

FP1: Control of the Variable Length Pendulum

Michele Cipriano, Karim Ghonim, Khaled Wahba

Control Design and Analysis for Underactuated Robotics:
Variable Length Pendulum

Authors: Xin Xin, Yannian Liu

Elective in Robotics: Underactuated Robotics
Department of Computer, Control and Management Engineering
Sapienza University of Rome

Introduction

- Variable Length Pendulum (VLP)
- Trajectory Tracking Control
- Total Energy Shaping
- Partial Energy Shaping
- Convergence of Energy
- Closed-Loop Equilibrium Points
- Simulink

Motion Equation

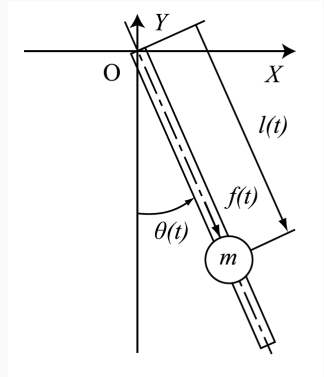
Euler-Lagrange equation:

$$\ddot{\theta}(t) + \frac{2\dot{l}(t)\dot{\theta}(t)}{l(t)} + \frac{g \sin \theta(t)}{l(t)} = 0$$

$$\ddot{l}(t) - ml(t)\dot{\theta}^2(t) - mg \cos \theta(t) = f(t)$$

Control input:

$$u = \ddot{l}(t)$$



Problem Formulation

Let $m = 1$. Total mechanical energy:

$$E_T = \frac{1}{2}\dot{l}^2(t) + \frac{1}{2}(l(t)\dot{\theta}(t))^2 - gl(t)\cos\theta(t)$$

Desired trajectory of swing described by:

$$E_r = \frac{1}{2}(l_r\dot{\theta}(t))^2 - gl_r\cos\theta(t)$$

with E_r and l_r desired energy and length of the VLP.

Moreover: $E_r = -gl_r\cos\theta_{max}$, $\theta_{max} \in (0, \pi]$.

Trajectory tracking control problem:

$$\lim_{t \rightarrow \infty} E_T = E_r \quad \lim_{t \rightarrow \infty} \dot{l} = 0 \quad \lim_{t \rightarrow \infty} l = l_r$$

Controller Design: Total Energy Shaping

Lyapunov candidate with $k_P > 0$, $k_D > 0$:

$$V_c = \frac{1}{2}(E_T - E_r)^2 + \frac{1}{2}k_P(l - l_r)^2 + \frac{1}{2}k_D\dot{l}^2$$
$$\dot{V}_c = \dot{l}((E_T - E_r + k_D)u - (E_T - E_r)(l\dot{\theta}^2 + g \cos \theta) - k_P(l - l_r))$$

Controller:

$$u = \frac{(E_T - E_r)(l\dot{\theta}^2 + g \cos \theta) - k_P(l - l_r) - k_V\dot{l}}{E_T - E_r + k_D}$$

with $k_V > 0$, then:

$$\dot{V}_c = -k_V\dot{l}^2 \leq 0$$

which holds only if:

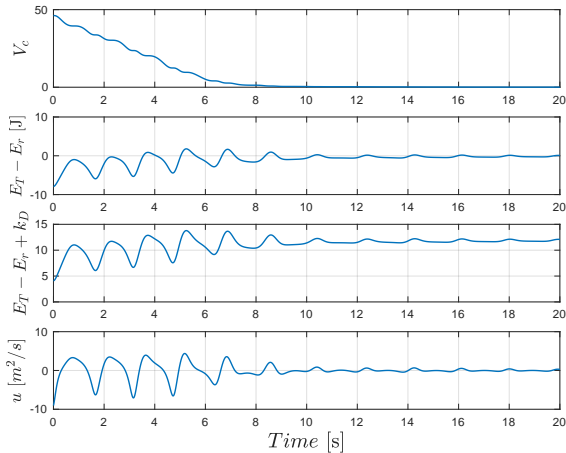
$$E_T - E_r + k_D \neq 0 \quad \forall t \geq 0$$

Controller Design: Total Energy Shaping

Consider the trajectory tracking control problem defined previously and $l_r = 3\text{m}$, $\theta_{\max} = 2\pi/5$, $g = 9.81\text{m/s}^2$. Choosing $k_D = 12$, $k_P = 30$ and $k_V = 12$ yields $l_{de} = 1.42\text{m}$.

