



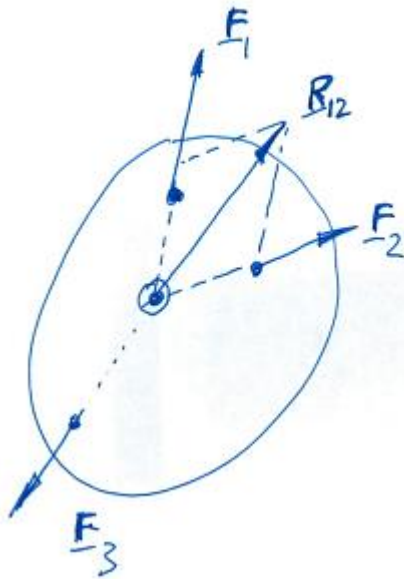
Lecture 7

Equilibrium of Rigid Bodies (3D)



Three-force member

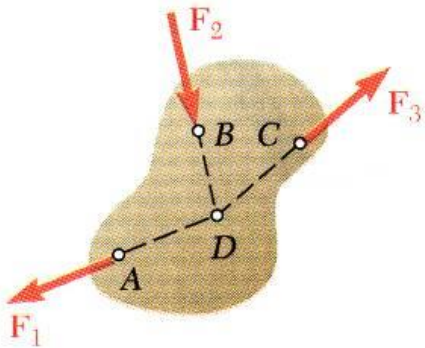
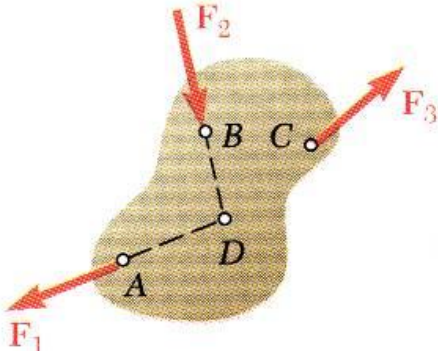
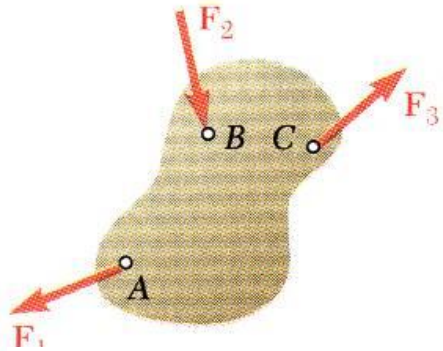
For three non-parallel forces acting on a rigid body, the line of action of the three forces must intersect at one common point.



Why ?

because the resultant of F_1 and F_2 , R_{12} , will form a two-force system with the remaining force F_3 .

Equilibrium of a Three-Force Body



- Consider a rigid body subjected to forces acting at only 3 points.
- Assuming that their lines of action intersect, the moment of F_1 and F_2 about the point of intersection represented by D is zero.
- Since the rigid body is in equilibrium, the sum of the moments of F_1 , F_2 , and F_3 about any axis must be zero. It follows that the moment of F_3 about D must be zero as well and that the line of action of F_3 must pass through D .
- The lines of action of the three forces must be **concurrent** or parallel.

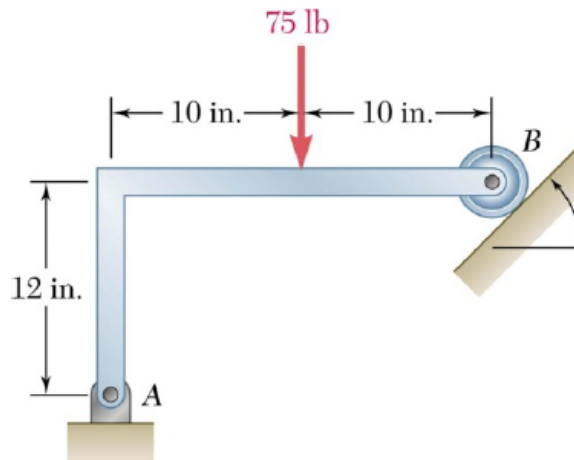
Definition:

If three **non-parallel** forces act on a rigid body in equilibrium, it is known as a three-force member.

Three-force member principle

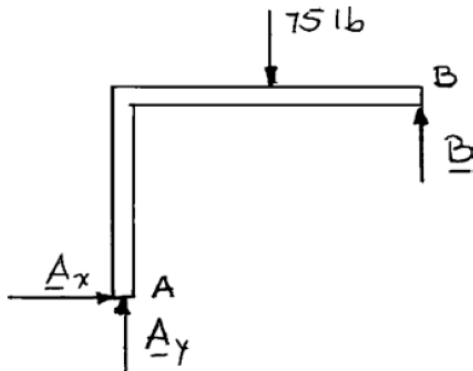
If a three-force member is in equilibrium, the line of action of all three forces must intersect at a common point; and the total resultant is zero. In other words, any single force is the equilibrant of the two other forces.

PROBLEM 4.13

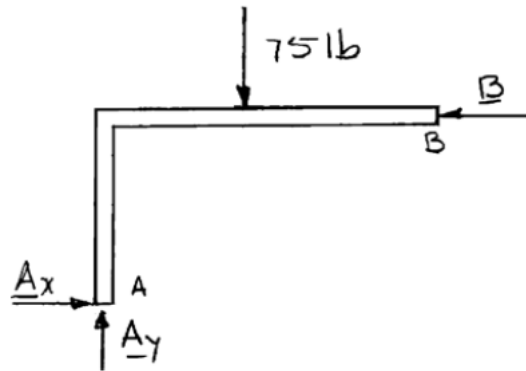


Determine the reactions at A and B when (a) $\alpha = 0$, (b) $\alpha = 90^\circ$, (c) $\alpha = 30^\circ$.

