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### A Fundamental Study on Particle Transport Through Rotary Dryers for Flight Design and System Optimisation

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**A FUNDAMENTAL STUDY ON PARTICLE TRANSPORT  
THROUGH ROTARY DRYERS FOR FLIGHT  
DESIGN AND SYSTEM OPTIMISATION**

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Key Words: differential calculus, flight geometry, rotary dryer,  
solid transport, optimisation

**ABSTRACT**

A model for particle transport in a flighted horizontal rotary dryer is developed in this paper. Mathematical principles applied to the current study are in the areas of differential calculus and analytical geometry. In contrast to the conventional approaches which are either based on empirical/semi-empirical correlations or obtained from the investigation of single particle trajectories, this paper develops rigorous mathematical analysis of the transport of bulk solids. A variety of important issues in rotary drying, such as axial flowrate of solids, retention time distribution and solid holdup are addressed and treated by using non-traditional methods. Since the model takes dimension, number and geometry of flights into account, it possesses the following two characteristics : (1) it is not only useful in the study of rotary drying dynamics, but also applicable to other processes employing flighted rotating cylinders (such as granulation drums and crushers); and (2) based on the model, an optimal drum configuration can be designed by using optimisation techniques. The model can be incorporated within a distributed parameter dryer model developed previously to form a more rigorous integrated dynamic model. A theoretical foundation for optimal flight design by using the current model is explained.

A pilot scale perspex rotary dryer equipped with a video camera has been constructed and used for model validation. Raw sugar was handled in the experiments. Particle transport was observed and measured by using a flow visualisation technique supplemented with traditional sampling methods. A significant model quality improvement has been observed through a comparative study between the newly developed model and conventional ones.

Flighted rotary drums are very widely used in the drying industry, amongst others. A cross sectional view of a typical flighted rotary dryer is schematically shown in Figure 1. A distributed parameter dryer model described by a set of partial differential equations has been developed recently by the authors (1 and 2). A key aspect in understanding the rotary dryer operation, is a knowledge of the solid transport behaviour, including retention time distribution, discharge rates from flights, axial solid flowrate, geometry and voidage of airborne curtains, and particle holdup. Consequently, a number of investigators such as Kelly et al. (3), Glikin (4), Kamke et al. (5), Matchett et al. (6), Gupta et al. (7) and Sherritt et al. (8) have been attracted by these topics. Notable developments have been reported in recent years but gaps in knowledge remain. These include:

(1) Rigorous mathematical derivations were not emphasised in many papers. For example, the following three defects can be detected in recent publications:

(a) The infinitesimal representation of discharge volume from a flight given by

$$dV = t^2 d\theta/2 \quad (1)$$

developed by Sherritt (8) will lead to considerable errors because of the geometry simplification. The Sherritt equation is correct for the auxiliary circle centred at the point C in Figure 2 rather than for the circle representing the drum wall.

(b) The average value approach (average falling angle and height) given by

$$\bar{\theta} = \frac{\int_0^{V_0} \theta dV}{V_0} ; \bar{h} = R_{eff} (1 + \sin \bar{\theta}) \quad (2)$$

is extensively used in particle transport simulations, but this is obviously inadequate where there are high dispersions about the average.

(c) Significant computational errors may occur by discretizing volume  $V$  in commonly used numerical schemes for the evaluation of integral equations (such as average angle representation in Equation (2)). This is because the variation of volume  $V$  may become a higher order infinitesimal than variation of angle  $\theta$  in some regions.

(2) A generalised approach handling flights with arbitrary geometry has not been developed.

(3) Particle transport and dryer dynamics were studied separately. A unified theory and algorithm to integrate these two stages has yet to be developed.

A generalised model for particle transport in flighted rotary dryers aimed at partly filling the identified gaps is developed in this work, by employing a rigorous mathematical analysis in areas of differential calculus and analytical geometry. This particle transport model is a constituent of the overall dynamic model described by partial differential equations.