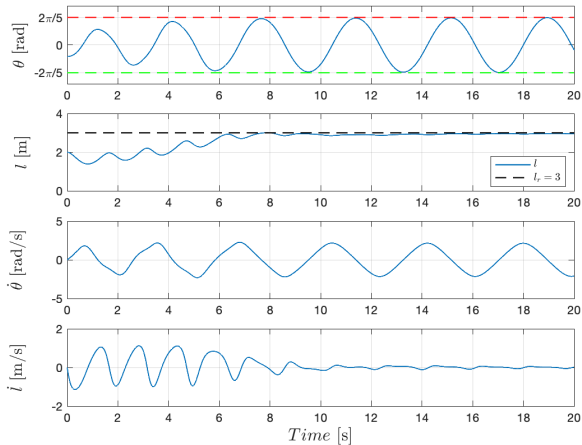
Controller Design: Total Energy Shaping

Figure 1: Time responses of V , $E_T - E_r$, $E_T - E_r + k_D$ and u with initial state $(\theta(0), l(0), \dot{\theta}(0), \dot{l}(0)) = (-\pi/6, 2, 0, 0)$.



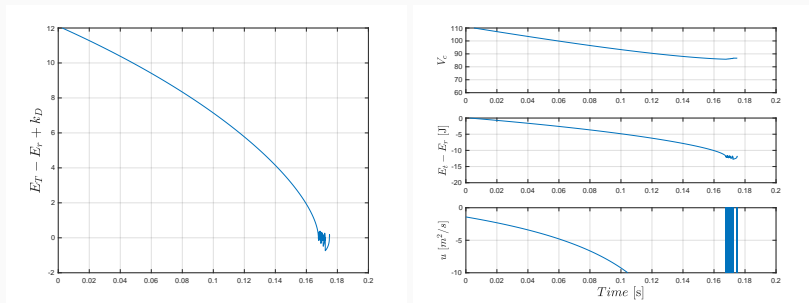
Controller Design: Total Energy Shaping

Figure 2: Time responses of $(\theta, l, \dot{\theta}, \dot{l})$ with initial state $(\theta(0), l(0), \dot{\theta}(0), \dot{l}(0)) = (-\pi/6, 2, 0, 0)$.



Controller Design: Total Energy Shaping

Figure 3: The controller encountered a singular point for initial state $(\theta(0), l(0), \dot{\theta}(0), \dot{l}(0)) = (-\pi/6, 2, 0, 4)$.



Controller Design: Partial Energy Shaping

E_P sum of kinetic energy of rotation and potential energy of VLP:

$$E_P = \frac{1}{2}(l(t)\dot{\theta}(t))^2 - gl(t)\cos\theta(t)$$

Lyapunov candidate:

$$V = \frac{1}{2}(E_P - E_r)^2 + \frac{1}{2}k_P(l - l_r)^2 + \frac{1}{2}k_D\dot{l}^2$$
$$\dot{V} = (-(E_P - E_r)(l\dot{\theta}^2 + g\cos\theta) + k_P(l - l_r) + k_D u)\dot{l}$$

Controller:

$$u = \frac{(E_P - E_r)(l\dot{\theta}^2 + g\cos\theta) - k_P(l - l_r) - k_V\dot{l}}{k_D}$$

which is free of singular points. Moreover:

$$\dot{V} = -k_V\dot{l}^2 \leq 0, \quad k_V > 0$$

Controller Design: Partial Energy Shaping

Consider:

$$\Gamma_c = \left\{ (\theta, l, \dot{\theta}, \dot{l}) \mid V(\theta, l, \dot{\theta}, \dot{l}) \leq c \right\}, \quad c > 0$$

