# Session #2 Gears: Basics, terminology, involute profile

## Today's Agenda

- Distribute homework #1
- Gears
  - Applications
  - Types
  - Terminology / nomenclature
  - Congugate action
  - Involute curve
  - Analysis & design

## Applications of Gears

Photo of small electric motor removed for copyright reasons.

Photo of an elaborate chronometer (ref: Dava Sobel's book *Longitude*) removed for copyright reasons.

Diagram removed for copyright reasons.
"AH Sprite/MG Midget Clutch, Transmission and Drive Train, Differential and Axles"

## Spur Gears

- Transmit motion between parallel shafts
- Teeth are parallel to the axis of rotation
- This is the simplest kind of gear we'll consider and most of today is dedicated to them



## Other Types of Gears

Rack



Courtesy of OSHA.

**Bevel** 

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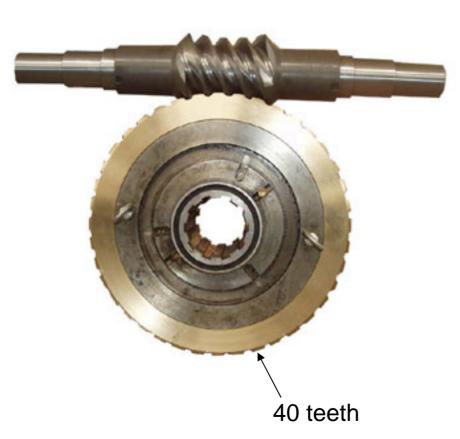
## Name That Gear



What type of worm gear set is this?

- Single-enveloping, single threaded
- Single-enveloping, multi-threaded
- Double enveloping single threaded worm gear
- Double enveloping multi- threaded

## Follow up



What is the reduction ratio of this gear set?

- 1) 10:1
- 2) 20:1
- 3) 40:1
- 4) 80:1

## Harmonic Gear Drive

- Not based on rigid body motion!
- Ellipse in center deforms the internal gear

Diagrams removed for copyright reasons.

## **Early Gears**

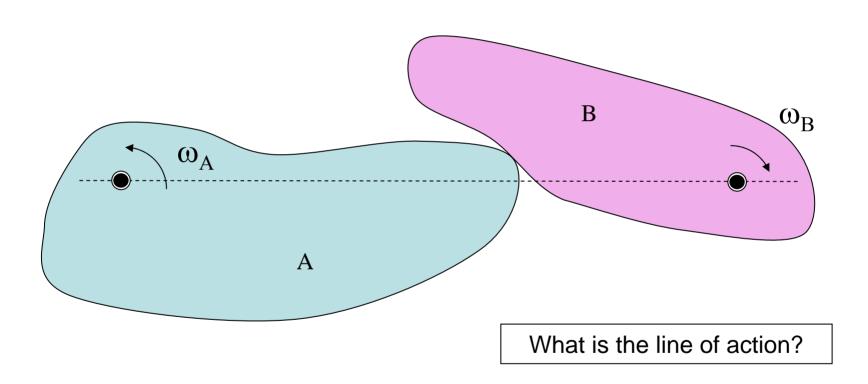
Drawing removed for copyright reasons.

Drawing removed for copyright reasons.

Roman watermills at Barbegal 300AD

Application for powering textile machinery 18<sup>th</sup> century

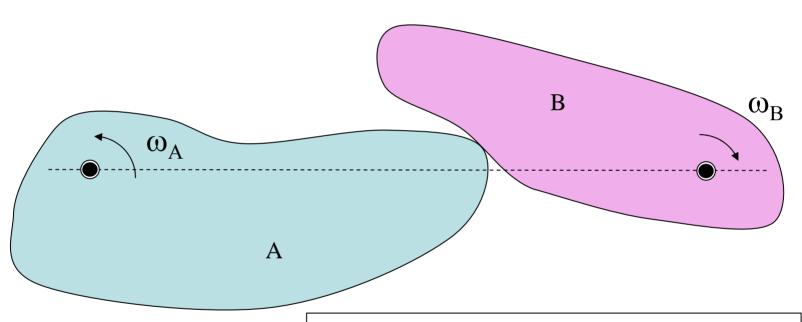
## Pitch Point



What is the pitch point?

What are the relationships among these?

## Sliding and Rolling

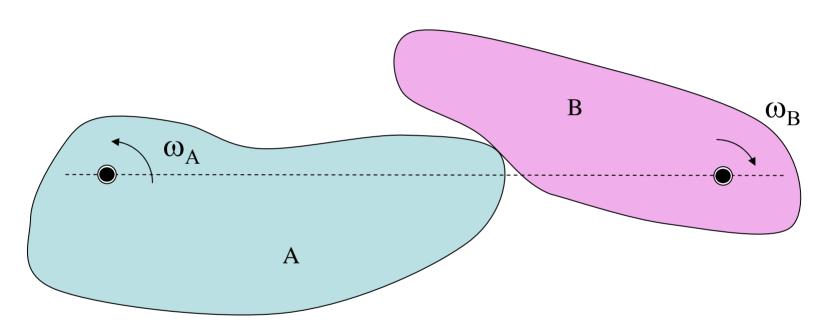


When one body is driving another, do the surfaces slide, roll, or both?

