

# 16 - 748 Problem Set

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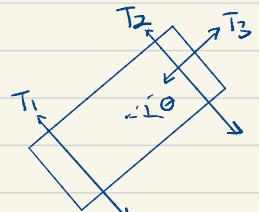
## Problem #1

a) Assume the system has no damping (single rigid body)

the dynamics of the system can be represented by  $[x \dot{x} \theta]^T$

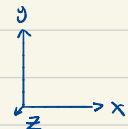
2nd order diff equation:

$$\begin{bmatrix} \ddot{x} \\ \ddot{y} \\ \ddot{\theta} \end{bmatrix} = A \begin{bmatrix} T_1 \\ T_2 \\ T_3 \end{bmatrix}$$



the force can be summarized:

$$F = \begin{bmatrix} F_x \\ F_y \\ F_z \end{bmatrix} = \begin{bmatrix} T_1 \cdot \sin\theta + T_2 \sin\theta + T_3 \cos\theta \\ T_1 \cdot \cos\theta + T_2 \cos\theta + T_3 \cdot \sin\theta \\ 0 \end{bmatrix}$$



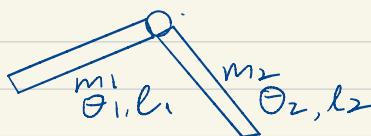
the torque acting on center

$$\begin{aligned} \tau &= L \cdot F = L \cdot T_1 + L \cdot T_2 \\ &= L (T_1 + T_2) \\ &\uparrow_{\text{Im}} \end{aligned}$$

$$\begin{bmatrix} \ddot{x} \\ \ddot{y} \\ \ddot{\theta} \end{bmatrix} = \begin{bmatrix} \sin\theta & \sin\theta & \cos\theta \\ \cos\theta & \cos\theta & \sin\theta \\ 1 & 1 & 0 \end{bmatrix} \cdot \begin{bmatrix} \ddot{T}_1 \\ \ddot{T}_2 \\ \ddot{T}_3 \end{bmatrix}$$

Since there exist  $\theta$   
where  $\text{rank}(A) = 2 < 3$   
thus the system is underactuated

b)



$$m_1 = m_2$$

$$\theta_1 = \theta$$

$$l_1 = l_2$$

kinetic energy  $T = \frac{1}{2}m_1 l_1 \dot{\theta}_1^2 + \frac{1}{2}m_2 l_2 \dot{\theta}_2^2$

potential energy  $V \rightarrow 0 \rightarrow$  planar system

$$L = T - V = \frac{1}{2}m l^2 (\dot{\theta}_1^2 + \dot{\theta}_2^2)$$

Dynamics

$$\frac{d}{dt} \left( \frac{\partial L}{\partial \dot{\theta}} \right) - \frac{\partial L}{\partial \theta} = F \leftarrow \text{force by rotational joint}$$

because two identical links are actuated  
by a single link

$$\therefore \dot{\theta}_1 = \dot{\theta}_2$$

$$\therefore m l^2 \dot{\theta}_1 = F$$

$$\dot{\theta} = \frac{1}{m l^2} F$$

$\nabla$   
 $\neq 0 \text{ rank}(1)$

Full rank Fully Actuated

Problem 2

a)  $\begin{bmatrix} \dot{\theta} \\ \ddot{\theta} \end{bmatrix} = \begin{bmatrix} 0 \\ (u - m \cdot g \cdot l \cdot \sin\theta - b\dot{\theta}) \cdot \frac{1}{ml^2} \end{bmatrix}$

to find stable points

$$\begin{bmatrix} \dot{\theta} \\ \ddot{\theta} \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix} \quad \begin{aligned} \dot{\theta} &= 0 \\ u - m \cdot g \cdot l \cdot \sin\theta - b\dot{\theta} &\cdot \frac{1}{ml^2} = 0 \end{aligned}$$

①  $\dot{b} = u = 0$

$$u - m \cdot g \cdot l \cdot \sin\theta - b\dot{\theta} = 0$$

$$\therefore m \cdot g \cdot l \cdot \sin\theta$$

fixed points are  $\begin{bmatrix} \dot{\theta} \\ \ddot{\theta} \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$   
where

$$\lambda = \underbrace{-\infty, \dots, -1, 0, 1, \dots, +\infty}_{\text{unstable equilibrium}}$$