Since $\dot{V} \leq 0$, any closed-loop solution starting in Γ_c remains in Γ_c for all $t \geq 0$. Let W be the largest invariant set in:

$$S = \{(\theta, l, \dot{\theta}, \dot{l}) \in \Gamma_c \mid \dot{V} = 0\}$$

Using **LaSalle's invariant principle**, every closed-loop solution starting in Γ_c approaches W as $t \rightarrow \infty$.

Controller Design: Partial Energy Shaping

Since $\dot{V} = 0$ holds for all elements of W , V and l are constant in W (let them be V^* and l^*). Moreover, since V is a constant in W , E_P is also a constant in W . Consequently:

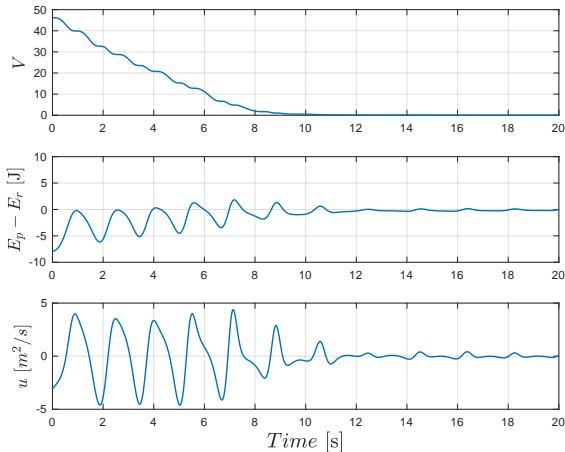
$$\lim_{t \rightarrow \infty} E_P = E^* \quad \lim_{t \rightarrow \infty} \dot{l} = 0 \quad \lim_{t \rightarrow \infty} l = l^*$$

Thus, the largest invariant set W can be defined as:

$$W = \left\{ (\theta, l, \dot{\theta}, \dot{l}) \left| \frac{1}{2} (l^* \dot{\theta})^2 - g l^* \cos \theta \equiv E^*, l \equiv l^* \right. \right\}$$

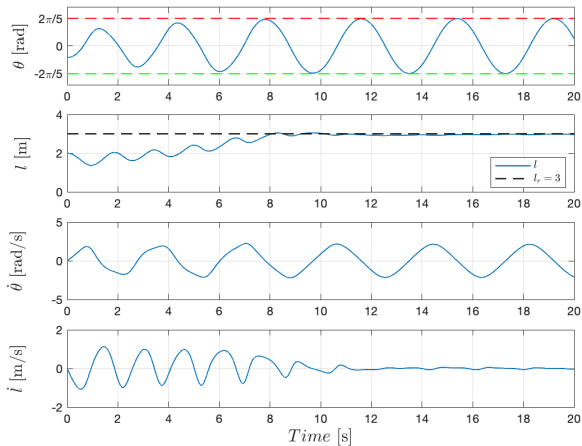
Controller Design: Partial Energy Shaping

Figure 5: Time responses of V , $E_p - E_r$ and u with initial state $(\theta(0), l(0), \dot{\theta}(0), \dot{l}(0)) = (-\pi/6, 2, 0, 0)$.



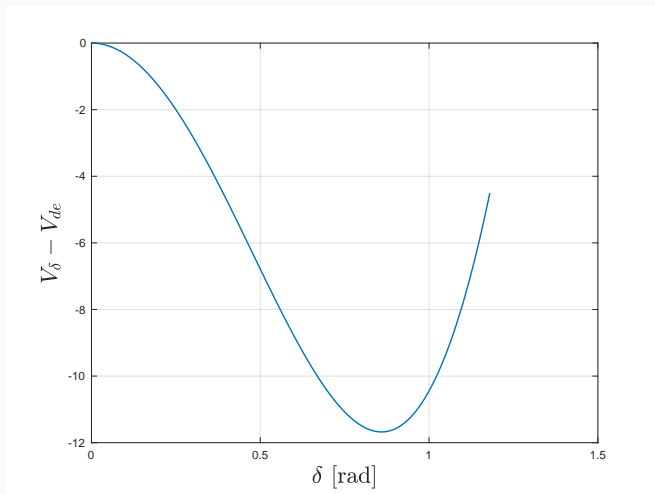
Controller Design: Partial Energy Shaping

