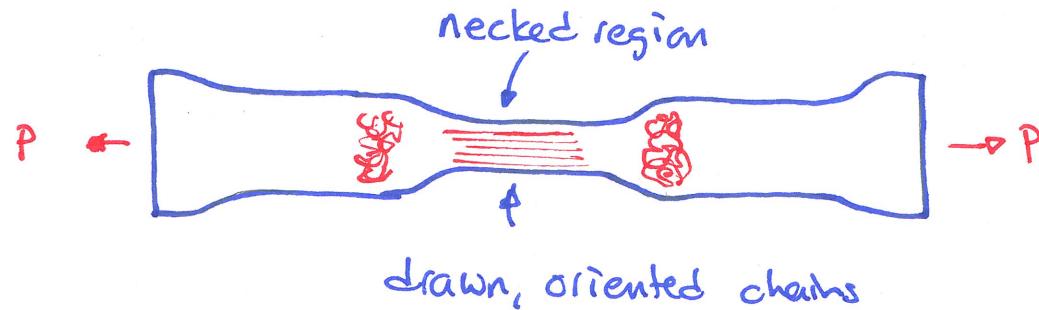
- Some polymers can be drawn



e.g. nylon, thermoplastics

- at local max. stress, neck forms
- but, as continue def'm, polymer chains become aligned, which strengthens the necked region
- if strengthening > area reduction, the neck propagates
- "drawing"

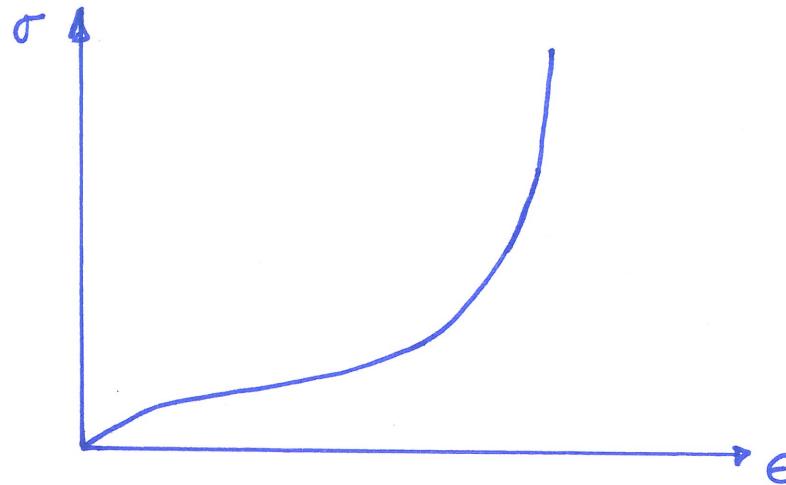
- drawing can extend over almost entire length of specimen
- $\epsilon$  of several hundred percent possible.



$\leftarrow$  waisted section  $\rightarrow$

## Rubbers

⑥



rubbers - polymer chains, occasional secondary cross-links

- at low  $\epsilon$ , chains slide over each other  $\Rightarrow$  low E
- as deform to large  $\epsilon$ , chains straighten
- as chains straighten, rubber stiffer; local slope increases

---

Above curves, for metals, ceramics, polymers, rubbers

all used: engineering stress,  $\sigma_{\text{eng}} = \frac{P}{A_0}$

engineering strain,  $\epsilon_{\text{eng}} = \frac{\delta}{l_0}$

Also have true stress + true strain

- useful for materials at large deformations
- used in plasticity.

## True stress + strain

$$\sigma_{\text{true}} = \frac{P}{A} \leftarrow \begin{matrix} \text{instantaneous area} \\ (\text{current}) \end{matrix}$$

relate  $\sigma_{\text{true}}$  to  $\sigma_{\text{eng}}$  &  $\epsilon_{\text{eng}}$

plastic deformation: Volume = constant

$$A_{l_0} = Al$$

↑              ↴  
initial values    instantaneous values

true strain

$$d\epsilon_{\text{true}} = \frac{dl}{l} \leftarrow \begin{matrix} \text{instantaneous (current)} \\ \text{length} \end{matrix}$$

$$\epsilon_{\text{true}} = \int_{l_0}^l \frac{dl}{l} = \ln \frac{l}{l_0} = \ln \frac{l}{l_0} \quad (\text{logarithmic strain})$$

$$\frac{l}{l_0} = \exp(\epsilon_{\text{true}}) = \frac{A_0}{A} = 1 + \epsilon_{\text{eng}}$$

$$\epsilon_{\text{true}} = \ln(1 + \epsilon_{\text{eng}}) \quad \text{for plasticity.}$$

$$\epsilon_{\text{eng}} = \frac{l - l_0}{l_0} = \frac{A_0}{A} - 1 ; \quad \frac{A_0}{A} = 1 + \epsilon_{\text{eng}}$$

$$\frac{\sigma_{\text{true}}}{\sigma_{\text{eng}}} = \frac{P}{A} \frac{A_0}{P} = \epsilon_{\text{eng}} + 1$$

$$\therefore \sigma_{\text{true}} = \sigma_{\text{eng}} (1 + \epsilon_{\text{eng}})$$

for plasticity.