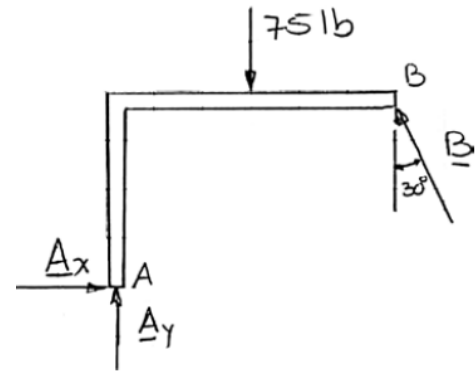
$$\alpha = 0$$



$$\alpha = 90^\circ$$



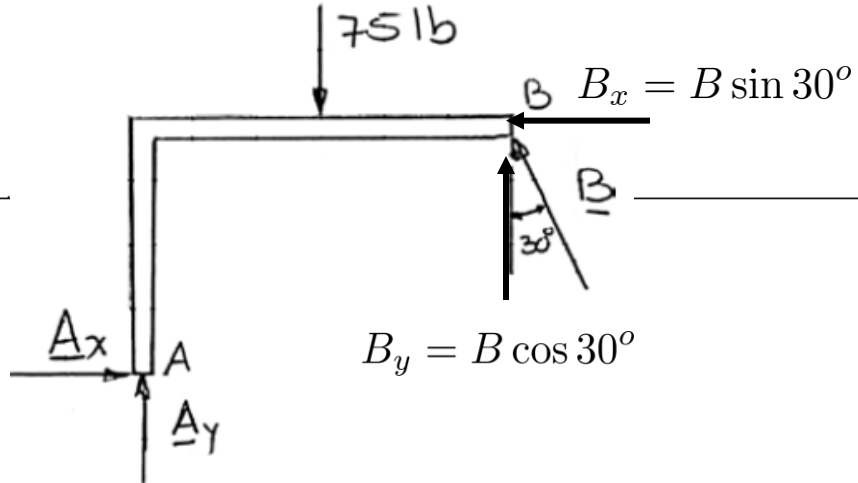
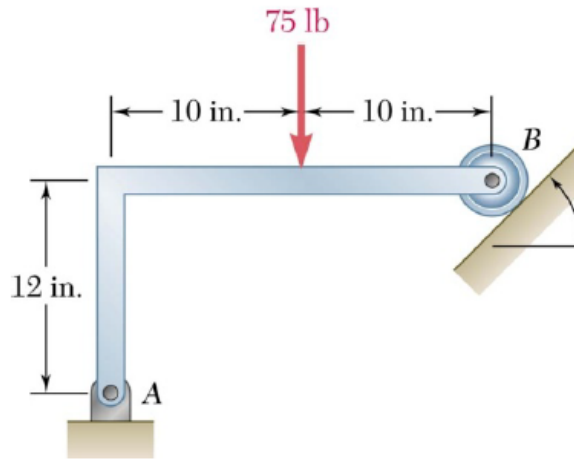
$$\alpha = 30^\circ$$



PROBLEM 4.13

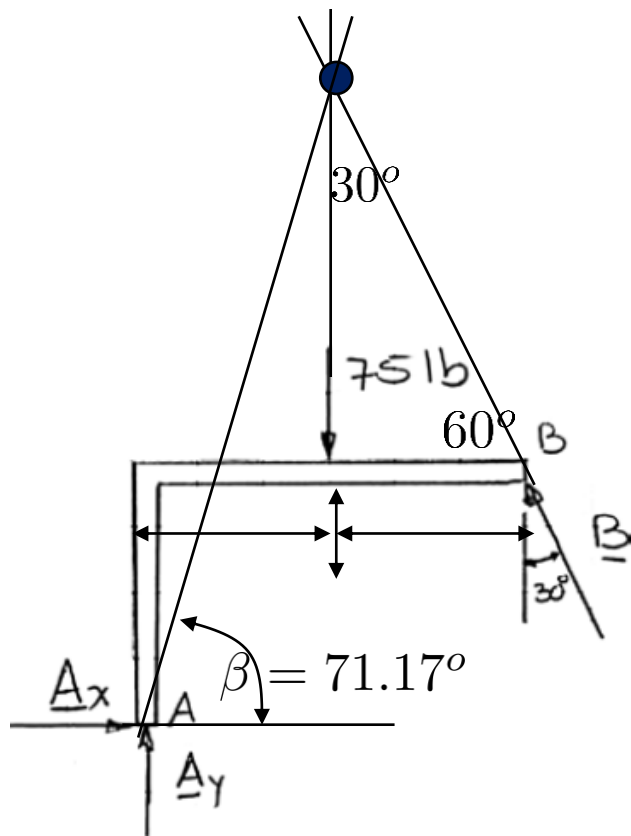
Determine the reactions at A and B when (a) $\alpha = 0$, (b) $\alpha = 90^\circ$, (c) $\alpha = 30^\circ$.

$$\alpha = 30^\circ$$

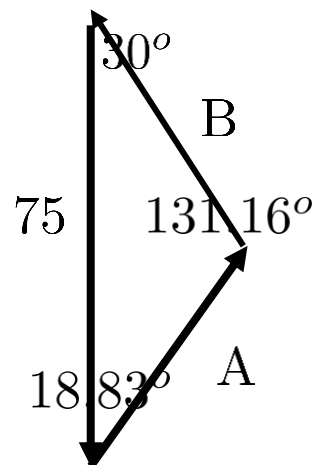


$$\sum M_A = 0 \rightarrow 12B_x + 20B_y - 75 \times 10 = 0$$

$$B = \frac{750}{12 \sin 30^\circ + 20 \cos 30^\circ}$$



$$\tan \beta = \frac{10 \tan 60^\circ + 12}{10} = 2.932$$



$$\frac{75}{\sin 90.57^\circ} = \frac{B}{\sin 59.43^\circ} = \frac{A}{\sin 30^\circ}$$

Equilibrium of a Rigid Body in Three Dimensions

$$\sum \vec{F} = 0 \quad \sum \vec{M}_O = \sum (\vec{r} \times \vec{F}) = 0$$

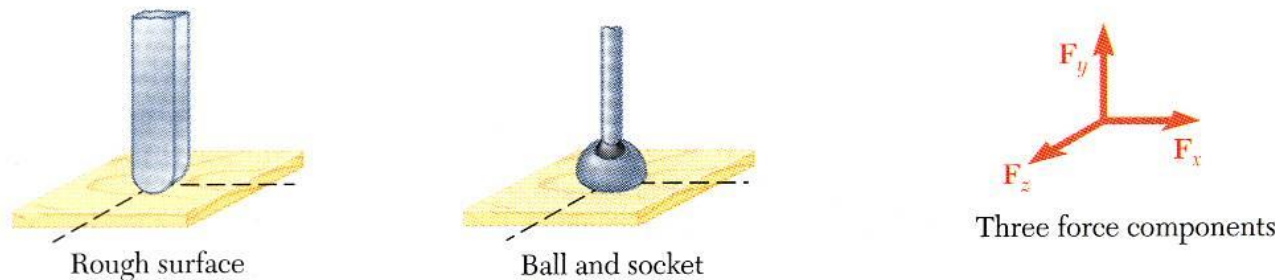
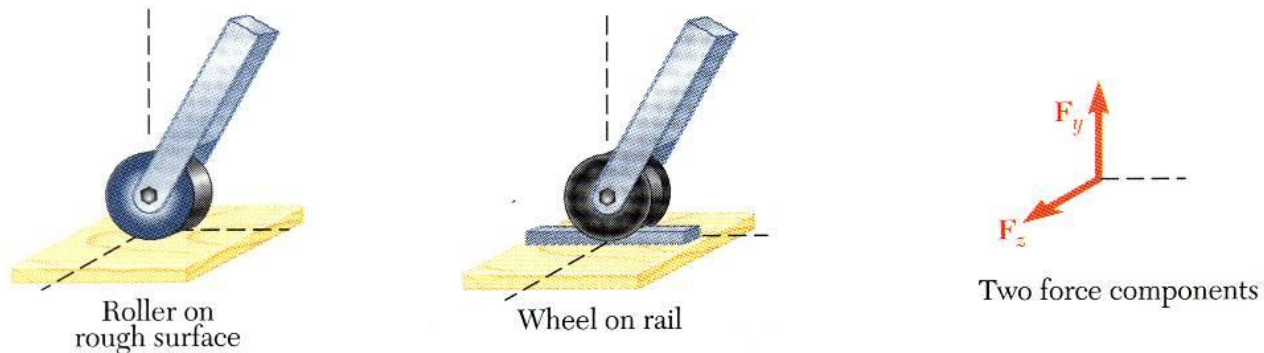
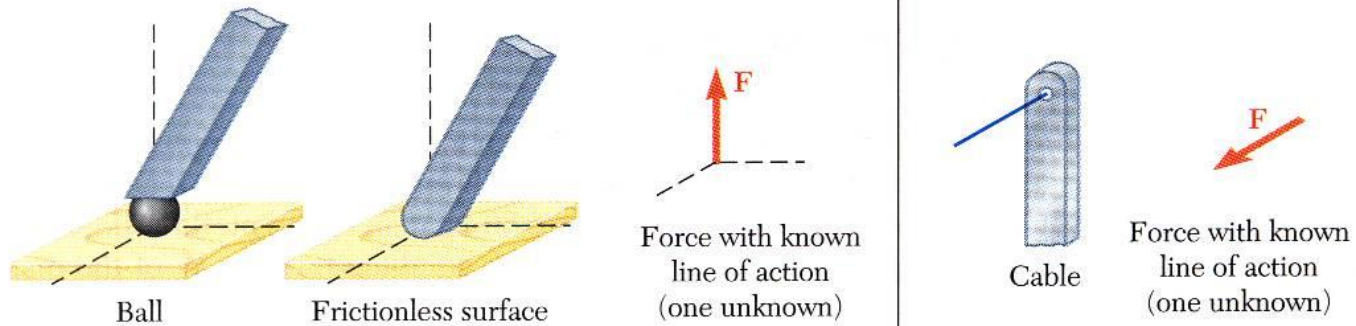
- Six scalar equations are required to express the conditions for the equilibrium of a rigid body in the general three dimensional case.