Figure 6: Time responses of $(\theta, l, \dot{\theta}, \dot{l})$ with initial state $(\theta(0), l(0), \dot{\theta}(0), \dot{l}(0)) = (-\pi/6, 2, 0, 0)$.



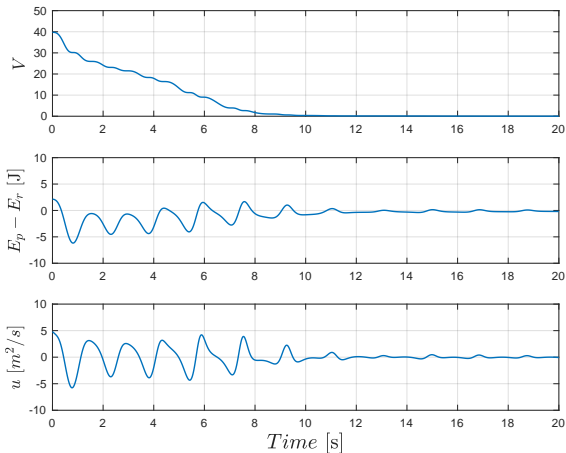
Controller Design: Partial Energy Shaping

Figure 7: $V_\delta - V_{de}$.



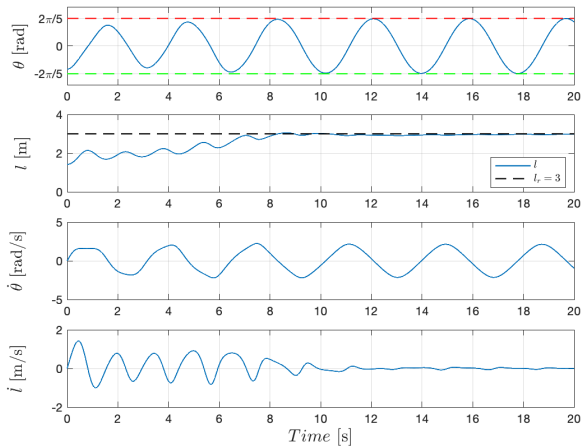
Controller Design: Partial Energy Shaping

Figure 8: Time responses of V , $E_p - E_r$ and u with initial state $(\theta(0), (0), \dot{\theta}(0), \dot{l}(0)) = (-\pi/3, l_{de}, 0, 0)$.



Controller Design: Partial Energy Shaping

Figure 9: Time responses of $(\theta, l, \dot{\theta}, \dot{l})$ with initial state $(-\pi/3, l_{de}, 0, 0)$.



Motion Analysis: Convergence of Energy

Trajectory Tracking Control Problem achieved iff $V^* = 0$, with:

$$V^* = \frac{1}{2}(E^* - E_r)^2 + \frac{1}{2}k_p(l^* - l_r)^2$$

Consider $E_p \equiv E^*$, $l \equiv l^*$, $\dot{l} \equiv 0$ and $\ddot{l} \equiv 0$ and $u = \ddot{l} \equiv 0$. Then, from the controller based on partial energy shaping:

$$(E^* - E_r)(l^* \dot{\theta}^2 + g \cos \theta) - k_p(l^* - l_r) \equiv 0$$

Motion Analysis: Convergence of Energy

Case 1: $E^* = E_r$

In this case $l^* = l_r$. The largest invariant set W becomes:

$$W_r = \left\{ (\theta, l, \dot{\theta}, \dot{l}) \left| \frac{1}{2}(l_r \dot{\theta})^2 - gl_r \cos \theta \equiv E_r, l \equiv l_r \right. \right\}$$

Hence, as $t \rightarrow \infty$, the closed-loop solution $(\theta(t), l(t), \dot{\theta}(t), \dot{l}(t))$ achieves the tracking control objective.