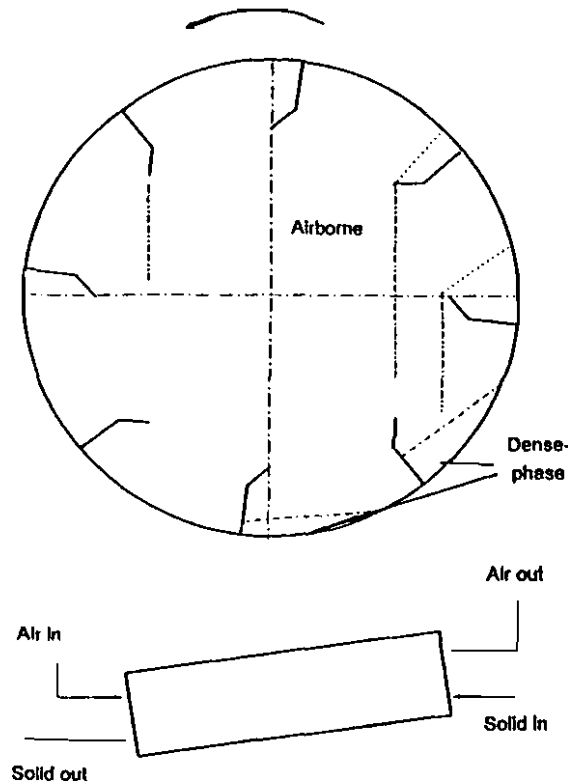


Figure 1: A Flighted Rotary Dryer

## SOLID FLOWRATE AND RETENTION TIME

A simplified cross sectional view of a rotary dryer is shown in Figure 3 in which  $\phi$  is denoted as the kinetic angle of repose and the angle  $\theta$  determines the position of a flight. If both  $\phi$  and  $\theta$  are specified, the cross sectional area filled with particles in the flight is computable by using analytical geometry for irregularly shaped flights or plane geometry for regularly shaped ones. Consequently, the solid holdup in the flight is uniquely determined in terms of the flight geometry, position and kinetic angle of repose. Normally at least three parameters are required to describe a flight with simple geometry: two side lengths  $h$  and  $d$ , and angle  $\alpha$  formed by these two sides (Figure 3). The kinetic angle of repose  $\phi$  is a function of the static friction coefficient  $\mu$ , position  $\theta$  and angular velocity  $\omega$ .

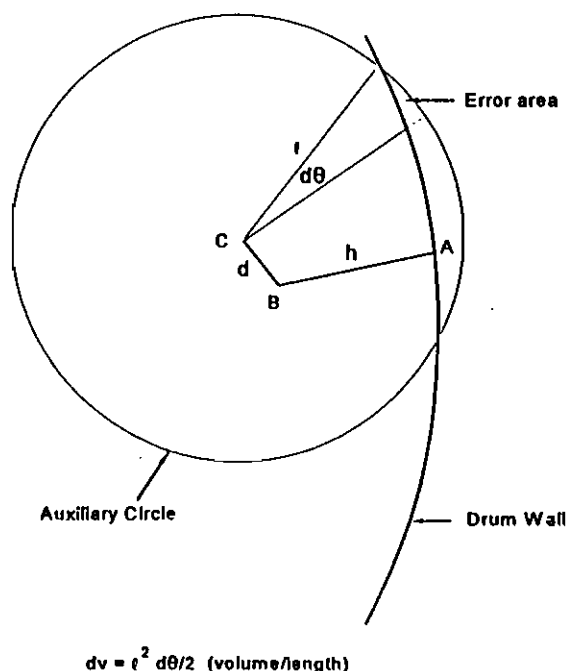


Figure 2: Error Detection for Simplified Method

The following correlation is well regarded and widely accepted (4 and 10):

$$\tan \phi = \frac{\mu - \mu (\omega^2 R/g) \sin \theta + (\omega^2 R/g) \cos \theta}{1 - \mu (\omega^2 R/g) \cos \theta - (\omega^2 R/g) \sin \theta} \quad (3)$$

That is, if the flight geometry is specified, the cross sectional area of a flight filled with solids is a composite function of  $\theta$  ( $S = S(\theta, \phi(\theta))$ ). The correlation given by Equation (3) has been validated for sugar particles (10) which will be used in an application example reported in this work. In optimisation studies, the number, dimensions and geometry of flights are treated as parameters to be determined by using mathematical programming. In this case, the cross section area  $S$  should be represented as  $S = S(\theta, \phi(\theta); h, d, \alpha)$ .