$$\begin{array}{lll} \sum F_x = 0 & \sum F_y = 0 & \sum F_z = 0 \\ \sum M_x = 0 & \sum M_y = 0 & \sum M_z = 0 \end{array}$$

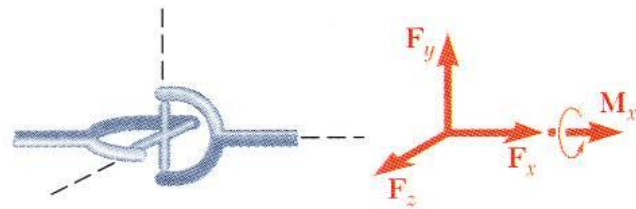
- These equations can be solved for no more than 6 unknowns which generally represent reactions at supports or connections.

Today's Lecture Attendance Password is: 3D Problem

Reactions at Supports and Connections for a 3D Structure



Reactions at Supports and Connections for a 3D Structure



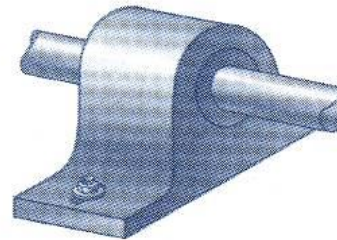
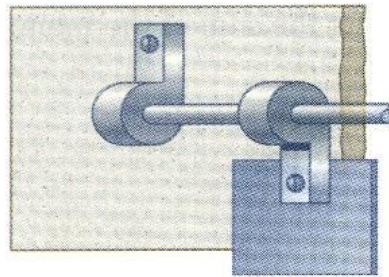
Universal joint

Three force components and one couple

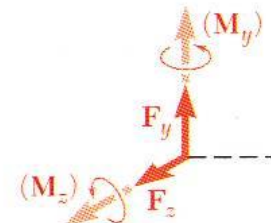


Fixed support

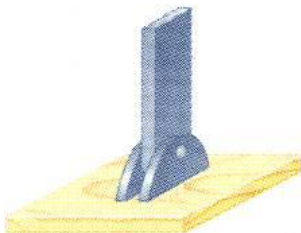
Three force components and three couples



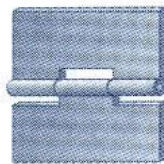
Hinge and bearing supporting radial load only



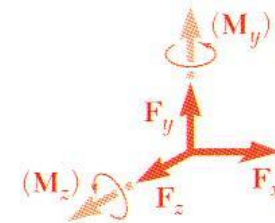
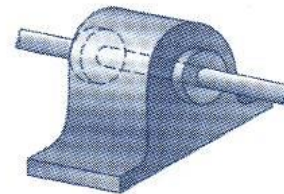
Two force components (and two couples)



Pin and bracket

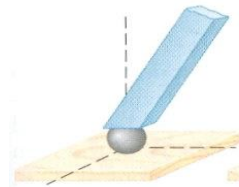
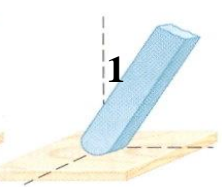
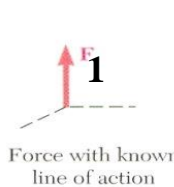
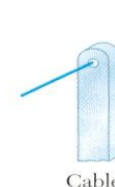


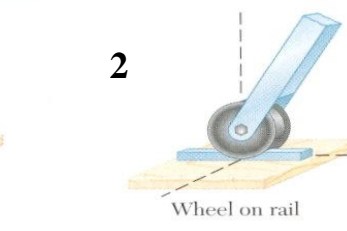
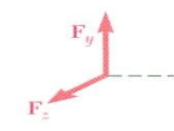
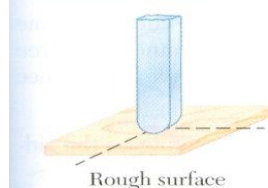
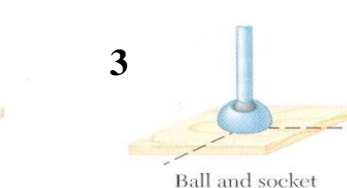
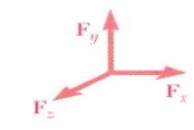
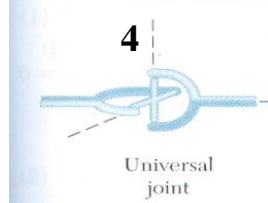
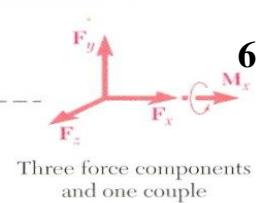
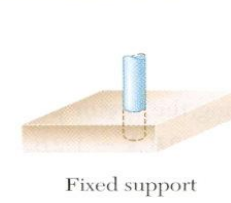
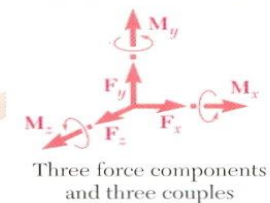
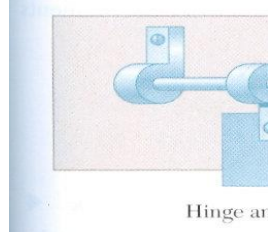
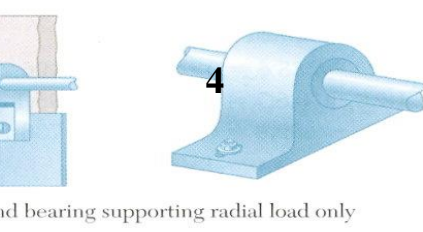
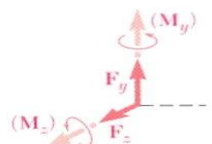
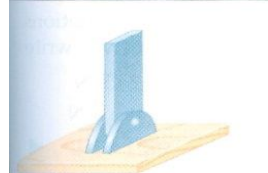
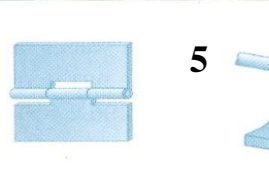
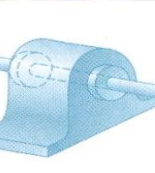
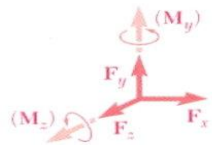