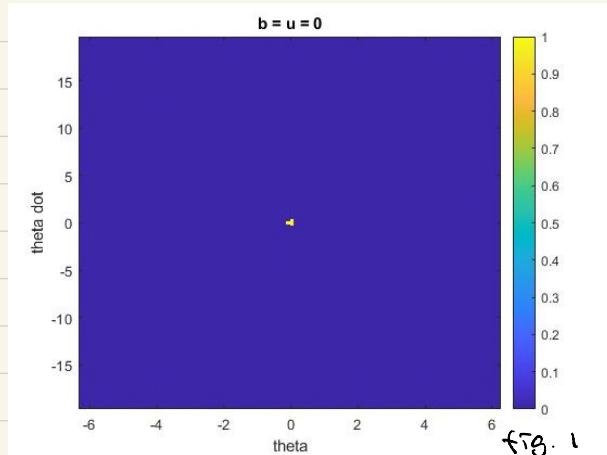


fig. 1

the basin attraction of the fixed point  
is a point which is  $\begin{bmatrix} 0 \\ 0 \end{bmatrix}$

$$\textcircled{2} \quad \left\{ \begin{array}{l} b = 0.25 \\ u = 0 \end{array} \right.$$

solve the equation:  $\dot{\theta} = 0$

$$\mu - m \cdot g \cdot l \cdot \sin \theta - b \dot{\theta} = 0$$

$\therefore$  fixed point is equal to 0

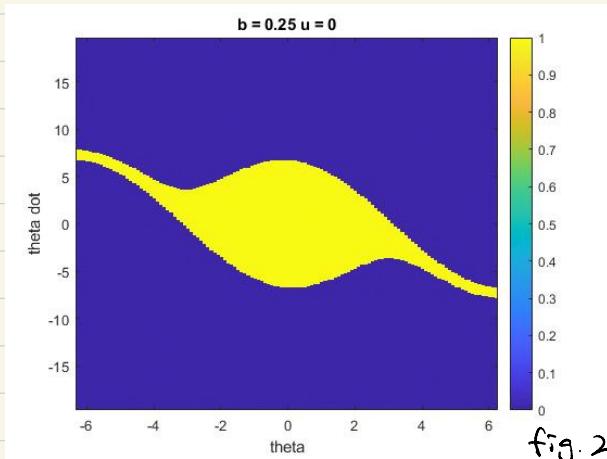


fig. 2

the basin attraction is show in figure 2,  
yellow region represents converge, purple region  
represents diverge

$$\textcircled{3} \quad \left\{ \begin{array}{l} b = 0.25 \\ u = \frac{g}{2L} y \end{array} \right.$$

solve equations

$$\frac{g}{2L} - m \cdot g \cdot l \cdot \sin \theta - b \dot{\theta} = 0$$

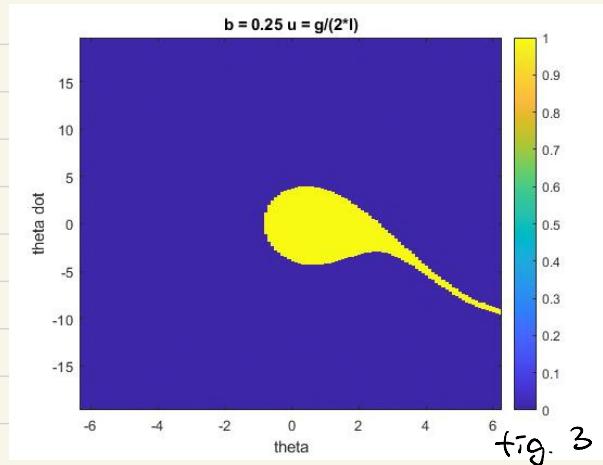
$$\frac{9.8}{2} - 9.8 \cdot \sin \theta - 0 = 0$$

$$\sin \theta = \frac{1}{2}$$

$$\theta = \frac{\pi}{6}, \frac{5}{6}\pi, \frac{7}{6}\pi, \dots$$

$\therefore$  fixed point

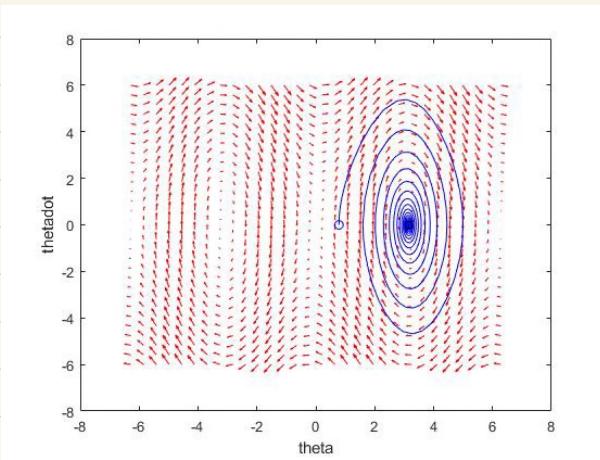
$$\begin{bmatrix} \frac{\pi}{6} \\ 0 \end{bmatrix}$$