When the drum is rotating, the infinitesimal angle and time are related by:

$$d\theta = \omega dt \quad (4)$$

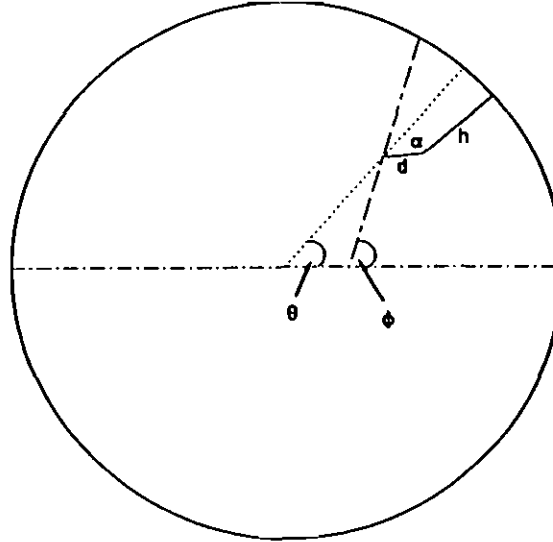


Figure 3: View of A Simplified Rotary Drum

The particle discharge rate from the  $i$ th flight is then given by:

$$R_{di} = -\rho_b \omega \frac{dS_i}{d\theta_i} = -\rho_b \omega \left( \frac{\partial S_i}{\partial \theta_i} + \frac{\partial S_i}{\partial \phi_i} \frac{\partial \phi_i}{\partial \theta_i} \right) \quad (5)$$

The axial solid flowrate can be obtained by using the discharge rate in the vertical direction given by Equation (5). Figure 4 shows the side view of a dryer. The particle transport is illustrated by two disks (Disks 1 and 2) displayed in the figure, with one (Disk 1) perpendicular to the drum axis and the other vertical (Disk 2). The infinitesimal slices used in the development of heat and mass transfer equations (1 and 2) are perpendicular to the drum axis. It is easy to see from Figure 4 that the far extreme points in flights delivering particles to the next slice bounded by Disk 1 are on Disk 2. That is, the mass passing through Disk 1 is determined by the amount of particles discharged from the segments of flights between Disks 1 and 2 with the lengths  $L_i$ ,  $i = 1, 2, \dots, n$ . In the example shown in Figure 4, since there are four active flights, the lengths  $L_1$ ,  $L_2$ ,  $L_3$  and  $L_4$  should be determined. The axial solid flowrate consists of both the airborne particle delivery (particles penetrating the inclined Disk 1) and the dense phase kilning. If the axial drag force is negligible, the solid flowrate is given by:

$$V_s = \Gamma \sum_1 (R_{di} L_i) = \Gamma \omega \rho_b \sum_1 \left( \frac{dS_i}{d\theta_i} L_i \right) \quad (6)$$

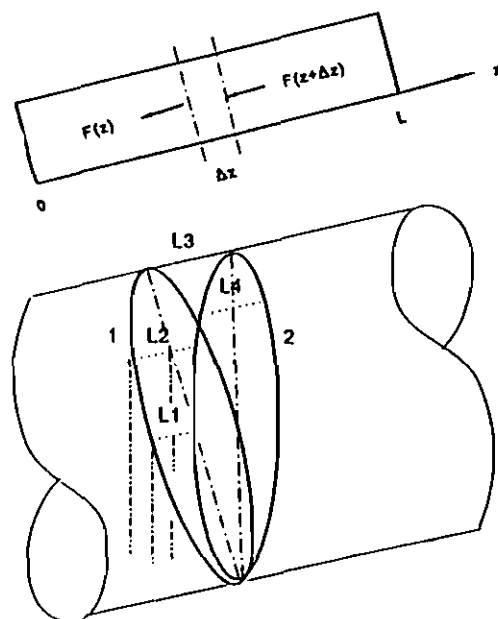


Figure 4: Side View of Rotary Dryer

where  $\Gamma \geq 1$  is a correction factor taking the kilning effect into account, and the length  $L_1$  in the  $i$ th flight delivering particles to the next infinitesimal slice can be computed as follows:

$$L_1 = h_i \tan \gamma \quad (7)$$

A more rigorous mathematical treatment on the kilning effect is continuing which will be reported in our subsequent papers. It should be pointed out that the height  $h_i$ , defined as the distance between the discharging point from the  $i$ th flight and the bottom dense phase, must also be computed dynamically. A commonly used simple equation given by  $h_i = R_{\text{eff}}(1 + \sin \theta_i)$  is a poor approximation in which the shape complexity of the dense phase is not taken into account. When axial drag force is not negligible, the following equations can be used to correct the length  $L_1$ :

$$\begin{aligned} \bar{L}_1 &= L_1 - \frac{1}{2} a t_1^2 \\ t_1 &= \sqrt{\frac{2h_i}{g}} \\ a &= C_D \frac{\pi \rho_s V^2 D^2}{8m_p} \end{aligned} \quad (8)$$

where the drag coefficient can be determined by using correlations explained in Denn (11).