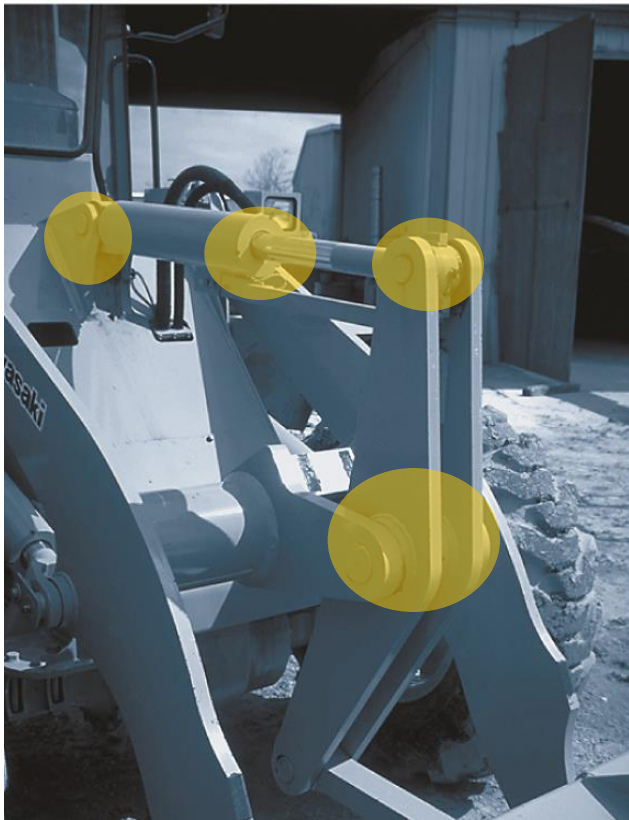
Hinge and bearing supporting axial thrust and radial load



Three force components (and two couples)

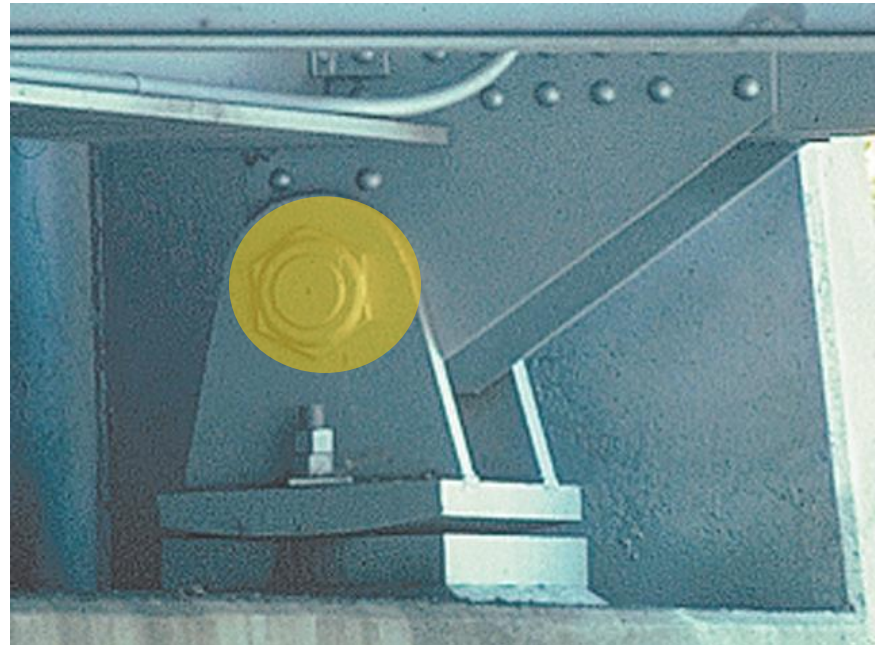
Boundary Support Summary

 <p>Ball</p>  <p>Frictionless surface</p>  <p>Force with known line of action (one unknown)</p>	 <p>Cable</p>  <p>Force with known line of action (one unknown)</p>
 <p>Roller on rough surface</p>  <p>Wheel on rail</p>	 <p>Two force components</p>
 <p>Rough surface</p>  <p>Ball and socket</p>	 <p>Three force components</p>
 <p>Universal joint</p>  <p>Three force components and one couple</p>	 <p>Fixed support</p>  <p>Three force components and three couples</p>
 <p>Hinge and bearing supporting radial load only</p>  <p>4</p>	 <p>Two force components (and two couples)</p>
 <p>Pin and bracket</p>  <p>Hinge and bearing supporting axial thrust and radial load</p>  <p>5</p>	 <p>Three force components (and two couples)</p>



Pin connections allow rotation. Reactions at pins are forces and NOT MOMENTS.

Rocker Bearing used to Support the Roadway of a Bridge



4.53 A 4×8 -ft sheet of plywood weighing 40 lb has been temporarily propped against column CD . It rests at A and B on small wooden blocks and against protruding nails. Neglecting friction at all surfaces of contact, determine the reactions at A , B , and C .

Freebody-Diagram

We have five unknowns and six equations.

The plywood sheet is free to move in x -direction, but ($\sum F_x = 0$).

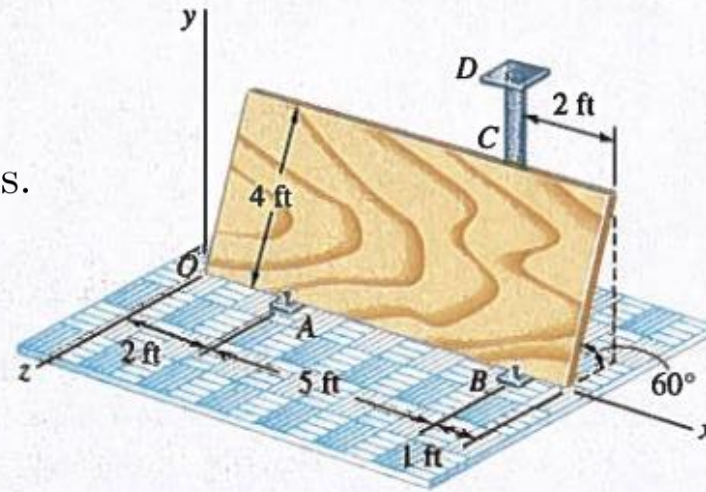
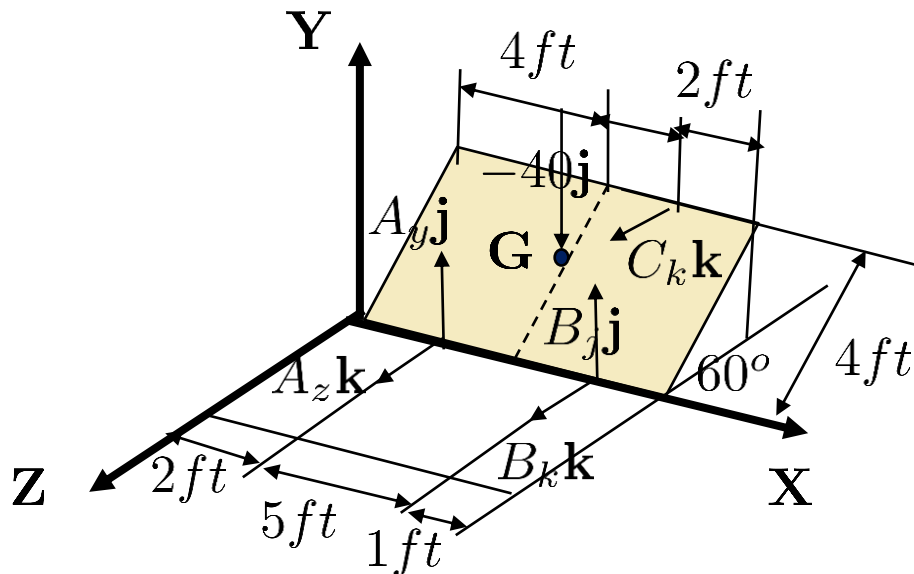