$$b) m l^2 \ddot{\theta} + m \cdot g \cdot l \cdot \sin \theta + b \dot{\theta} = u$$

when  $u = 2 \cdot m \cdot g \cdot l \cdot \sin \theta$

$$m l^2 \ddot{\theta} = -b \dot{\theta} + m \cdot g \cdot l \cdot \sin \theta$$



From the graph pendulum eventually stabilizes at  $[\theta, \dot{\theta}]^T = [\pi, 0]^T$

total energy  $E = \int_{t_1}^{t_2} \tau \dot{\theta} dt$

$$= \int_{t_1}^{t_2} 2 \cdot m \cdot g \cdot l \cdot \sin \theta \cdot \dot{\theta} dt \quad \leftarrow$$

$$= 10.8851$$

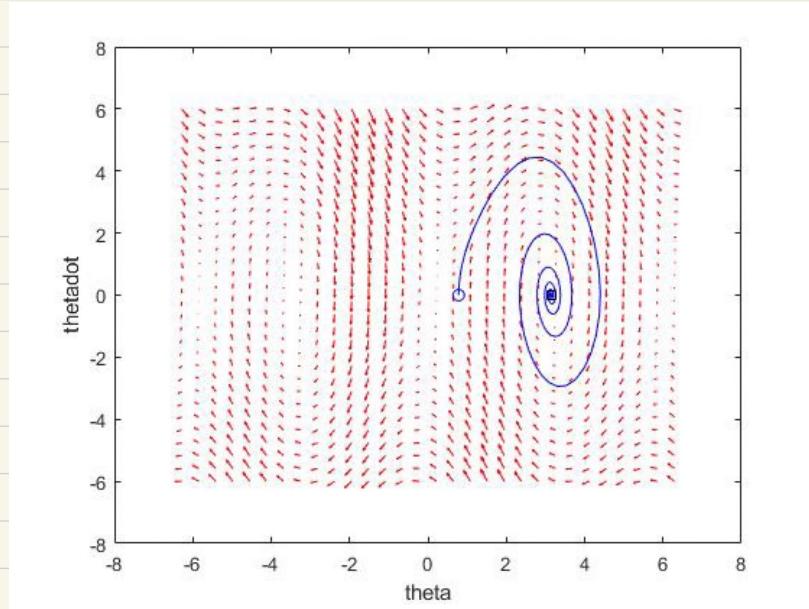
from  
matlab

$$c) m l^2 \ddot{\theta} + m \cdot g \cdot l \cdot \sin(\theta) + b\dot{\theta} = u$$

PD control feedback controller

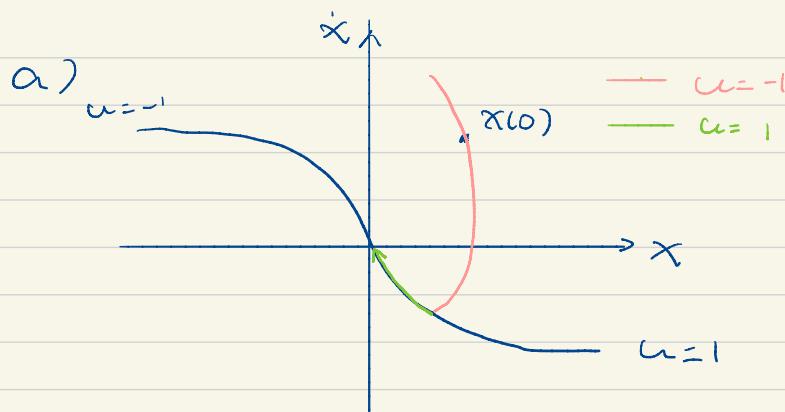
$$u = k_d (\theta - \pi) + k_p (\dot{\theta})$$

$$\text{where } k_d = -0.5 \quad k_p = -0.5$$



$$\begin{aligned} \text{total energy } E &= \int_{t_1}^{t_2} \frac{1}{2} I \dot{\theta}^2 dt \\ &= (-0.5 \cdot (\theta - \pi) - 0.5 \dot{\theta}) \dot{\theta} \\ &= \boxed{-2.4675} \end{aligned}$$