The solid transport model developed this way is incorporated with the heat and mass transfer models developed earlier by the authors (1 and 2). This forms a combined distributed parameter model. All the distributed variables can be computed more accurately by using the combined model than previously done. For example, the solid phase mass balance given by:

$$\frac{\partial M_s}{\partial t} = \frac{\partial V_s}{\partial z} \quad (9)$$

can be used to compute the solid holdup in combination with the axial solid flowrate as explained above. From Equation (9), a mathematical representation of the retention time is derived as:

$$t_r(t) = \int_0^L \frac{\partial M_s}{\partial V_s} dz \quad (10)$$

Since the axial solid flowrate can be computed rigorously, the retention time described by Equation (10) is not explicitly used in dynamic simulations. It is useful in the estimation of operational conditions by practitioners and in the equipment design.

In this section, most of the derivations are done using differential calculus. Symbolic mathematics software, such as MAPLE, MATHEMATICA, MATLAB with Symbolic Math Toolbox, and Calculus Calculator (CC4) is needed for the calculations, in particular for solving Equation (5).

#### AIRBORNE HOLDUP

In our previous work, the airborne holdup was determined by using the Hironue's correlation (2) which was originally developed for some special cases. More recently, Sherritt (8) proposed a computational method in which the total airborne holdup was computed as the product of the average airborne holdup for one flight and the total number of flights. As pointed out previously, considerable errors can be incurred by using an average value approach. A method for accurate computation of airborne holdup should be developed.

Since the particle discharge rate from a flight at any position is computable by using the techniques developed in the previous section, the instantaneous total airborne holdup can be obtained as follows:

$$G_s = \sum_1 R_{D1} t_1 \quad (11)$$

where the particle falling time,  $t_i$  from the  $i$ th flight is given in Equation (8). This assumes



that (a) the initial vertical velocity is zero; (b) the vertical drag force is negligible; and (c) there is no hindrance from other particles. If Assumptions (a) and (b) are not valid, the following equation should be used:

$$h_1 = v_0 t_1 + \frac{1}{2} (g - a_v) t_1^2 \quad (12)$$

where  $a_v$  is computed in a similar way as the last equation in Equation (8) and the extreme values of the initial vertical velocity  $v_0$  will be determined in the next section.

### CURTAIN GEOMETRY AND VOIDAGE

The size and shape of curtains formed by discharged particles from flights are determined by using the extreme value theory based on an analysis of single particles in extreme conditions. The extreme single particle motions are shown schematically in Figure 5. The maximum initial velocity of a single particle from a flight is the result of the angular velocity and a sliding/rolling velocity along the solid surface in the flight. This resultant velocity can be further decomposed into horizontal and vertical velocities,  $v_h$  and  $v_v$ , described by the following equation:

$$\begin{aligned} v_h &= \omega R_t \sin \theta + \sqrt{2gl(\sin \phi - \zeta \mu \cos \phi)} \cos \phi \\ v_v &= -\omega R_t \cos \theta + \sqrt{2gl(\sin \phi - \zeta \mu \cos \phi)} \sin \phi \end{aligned} \quad (13)$$

where  $R_t$  is the distance from the centre of the drum to the tip of a flight and the parameter  $\zeta \leq 1$  is determined based on the relative importance between sliding and rolling motions. The width of the curtain at any specified vertical distance  $h_i$  from the flight lip is given by:

$$\begin{aligned} W_c &= v_h t_v \\ v_v t_v + \frac{1}{2} g t_v^2 &= h_i \end{aligned} \quad (14)$$

The average voidage of the  $i$ th curtain,  $\epsilon_i$ , is described as:

$$\epsilon_i = \frac{A - \frac{R_{di} t_i}{\rho_s}}{A} = 1 - \frac{R_{di} t_i}{\rho_s A} \quad (15)$$

where  $\rho_s$  is the actual solid density,  $t_i$  is the particle falling time from the  $i$ th flight and  $A$  is the cross section area of the  $i$ th curtain. The equations formulated in this section will be validated against the experimental data.