Fig. P4.53

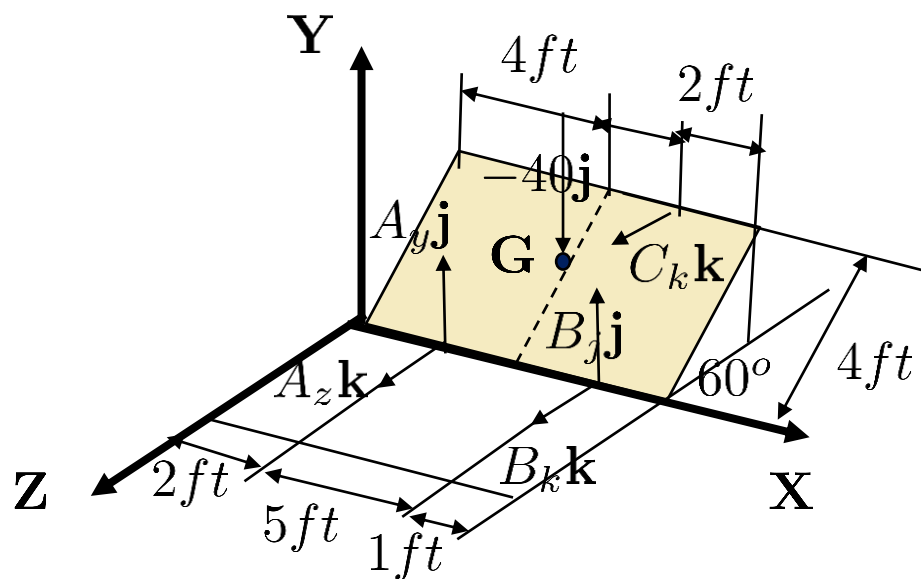


$$A:(2,0,0)$$

$$B:(7,0,0)$$

$$C : (6, 2\sqrt{3}, -2)$$

$$G : (4, \sqrt{3}, -1)$$



$$\mathbf{A}:(2,0,0)$$

$$\mathbf{B}:(7,0,0)$$

$$\mathbf{C}:(6, 2\sqrt{3}, -2)$$

$$\mathbf{G}:(4, \sqrt{3}, -1)$$

$$\mathbf{r}_{AB} = 5\mathbf{i}$$

$$\mathbf{r}_{AC} = 4\mathbf{i} + 4\sin 60^\circ\mathbf{j} - 4\cos 60^\circ\mathbf{k}$$

$$\mathbf{r}_{AG} = 2\mathbf{i} + 2\sin 60^\circ\mathbf{j} - 2\cos 60^\circ\mathbf{k}$$

$$\Sigma M_A = 0: \mathbf{r}_{B/A} \times (B_y\mathbf{j} + B_z\mathbf{k}) + \mathbf{r}_{C/A} \times C\mathbf{k} + \mathbf{r}_{G/A} \times (-40\text{ lb})\mathbf{j} = 0$$

$$\begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ 5 & 0 & 0 \\ 0 & B_y & B_z \end{vmatrix} + \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ 4 & 4\sin 60^\circ & -4\cos 60^\circ \\ 0 & 0 & C \end{vmatrix} + \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ 2 & 2\sin 60^\circ & -2\cos 60^\circ \\ 0 & -40 & 0 \end{vmatrix} = 0$$

$$(4C \sin 60^\circ - 80 \cos 60^\circ)\mathbf{i} + (-5B_z - 4C)\mathbf{j} + (5B_y - 80)\mathbf{k} = 0$$

$$\mathbf{i}: \quad 4C \sin 60^\circ - 80 \cos 60^\circ = 0 \quad C = 11.5470 \text{ lb}$$

$$\mathbf{j}: \quad -5B_z - 4C = 0 \quad B_z = 9.2376 \text{ lb}$$

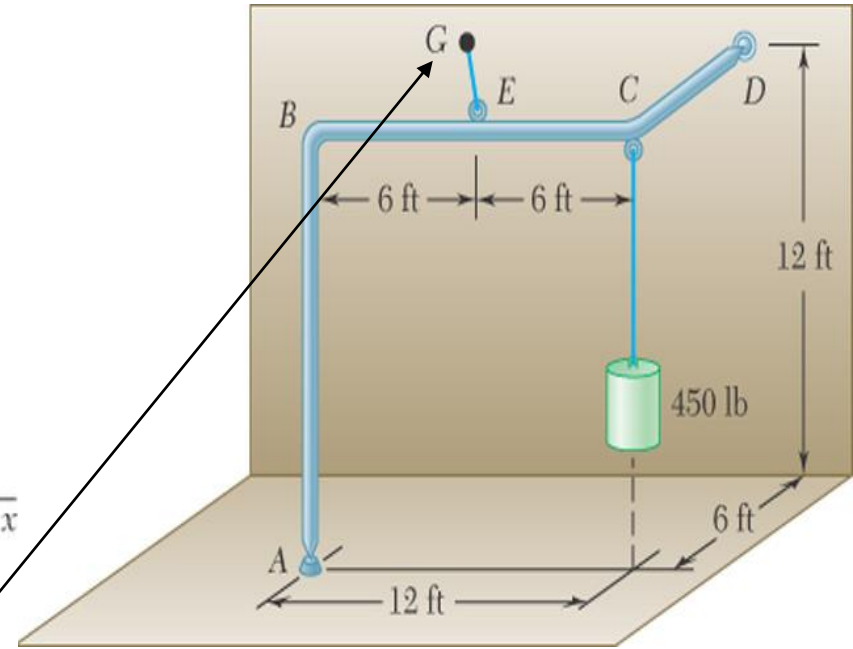
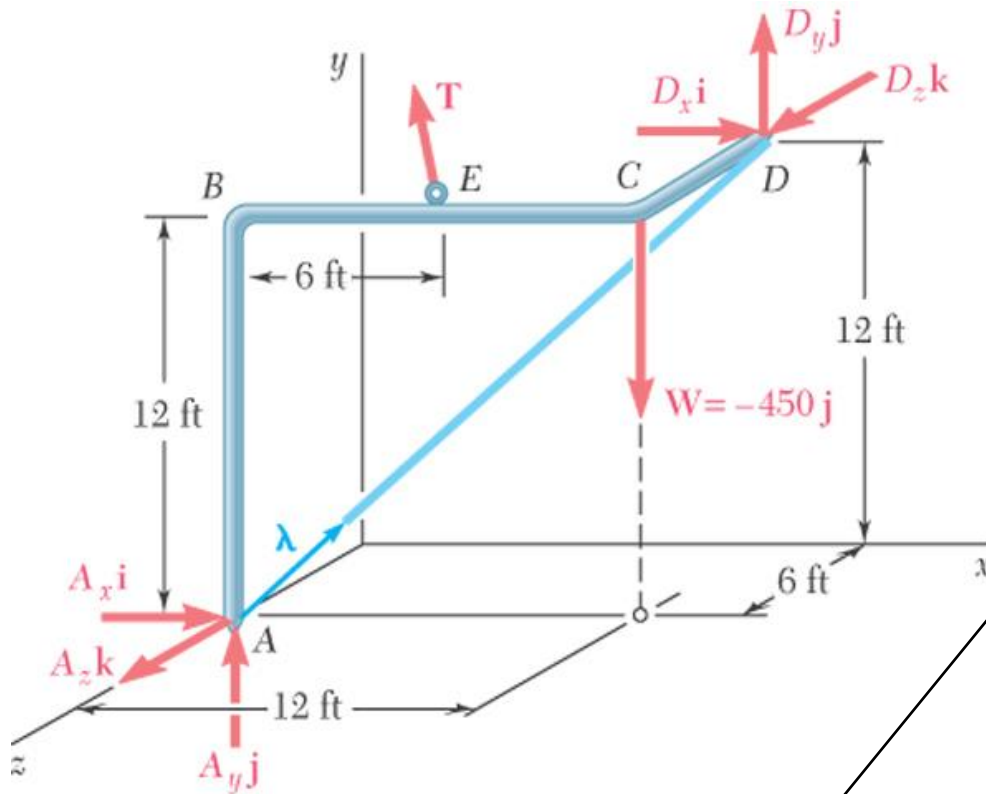
$$\mathbf{k}: \quad 5B_y - 80 = 0 \quad B_y = 16.0000 \text{ lb}$$