Motion Analysis: Convergence of Energy

Case 2 $E^* \neq E_r$

In this case:

$$l^* \dot{\theta}^2 + g \cos \theta \equiv \frac{k_p(l^* - l_r)}{E^* - E_r}$$

Considering $E_p \equiv E^*$ and $l \equiv l^*$ it is possible to prove that θ is a constant. Let $\theta \equiv \theta^*$. Moreover, since $E_r = -gl_r \cos \theta_{\max}$:

$$l^* = \frac{l_r(k_p + g^2 \cos \theta_{\max} \cos \theta^*)}{k_p + g^2}$$

Motion Analysis: Convergence of Energy

Since $\sin \theta^* = 0$ admits a solution only in $\{0, \pi\}$, either $(\theta^*, l^*) = (\pi, l_{ue})$ or $(\theta^*, l^*) = (0, l_{de})$ where:

$$l_{ue} = l^*|_{\theta^*=\pi} = \frac{l_r(k_p - g^2 \cos \theta_{\max})}{k_p + g^2}$$
$$l_{de} = l^*|_{\theta^*=0} = \frac{l_r(k_p + g^2 \cos \theta_{\max})}{k_p + g^2}$$

To guarantee that the length of the pendulum l_{ue} and l_{de} are positive in the two cases respectively assume $k_p > -g^2 \cos \theta_{\max}$ and $k_p > g^2 \cos \theta_{\max}$, which can be rewritten as:

$$k_p > g^2 |\cos \theta_{\max}|$$

Motion Analysis: Convergence of Energy

Considering also that $0 < \theta_{\max} \leq \pi$, it is easy to prove that:

$$0 < l_{ue} \leq l_r$$

$$0 < l_{de} < l_r$$

Let Ω be the equilibrium point set (which is an invariant set for the considered system) defined as follows:

$$\Omega = \{(\pi, l_{ue}, 0, 0), (0, l_{de}, 0, 0)\}$$

Motion Analysis: Closed-Loop Equilibrium Points

Consider the two equilibrium points defined in Ω . Let the state variable vector be $x = (\theta, l, \dot{\theta}, \dot{l})^T$. The state space representation is $\dot{x} = f(x)$, which, considering the dynamics of the VLP and the controller based on partial energy shaping, can be rewritten as:

$$\dot{x}_1 = x_3$$

$$\dot{x}_2 = x_4$$

$$\dot{x}_3 = -\frac{2x_3x_4 + g \sin x_1}{x_2}$$

$$\dot{x}_4 = \frac{(E_p - E_r)(x_2x_3^2 + g \cos x_1) - k_p(x_2 - l_r) - k_vx_4}{k_D}$$

where $E_p = x_2^2x_3^2/2 - gx_2 \cos x_1$.

Motion Analysis: Closed-Loop Equilibrium Points

The characteristic equation of the Jacobian matrix evaluated at the upper equilibrium point $x_{ue} = (\pi, l_{ue}, 0, 0)^T$ yields:

$$\det(sI - A_{ue}) = \left(s^2 + \frac{k_V}{k_D}s + \frac{g^2 + k_P}{k_D} \right) \left(s^2 - \frac{g}{l_{ue}} \right)$$

The jacobian A_{ue} has three eigenvalues in the open LHP and one eigenvalue in the open RHP. Thus, the upward equilibrium point is unstable and **hyperbolic** (since all the eigenvalues of A_{ue} have non-zero real parts).