### OUTLINE OF A POSSIBLE OPTIMISATION STRATEGY

Since the number and geometry of flights are considered in the development of the mathematical models for particle transport in rotary dryers, the models can be used in the design of optimal dryer configurations. A detailed description of system optimisation by

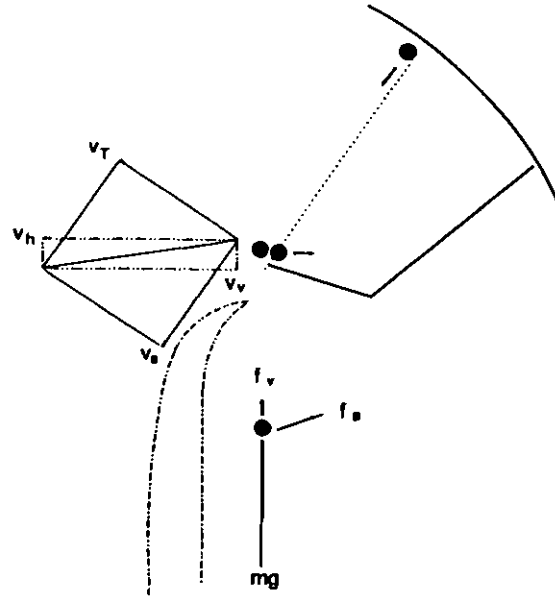


Figure 5: Curtain Geometry Analysis—  
Extreme Value Theory

using the current models exceeds the scope of this paper. Only the principles used in the system optimisation are outlined. As pointed out previously, the discharge rate  $R_{Di}$  from the  $i$ th flight is not only a function of position  $\theta_i$  but also a function of the flight configuration in terms of  $h$ ,  $d$  and  $\alpha$ . Consequently, the width of the  $i$ th curtain,  $W_{ci}$ , given by Equation (14) is also a function of  $h$ ,  $d$  and  $\alpha$ . In practice, the active heat and mass transfer volume in the dryer should be as large as possible to achieve maximum drying efficiency. The constrained optimisation problem can be formulated as follows:

$$\text{Maximize}_{h,d,\alpha} \left\{ \sum_{i=1}^n (W_{ci}(\theta_i; h, d, \alpha) + w_i R_{Di}(\theta_i; h, d, \alpha)) \right\} \quad (16)$$

subject to the constraints given by Equations (5) and (14). The active flight number  $n$  in Equation (16) depends on the total flight number  $N$ . The optimisation problem can be solved by employing a mathematical programming approach with four decision variables:  $h$ ,  $d$ ,  $\alpha$  and  $N$ .

#### PILOT PLANT CONFIGURATION

A pilot scale flighted rotary dryer equipped with a video camera, video recording system and television has been built to validate the model. An air chamber and a fan are also

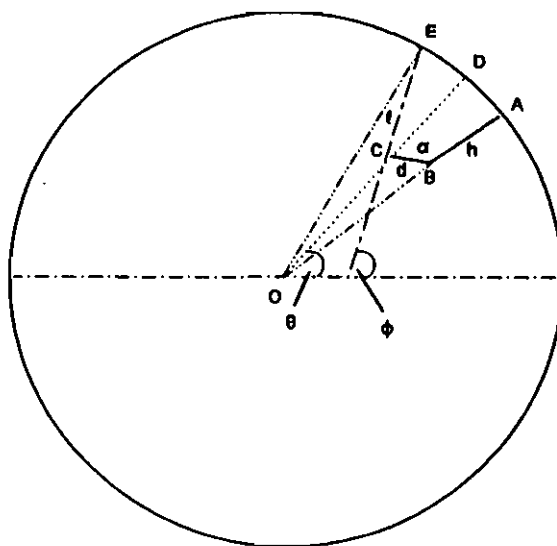


Figure 6(a): Cross Section Area at Stage 1

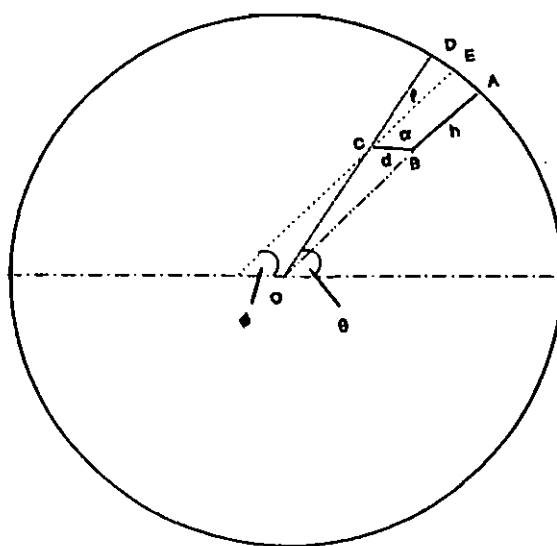


Figure 6(b): Cross Section Area at Stage 2



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connected to the dryer to introduce air flow through the drum. Raw sugar is used as the testing material. The pilot dryer parameters are as follows (refer to Figure 3):