What is the relationship to the pitch circles?

How could you determine this?

## Conjugate Action



Let's say  $\omega_A$  is a known. How can we determine  $\omega_B$ ?

Let's say  $\omega_A$  is a constant with time. Can we synthesize a shape of body B so that  $\omega_B$  is also constant with time?

## Rack Cutting

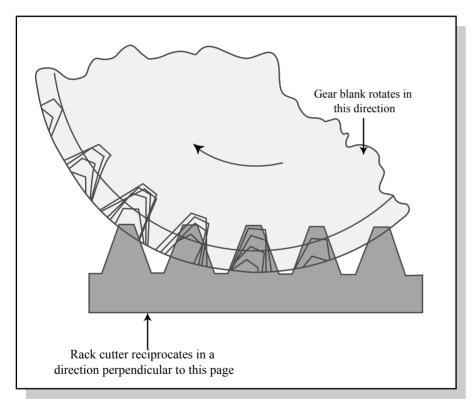
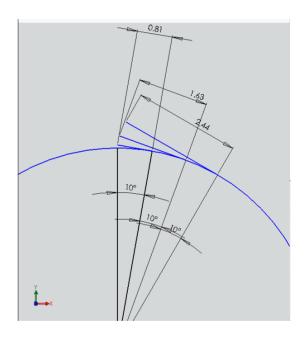


Figure by MIT OCW.

- A way to get the relative motion you want
- Pick one shape as you wish
- Enforce the motion you want
- Cut away
   everything that
   interferes

## Involute Profile

- How it is constructed
  - Demo
- Properties
  - Conjugate action
  - Allows design of whole sets of compatible gears
  - Conjugate action not sensitive to center distance variations



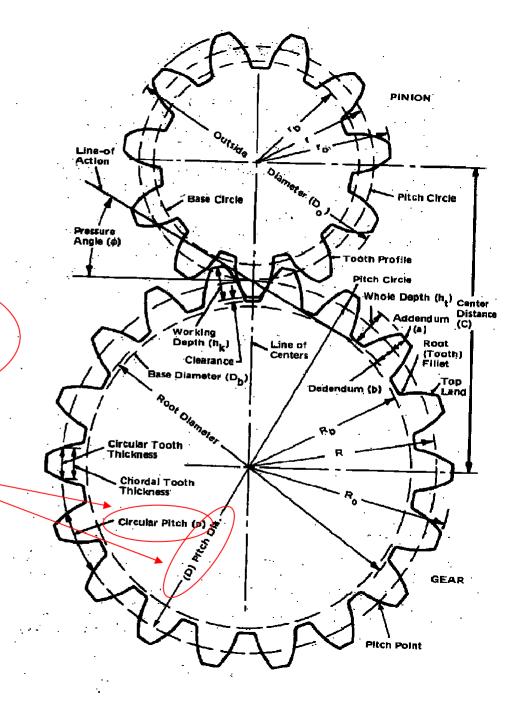
## Gear Terminology

Diametral pitch (teeth per inch)
# of teeth on a gear with a
1 inch pitch diameter

Easily confused

Source: Buchsbaum, Frank, *Design and Application of Small Standardized components Data Book 757* Vol. 2, Stock Drive Products, 1983.

See "Handbook of Gears." <a href="http://www.sdp-si.com/D190/D190cat.htm">http://www.sdp-si.com/D190/D190cat.htm</a> (accessed 28 June 2006.) Courtesy of Stock Drive Products/Sterling Instrument.



## More Gear Terminology

From Shigley and Mischke

Figure removed for copyright reasons. Source: Shigley and Mischke, Figure 13-5.

This geometry is not an involute.

### **Contact Ratio**

Figure removed for copyright reasons. Source: Shigley and Mischke, Figure 13-16.

contact ratio = length of arc of action / pitch = average number of teeth engaged

## Interference

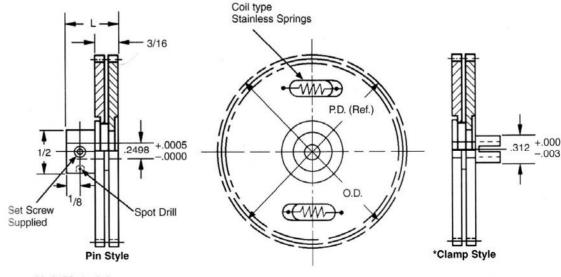
- Part of every tooth is not an involute
- This could in principle cause interference
- Due to the way gears are made, it really causes undercutting

Figure removed for copyright reasons. Source: Shigley and Mischke, Figure 13-17.