3



the minimum-time policy is presented above  
(check matlab function)

derivation:

the policy is to accelerate then brake ("bang-bang")  
in this case:  $u = -1$  then  $u = 1$

$$\ddot{q} = u \quad u = -1$$

$$\dot{q}(t) = q(0) - t$$

$$q(t) = q(0) + t\dot{q}(0) - \frac{1}{2}t^2$$

$$u = 1 \quad \dot{q}(t) = \dot{q}(0) + t$$

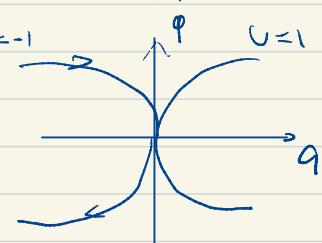
$$q(t) = q(0) + t\dot{q}(0) + \frac{1}{2}t^2$$

when  $x \geq 0$  :

$$\begin{cases} u = -1 & \dot{x} > -|\sqrt{2x}| \\ u = 1 & \dot{x} \leq -|\sqrt{2x}|\end{cases}$$

when  $x < 0$  :

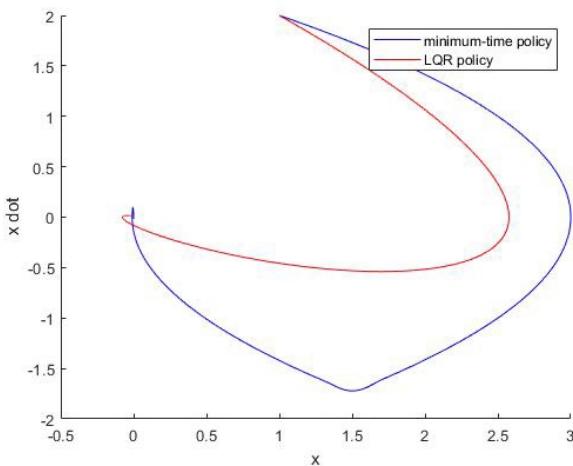
$$\begin{cases} u = -1 & \dot{x} > |\sqrt{2x}| \\ u = 1 & \dot{x} \leq |\sqrt{2x}|\end{cases}$$



b) Plot two policies in the same graph

$$Q = 0.25 I$$

$$R = 5$$



c)

using ode45, time to get to 10.051 within the goal is calculated.

$$t = 5.42 \text{ s} \quad \text{using minimum-time}$$
$$t = 14.38 \text{ s} \quad \text{using LQR}$$

time decreases as Q is increased

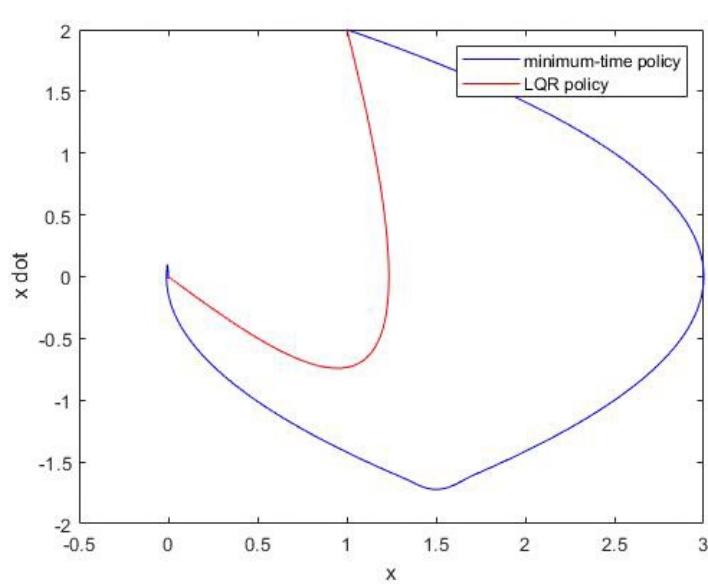
because the system punishes more when the x is not at the desired location.

time increases as R is increased  
because the system focuses more on the control input than the states. Thus less input can result in more converge time.

$$\text{when } Q = 100I \quad R = 10I$$

$$t = 4.08s \quad \text{using LQR}$$

the result make sense because LQR doesn't have constrain on  $u$  while minimum-time policy must satisfy  $|u| \leq 1$ . Thus, a higher input can result in less time to converge.



$$Q = 100I$$

$$R = 10$$