$D = 0.3\text{m}$ ,  $L = 1.75\text{m}$ ,  $h = 0.03\text{m}$ ,  $d = 0.02\text{m}$ ,  $\alpha = 105^\circ$ , Number of Flights = 10  
 Rotation Rate = 0 to 16 rpm, Inclination Angle ( $\gamma$ ) =  $0^\circ$  to  $8^\circ$   
 Friction Coefficient of Sugar =  $\tan 40^\circ$  to  $\tan 60^\circ$  (from (10))  
 Air velocity = 0 to 1.5 m/s

A more detailed description on the pilot plant design and experimental procedures is available elsewhere (12 and 13).

A time consuming step in the application of techniques developed in this paper is to formulate representations of cross section areas filled with particles in order to use Equation (5). The flight design shown in Figure 6 is used to demonstrate the computational procedures. The following three cases may be distinguished in computations:  $0 \leq \theta \leq \phi$ ;  $\phi \leq \theta \leq \theta_1$ ; and  $\theta_1 \leq \theta \leq \theta_r$ , where  $\theta_1$  represents the bounding angle at which the shape of cross section filled with solids, changes from a quadrilateral to a triangle (Figure 6(c)), and  $\theta_r$  is the final angle at which the particles are completely discharged. The most important parameter to be determined is the side length  $\ell$  (CE in Figure 6). In Stage 1 (Figure 6(a)),  $\ell = CE$  is computed by using:

$$\overline{CE}^2 + 2 \overline{CO} \overline{CE} \cos(\phi - \theta) + (\overline{CO}^2 - R^2) = 0 \quad (17)$$

where

$$\overline{CO}^2 = \overline{BC}^2 + (R - \overline{AB})^2 + 2\overline{BC}(R - \overline{AB}) \cos \alpha \quad (18)$$

The cross section area filled by particles in Stage 1 is given by:

$$S = \frac{1}{2} R^2 \Delta AOE - \text{area}(\Delta COB + \Delta COE) \quad (19)$$

where

$$\begin{aligned} \text{area}(\Delta COB) &= \frac{1}{2} \overline{CE} \overline{CO} \sin(\Delta OCE) \\ \text{area}(\Delta COE) &= \frac{1}{2} \overline{BC} \overline{CO} \sin(\Delta OCB) \end{aligned} \quad (20)$$

In Stage 2 (Figure 6(b)),  $\ell = CE$  is determined as:

$$\overline{CE}^2 + 2 \overline{CO} \overline{CE} \cos(\phi - \theta) + (\overline{CO}^2 - R^2) = 0 \quad (21)$$

The shifting angle  $\theta_1$  (Figure 6(c), E coincides with A) is computable by using the following equation:

$$\overline{AB}^2 + \overline{BC}^2 - 2 \overline{AB} \overline{BC} \cos \alpha = -[2 \overline{AC} \overline{CO} \cos (\theta - \phi) + (\overline{CO}^2 - R^2)] \quad (22)$$

The area filled with particles in Stage 2 is represented by:

$$S = \left[ r (r - \overline{CE}) (r - \overline{CO}) (r - R) \right]^{1/2} + \frac{1}{2} R^2 \Delta AOE - \text{area} (\Delta BOC) \quad (23)$$

where

$$r = \frac{1}{2} (\overline{CE} + \overline{CO} + R) \quad (24)$$

$$\Delta AOE = \Delta AOD - \arcsin (\overline{CE} \sin (\theta - \phi) / R)$$

Stage 3 is shown in Figure 6(d) with side length  $\ell = CE$  described by:

$$\ell = \overline{CE} = \frac{\overline{BC} \sin \alpha}{\sin (\theta - \phi - \Delta AOC)} \quad (25)$$

where

$$\sin (\Delta AOC) = \frac{\overline{BC} \sin \alpha}{\sqrt{\overline{BC}^2 + (R - \overline{AB})^2 + 2 \overline{BC} (R - \overline{AB}) \cos \alpha}} \quad (26)$$

The area filled with particles in Stage 3 is given by:

$$S = \frac{1}{2} \overline{CE} \overline{BC} \sin (\Delta BCE) \quad (27)$$

where

$$\Delta BCE = \pi - (\alpha + (\theta - \phi - \Delta AOD)) \quad (28)$$

Since the cross sectional areas filled with particles at various stages can be determined rigorously, the discharge rate from a flight at any position can be computed by using Equation (5) with the aid of suitable symbolic mathematical software. In this work, the CC - Calculus Calculator software (14) is employed.