## Backlash

- If you want any tolerance for
  - Center distance errors
  - Thermal growth
- There will be backlash when

gears reverse



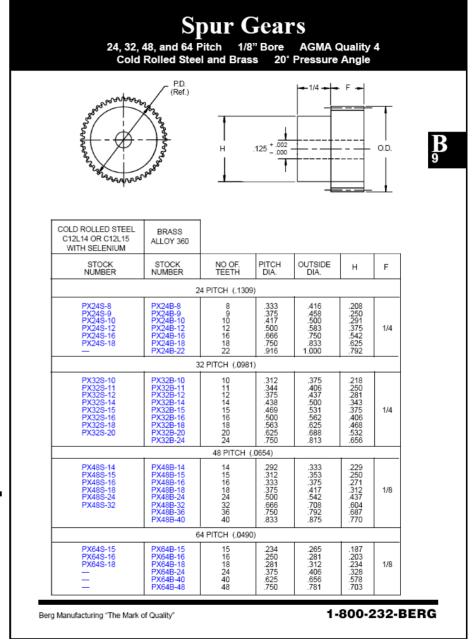
Hub Material: 303 Stainless Steel

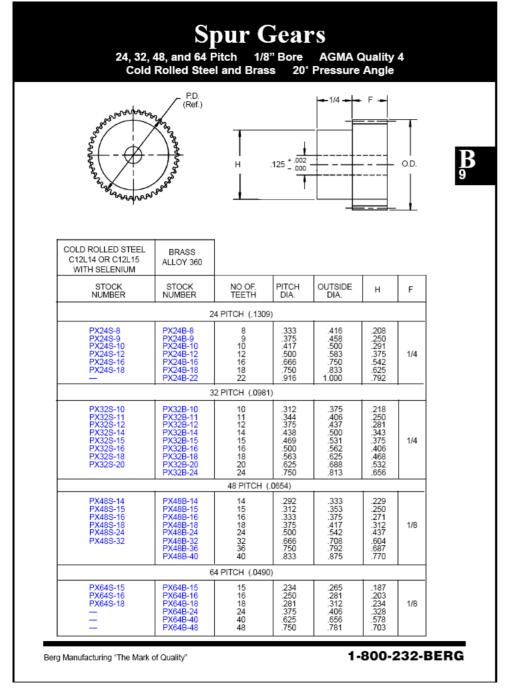
An "anti-backlash" gear

Courtesy of W. M. Berg, Inc. Used with permission.

## Gear Selection

- Pitch
- Face width
- Material
- Pressure angle
- # of teeth
- Hub style, bore, etc.

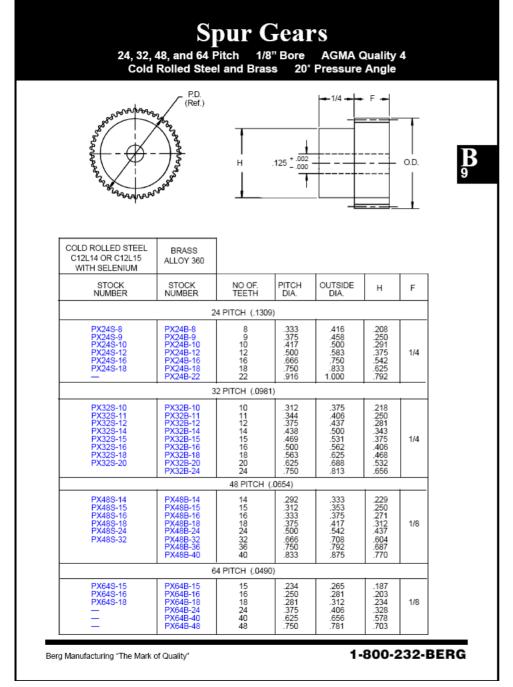




You call up the number 1-800-232-BERG and ask that, for a special application, you want a 48 pitch spur gear, but with a pitch dia of 0.32 inches. They will probably say:

- OK, no problem
- 2. OK, but it will cost a lot
- No, this is not technically possible

Courtesy of W. M. Berg, Inc. Used with permission.



You call up the number 1-800-232-BERG and ask that, for a special application, you want a 48 pitch spur gear, but with a pitch dia of half the smallest one in the catalog. They will probably say:

- 1. OK, no problem
- 2. OK, but it will cost a lot
- 3. OK, but it will be weak
- 4. No, this is not technically possible

Courtesy of W. M. Berg, Inc. Used with permission.

## Strength of Gears

- Any good catalog will have a formula and tables
- What factors must enter the equation?

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Where do the teeth wear the most?