Motion Analysis: Closed-Loop Equilibrium Points

The characteristic equation of the Jacobian matrix evaluated at the downward equilibrium point $x_{de} = (0, l_{de}, 0, 0)^T$ yields:

$$\det(sI - A_{de}) = \left(s^2 + \frac{k_V}{k_D}s + \frac{g^2 + k_P}{k_D} \right) \left(s^2 + \frac{g}{l_{ue}} \right)$$

The Jacobian A_{de} has two eigenvalues in the open LHP and two eigenvalues on the imaginary axis. Thus, the downward equilibrium point is **non-hyperbolic** (since there is at least one eigenvalue on the imaginary axis).

Motion Analysis: Closed-Loop Equilibrium Points

Consider the Lyapunov function V used for partial energy shaping:

$$V = \frac{1}{2}(E_P - E_r)^2 + \frac{1}{2}k_P(l - l_r)^2 + \frac{1}{2}k_D\dot{l}^2$$

Consider the following set:

$$\Gamma_d = \left\{ (\theta, l, \dot{\theta}, \dot{l}) \mid V(\theta, l, \dot{\theta}, \dot{l}) < V(0, l_{de}, 0, 0) \right\}$$

Let the values of V at the downward equilibrium point x_{de} and a state $(\delta, l_{de}, 0, 0)$ be respectively V_{de} and V_δ :

$$V_{de} = V(0, l_{de}, 0, 0) = \frac{1}{2}(gl_{de} + E_r)^2 + \frac{1}{2}k_P(l_{de} - l_r)^2$$

$$V_\delta = V(\delta, l_{de}, 0, 0) = \frac{1}{2}(gl_{de} \cos \delta + E_r)^2 + \frac{1}{2}k_P(l_{de} - l_r)^2$$

Motion Analysis: Closed-Loop Equilibrium Points

Compute the difference between V_δ and V_{de} yields:

$$V_\delta - V_{de} = -\frac{g^2 l_{de} l_r (1 - \cos \delta) \Xi}{k_P + g^2}$$

where:

$$\Xi = k_P(1 - \cos \theta_{\max}) - (k_P + g^2 \cos \theta_{\max}) \sin^2 \left(\frac{\delta}{2} \right)$$

Using $k_P > -g^2 \cos \theta_{\max}$ and $|\sin \delta| < |\delta|$ for $\delta \neq 0$:

$$\Xi > k_P(1 - \cos \theta_{\max}) - \frac{(k_P + g^2 \cos \theta_{\max}) \delta^2}{4}$$

with $\delta \neq 0$.

Motion Analysis: Closed-Loop Equilibrium Points

If δ satisfies:

$$0 < |\delta| \leq \delta_m = 2\sqrt{\frac{k_P(1 - \cos \theta_{\max})}{k_P + g^2 \cos \theta_{\max}}}$$

then $\Xi > 0$ (proof by simply noticing that $-\delta^2 \geq -\delta_m^2$), which, using $V_\delta - V_{de}$, implies:

$$V_\delta < V_{de}, \quad (\delta, l_{de}, 0, 0) \in \Gamma_d$$

which itself implies that $\Gamma_d \neq \emptyset$.

Motion Analysis: Closed-Loop Equilibrium Points

Since V is monotonically decreasing under the controller based on partial energy shaping, for any δ satisfying $0 < |\delta| \leq \delta_m$, closed-loop solution starting from Γ_d approaches W_r due to the results obtained in Convergence of Energy (Case 1). Thus, the downward equilibrium point $(0, l_{de}, 0, 0)$ is unstable in the Lyapunov sense.

Every closed-loop solution under the closed-loop system consisted of the dynamic equation of the VLP and the controller based on partial energy shaping, supposing $k_P > g^2 |\cos \theta_{\max}|$, $k_D > 0$, $k_V > 0$, $0 < \theta_{\max} \leq \pi$, approaches $W = W_r \cup \Omega$.

Conclusion

- Total Energy Shaping
- Partial Energy Shaping
- Motion Analysis
- Unstable Equilibrium Points

Q&A

References



X. Xin and Y. Liu, *Control Design and Analysis for Underactuated Robotic Systems*.

Springer Science & Business Media, 2014.