## EXPERIMENTS AND MODEL VALIDATION

In the experiments using the pilot plant, the following variables were measured: kinetic angle of repose, flight discharge rate, solid axial flowrate, air velocity, average retention time and retention time distribution, and airborne curtain width. The video recorder system and tracers (glass beads and coloured sugar) supplemented by the conventional weighting techniques were employed for the measurements. The detailed experimental procedures have been well documented (12 and 13).

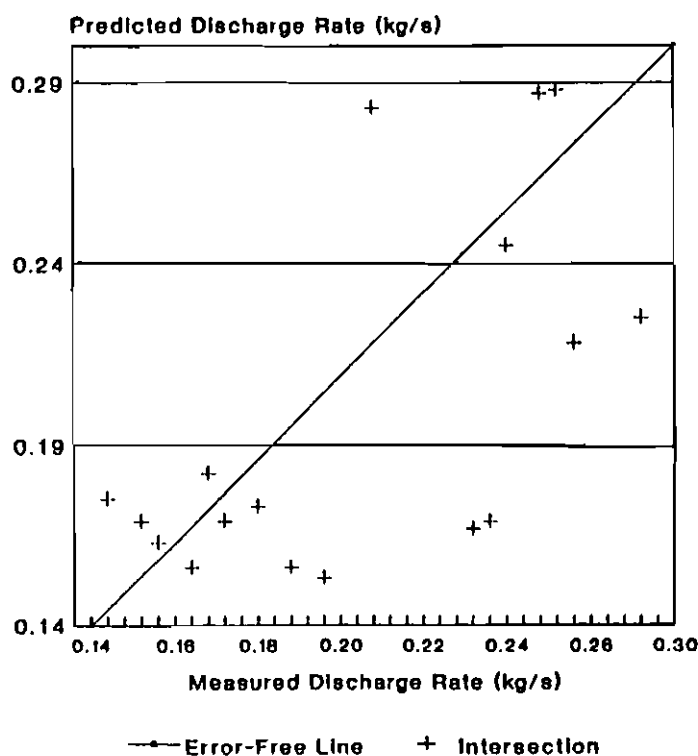


Figure 7: Prediction vs. Measurement  
(Discharge Rate)

The heart of the model is the flight discharge rate given by Equation (5). It is used for the determination of a variety of important system variables such as solid axial flowrate, airborne solid holdup, voidage and retention time. The computed flight discharge rates are compared with the measured ones in Figure 7. Since the deviations were caused by both the model mismatch as well as the inaccurate measurements (overall measurement error is as high as 33%), the model accuracy could be regarded as acceptable.

The computed retention times are close to, but about 10% longer than the average measured retention times which can be seen from Figure 8. The major factors leading to this error seem to come from bouncing and kilning, which have so far not been rigorously considered in the model. Only a correction factor  $\Gamma$  was introduced to try and handle the bouncing and kilning effects. Obviously, if these two actions are significant, more rigorous methods should be investigated. Further theoretical development and experimental validation work is continuing.