#### Gear Reference Guide

#### **GEAR TOOTH STRENGTH**

Many factors must be considered when designing a gear train. The information listed on this pag should be used as a general guideline for your application. If more critical strength calculation required W.M. Berg suggests that you consult our engineering department or any one of the many geal handbooks that are readily available.

When a gear train is transmitting motion, it is save to assume that all of the load is being carried be one tooth. This is because as the load approaches the end of the tooth, where the bending force would be the greatest, a second tooth comes into mesh to share the load. Simple results can be obtained from the Lewis bending strength equation.

 $W_t = \frac{SFY}{D.P}$ 

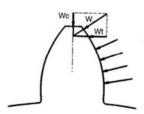
W<sub>t</sub> = Maximum transmitted load (lbs or N)

S = Maximum bending tooth stress (taken as 1/3 of the tensile strength) See Table C on Page 5

F = Face width of gear (in. or mm)

D.P. = Diametral Pitch = 1/module (for equation only)

Y = Lewis Factor (See Table)



NOTE: The maximum bending tooth stress (S) is valid for well lubricated, low shock applications. For high shock, poorly lubricated applications, the safe stress could be as low as .025S. If your design calls for an unfriendly environment for gears, you might want to lower S to assure a reasonable amount of gear life.

	NO. OF TEETH	14 1/2* INVOLUTE	20° INVOLUTE
LEWIS FACTOR - Y	TEETH  10 11 12 13 14 15 16 17 18 19 20 22 24 26 28 30 32 34 36 38 40 45	INVOLUTE  0.176 0.192 0.210 0.223 0.236 0.245 0.255 0.264 0.277 0.283 0.292 0.302 0.308 0.314 0.318 0.322 0.325 0.325 0.329 0.332 0.336 0.336 0.340	0.201 0.226 0.245 0.245 0.264 0.276 0.289 0.302 0.308 0.314 0.320 0.337 0.337 0.344 0.352 0.358 0.364 0.377 0.383
	26 28 30 32 34 36 38 40	0.308 0.314 0.318 0.322 0.325 0.329 0.332 0.336	0.344 0.352 0.358 0.364 0.370 0.377 0.383 0.389
	45 50 55 60 65 70 75 80 90 100	0.340 0.346 0.352 0.355 0.358 0.360 0.361 0.363 0.366 0.368	0.399 0.408 0.415 0.421 0.425 0.429 0.433 0.436 0.442 0.446 0.458
	200 300 RACK	0.378 0.382 0.390	0.458 0.463 0.471 0.484



1-800-232-BERG

Berg Manufacturing "The Mark of Quality"

Courtesy of W. M. Berg, Inc. Used with permission.

## Concept Question

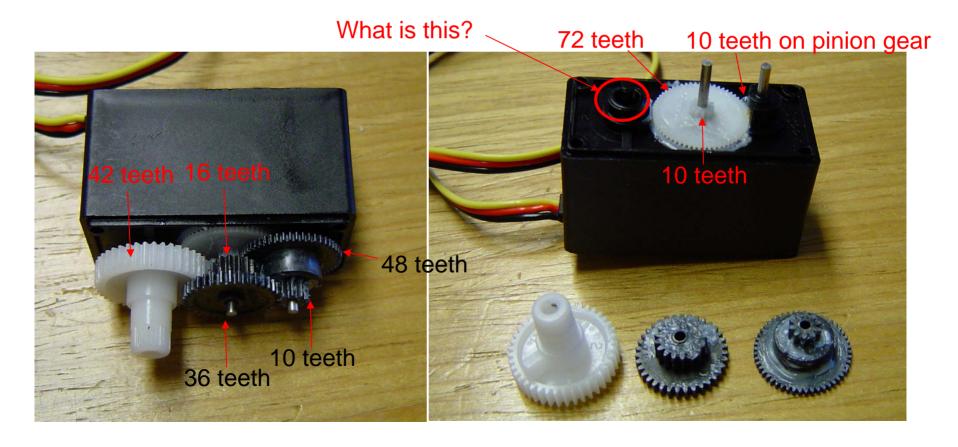
A compound gear train is formed of eight gears. As we proceed from the pinion on the electric motor to the gear on the output shaft, how do the pitch and face width vary?



- Pitch rises, face width rises
- 2. Pitch rises, face width falls
- 3. Pitch falls, face width rises
- 4. Pitch falls, face width falls

## Homework Question #3

 Given the top output shaft speed, what is the motor shaft speed (in rpm)?



## Next Steps

- Begin to develop project ideas
  - Email your slides to TA by Friday noon
- On Friday 10 FEB (right here) you will present
  - A project idea OR what you're looking for in a project
  - Who are you? What do you bring to a project team?
  - Then fill out a project preference form
- On Tues 14 FEB (right here) project teams will be announced
- Start HW#1