$$\Sigma F_y = 0: \quad A_y + B_y - 40 = 0 \quad A_y = 40 - 16.0000 = 24.0000 \text{ lb}$$

$$\Sigma F_z = 0: \quad A_z + B_z + C = 0 \quad A_z = 9.2376 - 11.5470 = -2.3094 \text{ lb}$$

$$\mathbf{A} = (24.0 \text{ lb})\mathbf{j} - (2.31 \text{ lb})\mathbf{k}; \quad \mathbf{B} = (16.00 \text{ lb})\mathbf{j} - (9.24 \text{ lb})\mathbf{k}; \quad \mathbf{C} = (11.55 \text{ lb})\mathbf{k} \quad \blacktriangleleft$$

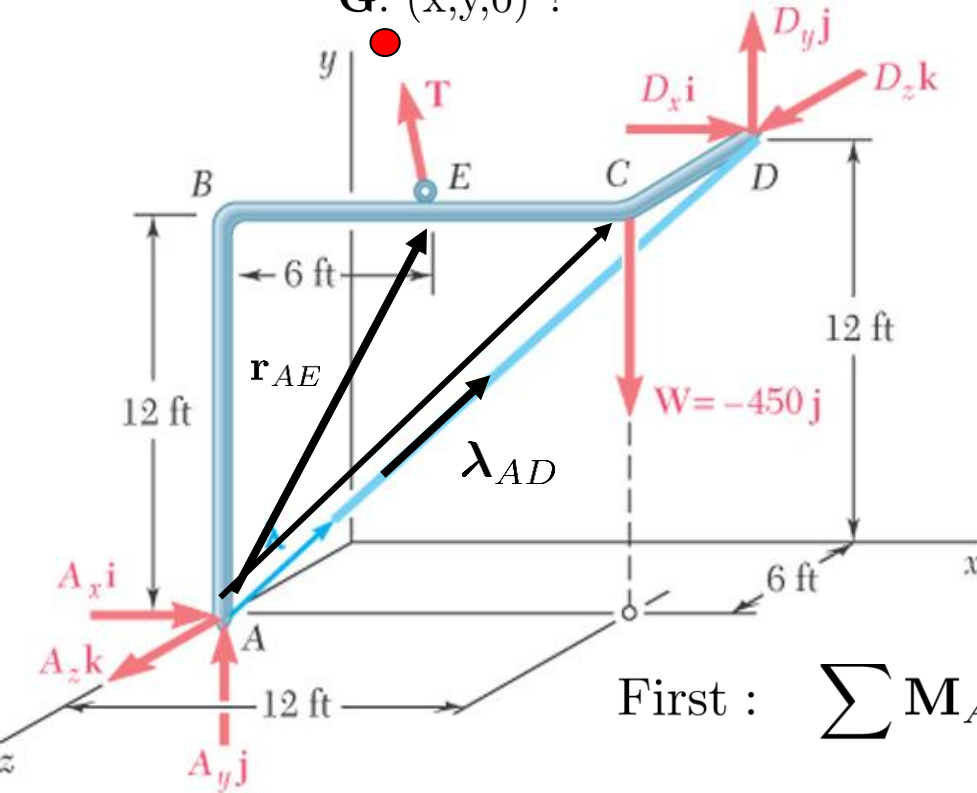
Sample Problem 4.10



Determine (a) where G should be located if the tension in the cable is to be minimum, (b) the corresponding minimum value of the tension.

$$A : (0, 0, 6), \quad C : (12, 12, 6), \quad D : (12, 12, 0) \quad E : (6, 12, 6)$$

$$G : (x, y, 0) ?$$



$$\begin{aligned} \mathbf{r}_{AE} &= (6 - 0)\mathbf{i} + (12 - 0)\mathbf{j} + (6 - 6)\mathbf{k} \\ &= 6\mathbf{i} + 12\mathbf{j} \end{aligned}$$

$$\begin{aligned} \mathbf{r}_{AC} &= (12 - 0)\mathbf{i} + (12 - 0)\mathbf{j} + (6 - 6)\mathbf{k} \\ &= 12\mathbf{i} + 12\mathbf{j} \end{aligned}$$

$$\begin{aligned} \mathbf{r}_{AD} &= (12 - 0)\mathbf{i} + (12 - 0)\mathbf{j} + (0 - 6)\mathbf{k} \\ &= 12\mathbf{i} + 12\mathbf{j} - 6\mathbf{k} \quad |\mathbf{r}_{AD}| = 18 \end{aligned}$$

$$\lambda_{AD} = \frac{\mathbf{r}_{AD}}{|\mathbf{r}_{AD}|} = \frac{2}{3}\mathbf{i} + \frac{2}{3}\mathbf{j} - \frac{1}{3}\mathbf{k}$$

$$\text{First : } \sum \mathbf{M}_A = 0 \rightarrow \sum M_{AD} = 0 \rightarrow$$

$$\lambda_{AD} \cdot \{ \mathbf{r}_{AE} \times \mathbf{T}_{EG} + \mathbf{r}_{AC} \times \mathbf{W} + \mathbf{r}_{AD} \times \mathbf{D} \} = 0 .$$

$$\mathbf{W} = -450\mathbf{j}$$

$$\text{where } \lambda_{AD} = \frac{\mathbf{r}_{AD}}{|\mathbf{r}_{AD}|} = \frac{1}{18} (12\mathbf{i} + 12\mathbf{j} - 6\mathbf{k}) \text{ and}$$

$$\mathbf{r}_{AC} \times \mathbf{W} = (12\mathbf{i} + 12\mathbf{j}) \times (-450\mathbf{j}) = -5400\mathbf{k}$$

Why ?

