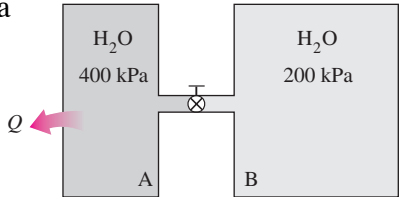
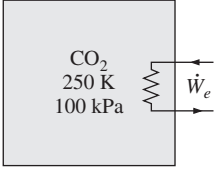


## CHE 260F – Thermodynamics and Heat Transfer

### Mid-Term Exam – 2018

*You have 110 minutes to do the following five problems. You may use any type of non-communicating calculator. All questions are worth equal marks.*

- 1) A piston–cylinder device contains helium gas initially at 150 kPa, 20°C, and 0.5 m<sup>3</sup>. The helium is now compressed in a polytropic process ( $PV^n = \text{constant}$ ) to 400 kPa and 140°C. Determine the heat loss or gain during this process.  
Assume that for helium  $R = 2.0769 \text{ kJ/kgK}$  and  $c_v = 3.1156 \text{ kJ/kgK}$ .
- 2) Two rigid tanks are connected by a valve. Tank A contains 0.2 m<sup>3</sup> of water at 400 kPa and 80 percent quality. Tank B contains 0.5 m<sup>3</sup> of water at 200 kPa and 250°C. The valve is now opened, and the two tanks allowed to lose heat until the system reaches thermal equilibrium with the surrounding air at 25°C. Determine the amount of heat transfer during this process.
 
- 3) A 0.8 m<sup>3</sup> rigid tank contains carbon dioxide (CO<sub>2</sub>) gas at 250 K and 100 kPa. A 500-W electric resistance heater placed in the tank is now turned on and kept on for 40 min after which the pressure of CO<sub>2</sub> is measured to be 175 kPa. Assuming the surroundings to be at 300 K and using constant specific heats, determine (a) the final temperature of CO<sub>2</sub>, (b) the net amount of heat transfer from the tank, and (c) the entropy generation during this process.  
Assume that for CO<sub>2</sub>:  $R = 0.1889 \text{ kJ/kgK}$ ,  $c_p = 0.895 \text{ kJ/kgK}$ ,  $c_v = 0.706 \text{ kJ/kgK}$ .
 
- 4) Air enters a nozzle steadily at 280 kPa and 77°C with a velocity of 50 m/s and exits at 85 kPa and 320 m/s. The heat losses from the nozzle (per kilogram of air flowing through the nozzle) to the surrounding atmosphere at 20°C are estimated to be 3.2 kJ/kg. Determine: (a) the exit air temperature and (b) the entropy generated per kilogram of airflow.  
Assume that for air:  $R = 0.2870 \text{ kJ/kgK}$ ,  $c_p = 1.005 \text{ kJ/kgK}$ ,  $c_v = 0.718 \text{ kJ/kgK}$ .
- 5) Steam enters an adiabatic turbine steadily at 3.5 MPa, 500°C, and 45 m/s, and leaves at 100 kPa and 75 m/s. If the measured power output of the turbine is 5 MW and the isentropic efficiency is 77%, determine the mass flow rate of steam.

**Ideal gas equation**

$$PV = NR_u T \quad R_u = 8.314 \text{ kJ/kmol K}$$

$$PV = mRT \quad R = R_u/M$$

**Boundary Work**

$$W_{12} = - \int_{V_1}^{V_2} P dV$$

**Flow work per unit mass of fluid**

$$w_{\text{flow}} = Pv$$

**First law**  $Q + W = \Delta E$ **Enthalpy**  $h = u + Pv$ **Specific heats**

$$c_v(T) \equiv \left( \frac{\partial u}{\partial T} \right)_v \text{ and } c_p(T) \equiv \left( \frac{\partial h}{\partial T} \right)_p$$

**For an ideal gas**

$$c_p = c_v + R$$

$$\Delta u = u_2 - u_1 = c_{v, \text{avg}} (T_2 - T_1)$$

$$\Delta h = h_2 - h_1 = c_{p, \text{avg}} (T_2 - T_1)$$

$$\text{Specific heat ratio } \gamma = \frac{c_p}{c_v} = \frac{\overline{c_p}}{\overline{c_v}}$$

**For a liquid or solid**

$$\Delta h = h_2 - h_1 = c(T_2 - T_1) + v(P_2 - P_1)$$

**For a control volume**

$$\dot{Q} + \dot{W} = \dot{m} \left[ (h_2 - h_1) + \frac{V_2^2 - V_1^2}{2} + g(z_2 - z_1) \right]$$

**Second Law**

$$\begin{aligned} dS_{\text{isolated}} &> 0 && \text{not in equilibrium} \\ dS_{\text{isolated}} &= 0 && \text{at equilibrium} \end{aligned}$$

$$T \equiv \left( \frac{\partial U}{\partial S} \right)_{m, V}$$

$$\frac{P}{T} \equiv \left( \frac{\partial S}{\partial V} \right)_{m, U}$$

**Gibbs equation**

$$ds = \frac{1}{T} du + \frac{P}{T} dv$$

**Entropy change**

$$dS = \left( \frac{\delta Q_{\text{rev}}}{T} \right)$$

$$\Delta S = S_2 - S_1 = \int_1^2 \left( \frac{\delta Q_{\text{irr}}}{T} \right) + S_{\text{gen}}$$

**For a liquid or solid**

$$\Delta s = s_2 - s_1 = c_{\text{avg}} \int_{T_1}^{T_2} \frac{dT}{T} = c_{\text{avg}} \ln \frac{T_2}{T_1}$$

**For an ideal gas**

$$\Delta s = s_2 - s_1 = c_v \ln \frac{T_2}{T_1} + R \ln \frac{v_2}{v_1}$$

$$\Delta s = s_2 - s_1 = c_v \ln \frac{P_2}{P_1} + c_p \ln \frac{v_2}{v_1}$$

$$\Delta s = s_2 - s_1 = c_p \ln \frac{T_2}{T_1} - R \ln \frac{P_2}{P_1}$$

**For an isentropic process in an ideal gas**

$$\frac{T_2}{T_1} = \left( \frac{v_1}{v_2} \right)^{(\gamma-1)} ; \quad \frac{T_2}{T_1} = \left( \frac{P_2}{P_1} \right)^{(\gamma-1)/\gamma} ;$$

$$\frac{P_2}{P_1} = \left( \frac{v_1}{v_2} \right)^{\gamma} ; \quad Pv^{\gamma} = \text{constant}$$

**For a saturated liquid-vapour mixture**

$$x = \frac{\text{mass of vapour}}{\text{mass of mixture}} = \frac{m_g}{m}$$

$$u = \frac{m_g}{m} u_g + \frac{m_f}{m} u_f = x u_g + (1-x) u_f$$

$$h = \frac{m_g}{m} h_g + \frac{m_f}{m} h_f = x h_g + (1-x) h_f$$

$$h = h_f + x(h_g - h_f) = h_f + x h_{fg}$$

$$s = \frac{m_g}{m} s_g + \frac{m_f}{m} s_f = x s_g + (1-x) s_f$$