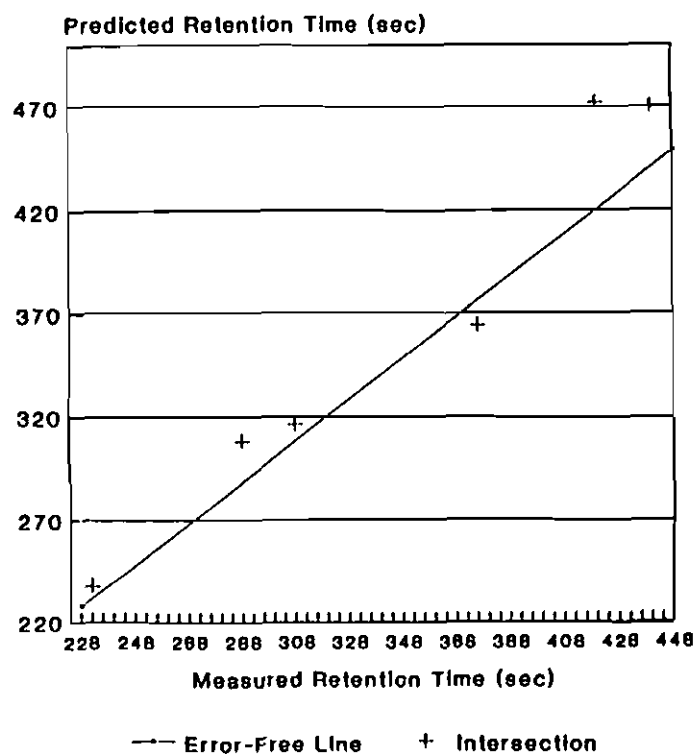


Figure 8: Prediction vs. Measurement  
(Retention Time)

Table 1: Computed and Measured Curtain Widths

Rotation Rate (rpm)	Flight Position $\theta$ (deg)	Falling Height $h_f$ (mm)	Computed Width $W_c$ (mm)	Measured Width $W_m$ (mm)
3	50	107	16.5	15
3	80	107	20.5	22
3	115	107	15.7	13
4.5	50	110	18.8	17
4.5	80	110	23.4	23
4.5	100	110	20.7	25



The computed and measured widths of curtains at various positions and operational conditions are listed in Table 1. A good agreement has been achieved which has confirmed the validity of the application of the extreme value theory. However, it has been observed from the video pictures that the further the position away from a curtain centre, the lower the particle density. That is, the solid density is not uniform within a curtain. This observation has certain implications in rotary drum behaviour.

### CONCLUSIONS

The rigorous mathematical model developed in this paper provides an improved prediction of particle transport in rotary dryers in terms of flight discharge rate, retention time, airborne and dense phase solid holdup, particle curtain geometry and voidage, and solid flowrate. The model can be integrated into an overall dynamic model described by a set of partial differential equations developed earlier by the authors (1 and 2). Consequently, a unified theory can be established for investigation of rotary drying dynamics. The model is not only useful in the dynamic study of rotary drying processes, but also provides a sound basis for design of optimal drum configurations by using optimisation techniques. A number of important modelling issues have been verified by using experimental data. The model is in the process of being further validated against more accurate measurements.

Additional studies on bouncing and kilning, particle density distribution within a curtain, shielding effects, and drag forces are needed to address the deficiencies in the current approach.

### ACKNOWLEDGMENT

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### NOTATION

$a$	acceleration, m/sec <sup>2</sup>
$C_D$	drag coefficient
$D_p$	particle diameter, m
$d$	side length of flight, m
$G_s$	airborne holdup per unit length, kg/m
$g$	gravitational acceleration, m/sec <sup>2</sup>
$h$	side length of flight, m; particle falling height, m
$h_i$	particle falling height from the $i$ th flight, m
$h_s$	specified falling distance, m
$L$	dryer length
$L_i$	length in the $i$ th flight delivering particle to next slice, m
$M$	mass holdup per unit length, kg/m
$m_p$	single particle mass, kg
$n$	number of active flights
$N$	total number of flights
$R$	dryer radius, m
$R_{eff}$	effective dryer radius, m
$R_D$	particle discharge rate from flight with unit length, kg/sec m

$R_e$	Reynold number
$R_i$	radius of inner circle (flight-free space), m
$S$	cross sectional area of flight filled with solids, $m^2$
$t_i$	particle falling time from the $i$ th flight, sec
$t_r$	retention time, sec; min
$t_s$	particle falling time to reach the specified position, sec
$V$	volume filled with particle in flight, $m^3$
$V_p$	velocity, m/sec
$V_s$	solid flowrate, kg/sec
$v_h$	horizontal velocity, m/sec
$v_v$	vertical velocity, m/sec
$W_c$	computed curtain width, mm
$W_m$	measured curtain width, mm
$w_i$	weighting factor
$z$	spatial variable, m
$l$	side length of solids in flight, m
$\alpha$	flight angle, deg
$\theta$	flight position angle, rad
$\phi$	angle of repose, rad
$\gamma$	drum slope to horizontal, deg
$\epsilon$	voidage
$\Gamma$	correction factor
$c$	sliding-rolling ratio
$\mu$	coefficient of static friction
$\rho_b$	bulk density, $kg/m^3$
$\rho_s$	solid density, $kg/m^3$
$\omega$	angle velocity, rad/sec

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