$$\lambda_{AD} \cdot (\mathbf{r}_{AE} \times \mathbf{T}_{EG}) + \lambda_{AD} \cdot (\mathbf{r}_{AC} \times \mathbf{W}) = 0, \Rightarrow$$

$$\lambda_{AD} \cdot (\mathbf{r}_{AE} \times \mathbf{T}_{EG}) = -\left(\frac{2\mathbf{i} + 2\mathbf{j} - 1\mathbf{k}}{3}\right) \cdot (-5400\mathbf{k}) = -1800.$$

$$\lambda_{AD} \cdot (\mathbf{r}_{AE} \times \mathbf{T}_{EG}) = \mathbf{T}_{EG} \cdot (\lambda_{AD} \times \mathbf{r}_{AE}) = 1800 \quad \lambda_{AD} = \frac{2}{3}\mathbf{i} + \frac{2}{3}\mathbf{j} - \frac{1}{3}\mathbf{k}$$

$$\mathbf{r}_{AE} = 6\mathbf{i} + 12\mathbf{j}$$

$$\lambda_{AD} \times \mathbf{r}_{AE} = 4\mathbf{i} - 2\mathbf{j} + 4\mathbf{k}, \quad \text{and} \quad |\lambda_{AD} \times \mathbf{r}_{AE}| = 6.$$

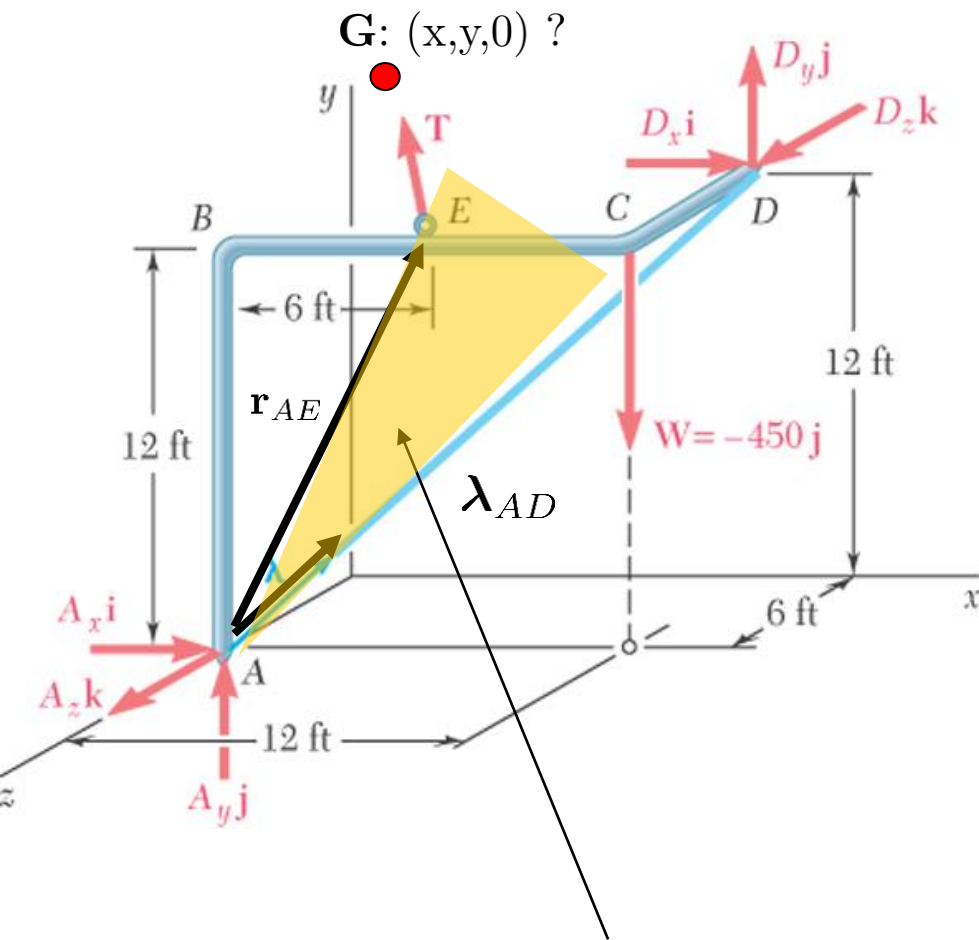
$$\mathbf{T}_{EG} \cdot (\lambda_{AD} \times \mathbf{r}_{AE}) = T_{EG} |\lambda_{AD} \times \mathbf{r}_{AE}| \cos \theta = 6T_{EG} \cos \theta = 1800$$

$$\mathbf{T}_{EG} = T_{EG} \lambda_{EG} \quad \lambda_{EG} \cdot \frac{\lambda_{AD} \times \mathbf{r}_{AE}}{|\lambda_{AD} \times \mathbf{r}_{AE}|} = \cos \theta$$

$$\mathbf{T}_{EG} = T_{min} \frac{(\lambda_{AD} \times \mathbf{r}_{AE})}{|\lambda_{AD} \times \mathbf{r}_{AE}|} = T_{min} \frac{(\lambda_{AD} \times \mathbf{r}_{AE})}{6}$$

Why?

$$\cos \theta = 1 \rightarrow \lambda_{EG} // \frac{\lambda_{AD} \times \mathbf{r}_{AE}}{|\lambda_{AD} \times \mathbf{r}_{AE}|} \rightarrow \theta = 0 \quad T_{EG} = T_{min}$$



If T_{EG} is minimum, \mathbf{T}_{EG} should be perpendicular to the plane spanned by λ_{AD} and \mathbf{r}_{AE} .

Hence : $\mathbf{T}_{EG} = T_{min} \frac{(\boldsymbol{\lambda}_{AD} \times \mathbf{r}_{AE})}{|\boldsymbol{\lambda}_{AD} \times \mathbf{r}_{AE}|} = T_{min} \frac{(\boldsymbol{\lambda}_{AD} \times \mathbf{r}_{AE})}{6}$

$$\mathbf{T}_{EG} \cdot (\boldsymbol{\lambda}_{AD} \times \mathbf{r}_{AE}) = T_{min} |\boldsymbol{\lambda}_{AD} \times \mathbf{r}_{AE}| = 6T_{min}$$

Recall : $\boldsymbol{\lambda}_{AD} \cdot (\mathbf{r}_{AE} \times \mathbf{T}_{EG}) = \mathbf{T}_{EG} \cdot (\boldsymbol{\lambda}_{AD} \times \mathbf{r}_{AE}) = 1800$

$$6T_{min} = 1800 \rightarrow T_{min} = 300lb;$$

$$\mathbf{T}_{EG} = T_{min} \frac{(\boldsymbol{\lambda}_{AD} \times \mathbf{r}_{AE})}{|\boldsymbol{\lambda}_{AD} \times \mathbf{r}_{AE}|} = T_{min} \frac{(\boldsymbol{\lambda}_{AD} \times \mathbf{r}_{AE})}{6}$$

Recall $\boldsymbol{\lambda}_{AD} \times \mathbf{r}_{AE} = 4\mathbf{i} - 2\mathbf{j} + 4\mathbf{k}$

$$\mathbf{T}_{EG} = T_{min} \frac{(\boldsymbol{\lambda}_{AD} \times \mathbf{r}_{AE})}{6} = (50) (4\mathbf{i} - 2\mathbf{j} + 4\mathbf{k})$$

$$\mathbf{T}_{EG} = T_{min} \frac{(\boldsymbol{\lambda}_{AD} \times \mathbf{r}_{AE})}{6} = (50)(4\mathbf{i} - 2\mathbf{j} + 4\mathbf{k})$$

