Fluid Dynamics + Turbulence (fall 2017) Homework Problems II + voluntary Exercises

Posted:

Friday September 8, 2017.

Deadline for submission of homework problem:

Tuesday September 19 at 01.15 pm (on paper at Navitas 04.041).

Homework problem 2.1: Why does an Airbus A380 fly?

(a) Use the Navier-Stokes equation in the static limit $\vec{u}=0$, the gravitational force $\vec{f}_{\rm ext}=-\rho g \vec{e}_z$, and the equation of state $p/p_0=\rho/\rho_0$ to derive the barometric height formula

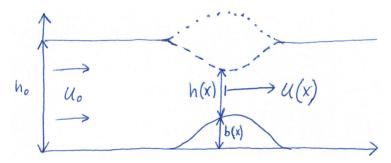
$$\frac{p(z)}{p_0} = \frac{\rho(z)}{\rho_0} = \exp\left(-\frac{g\rho_0 z}{p_0}\right),\,$$

where z is the height above ground, $p_0 = p(z=0) = 1.01 \times 10^5 \frac{\text{kg}}{\text{m}\,\text{sec}^2}$ is the air pressure at ground, $\rho_0 = \rho(z=0) = 1.20\,\text{kg/m}^3$ is the air density at ground, and $g=9.81\,\text{m/sec}^2$ is the acceleration of gravity. How much is the air density reduced at the height $z=10\,\text{km}$ above ground?

(b) An Airbus A380 is typically cruising with the velocity $u = 945 \,\mathrm{km/h}$ at a height 10 km above ground. The lift force acting on the air wings is balancing the weight force. Use Bernoulli's equation to make a rough estimate of the velocity difference above and below the air wings. All you need are the results from (a), the wing area $A_{wing} = 846 \,\mathrm{m}^2$ and the weight $m \approx 500 \,\mathrm{t}$ of the A380.

Homework problem 2.2: Surface bump or dip in a shallow river flow?

Consider an ideal two-dimensional shallow river flow over a small bump at the river bed; see sketch.



(a) Explain the two conservation laws:

$$u_0 h_0 = u(x)h(x) , \qquad (1)$$

$$\frac{u_0^2}{2} + gh_0 = \frac{u^2(x)}{2} + g(b(x) + h(x)) .$$
(2)

(b) Equations (1) and (2) are two equations for the two unknown functions h(x) and u(x). Eliminate u(x) and derive an equation only for h(x). Show that this equation is consistently solved by $h(x) = h_0$ when b(x) = 0. In order to solve the equation also for b(x) > 0, assume the bump $b(x) \ll h_0$ and the change of river surface height $\Delta h(x) = h(x) - h_0 \ll h_0$ to be small. Derive the solution

$$\Delta h(x) = \frac{b(x)}{\left(\frac{u_0^2}{gh_0} - 1\right)} \tag{3}$$

by keeping first-order terms in small $b(x)/h_0$ and $\Delta h(x)/h_0$ in a consistent way.

- (c) Use equation (3) in the two limits $u_0^2 \ll gh_0$ and $u_0^2 \gg gh_0$ to explain when a surface bump or dip occurs.
- (d) What is the catch with equation (3) when $u_0^2 \approx gh_0$?
- (e) Use the numbers $b_{\rm max}=0.1\,{\rm m}$, $h_0=1\,{\rm m}$, $g=9.81\,{\rm m/s^2}$ to calculate the extremum of $\Delta h(x)$ for $u_0=1\,{\rm m/s}$ and $u_0=10\,{\rm m/s}$.

Exercise 2.1

The friction force in the Navier-Stokes equation

$$\rho\left(\frac{\partial \vec{u}}{\partial t} + \left(\vec{u} \cdot \vec{\nabla}\right)\vec{u}\right) = \vec{f}_{\text{ext}} - \vec{\nabla}p + \mu\left(\vec{\nabla} \cdot \vec{\nabla}\right)\vec{u} + \left(\mu_v + \frac{\mu}{3}\right)\vec{\nabla}\left(\vec{\nabla} \cdot \vec{u}\right)$$
(4)

contains the term

$$\frac{\partial^2 u_y}{\partial x \partial y} \tag{5}$$

in x-direction. Why does this term represent a force pointing in the x-direction?

Exercise 2.2

An infinitesimal closed line integral can be approximated as

$$\oint \vec{v}(x,y) \cdot d\vec{s} \approx v_x \left(x, y + \frac{\Delta y}{2} \right) \Delta x
+ v_y \left(x + \frac{\Delta x}{2}, \Delta y \right) (-\Delta y)
+ v_x \left(x, y - \frac{\Delta y}{2} \right) (-\Delta x)
+ v_y \left(x - \frac{\Delta x}{2}, y \right) \Delta y.$$
(6)

Use a Taylor series expansion and show that

$$\oint \vec{v} \cdot d\vec{s} = -\left(\vec{\nabla} \times \vec{v}\right) \cdot \vec{e}_z \Delta x \Delta y \tag{7}$$

Given this result, illustrate what the irrotational flow condition $\vec{\nabla} \times \vec{v} = 0$ means.

Exercise 2.3

Consider the following two-dimensional velocity field:

$$u_{x} = u_{0} \tag{8}$$

$$u_{y} = \kappa x^{2} \tag{9}$$

- (a) Sketch the velocity vector at selected positions along the *x*-axis.
- **(b)** Calculate the divergence $\vec{\nabla} \cdot \vec{u}$ and curl $\vec{\nabla} \times \vec{u}$ of the velocity field.

(c) Use the differential equations

$$\frac{\mathrm{d}x}{\mathrm{d}t} = u_x, \quad \frac{\mathrm{d}y}{\mathrm{d}t} = u_y \tag{10}$$

to show that if a particle is in position (x_0, y_0) at time t_0 , then at time t its position is given by

$$x(t) = x_0 + u_0(t - t_0) (11)$$

$$y(t) = y_0 + \frac{\kappa}{3u_0} \left[x(t)^3 - x_0^3 \right]. \tag{12}$$

(d) Sketch some particle trajectories (path lines) with $t_0 = x_0 = 0$.

Exercise 2.4

When we derived the Bernoulli equation

$$\frac{\rho_0}{2}\vec{v}^2 + p = \text{constant},\tag{13}$$

we stumbled over the relation

$$\vec{\nabla} \left(\frac{\rho_0}{2} \vec{v}^2 + p \right) = \rho_0 \vec{v} \times \left(\vec{\nabla} \times \vec{v} \right), \tag{14}$$

where we simply evoked the irrotational flow condition to put the right-hand side to zero. Now we want to be more careful.

(a) Show that when we multiply (scalar product) the right-hand side with a segment $d\vec{s}$ of a streamline, that

$$d\vec{s} \cdot \left[\vec{v} \times \left(\vec{\nabla} \times \vec{v} \right) \right] = 0. \tag{15}$$

(b) Show that the integration of the left-hand side,

$$\int \vec{\nabla} \left(\frac{\rho_0}{2} \vec{v}^2 + p \right) \cdot d\vec{s} = 0, \tag{16}$$

along a streamline, leads to

$$\frac{\rho_0}{2}\vec{v}^2 + p = \text{constant} = c \tag{17}$$

along the streamline.

(c) What can you say about the constant *c* for different stream lines?