G: (x,y,0)

$$E : (6, 12, 6), \quad G(x, y, 0)$$

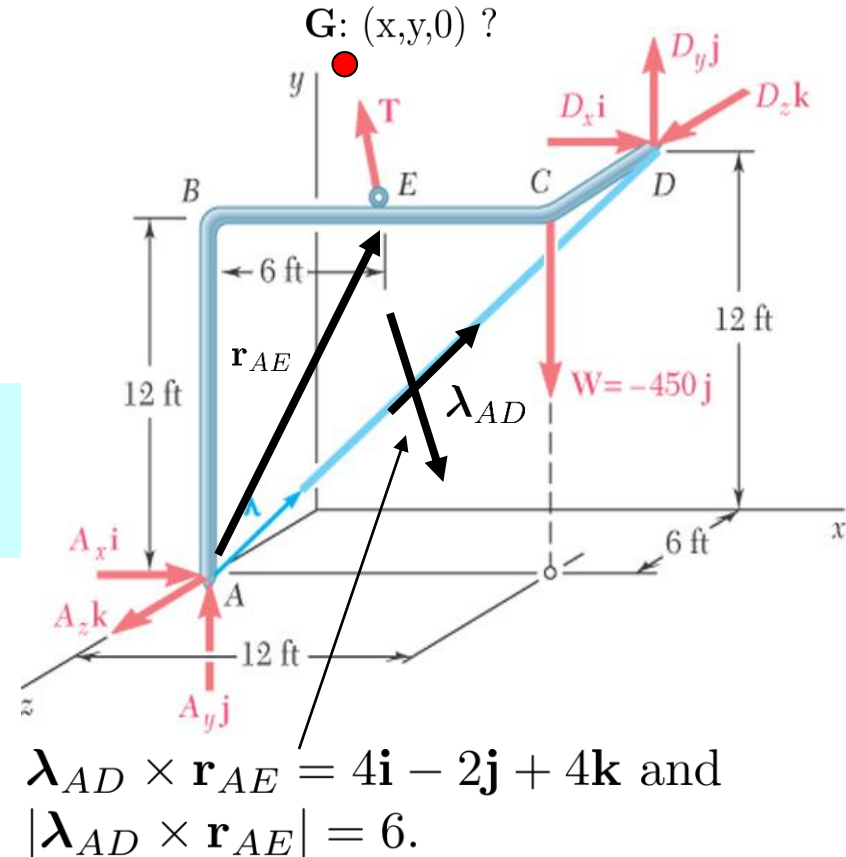
$$\mathbf{r}_{EG} = (x - 6)\mathbf{i} + (y - 12)\mathbf{j} + (0 - 6)\mathbf{k}$$

$$\mathbf{r}_{EG} // (4\mathbf{i} - 2\mathbf{j} + 4\mathbf{k})$$

**They must be in a parallel direction;
that is**

$$\frac{x - 6}{4} = \frac{y - 12}{-2} = \frac{0 - 6}{4}$$

Hence x=0, y=15 ft

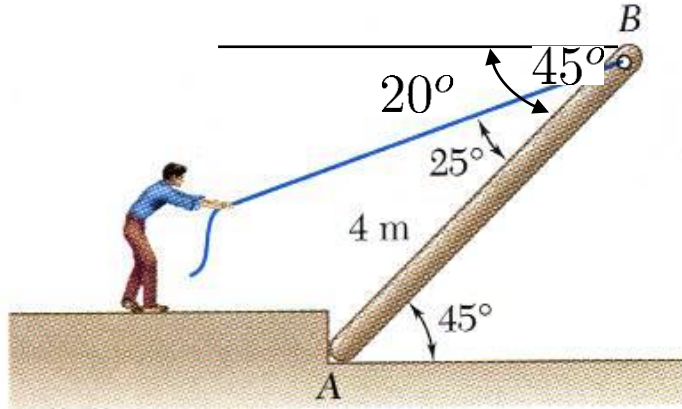


$$\boldsymbol{\lambda}_{AD} \times \mathbf{r}_{AE} = 4\mathbf{i} - 2\mathbf{j} + 4\mathbf{k} \text{ and } |\boldsymbol{\lambda}_{AD} \times \mathbf{r}_{AE}| = 6.$$

$$E : (6, 12, 6)$$

$$G : (x, y, 0)$$

Sample Problem 4.6



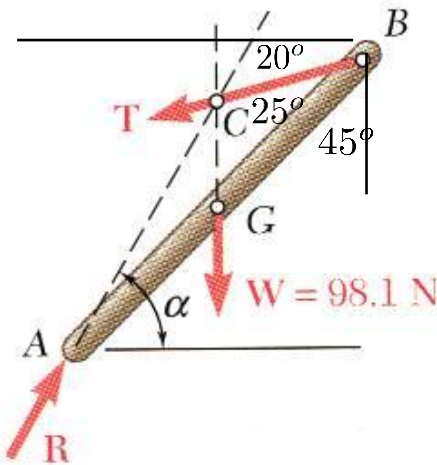
A man raises a 10 kg joist, of length 4 m, by pulling on a rope.

Find the tension in the rope and the reaction at A.

SOLUTION:

- Create a free-body diagram of the joist. Note that the joist is a 3 force body.
- The three forces must be concurrent for static equilibrium. Therefore, the reaction ***R*** must pass through the intersection of the lines of action of the weight and rope forces.
- Utilize a force triangle to determine the magnitude of the reaction force ***R***.

- Create a free-body diagram of the joist.



$$AF = AB \cos 45 = (4 \text{ m}) \cos 45 = 2.828 \text{ m}$$

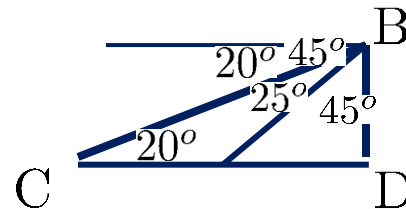
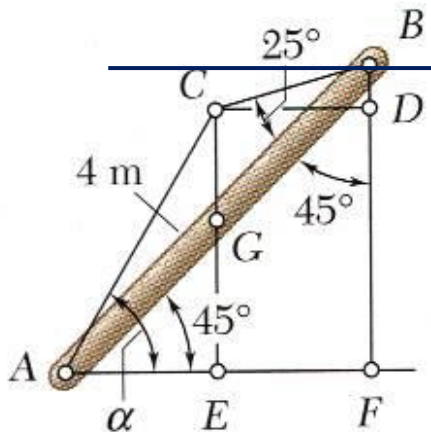
$$CD = AE = \frac{1}{2} AF = 1.414 \text{ m}$$

$$BD = CD \cot(45 + 25) = CD \tan 20 = 1.414 \tan 20 = 0.515 \text{ m}$$

$$CE = BF - BD = (2.828 - 0.515) \text{ m} = 2.313 \text{ m}$$

$$\tan \alpha = \frac{CE}{AE} = \frac{2.313}{1.414} = 1.636$$

$$\alpha = 58.6^\circ$$



- Determine the magnitude of the reaction force \mathbf{R} .

$$\frac{T}{\sin 31.4^\circ} = \frac{R}{\sin 110^\circ} = \frac{98.1 \text{ N}}{\sin 38.6^\circ}$$

$$R = 147.8 \text{ N}$$

